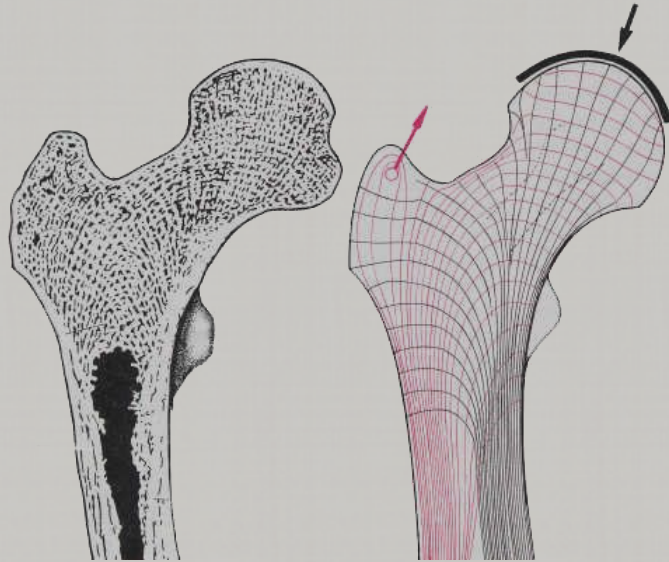


ARUP

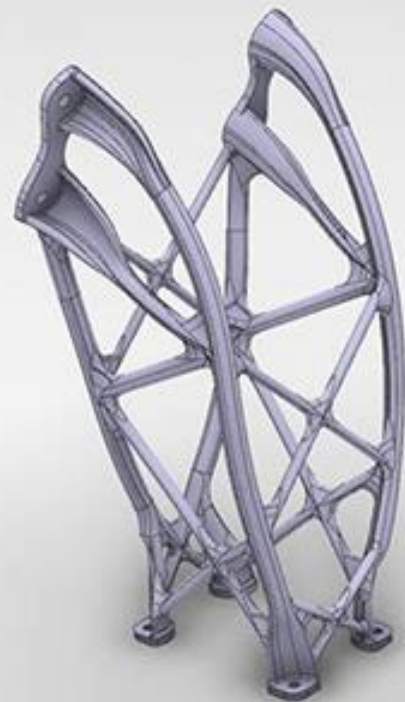
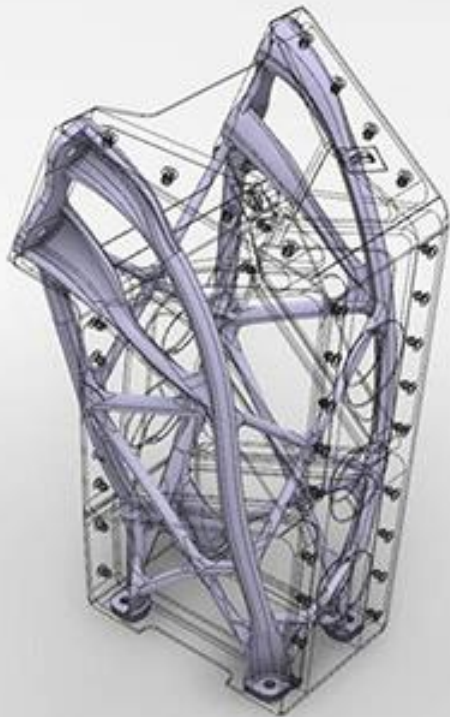
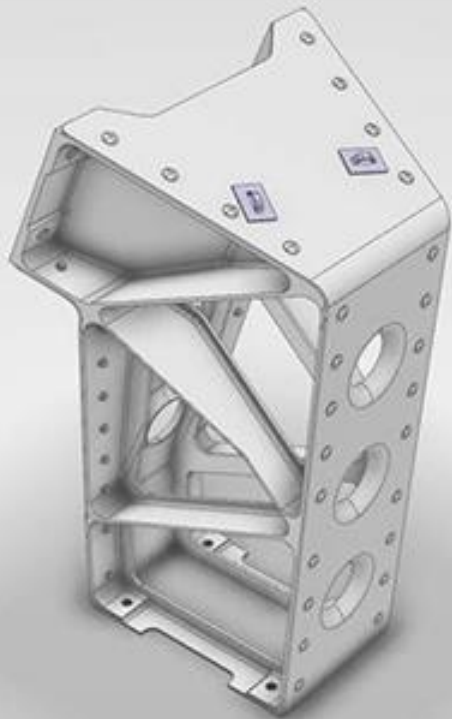
Bayu
Prayudhi

 TU Delft





The optimization of material usage plays a necessary role in nature; the principle is '**materials are expensive and shape is cheap**' as opposed to current technology in our manufacturing industry, the opposite tend to be the case (Pawlyn, 2011).





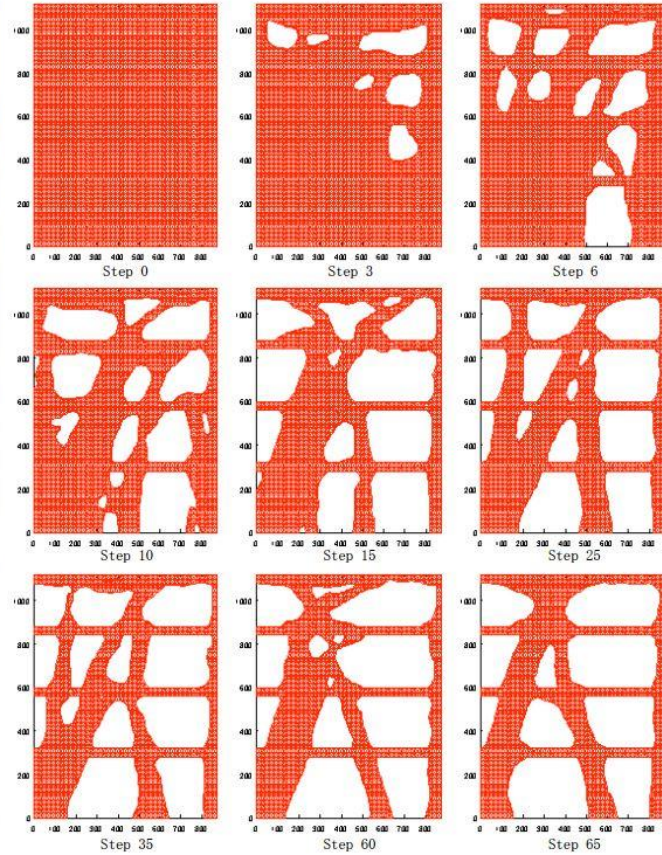
Jet Engine Bracket From Indonesia Wins 3D Printing Challenge

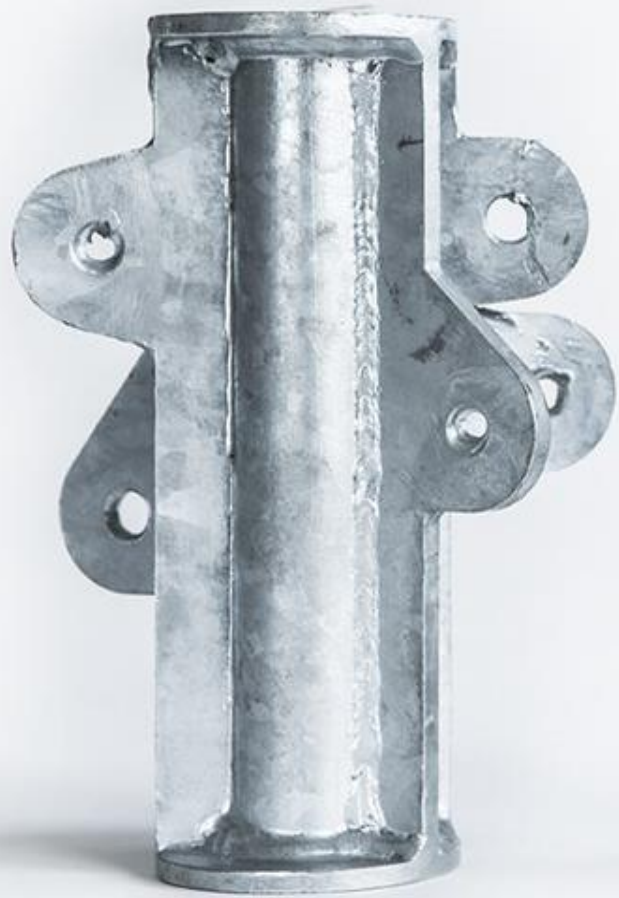
Dec 11, 2013 by Tomas Kellner





Architecture ?







3F3D

FORM FOLLOWS FORCE
with 3D PRINTING



TOPOLOGY OPTIMIZATION FOR FREEFORM ENVELOPE
DESIGN WITH ADDITIVE MANUFACTURING



TOPOLOGY OPTIMIZATION FOR FREEFORM ENVELOPE
DESIGN WITH **ADDITIVE MANUFACTURING**

design tool

**TOPOLOGY
OPTIMIZATION**

fabrication method

**ADDITIVE
MANUFACTURING**

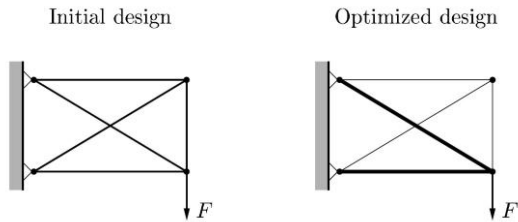
case study

**FREEFORM
ENVELOPE**

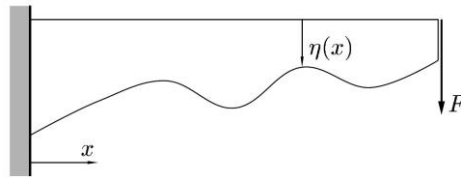
**TOPOLOGY
OPTIMIZATION**

**ADDITIVE
MANUFACTURING**

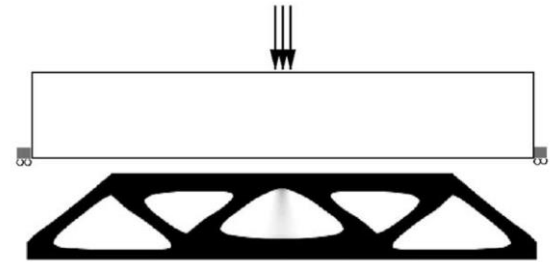
**FREEFORM
ENVELOPE**



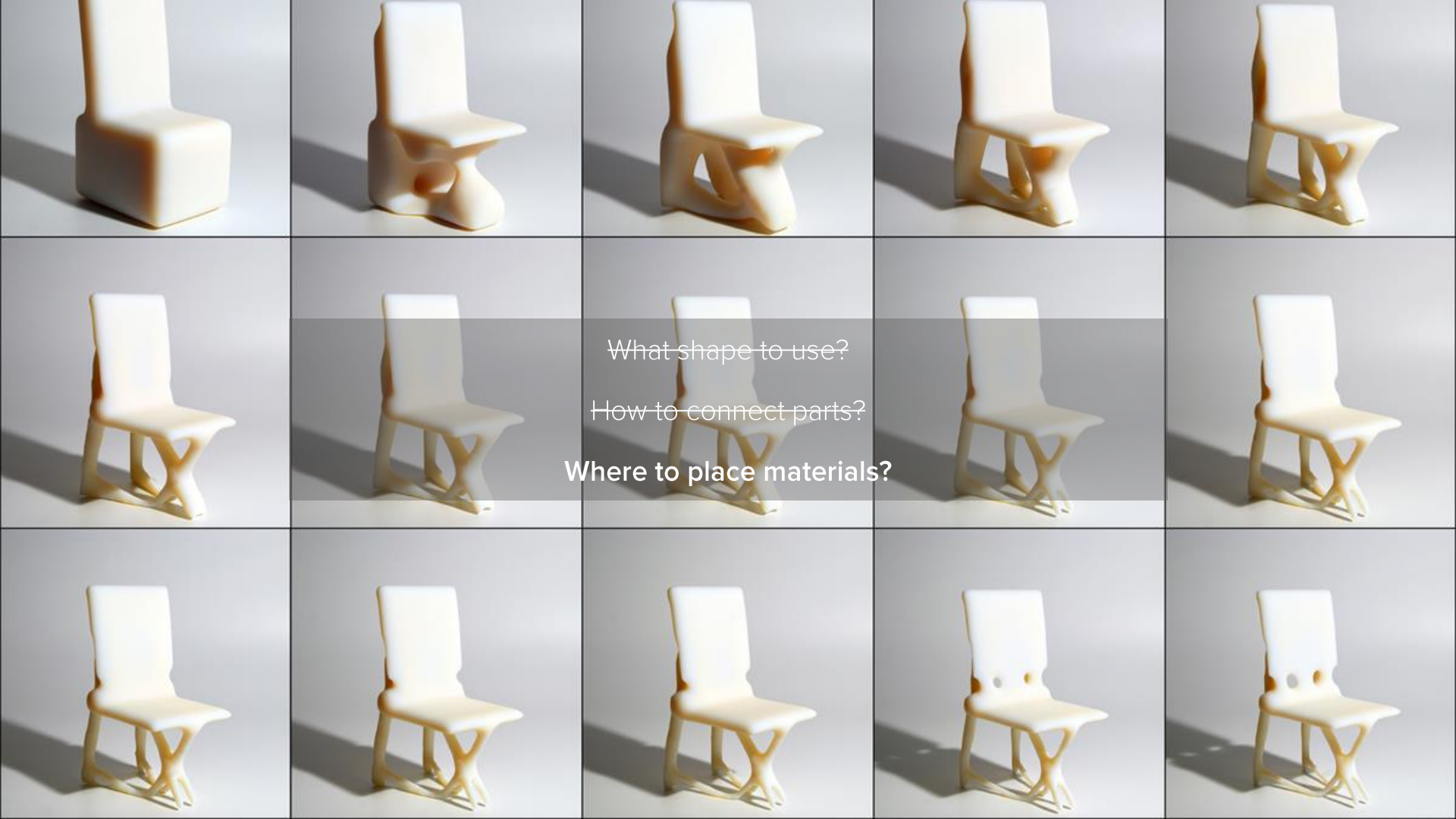
sizing optimization



shape optimization



topology optimization



What shape to use?

How to connect parts?

Where to place materials?

TO basic principle:

1. Define problem:

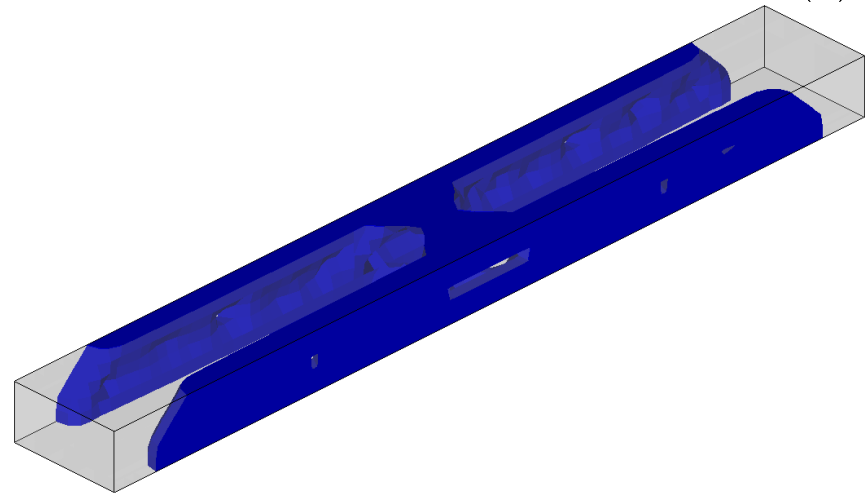
- Objective? Constraints?
- Domain? Boundary conditions?
- Loadcases?

2. Discretize and parameterize material distribution

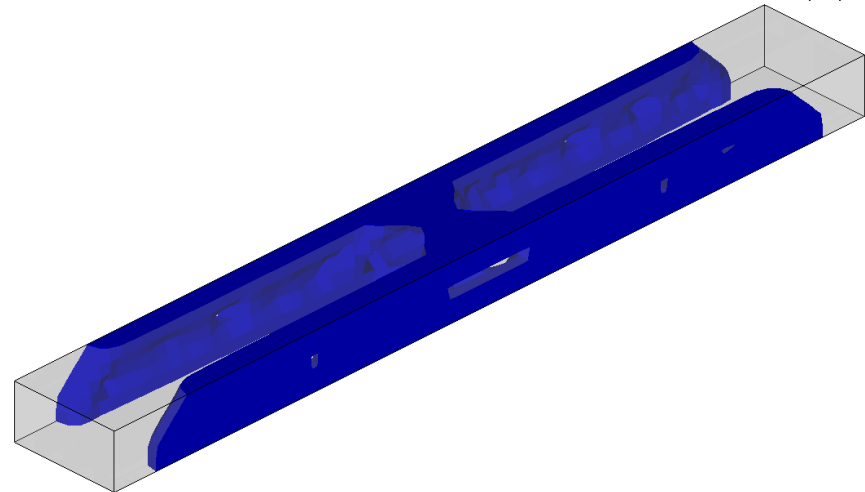
3. Optimize material distribution for best performance

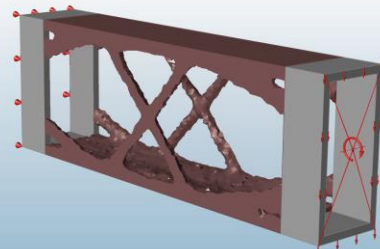
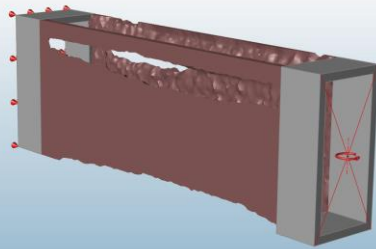
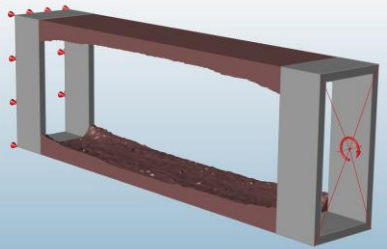
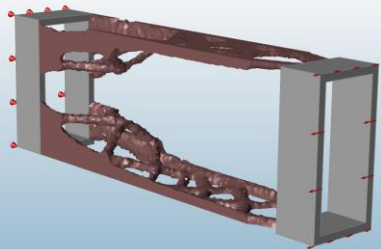
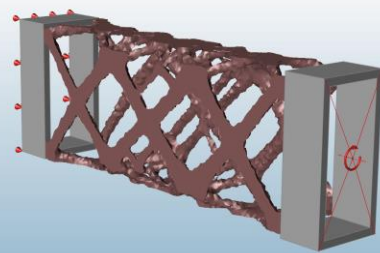
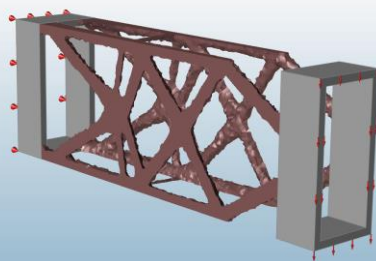
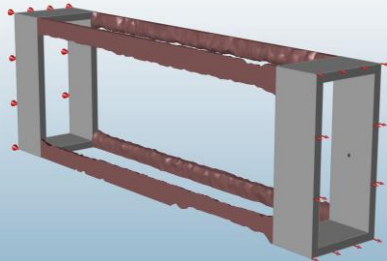
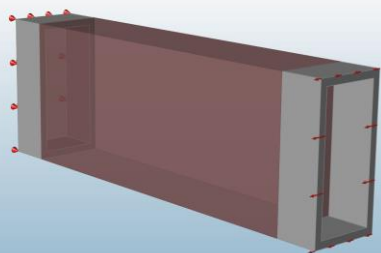
4. Evaluate / fine-tune result (post-processing, shape optimization)

