

Experimental investigation of wave tip variability of impacting waves

van Meerkerk, M.; Poelma, C.; Hofland, B.; Westerweel, J.

DOI

[10.1063/5.0016467](https://doi.org/10.1063/5.0016467)

Publication date

2020

Document Version

Accepted author manuscript

Published in

Physics of Fluids

Citation (APA)

van Meerkerk, M., Poelma, C., Hofland, B., & Westerweel, J. (2020). Experimental investigation of wave tip variability of impacting waves. *Physics of Fluids*, 32(8), Article 082110. <https://doi.org/10.1063/5.0016467>

Important note

To cite this publication, please use the final published version (if applicable).
Please check the document version above.

Copyright

Other than for strictly personal use, it is not permitted to download, forward or distribute the text or part of it, without the consent of the author(s) and/or copyright holder(s), unless the work is under an open content license such as Creative Commons.

Takedown policy

Please contact us and provide details if you believe this document breaches copyrights.
We will remove access to the work immediately and investigate your claim.

Experimental investigation of wave tip variability of impacting waves

M. van Meerkerk,¹ C. Poelma,^{1, a)} B. Hofland,² and J. Westerweel³

¹⁾ *3ME, Process and Energy, Multiphase Systems, Delft University of Technology, Leeghwaterstraat 39, 2628 CB, Delft, The Netherlands*

²⁾ *CEG, Hydraulic Engineering, Delft University of Technology, Stevinweg 1, 2628 CN, Delft, The Netherlands*

³⁾ *3ME, Process and Energy, Fluid Mechanics, Delft University of Technology, Leeghwaterstraat 39, 2628 CB, Delft, The Netherlands*

(Dated: July 23, 2020)

We present an experimental study on the variation in wave impact location and present a mechanism for the development of free surface instabilities on the wave crest for repeatable plunging wave impacts on a vertical wall. The existence of free surface instabilities on an impacting wave is well known, but their characteristics and formation mechanism are relatively unknown. The development of the global wave shape is measured using a visualization camera, whereas the local wave shape is measured with an accurate stereo-PLIF technique. A repeatable wave is generated with negligible system variability. The global wave behavior resembles that of a plunging breaker, with a gas pocket cross-sectional area defined by an ellipse of constant aspect ratio. The variability of the local wave profile increases significantly as it approaches the wall. The impact location varies by approximately 0.5% of the wave height or more than a typical pressure sensor diameter. Additionally, the wave tip accelerates to a velocity of $1.5\sqrt{gh_0}$ compared to the global wave velocity of $1.2\sqrt{gh_0}$. The difference in impact location and velocity can result in a pressure variation of approximately 25%. A mechanism for instability development is observed as the wave tip becomes thinner and elongates when it approaches the wall. A flapping liquid sheet develops that accelerates the wave tip locally and this triggers a spanwise Rayleigh-Taylor instability.

I. INTRODUCTION

In recent years, the liquefied natural gas (LNG) market showed significant growth with an increased demand for floating liquefaction facilities, storage facilities, and shipping solutions. Furthermore, LNG is proposed as an alternative shipping fuel, especially with the prospect of stricter emission standards.¹ New challenges arise with the widespread use of LNG, such as the growth in bulk capacity of containment systems, trading routes with extreme weather conditions, and the use of lower filling levels.² Lower filling levels evidently lead to an increase in extreme impact events, which have the potential to cause structural damage.^{3,4} Wave impact events are the basis of these extreme loads, which requires a fundamental understanding of wave impacts before studying increasingly complex phenomena.⁵

The study of wave impacts on a wall has been an active area of research for decades.^{6–11} Moreover, the impact of waves upon structures is relevant for many fields such as ocean, coastal, and maritime engineering. Bag-nold⁶ already showed significant variation of the wave impact pressure for carefully repeated wave impact experiments. The generation of repeatable waves is not trivial. Small changes in the input parameters, such as the water depth, the wave generation method, and even the weather conditions (for large-scale outdoor experiments), results in significant variability of the impact pressure.^{12,13} On the other hand, the pressure impulse