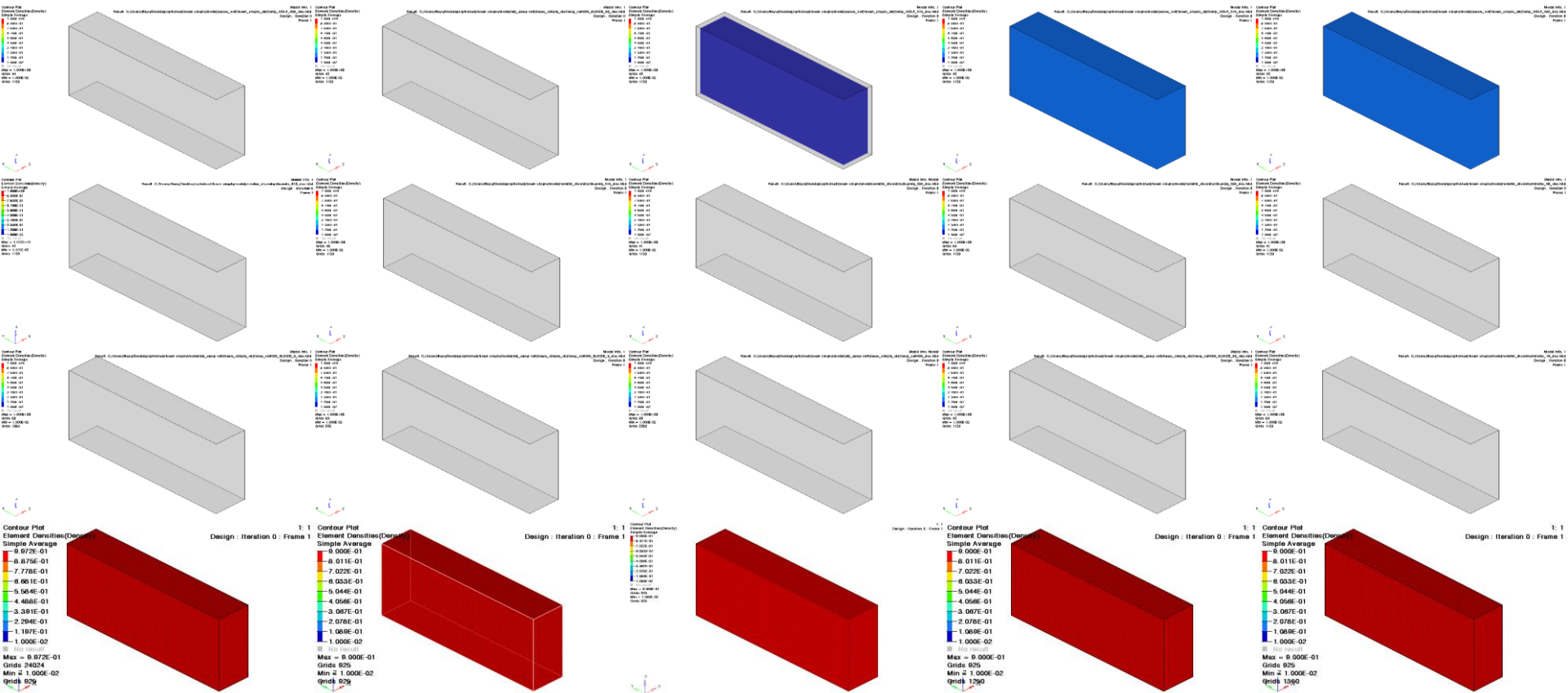
Contour Plot
Displacement(Mag)
Analysis system
-8.727E+00
-7.757E+00
-6.787E+00
-5.818E+00
-4.848E+00
-3.878E+00
-2.908E+00
-1.938E+00
-9.698E-01
-0.000E+00
No result
Max = 8.727E+00
Grids 2204
Min = 0.000E+00
Grids 85



Contour Plot
Element Stresses (2D & 3D)(vonMises)
Analysis system
Simple Average
1.097E+02
9.749E+01
8.531E+01
7.312E+01
6.093E+01
4.875E+01
3.658E+01
2.437E+01
1.219E+01
1.490E-05
No result
Max = 1.097E+02
Grids 199
Min = 1.490E-05
Grids 1



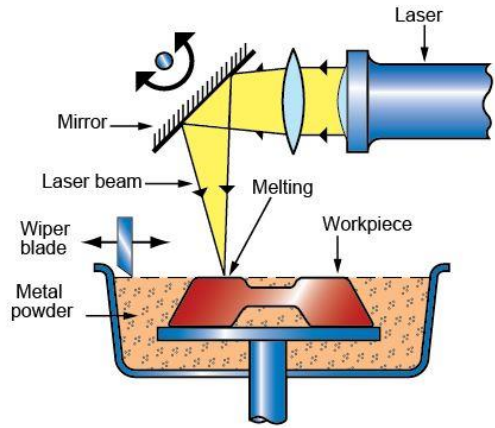




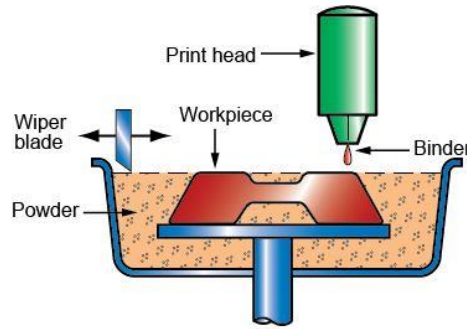
TOPOLOGY
OPTIMIZATION

**ADDITIVE
MANUFACTURING**

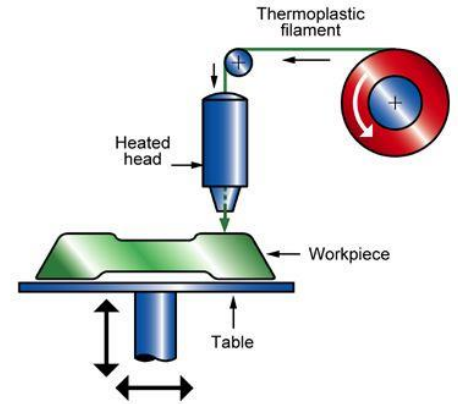
FREEFORM
ENVELOPE



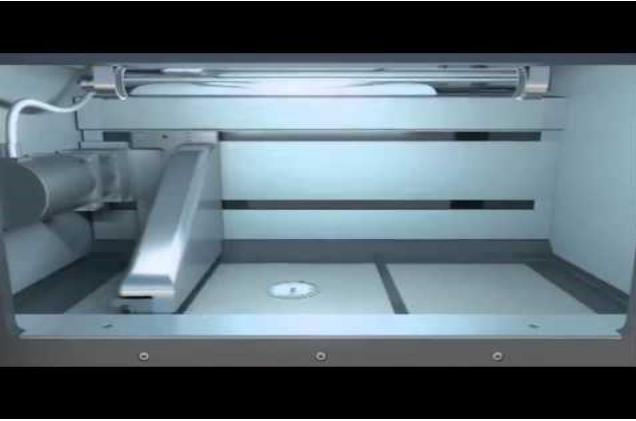
Direct metal laser sintering



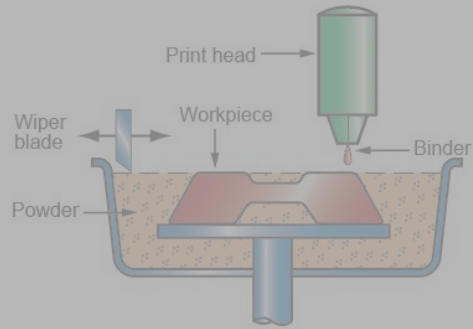
Binder jetting process



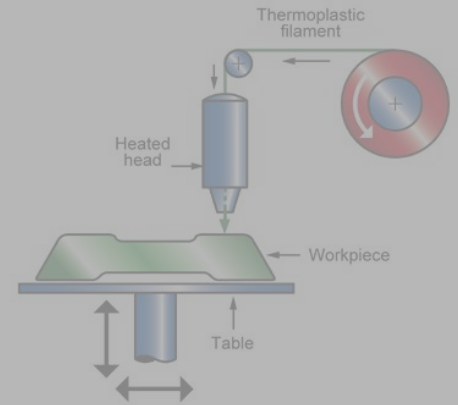
Fused deposition modeling



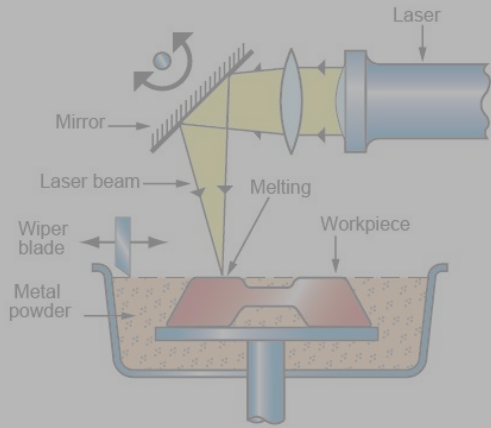
Direct metal laser sintering



Binder jetting process



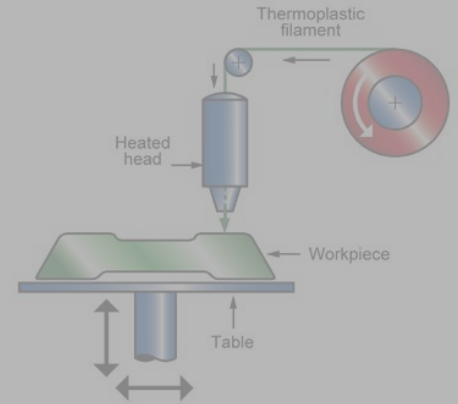
Fused deposition modeling



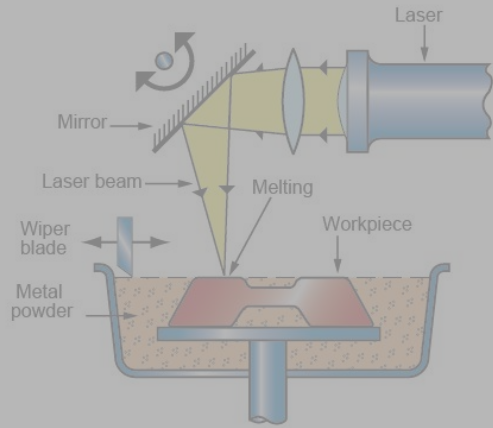
Direct metal laser sintering



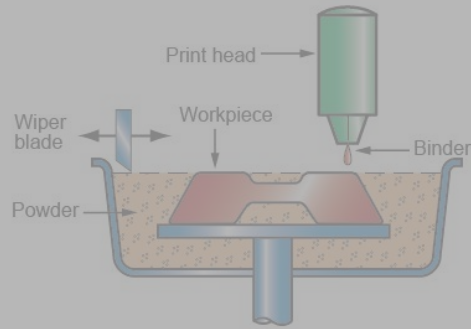
Binder jetting process



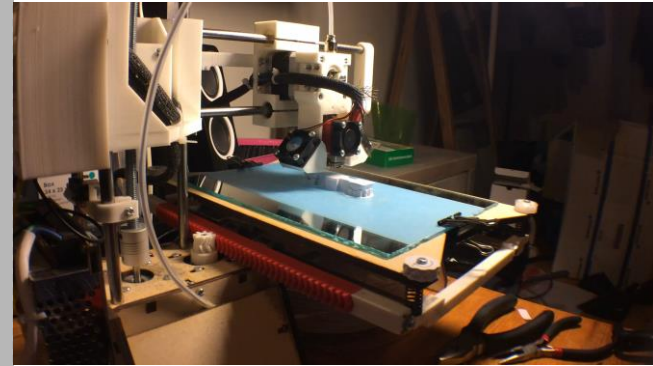
Fused deposition modeling



Direct metal laser sintering



Binder jetting process



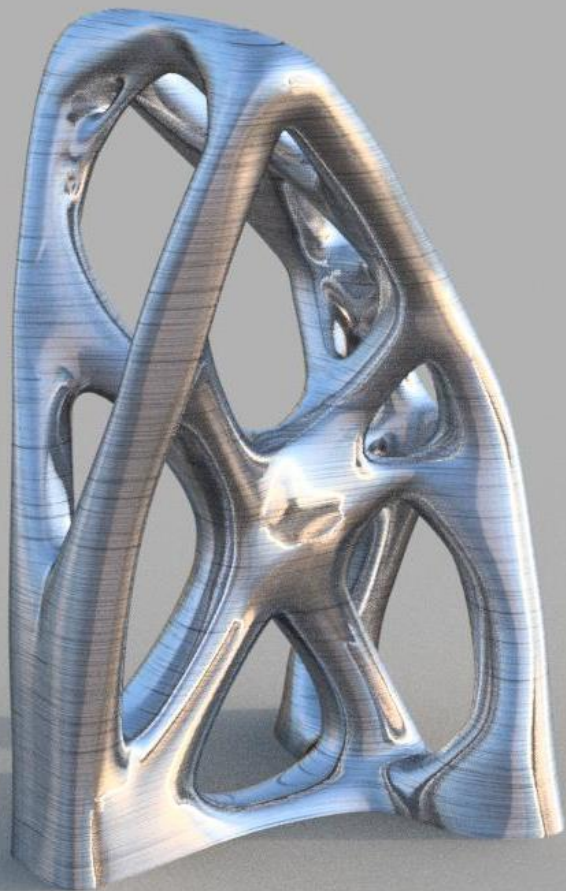
Fused deposition modeling



Metal Additive Manufacturing



3dsystem ProX400
Build chamber 500x500x500
Resolution range : 10 μm – 100 μm



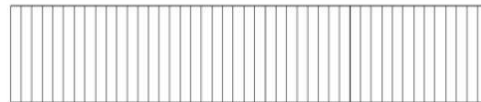
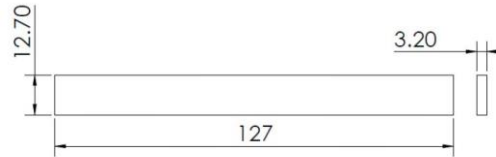
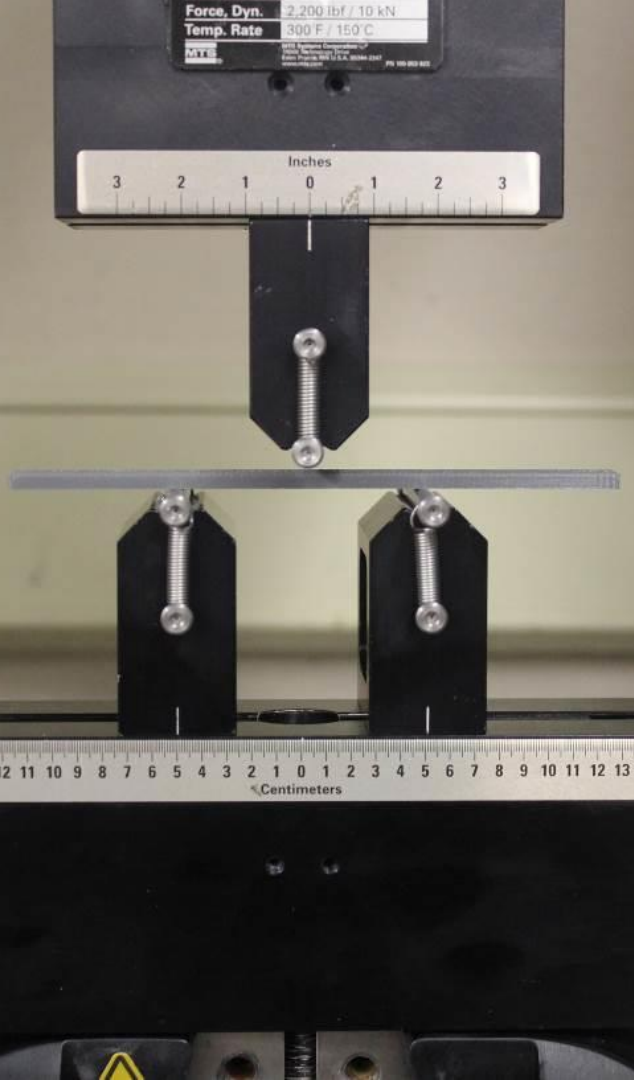
	Company	Country	Material	Ultimate Tensile Strength (MPa)	Price (EUR)
Binder jetting	Shapeways	NL	Stainless Steel 420 60% + Bronze 40%	682	229
	i.materialise	BE	Stainless Steel 420 60% + Bronze 40%	682	216
	ExOne	GER	Stainless Steel 420 60% + Bronze 40%	682	125
Direct Laser Melting	RapidObject	GER	Tool steel 1.2709	1100	757
	Stratasys	US	Stainless Steel 17-4 PH	980	1236
	Fit-Prototyping	GER	Stainless Steel 17-4 PH	980	930
	RapidObject	GER	Stainless steel 1.4542	930	924
	Stratasys	US	Stainless Steel 316L	675	1172
	Protolabs	UK	Stainless Steel 316L	675	1101
	Star-Prototype	CHN	Stainless Steel 316L	675	642
	Stratasys	US	Aluminum AlSi10Mg	400	1347
	Star-Prototype	CHN	Aluminum AlSi10Mg	400	1120
	Protolabs	UK	Aluminum AlSi10Mg	400	1044
	RapidObject	GER	Aluminum AlSi10Mg	400	428
	Fit-Prototyping	GER	Aluminum AlSi10Mg	400	350



Structure with FDM?