(i.e., the integral of pressure over time) is far more repeatable and is used to model and scale the pressure of wave impact experiments.^{9,10,14–16} In recent years, the study of liquid sloshing^{17–21} and slamming on both wave energy converters^{22–24} and floating offshore structures³ has received considerable attention. The peak impact pressure is especially relevant in these applications.^{3,4}

A number of reviews have been published both on extreme wave impact events and sloshing. For example, the effect of liquid sloshing impacts has been thoroughly reviewed by Ibrahim²⁵. A detailed review of water wave impacts on vertical walls is presented by Peregrine⁴, whereas Dias and Ghidaglia²⁶ present a detailed review on slamming. The impact of a wave can be divided into several elementary loading processes, such as the direct impact, the jet deflection, and the compression of the entrapped or escaping gas.²⁰ Different types of wave impact can be defined by a combination of elementary loading processes. The classification of wave impact type depends on the wave shape prior to impact, which is either classified as a slosh, a flip-through, a gas pocket, or an aerated type of wave impact.^{4,7,27} For example, the flip-through wave impact has been studied in detail with and without hydro-elasticity.^{28,29} The effect of hydro-elasticity is relevant for all wave impact types.³⁰ The flip-through wave impact only occurs for a limited parameter space.⁴ On the other hand, the impact of a plunging breaking wave occurs for a wider parameter space and often results in a gas pocket type wave impact. The impact type can easily be identified, but scaling of wave impacts from small-scale to large-scale experiments is not straightforward.

Obtaining dynamic similarity of liquid sloshing or wave

^{a)} Electronic mail: C.Poelma@tudelft.nl

This is the author's peer reviewed, accepted manuscript. However, the online version of record will be different from this version once it has been copyedited and typeset. PLEASE CITE THIS ARTICLE AS DOI:10.1063/1.50016467

85 impact events is complex.²⁵ The elementary loading pro-143
 cesses can be used to identify the required similarity144
 87 parameters.²⁰ For example, Froude scaling can be used
 88 for the global flow, where the wave is not influenced145
 89 by the presence of the impact wall (i.e., the increase146
 in pressure in the gas pocket and increase in flow from147
 90 the enclosed gas pocket).^{18,26} The global flow displays148
 92 remarkable similarities with a plunging breaking wave,149
 93 which allows a comparison of the wave crest velocity^{7,31},150
 94 the wave crest trajectory³², and the gas pocket cross-151
 95 sectional area^{33,34}. The gas pocket behavior *after* wave152
 96 impact has been studied in detail, which shows that the153
 97 enclosed gas pocket decreases in volume and starts to154
 98 oscillate.^{35–37} The decrease in volume of the gas pocket155
 99 after wave impact is not related to gas leakage at the156
 100 wave crest.³⁵ On the other hand, the local flow is sig-157
 101 nificantly altered by the strong gas flow over the wave158
 102 crest for a gas pocket type wave impact. The local flow159
 103 can be altered by the surface tension of the gas-liquid160
 104 interface³⁸, the gas-liquid density ratio^{39,40}, the com-161
 105 pressibility of the gas (i.e., the speed-of-sound)¹⁸, the162
 106 possibility of phase change^{5,41,42}, and the aeration of the163
 107 liquid.^{9,43,44} The scaling of the local flow is not well un-164
 108 derstood, but especially the formation of ligaments and165
 109 droplets are thought to be relevant for the variability in166
 110 wave impact pressures.^{26,45}167

111 The global features of a wave impact on a vertical168
 112 wall can be accurately represented by potential flow169
 113 models.^{4,46–49} Apart from ignoring viscous effects, these170
 114 simulations generally also ignore surface tension effects,171
 115 as the impact is inertia dominated.⁴⁶ The irrotational172
 116 flow assumption seems to be valid, as qualitative agree-173
 117 ment between experimental and numerical impact pres-174
 118 sures can be obtained.²⁶ Nonetheless, the gas phase175
 119 should not be neglected, especially when the flow sep-176
 120 arates near the wave crest.⁵⁰ Furthermore, the inertia177
 121 of the wave tip is small and consequently it is pushed178
 122 upward where it can eventually be blown off the wave179
 123 crest.^{46,49} Compressible multiphase simulations are re-180
 124 quired to capture this effect.^{26,51,52} However, the simu-181
 125 lations are often not able to capture the development of182
 126 instabilities on the wave crest.^{24,26,50,53}183

127 The source of impact pressure variability in repeated184
 128 wave impact experiments is thought to be the instability185
 129 development on the wave crest. However, the mechanism186
 130 that is responsible for the formation of these instabili-187
 131 ties is still largely unknown.⁴⁵ An approaching plunging188
 132 breaking wave that encloses a gas pocket forces a strong189
 133 gas flow over the wave crest, which results in a shear190
 134 force on the wave crest. The shear force of the expelled191
 135 gas is often postulated to result in a Kelvin-Helmholtz192
 136 type instability.^{40,45,53,54} Additionally, the wave tip of193
 137 the plunging breaking wave is deflected by the strong gas194
 138 flow prior to the impact on the wall.^{35,55} Prior to impact,195
 139 gas cushioning (i.e., the increase in pressure in front of196
 140 the wave tip) can also result in deformation of the wave197
 141 tip.^{49,56,57} The wave tip deflection is shown to depend198
 142 on the density ratio and the scale of the experiment.^{39,40,199}

However, accurate measurements of the wave tip deflec-
 tion have up to now not been reported.

In the present study, accurate free surface measurements
 at both the global and local scale were performed to in-
 vestigate the source of impact pressure variability in re-
 peated wave impact measurements. The variability in
 impact location of the wave crest is accurately deter-
 mined and a mechanism for the development of wave
 crest instabilities is proposed. Both the free surface
 instabilities and the deflection of the wave tip are im-
 portant in the context of sloshing induced loads, where
 also the extreme impact pressure needs to be taken into
 account.^{3,4} The global wave behavior is shown to be re-
 peatable for measurements that have negligible system
 variability. Additionally, the wave behavior prior to im-
 pact is shown to resemble a plunging breaking wave. The
 local flow is investigated with a stereo-PLIF technique,
 which shows both an acceleration and a deflection of the
 wave tip prior to impact. The wave tip shows a signifi-
 cant variation in impact location on the scale of typical
 pressure membrane diameters of $d \sim 1 - 5.5$ mm.^{19,58}
 Furthermore, the development of a span-wise instabil-
 ity is observed. The instability on the wave crest is
 remarkably similar to that of a flapping liquid sheet.⁵⁹
 The length scale of the instability depends on the wave
 shape, density ratio, and the surface tension, which was
 already suggested in previous work.^{38,40,51,52} Addition-
 ally, the study may also provide quantitative data of the
 wave shape, wave velocity, and wave instability for phys-
 ical and numerical model validations.

This paper is organized as follows. The experimental
 setup and equipment are introduced in section II. This
 section also introduces the experimental procedure re-
 quired for the generation of repeatable waves. There-
 after, the results are introduced and discussed. First in
 section III A, the system variability is quantified and re-
 peatable waves are identified. Then in section III B, the
 behavior of the global wave is identified. Finally in sec-
 tion III C, the local wave behavior is discussed and two
 sources of impact pressure variability are identified. The
 findings are summarized in section IV.

II. EXPERIMENTAL APPROACH

A. Wave flume

Figure 1 shows the experimental facility used in this
 study. The measurements are performed in the wave
 flume of the Hydraulic Engineering Laboratory at the
 Delft University of Technology. The flume is 39 m long
 with a cross-section of 0.79×1 m², and the water depth is
 maintained at $h_0 = 500.0 \pm 0.5$ mm for all measurements.
 The flume is equipped with a piston-type wavemaker that
 has a maximum stroke of 2 m. Additionally, the flume
 contains an active reflection compensation (ARC) sys-
 tem, which is designed to operate during continuous wave