Company	Country	Material	Ultimate Tensile Strength (MPa)
Markforged	US	Nylon + continuous carbon fibre	700
Arevolabs	US	PEEK (Polyetheretherketone) + milled carbon fibre	145
Indmatec	GER	PEEK (Polyetheretherketone)	98
Stratasys	GER	PEI (Polyetherimide)	81
Colorfabb	NL	Co-polyester + milled carbon fibre	76





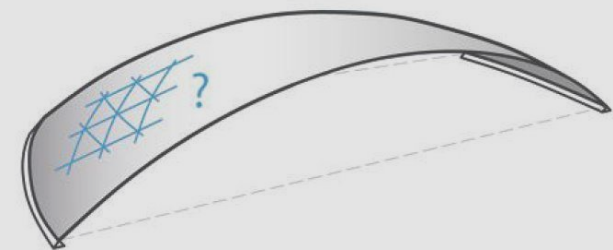
Raster Orientation (degrees)	Actual Width (mm)	Actual Thickness (mm)	Ultimate Stress (MPa)	Ultimate Strain (%)	Flexural Modulus (GPa)
0	12.63	3.34	99.34	6.64%	3.13
0	13.08	3.19	103.77	9.19%	3.17
0	13.08	3.09	100.90	13.29%	3.18
0	12.75	3.15	107.14	12.16%	3.48
0	12.76	3.23	99.87	11.82%	2.98
45	12.83	3.37	92.77	7.17%	2.98
45	12.69	3.36	92.29	7.48%	3.21
45	12.76	3.44	88.23	8.15%	2.93
45	13.10	3.34	89.43	8.81%	3.02
45	12.75	3.39	90.53	7.59%	2.78
90	12.74	3.38	85.77	4.73%	3.01
90	12.72	3.40	85.48	4.73%	2.88
90	12.77	3.33	87.32	4.75%	3.05
90	12.76	3.31	86.52	4.01%	3.00
90	12.67	3.37	85.59	4.30%	3.05

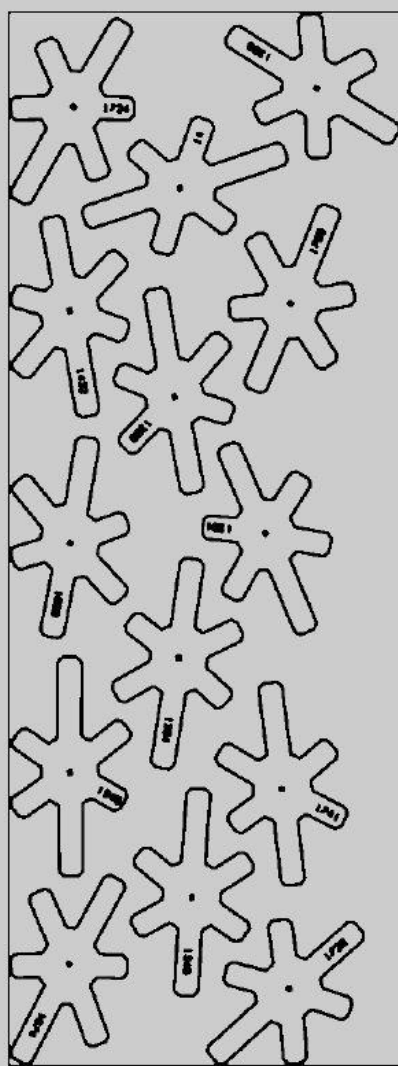
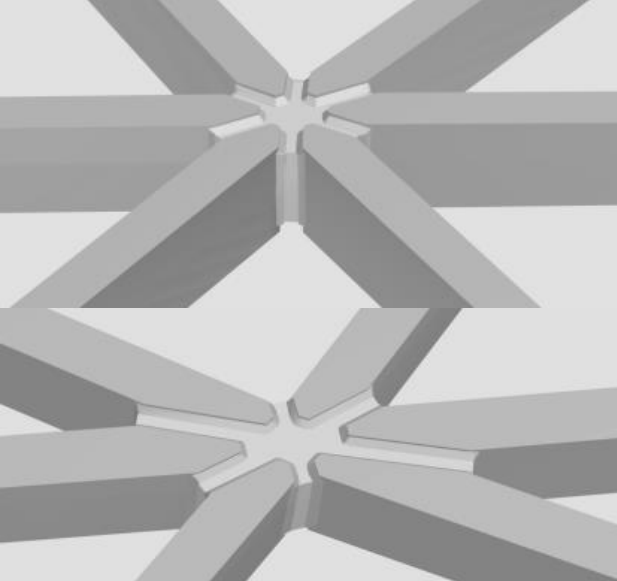
Raster Orientation (degrees)	Ultimate Stress (MPa)	Ultimate Strain (%)	Flexural Modulus of Elasticity (GPa)
0	102.203	0.106	3.187
45	90.649	0.078	2.985
90	86.136	0.045	3.000

TOPOLOGY
OPTIMIZATION

ADDITIVE
MANUFACTURING

**FREEFORM
ENVELOPE**







ROBERT AND ALLEN COOKE COURTYARD

ROBERT AND ALLEN COOKE COURTYARD

Although triangulated surfaces can describe any freeform shape, employed in construction they are **economically less advantageous** than equivalent surface structures built of quadrilateral (four-sided) facets: Quadrangular mesh constructions require fewer machining operations on the glass, and fewer mullions (as they eliminate the diagonal mullion from one side of the triangle). (Schober, 2002)

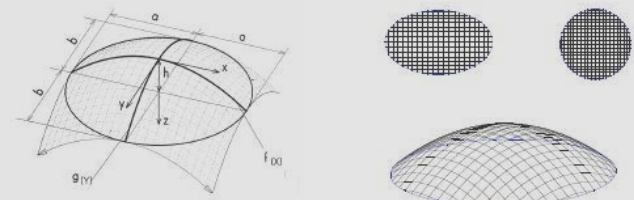


Figure 8. Translational surface covering an elliptical (top left) or circular plan

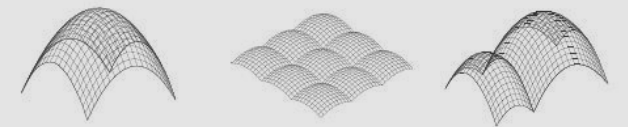


Figure 9. Joining of translational surfaces

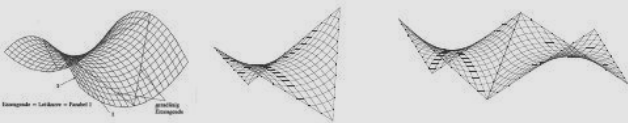


Figure 10. Hyperbolic paraboloid as translational surface (left) and joining possibilities (right)

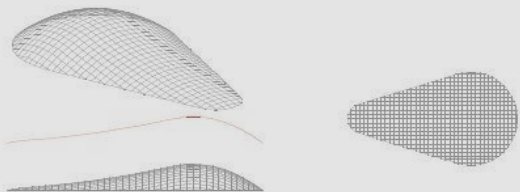
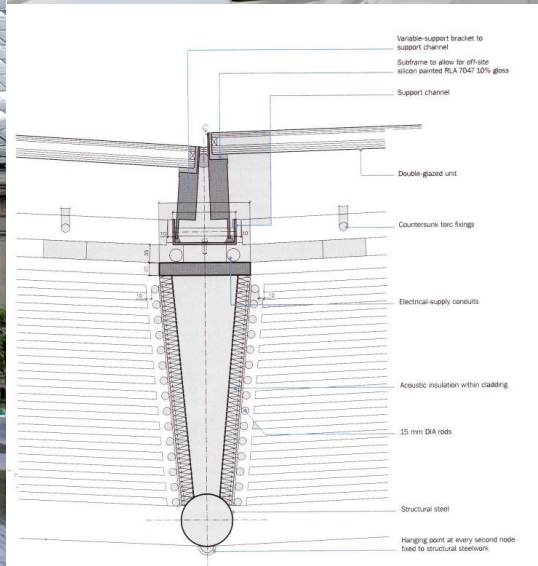
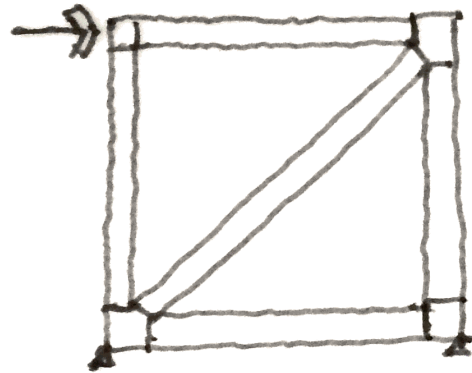


Figure 11. Translation surface covering two semi-circular plans connected with tangential edges.

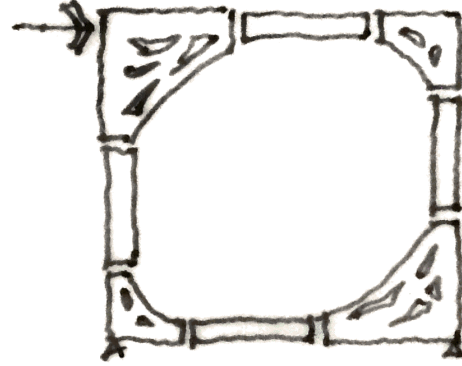




Design ideas

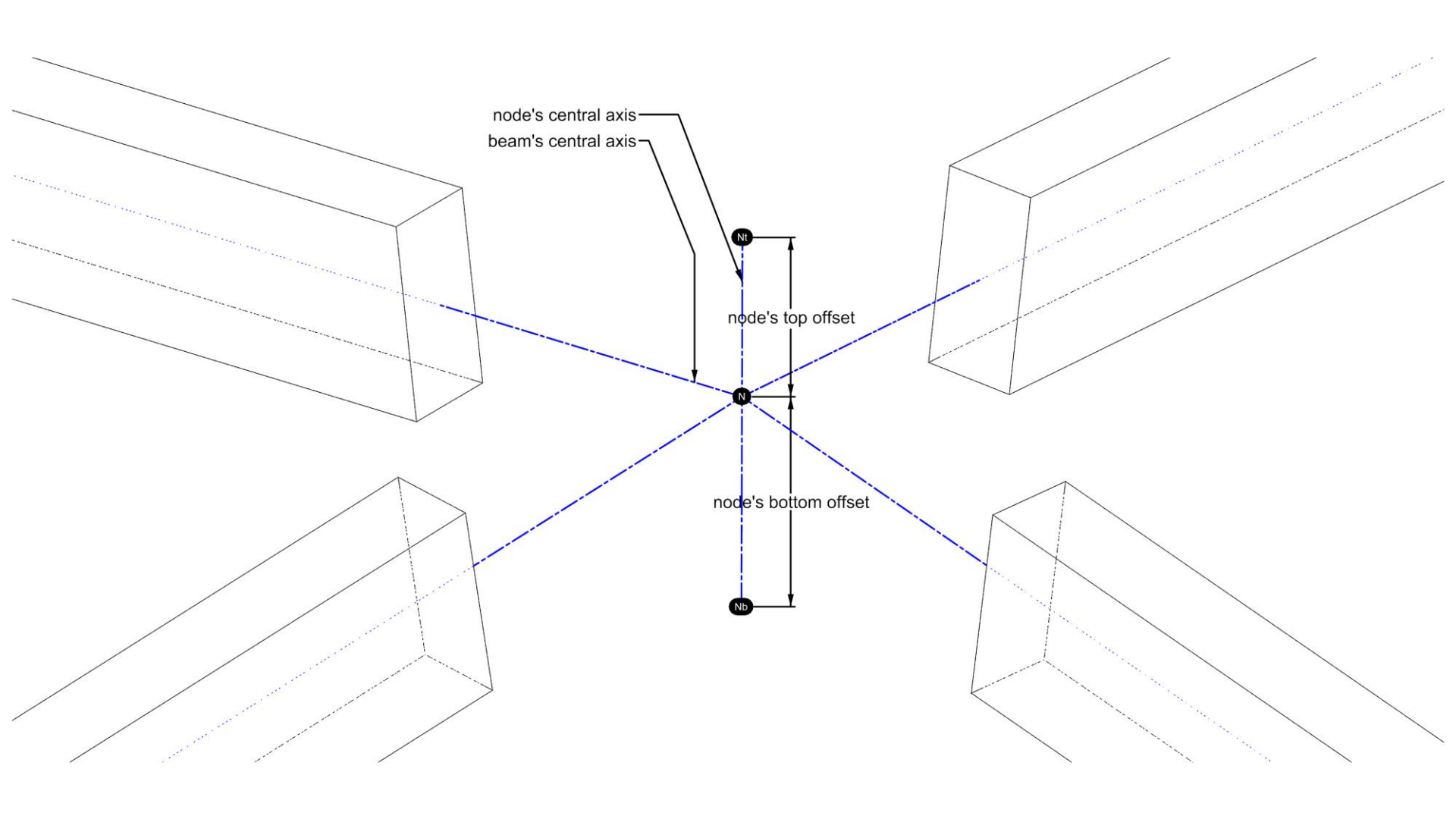


NORMAL



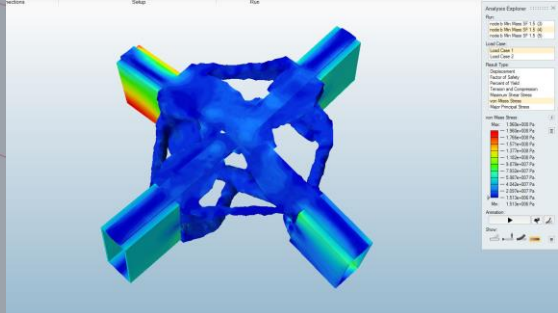
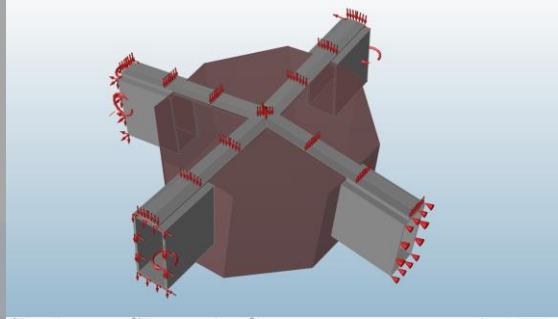
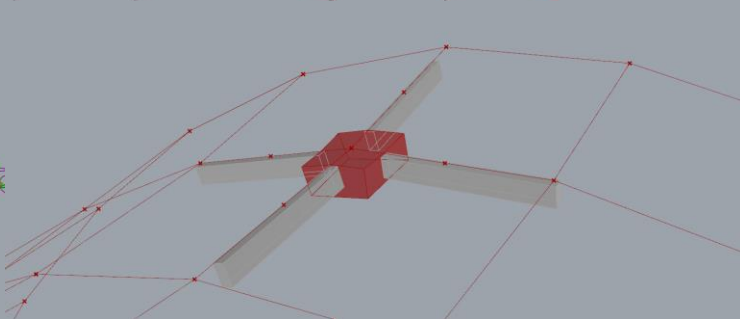
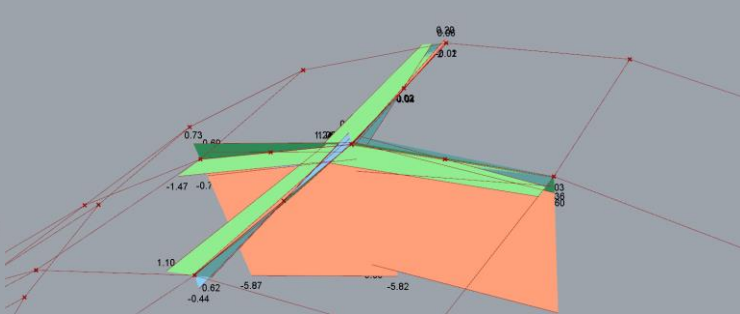
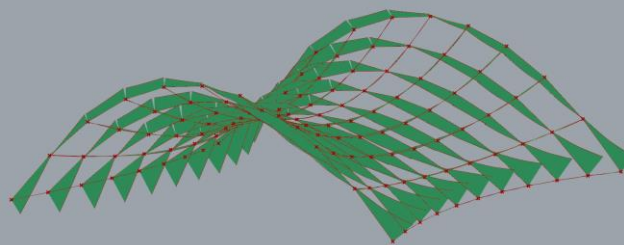
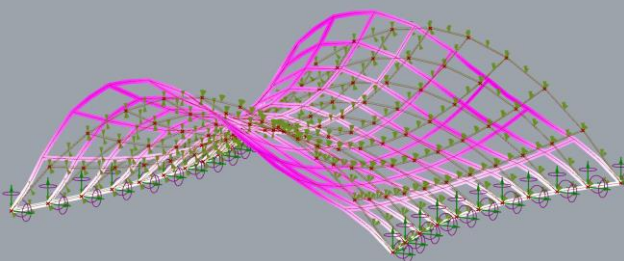
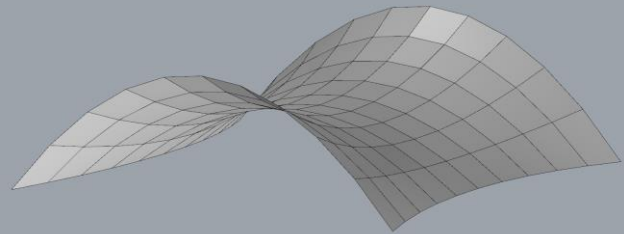
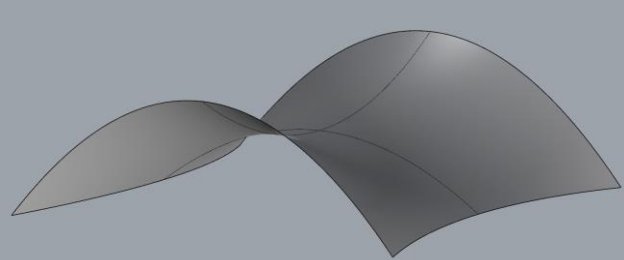
AM

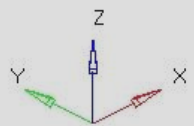
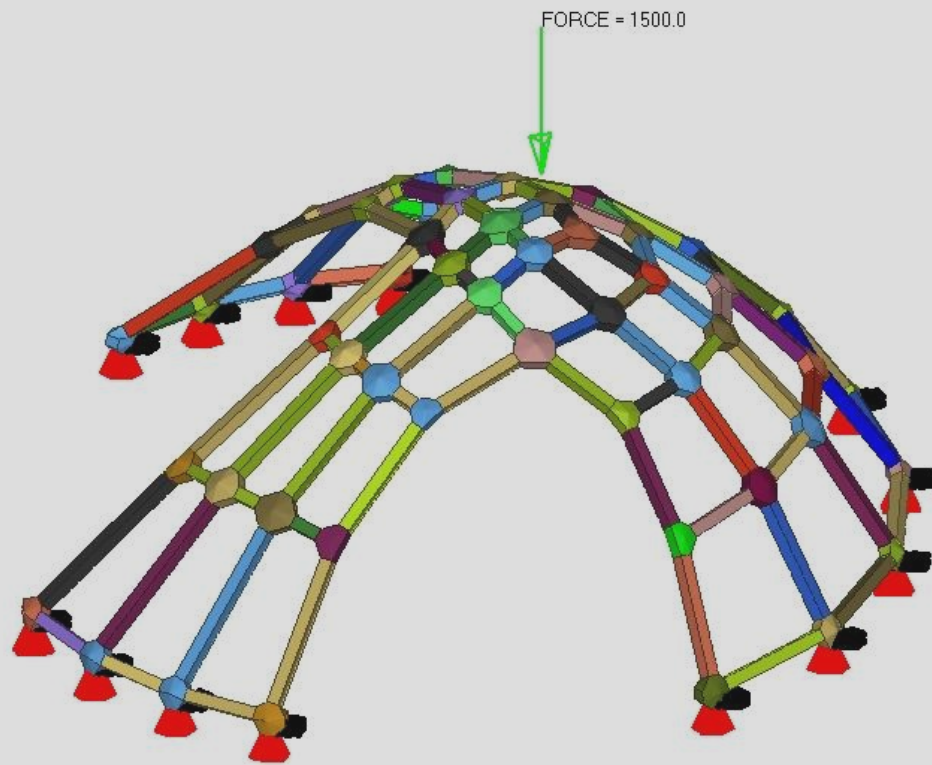


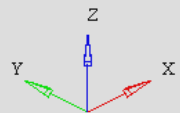
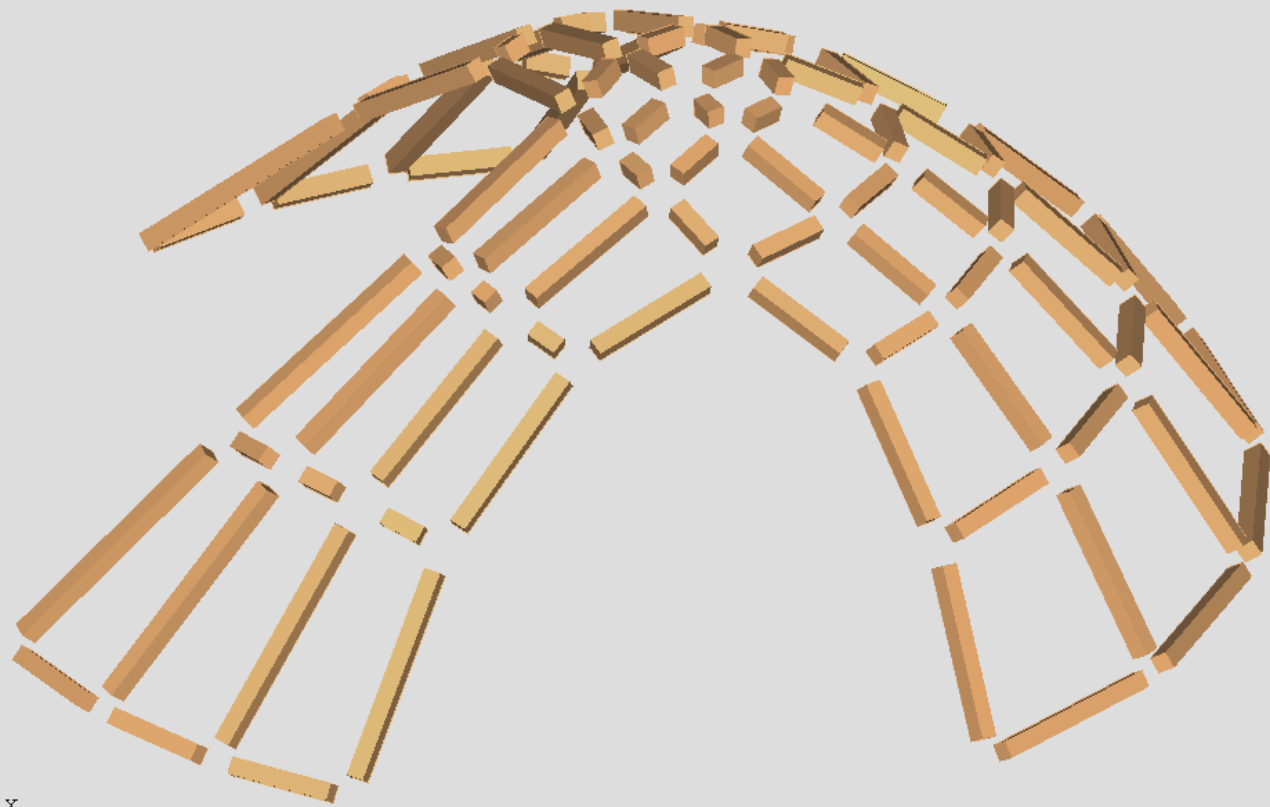




Localized Complexity



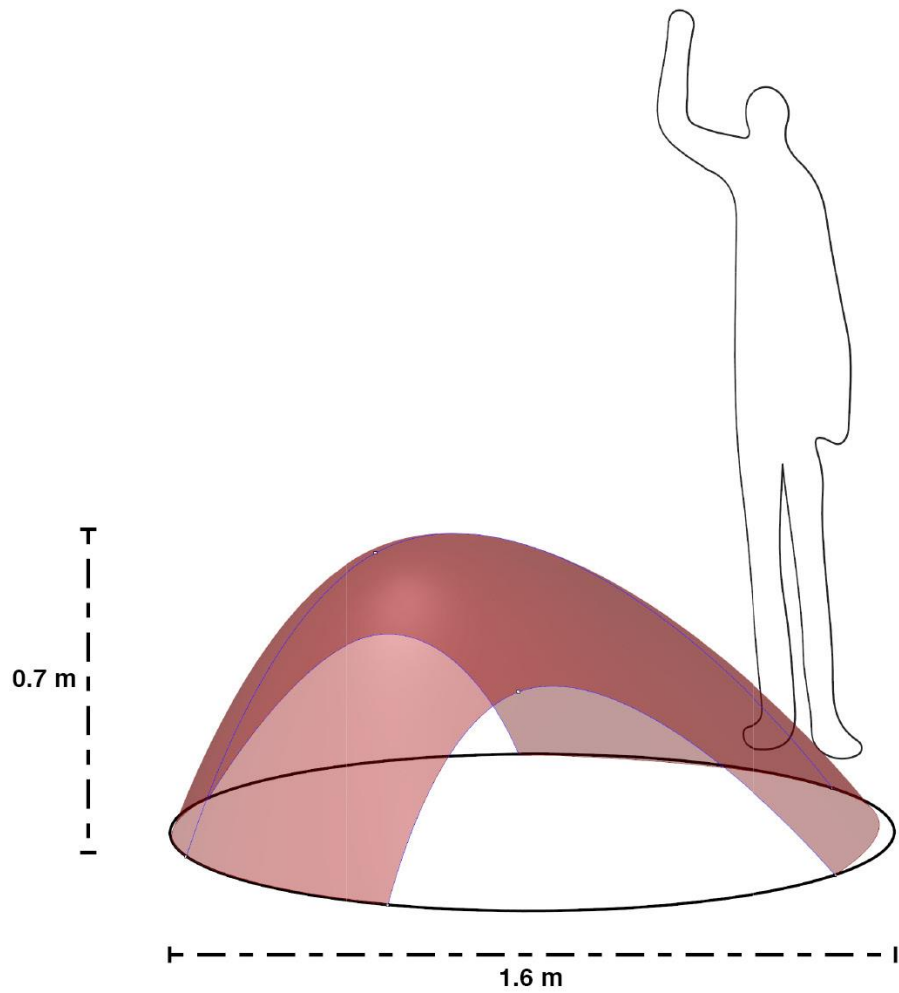


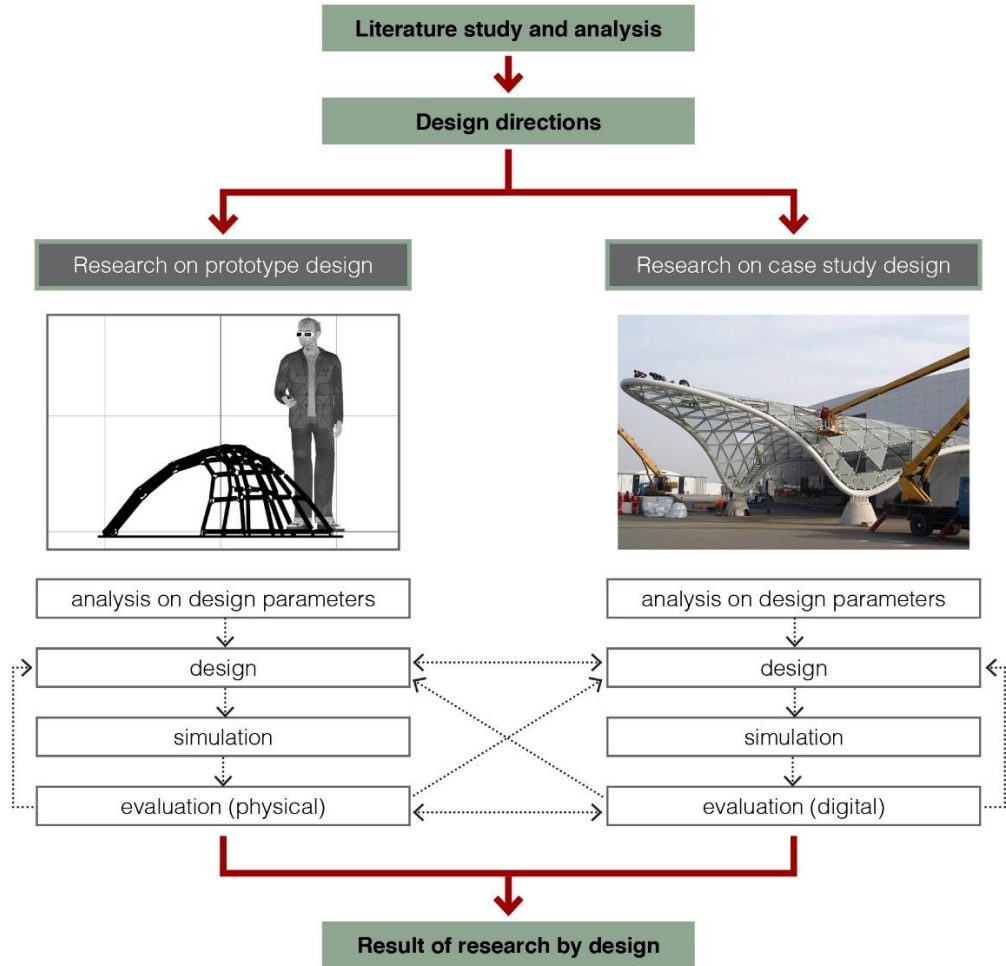


Global Calculation for Local Optimization



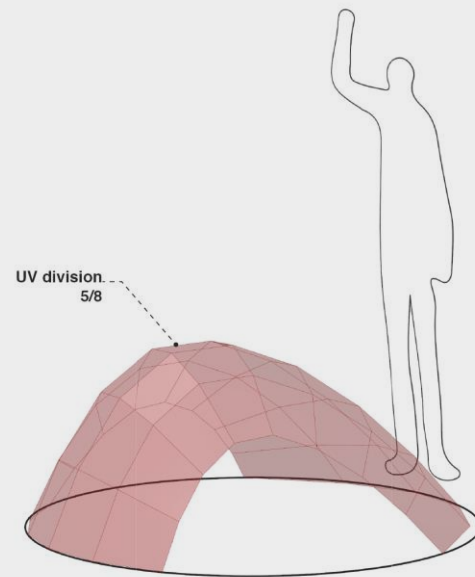
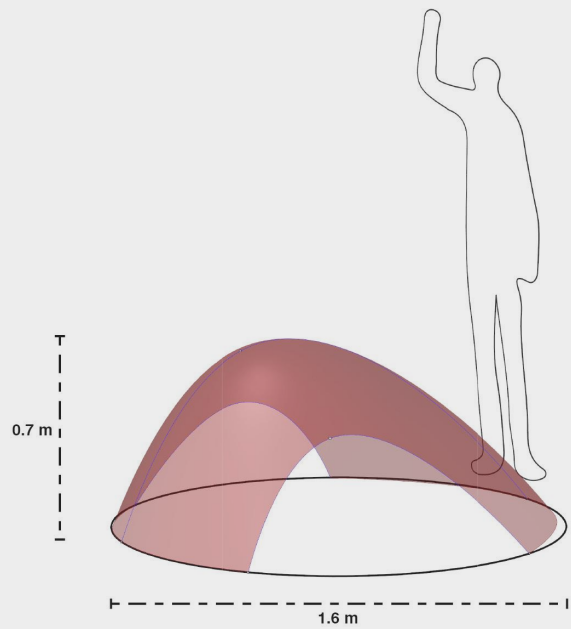


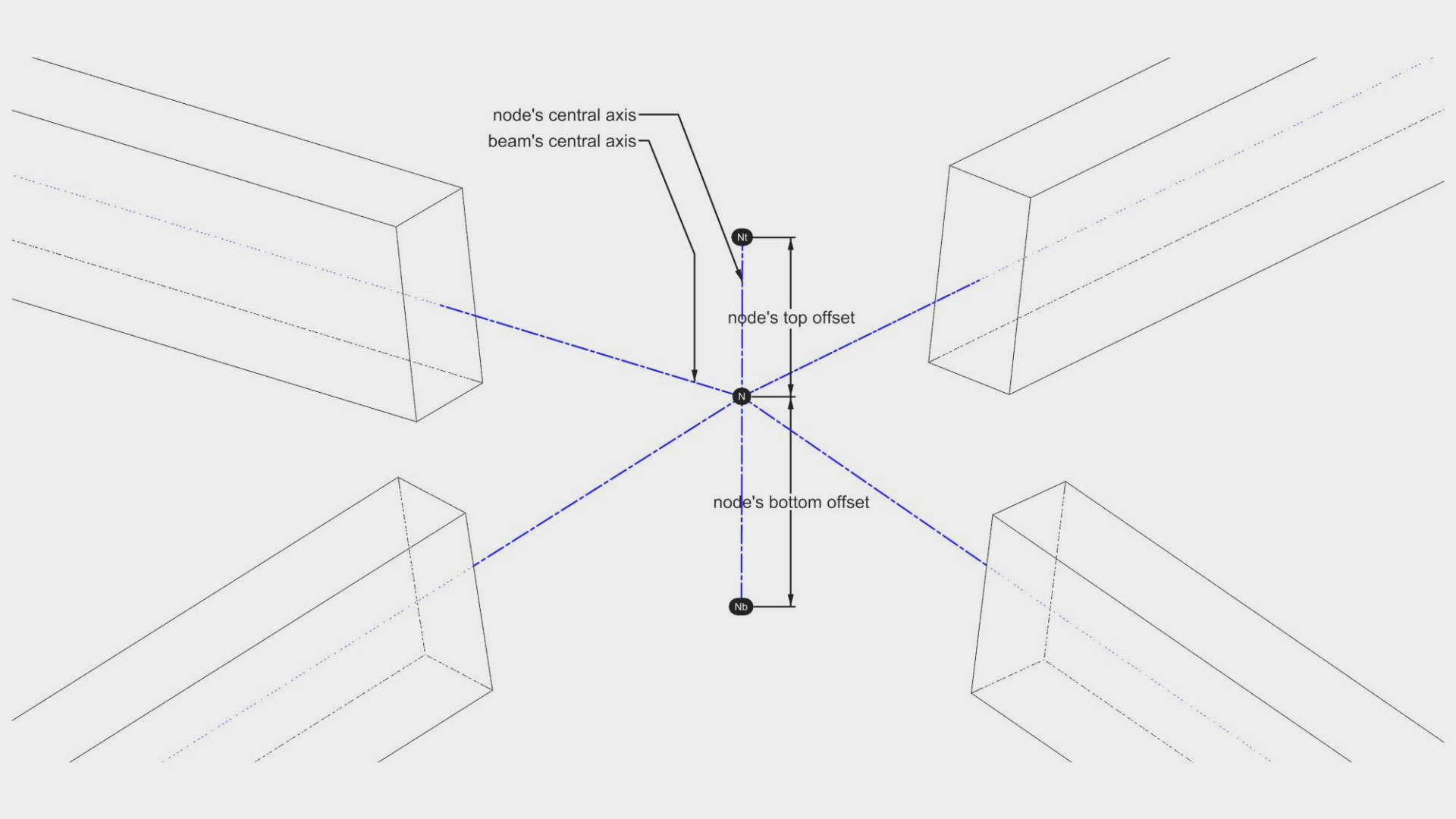


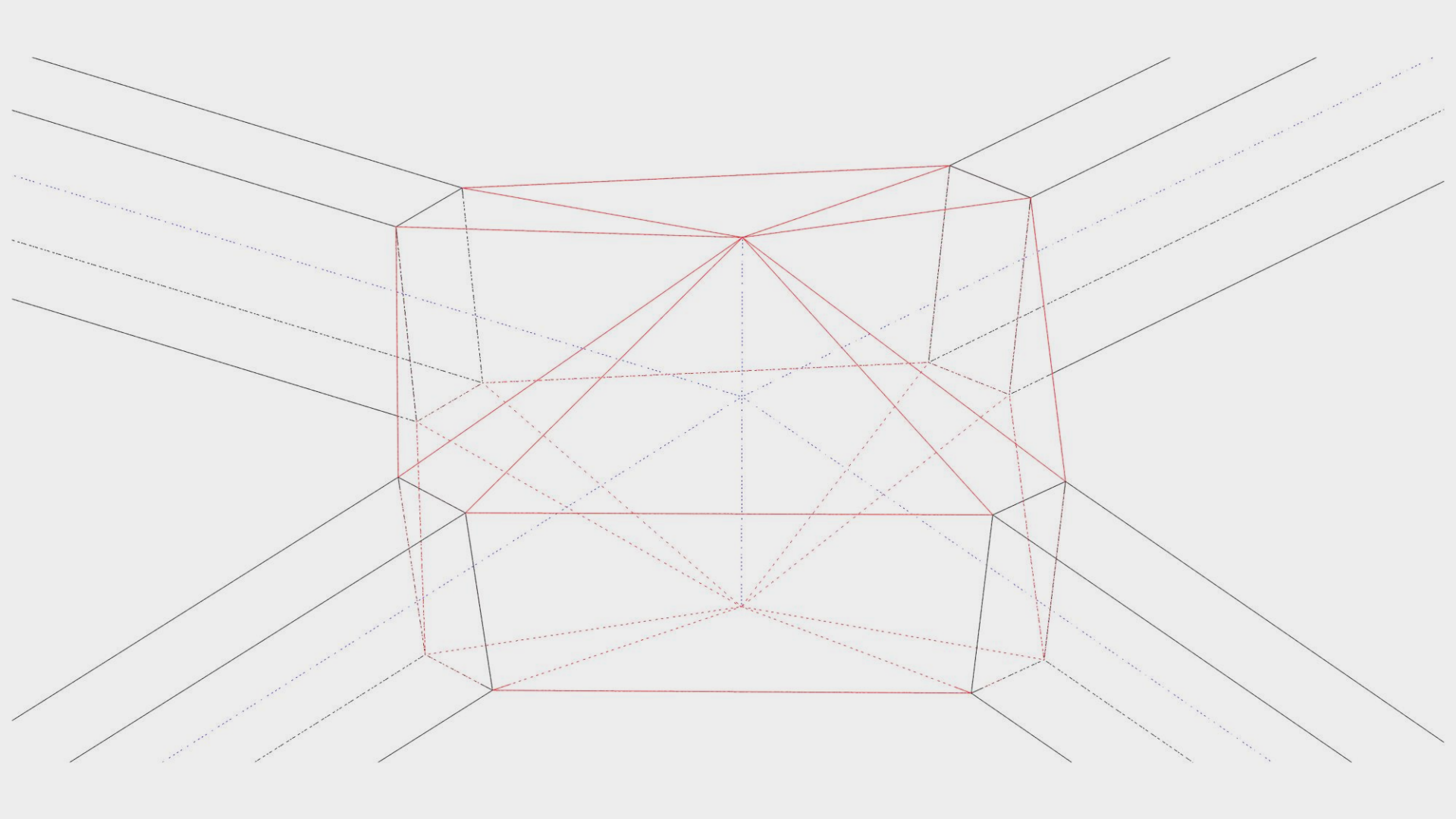


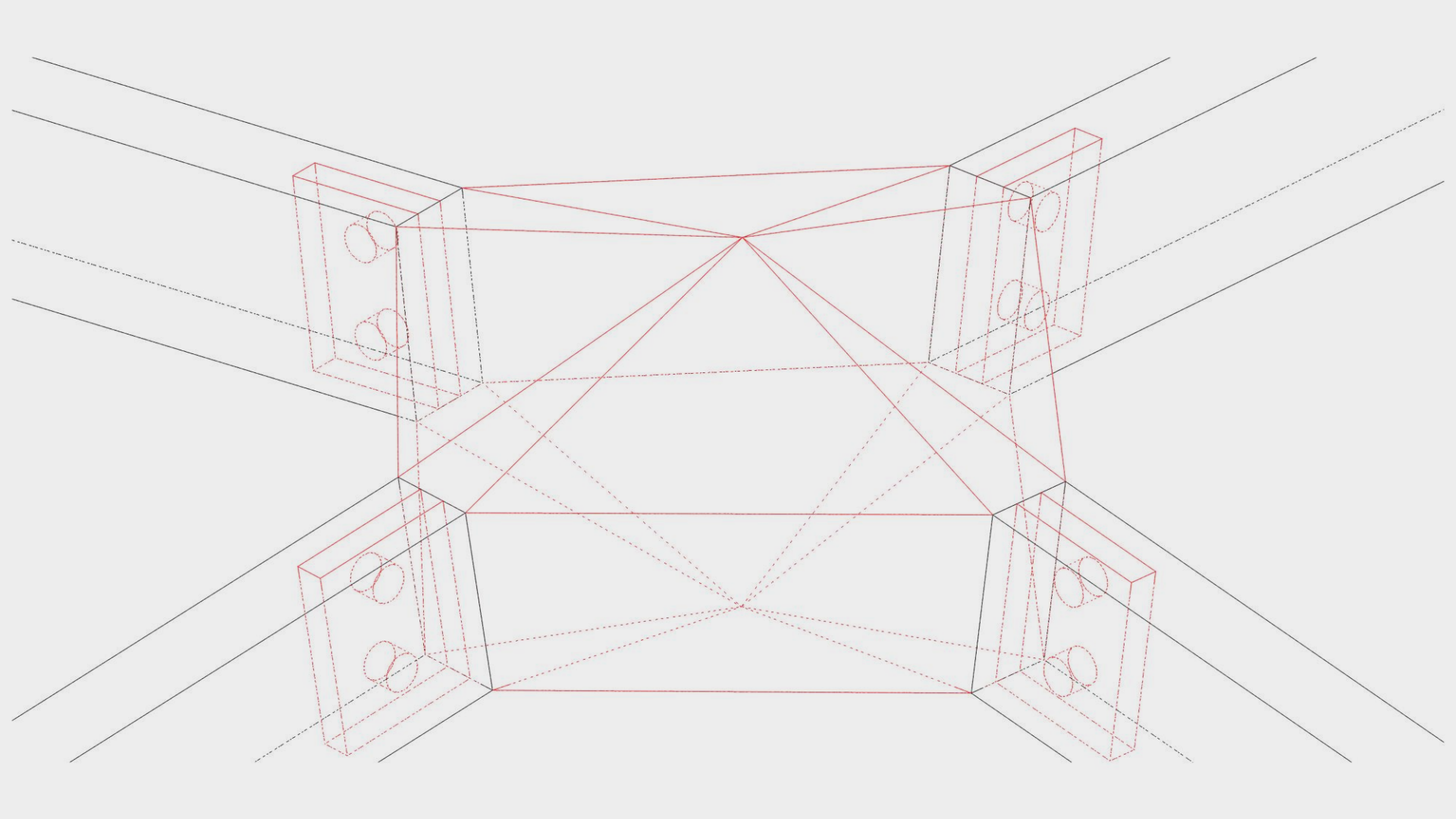
A prototype structure is shown, constructed from four rectangular cardboard beams connected by white, 3D-printed, lattice-like joints. The joints are designed with multiple openings and protrusions, suggesting they are meant to be connected to other parts of a larger assembly. The structure is laid out on a dark, textured surface, possibly a table or floor. The text "Prototype Structure" is overlaid in the center of the image.

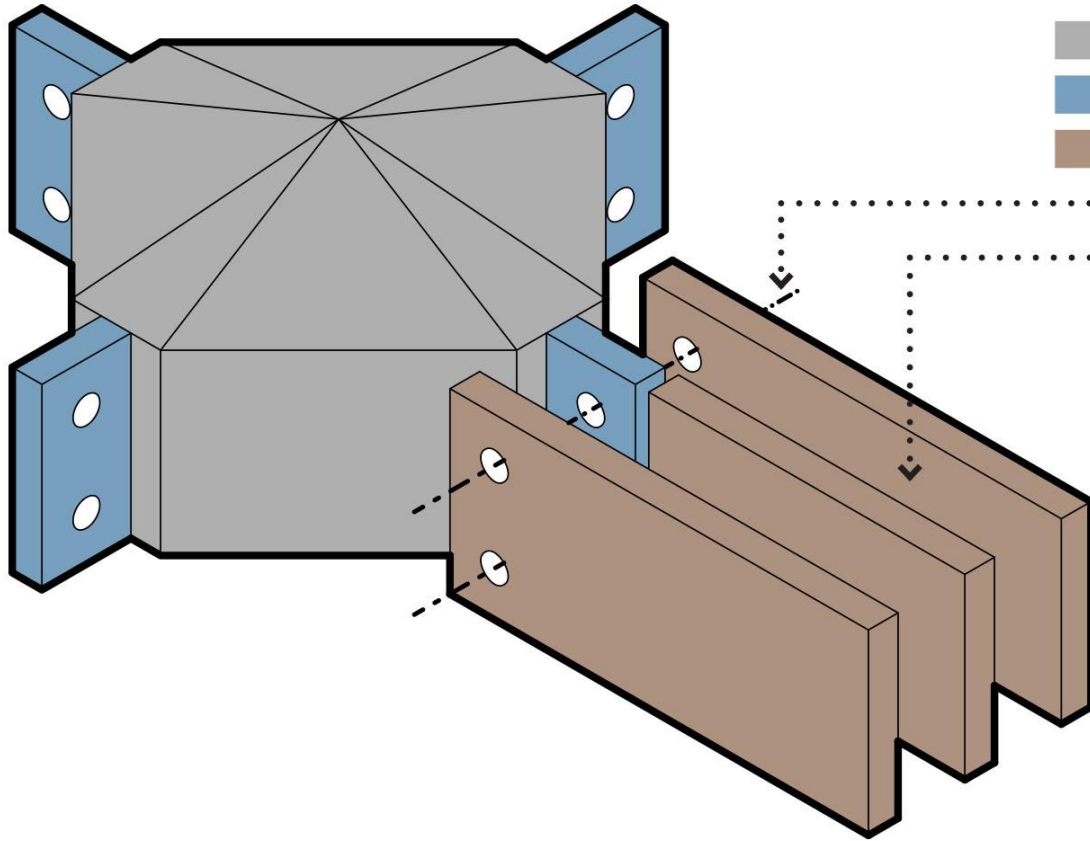
Prototype Structure



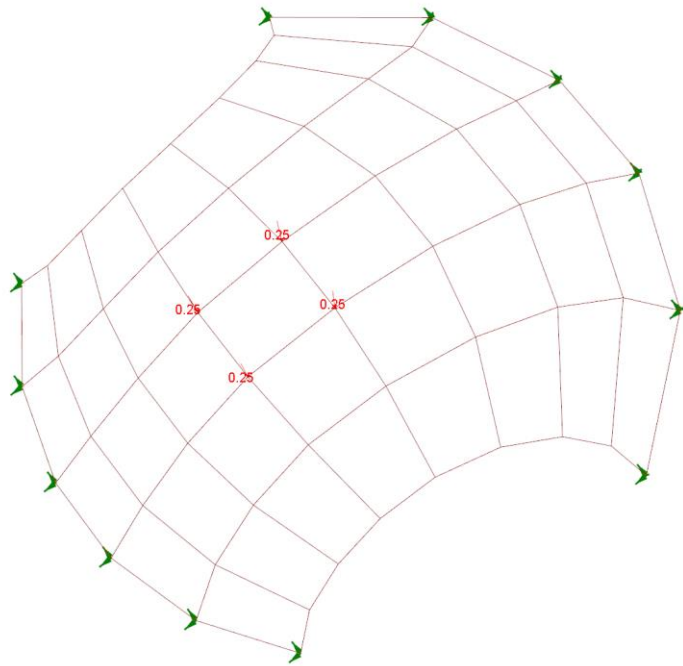


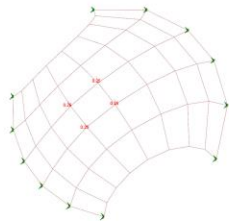




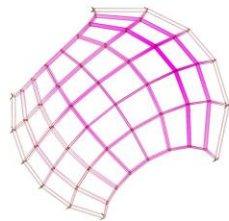


- Node element : TO design space
- Node element : non-design space
- Beam element : solid wood
- Bolt connection
- Three wood planks attached with glue

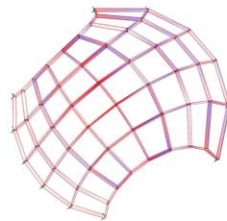




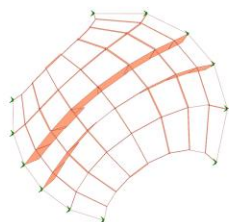
Loadcase and boundary conditions



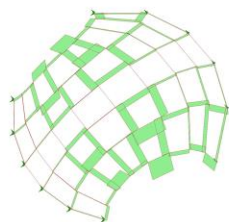
Displacement diagram



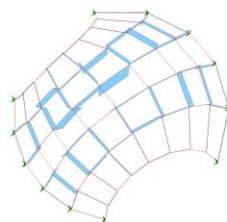
Beams utilization, Red: compression, Blue: tension



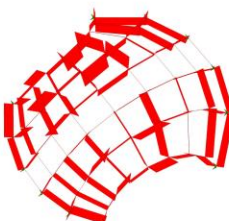
N_x : Normal (axial) forces on beam's local X-direction



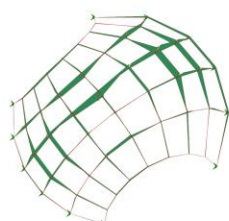
V_y : Shear forces on beam's local Y-direction (in plane shear)



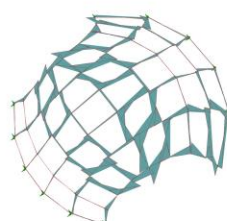
V_z : Shear forces on beam's local Z-direction (out of plane shear)



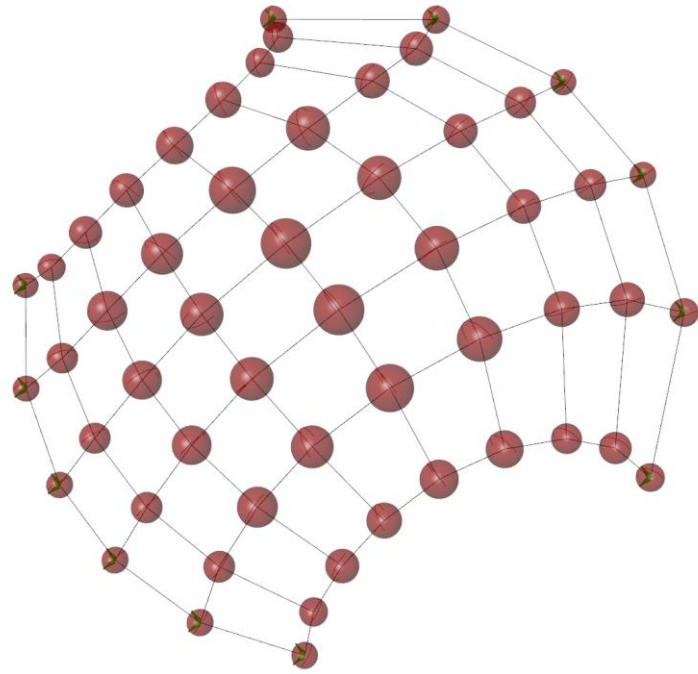
M_x : Torsional moment on beam's local X-axis

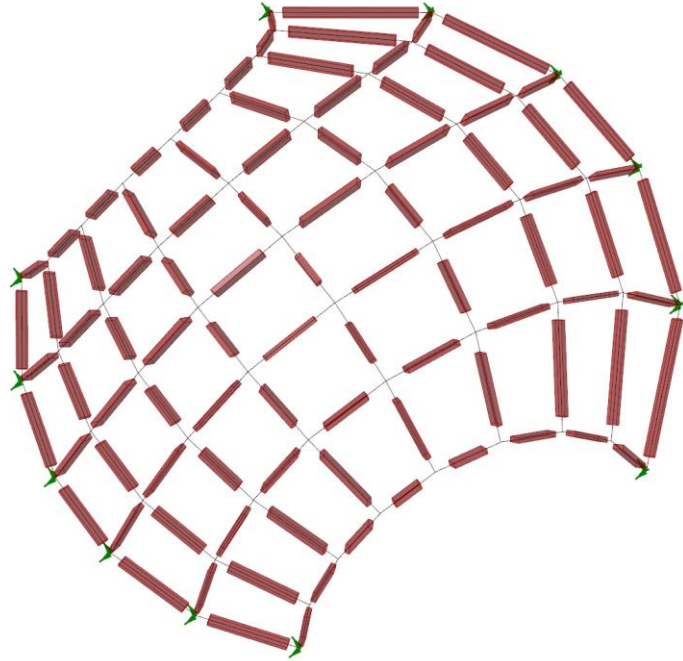


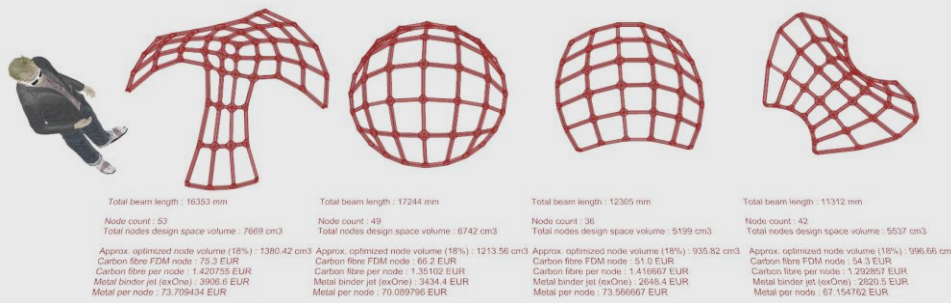
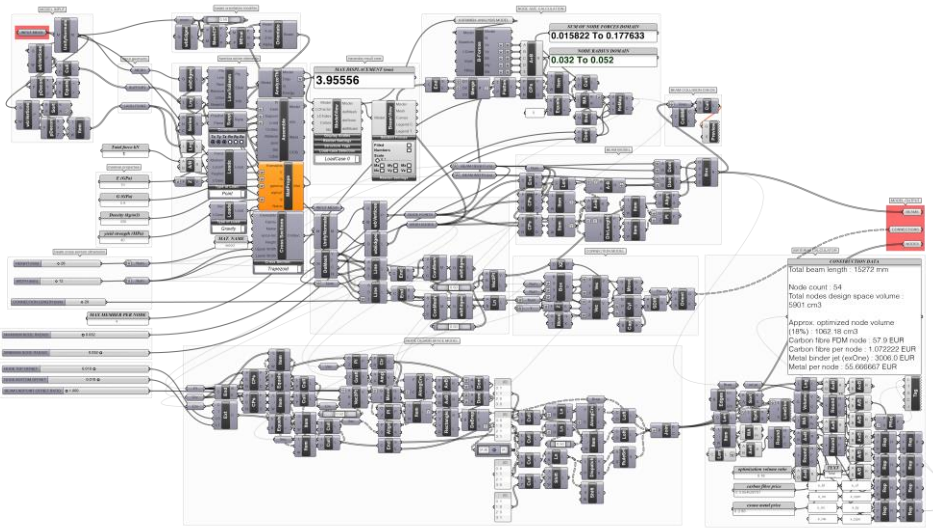
M_y : Bending moment on beam's local Y-axis (out of plane bending)

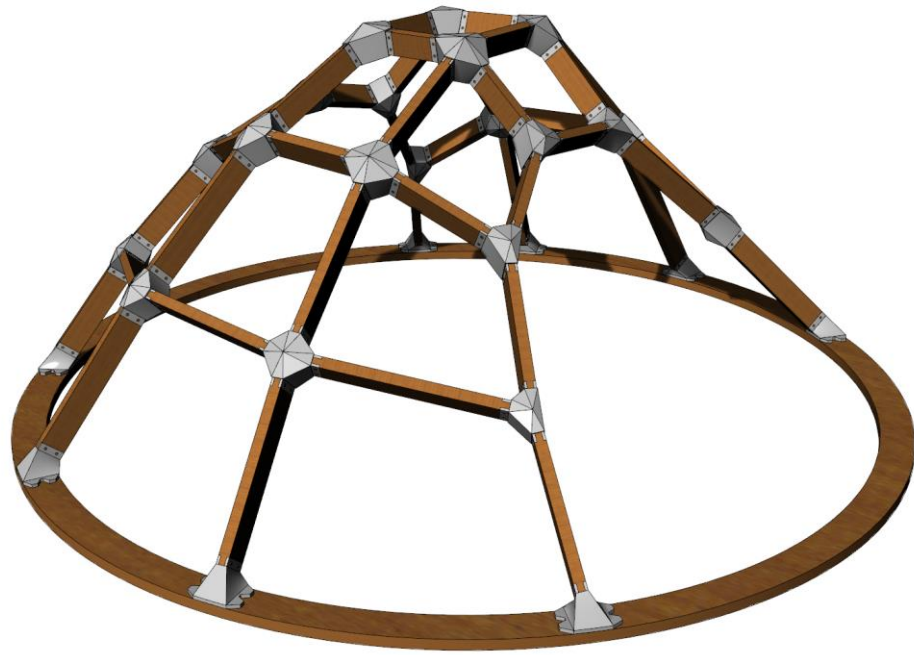


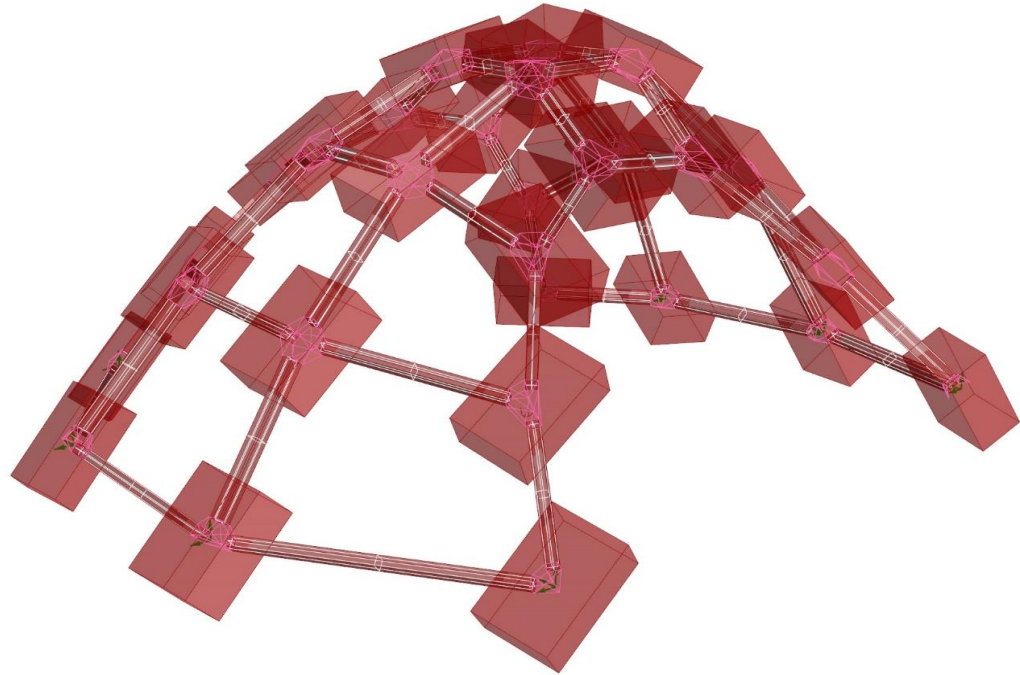
M_z : Bending moment on beam's local Z-axis (in plane bending)

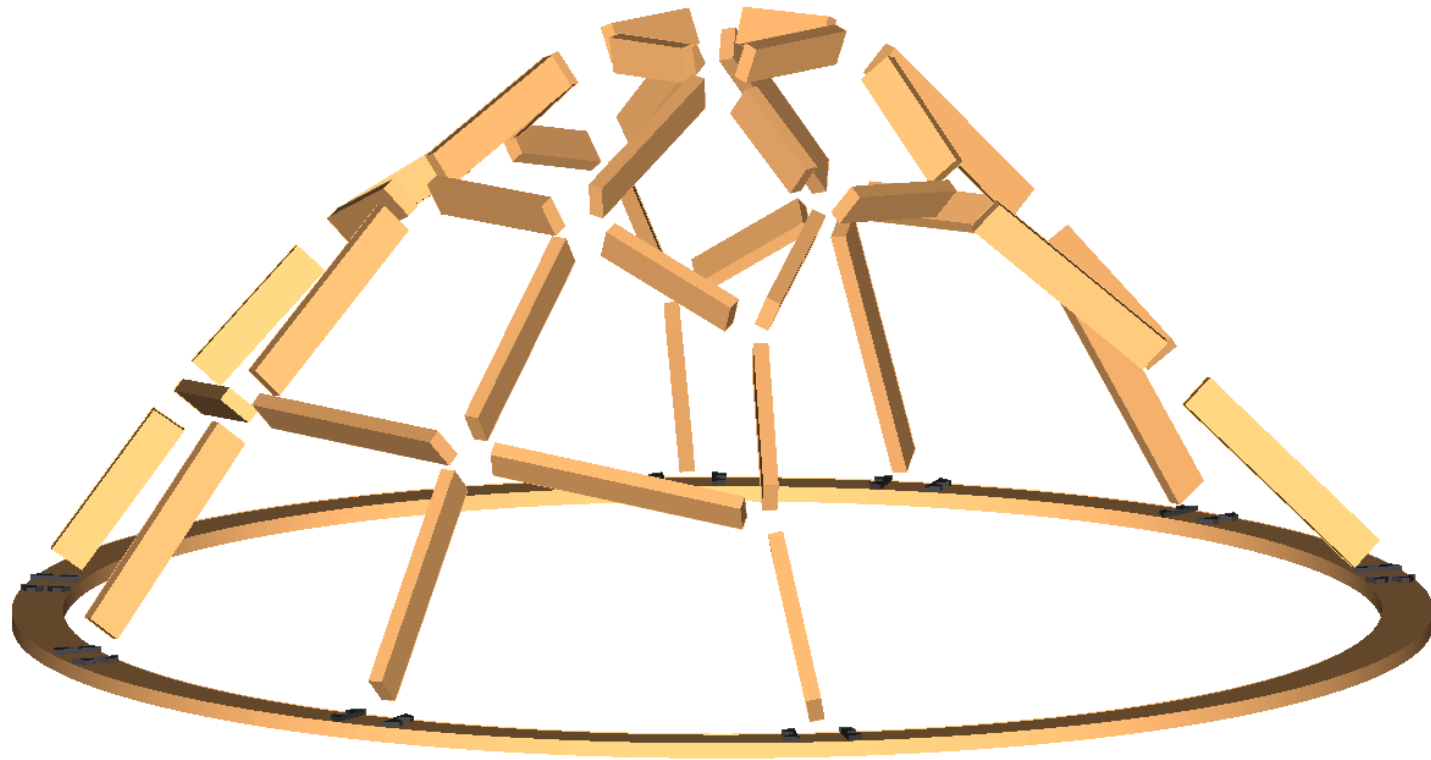




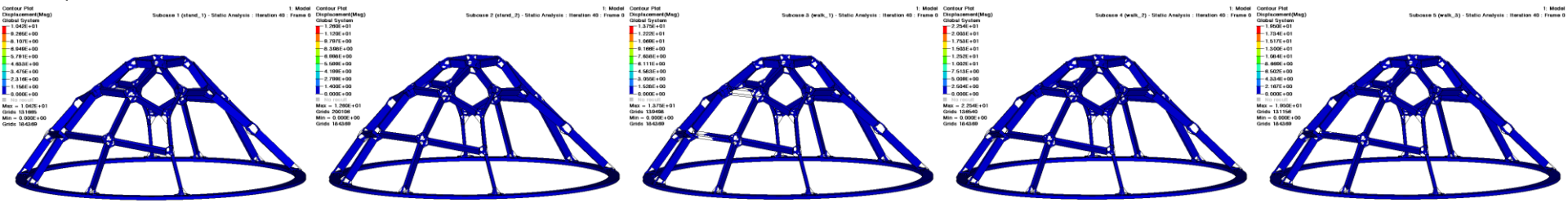




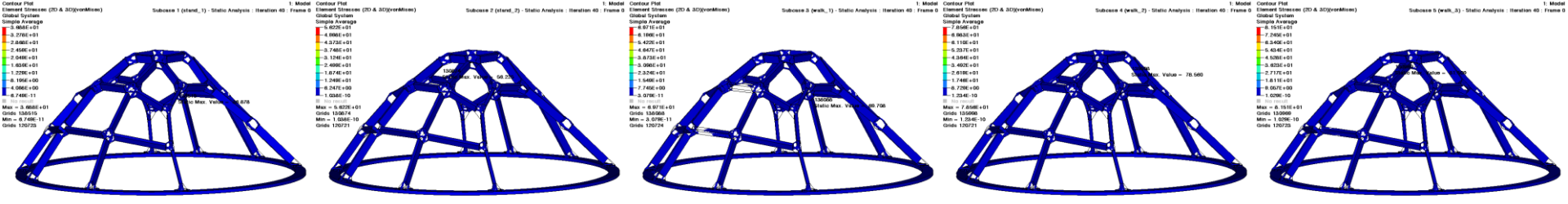


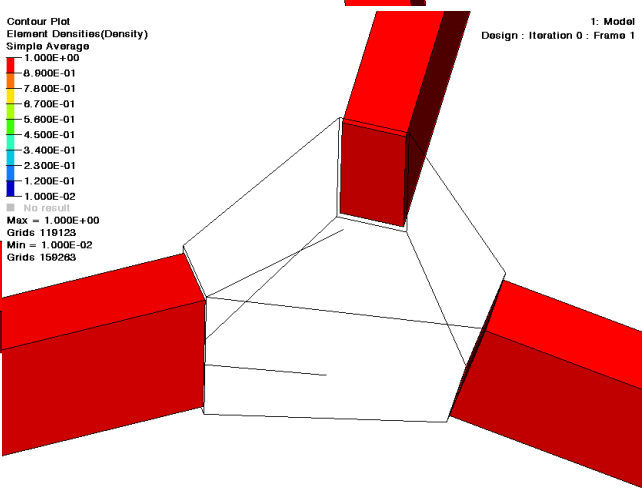
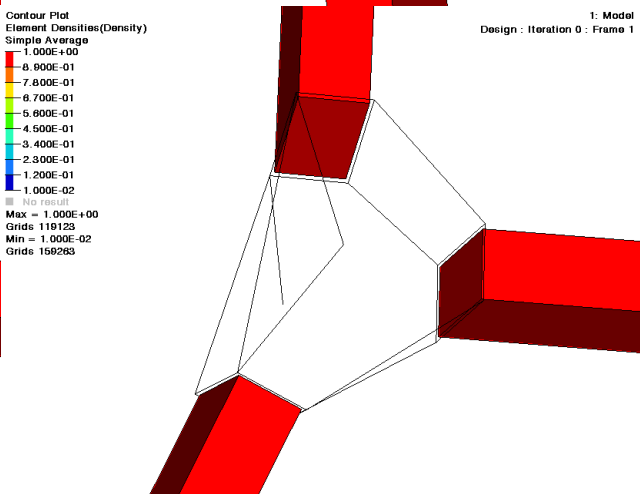
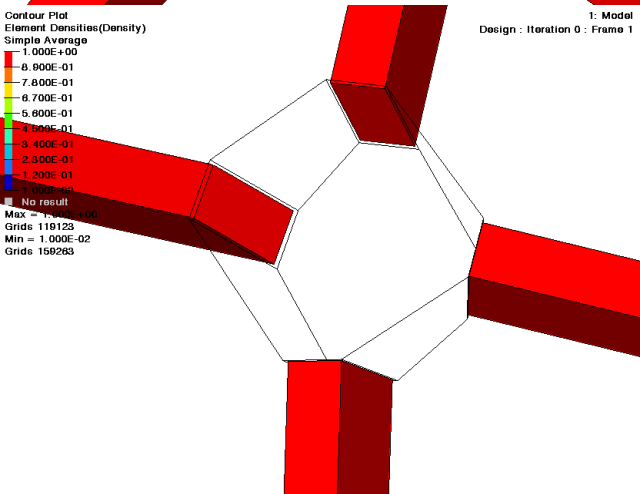
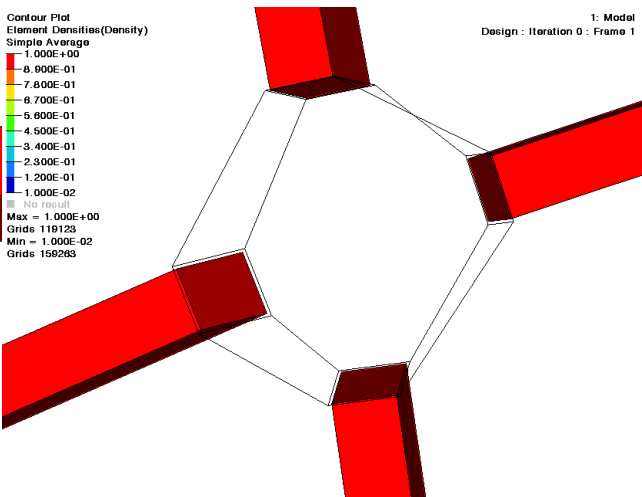
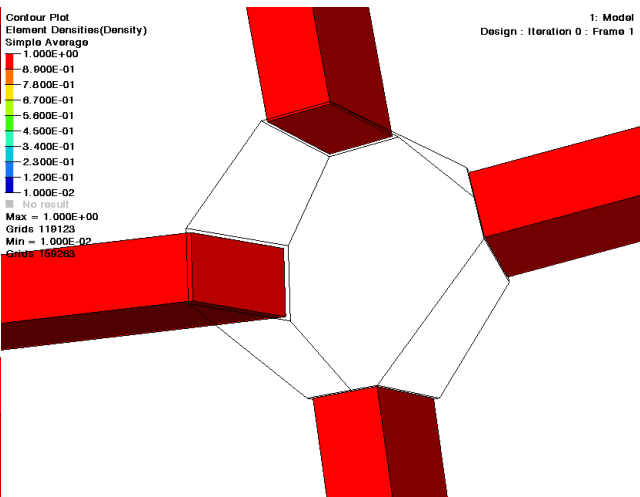
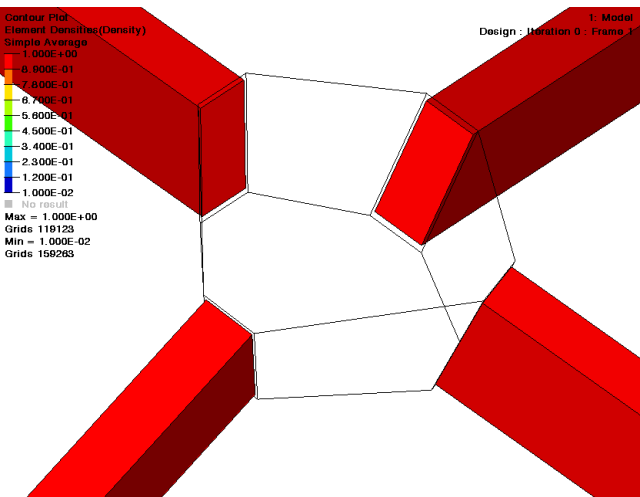


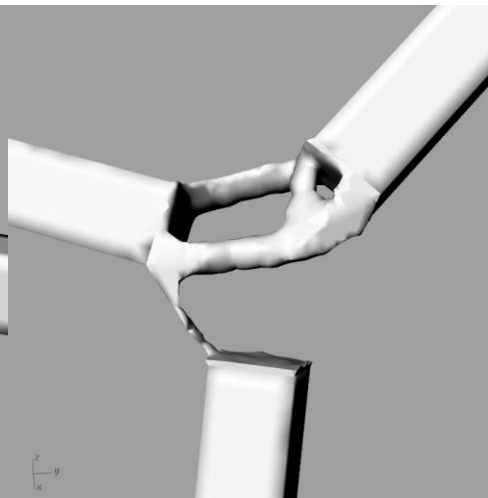
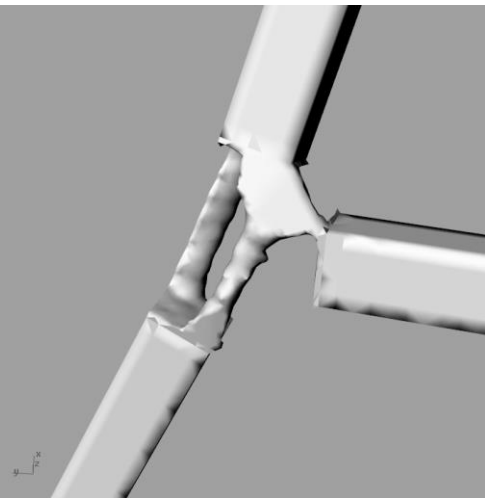
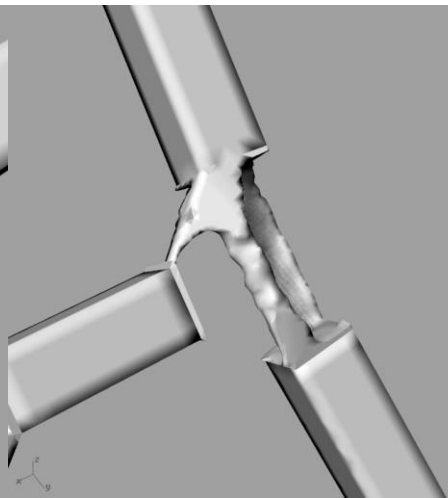
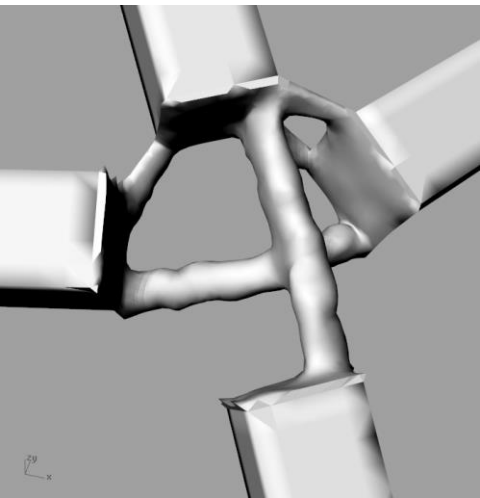
Displacement :



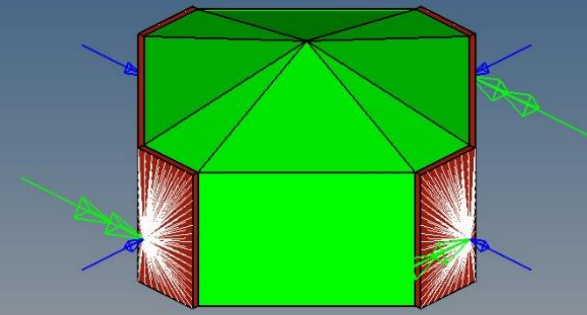
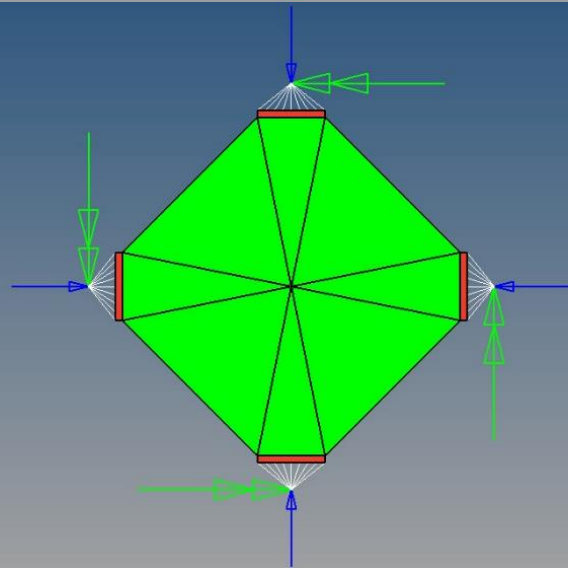
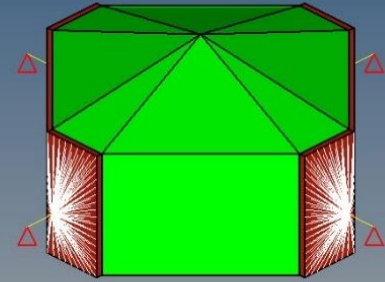
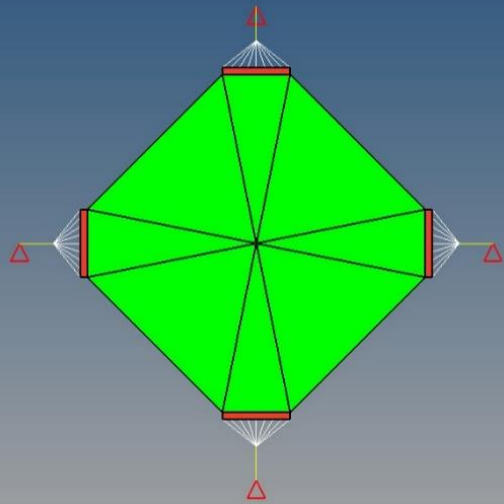
Von mises stresses



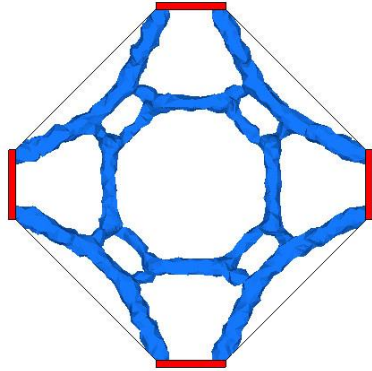




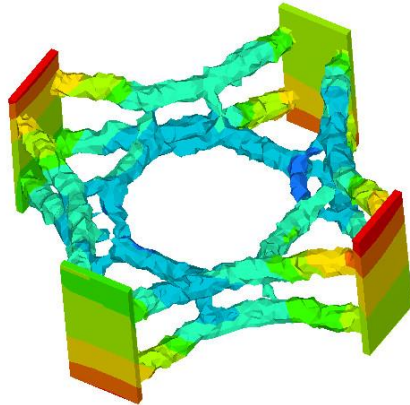
Generic design optimization



Contour Plot
Element Densities(Density)
Simple Average
1.000E+00
8.900E-01
7.800E-01
6.700E-01
5.600E-01
4.500E-01
3.400E-01
2.300E-01
1.200E-01
1.000E-02
No result
Max = 1.000E+00
Grids 9
Min = 1.000E-02
Grids 5824

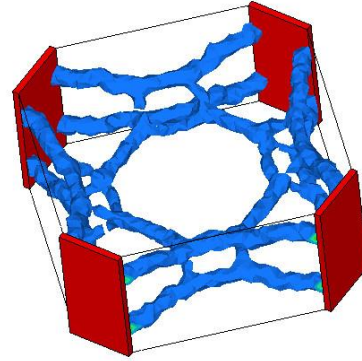


Contour Plot
Displacement(Meg)
Analysis system
6.212E+00
5.522E+00
4.832E+00
4.142E+00
3.451E+00
2.761E+00
2.071E+00
1.381E+00
6.903E-01
0.000E+00
No result
Max = 6.212E+00
Grids 926
Min = 0.000E+00
Grids 1

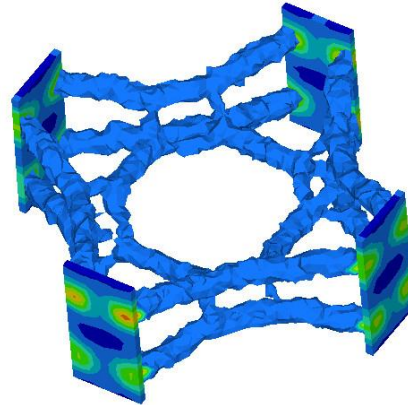


Subcase 1 (compress) - Static Analysis : Iteration 73 : Frame 25

1: Model
Design : Iteration 73 : Frame 25
Contour Plot
Element Densities(Density)
Simple Average
1.000E+00
8.900E-01
7.800E-01
6.700E-01
5.600E-01
4.500E-01
3.400E-01
2.300E-01
1.200E-01
1.000E-02
No result
Max = 1.000E+00
Grids 9
Min = 1.000E-02
Grids 5824



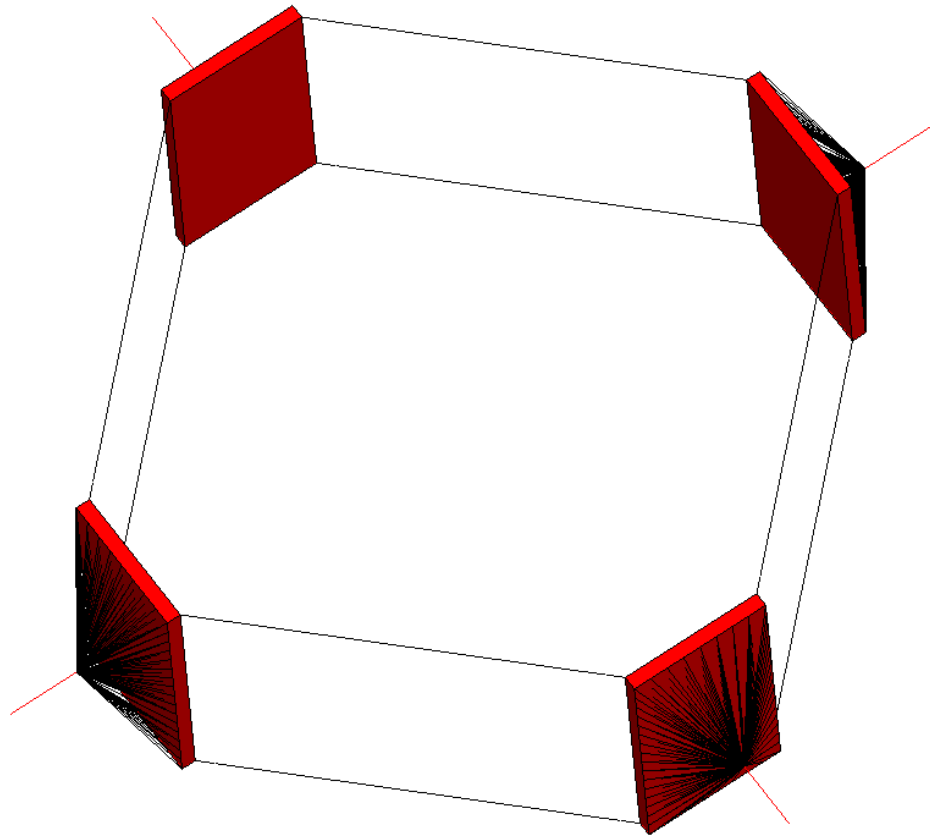
1: 1
Contour Plot
Element Stresses (2D & 3D)(vonMises)
Analysis system
Simple Average
1.636E+01
1.454E+01
1.272E+01
1.091E+01
9.089E+00
7.271E+00
5.453E+00
3.636E+00
1.818E+00
7.307E-08
No result
Max = 1.636E+01
Grids 377
Min = 7.307E-08
Grids 4722

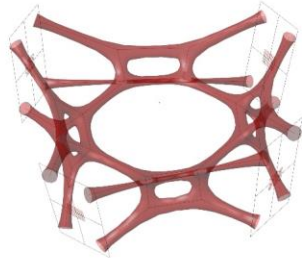
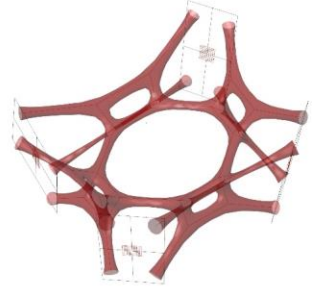
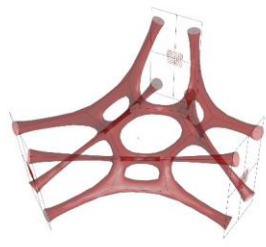
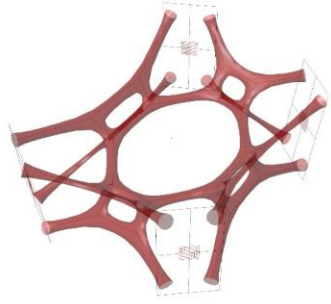


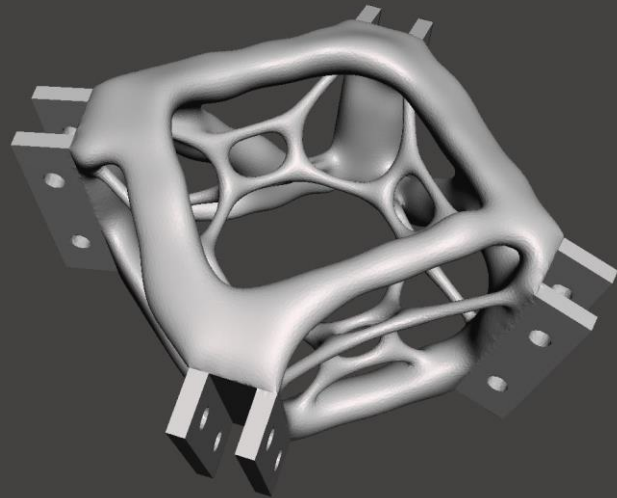
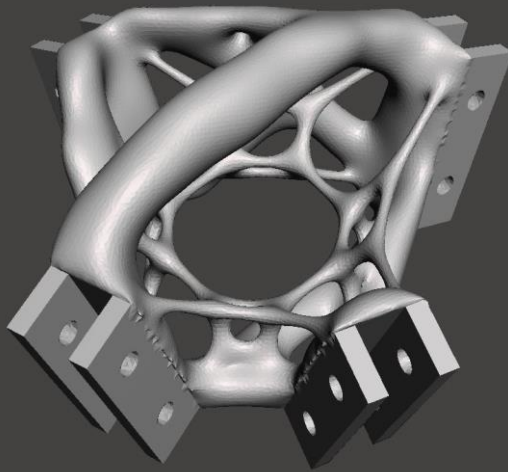
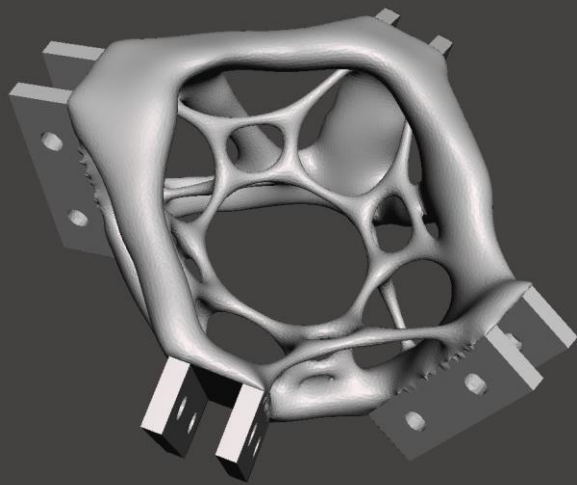
Subcase 1 (compress) - Static Analysis : Iteration 73 : Frame 25

1: Model
Design : Iteration 73 : Frame 25

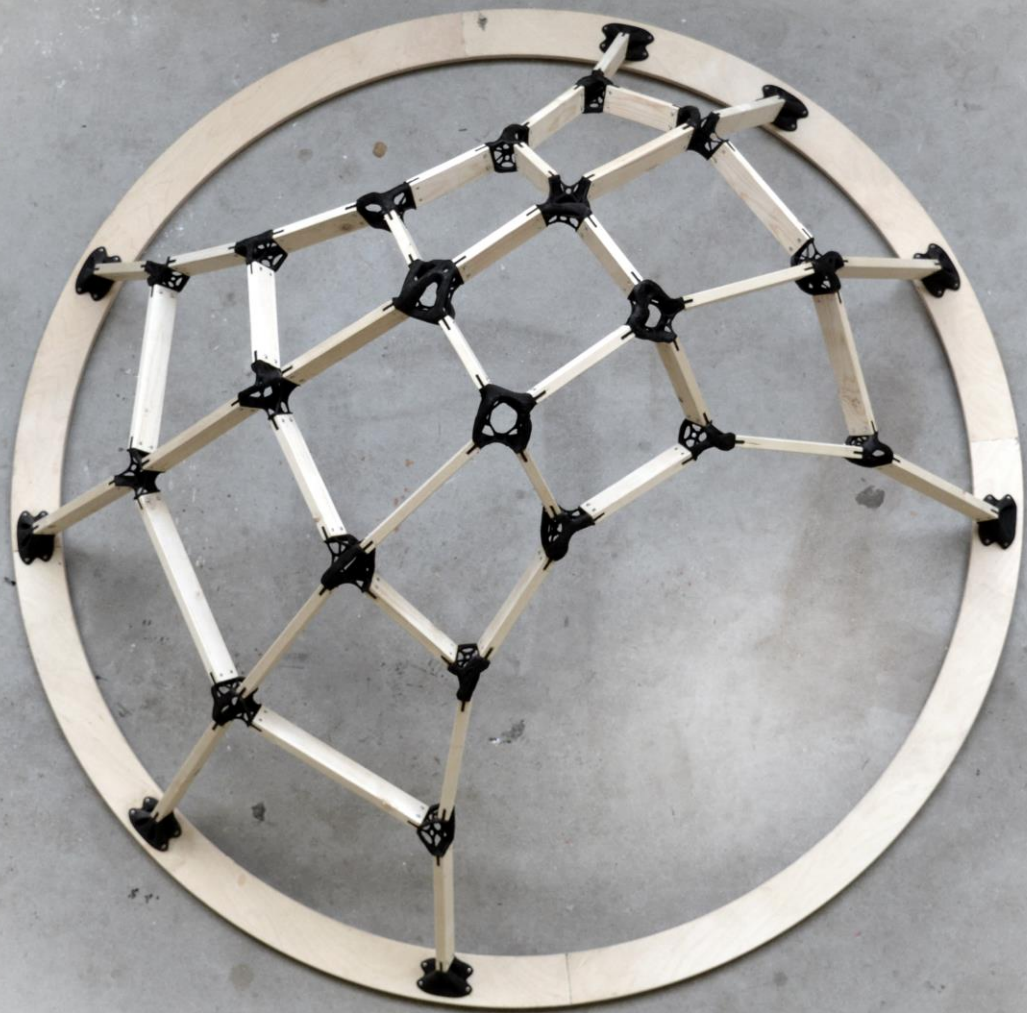
1: 1







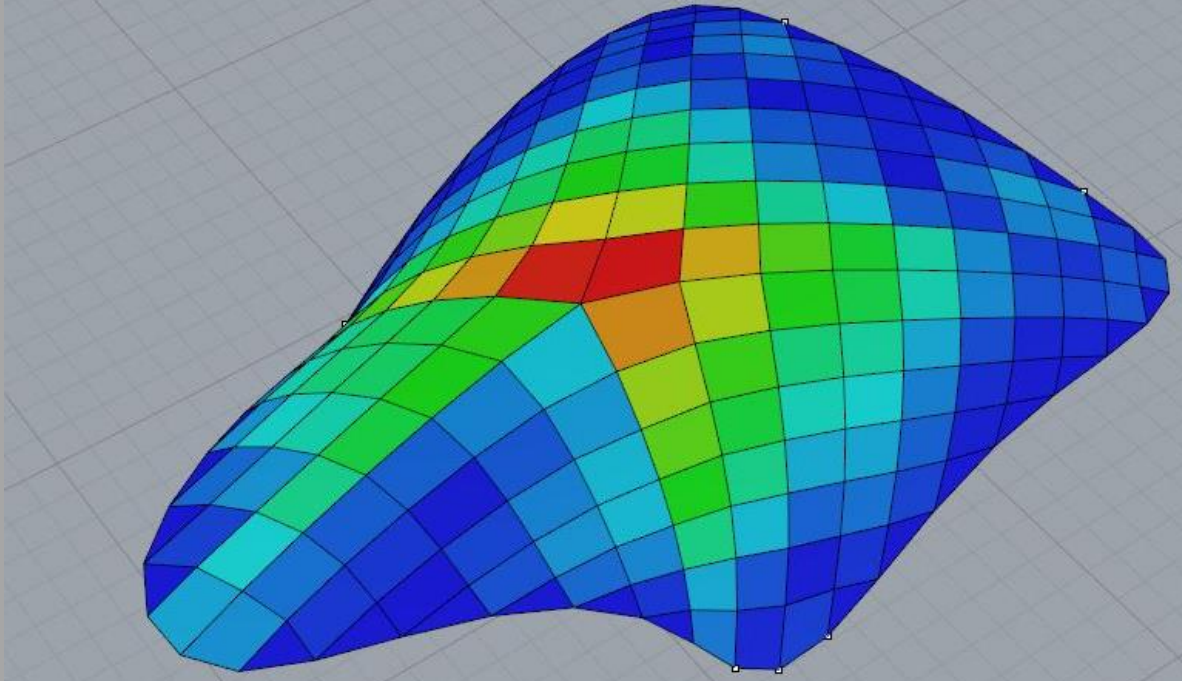




Case Study







Mode

Planarity

Range

49.042787108

0.0

0 - Max Range

Min - Max Range

Show range in viewport

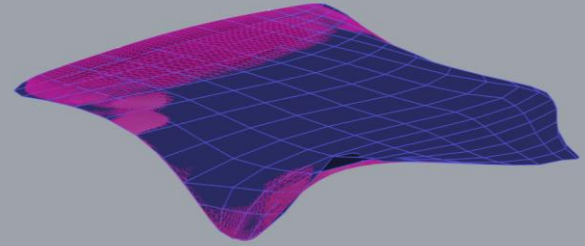
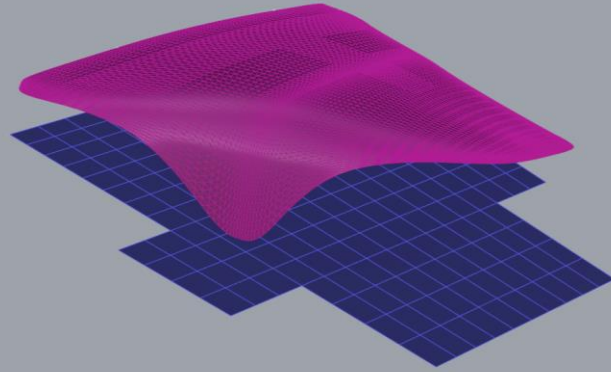
Unit

millimeters

Meshes

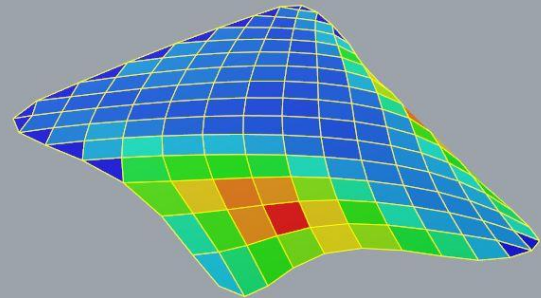
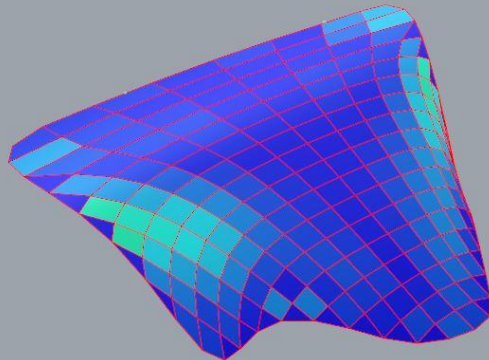
Add Object

Remove Object



Forces :

- Closeness to reference surface
- Closeness to boundary curve
- Planarize panels
- Equalize edges length
- Fairness curvature



Mesh Analysis

Mode: _____

Planarity

Range

179.587073581

0.0

0 - Max Range

Min - Max Range

Show range in viewport

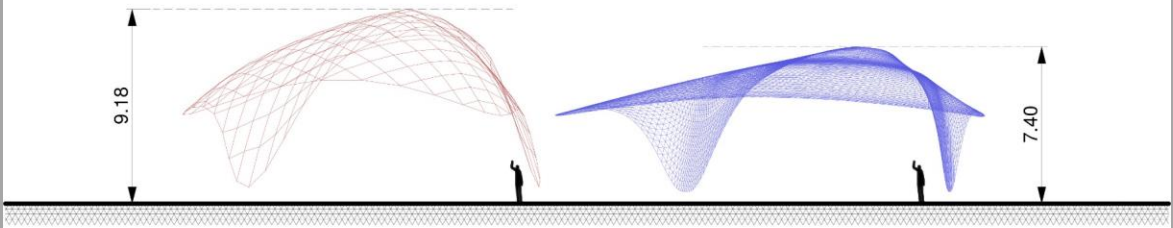
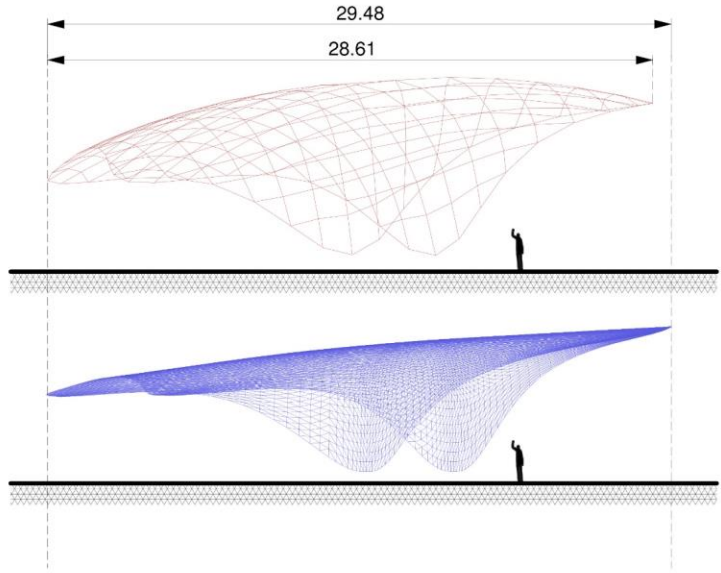
Unit

millimeters

Meshes

Add Object

Remove Object

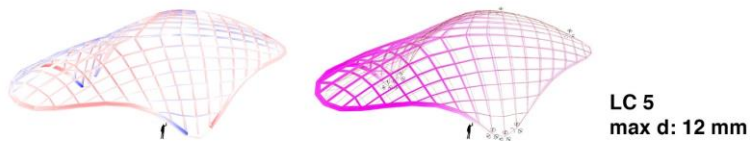
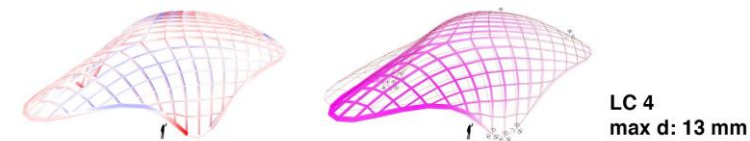
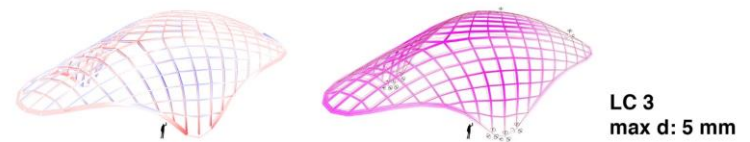
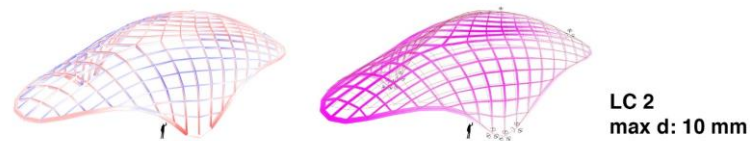
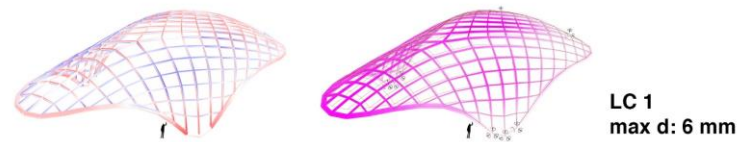


Loadcases:

LC1	: DL + LL	1.0
LC2	: DL + LL + Snow load	0.4
LC3	: DL + LL + Wind load (uplift)	0.8
LC4	: DL + LL + Lateral load (X)	0.4
LC5	: DL + LL + Lateral load (-X)	0.4

Beams:

RHS 200x100x8
CHS 400x10

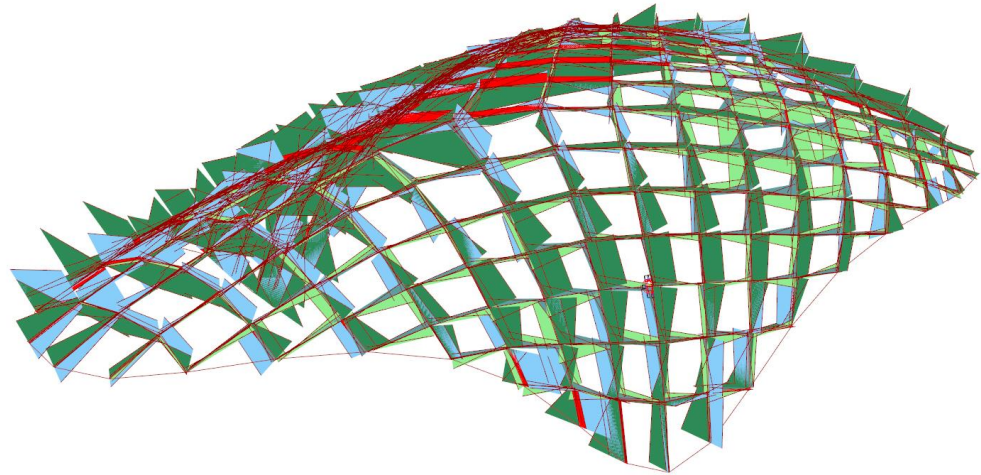


Loadcases:

LC1	: DL + LL	1.0
LC2	: DL + LL + Snow load	0.4
LC3	: DL + LL + Wind load (uplift)	0.8
LC4	: DL + LL + Lateral load (X)	0.4
LC5	: DL + LL + Lateral load (-X)	0.4

Beams:

RHS 200x100x8
CHS 400x10

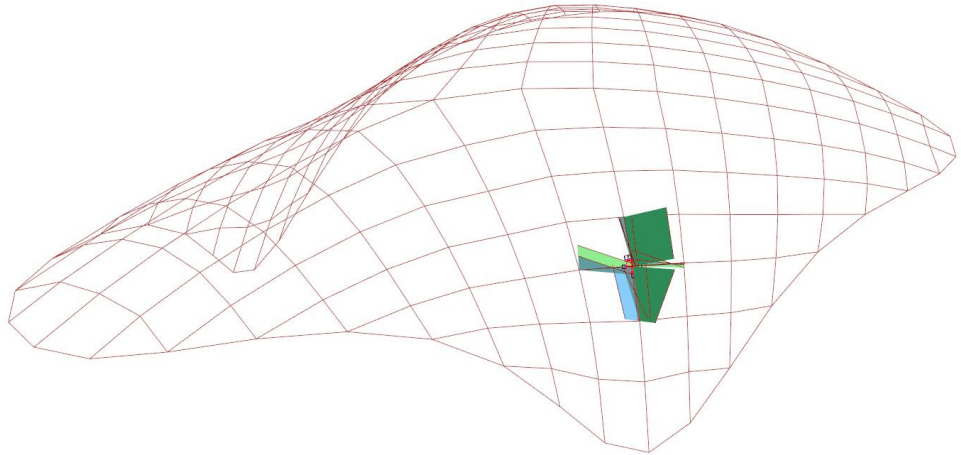


Loadcases:

LC1	: DL + LL	1.0
LC2	: DL + LL + Snow load	0.4
LC3	: DL + LL + Wind load (uplift)	0.8
LC4	: DL + LL + Lateral load (X)	0.4
LC5	: DL + LL + Lateral load (-X)	0.4

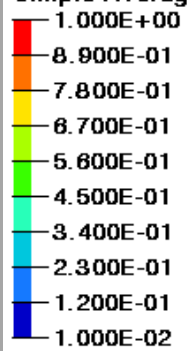
Beams:

- RHS 200x100x8
- CHS 400x10



Contour Plot
Element Densities(Density)

Simple Average



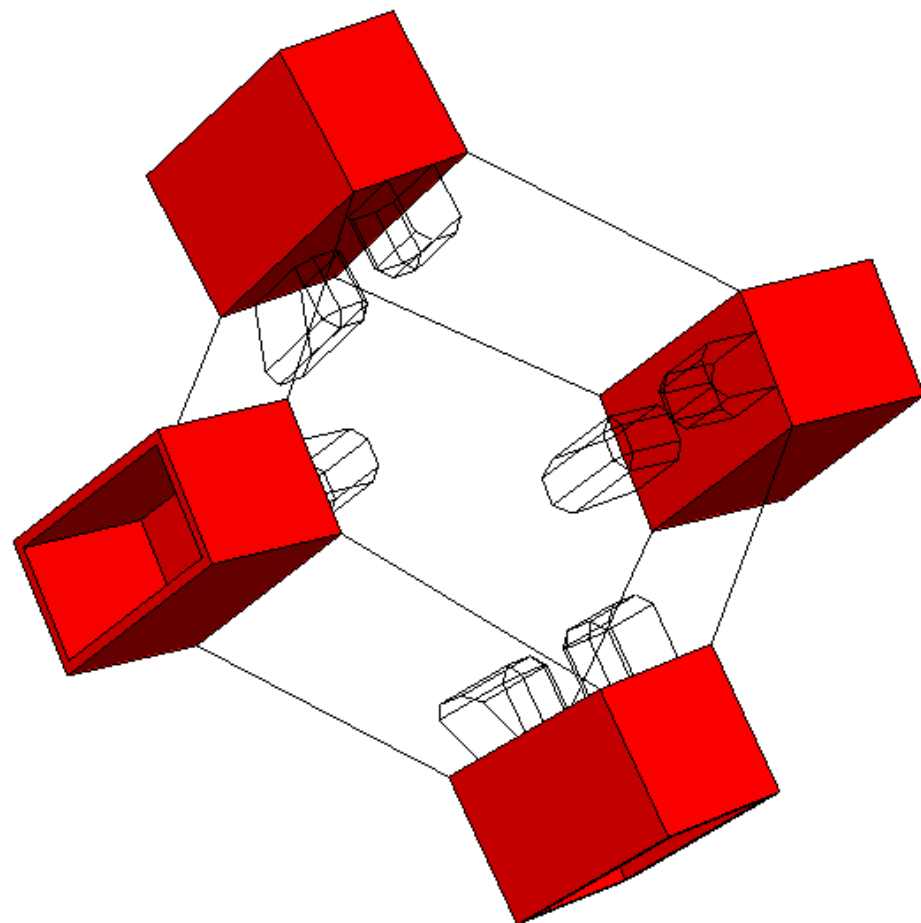
■ No result

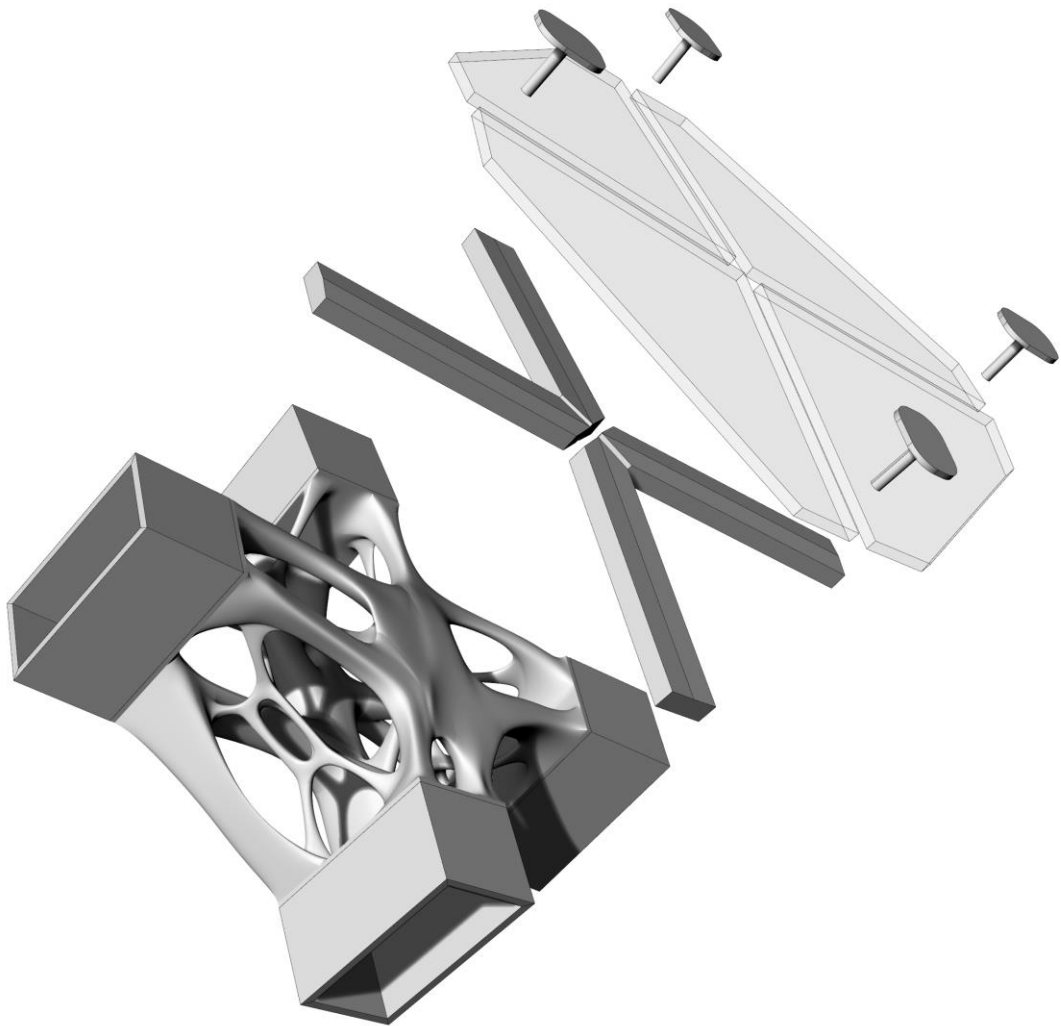
Max = 1.000E+00

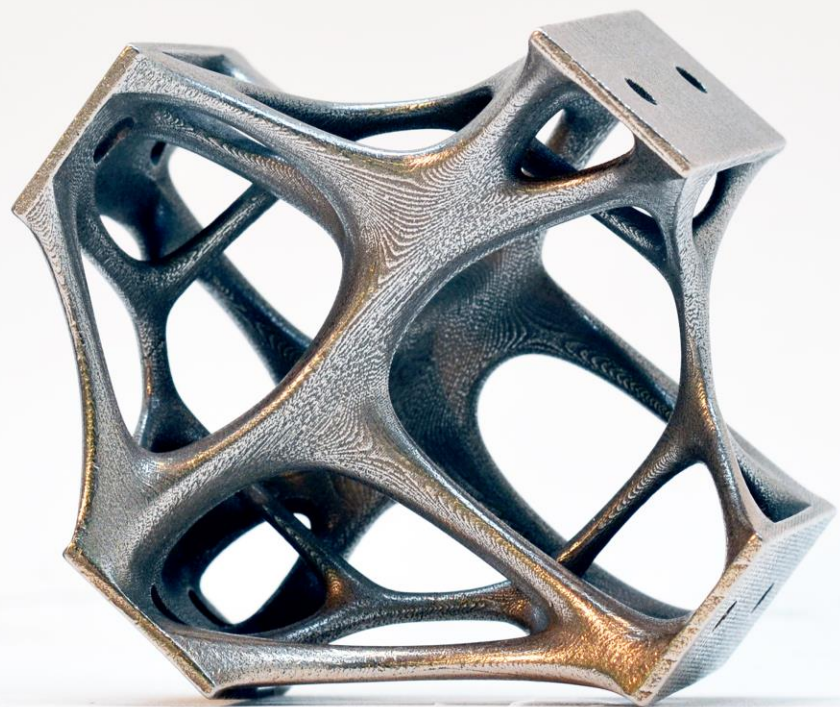
Grids 18377

Min = 1.000E-02

Grids 87008







Further research

- TO and AM in other architecture elements
- Quad glazing assembly detail integration with AM
- Extensive advanced structural analysis
- Extensive physical test for AM materials
- Extensive cost analysis

Conclusion

**AM as design tool :
Localized Complexity**

Conclusion

TO : designed by computer?