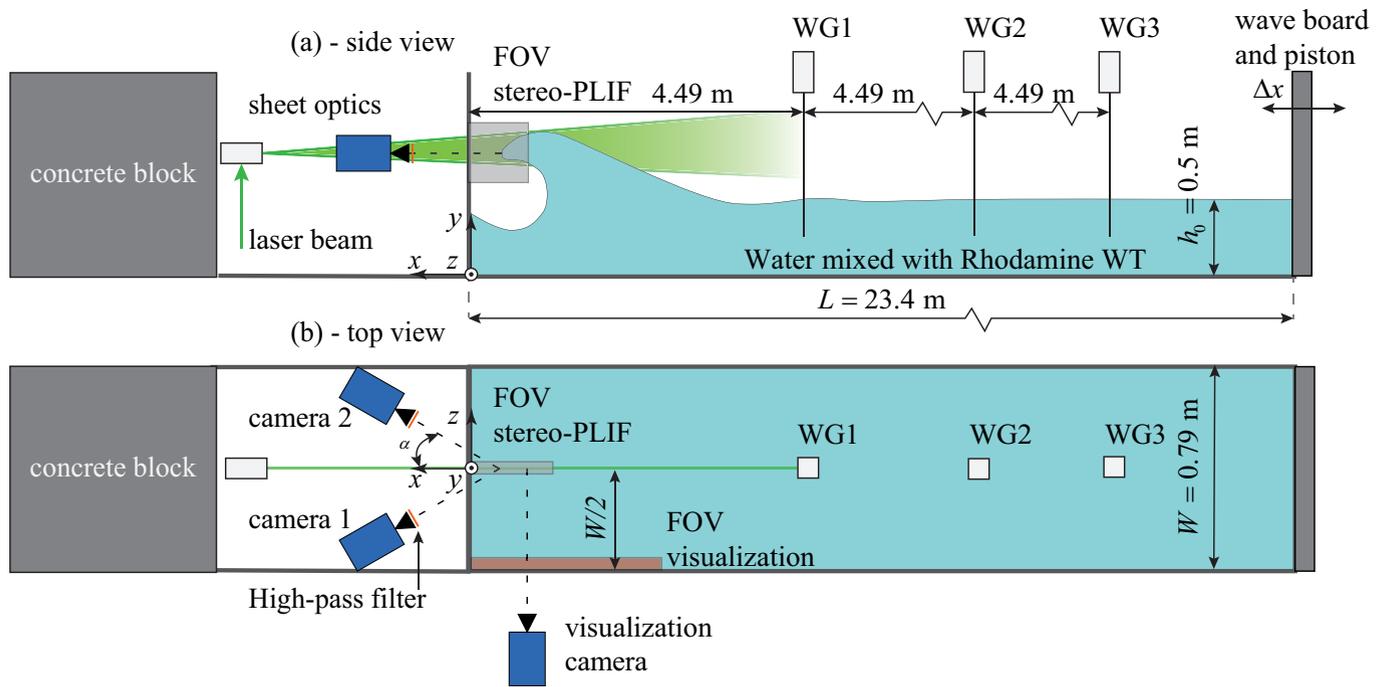


Figure 1. Schematic of the experimental facility. The origin of the coordinate system is located at the center plane of the wave flume on the bottom of the flume. The positive x -direction is from the wave board towards the impact wall. (a) Side-view of the wave flume. The cameras for the stereo-PLIF are aligned on a xz -plane. A vertical light sheet (xy -plane) is created at the center plane of the flume. A focused wave, generated at the wave board ($x = -L$), impacts the impact wall at $x = 0$. (b) Top view of the set-up. The visualization camera measures the global wave shape, either at the sidewall or the light-sheet location. The stereo-PLIF system measures wave crest details in a smaller field-of-view at the light-sheet location.

196 or wave spectrum generation. In this work, the ARC sys-223
 197 tem is disabled during generation of the single focused224
 198 wave. After impact of the focused wave, the system is225
 199 enabled to dampen the reflected waves and to reduce the226
 200 downtime between experiments.227

201 The current method of wave generation is similar to228
 202 the *large scale* tests of the Sloskel project, where the ef-229
 203 fective flume length is scaled with the length-scale (λ) of230
 204 the depth-based Froude number (i.e., $\lambda = h_0/H_\lambda = 1/6$ 231
 205 with h_0 the current water depth and H_λ the full-scale232
 206 water depth).²⁰ A Froude scaled experiment requires a233
 207 reduced effective flume length, which is obtained by plac-234
 208 ing a 20 mm thick transparent perspex wall at a distance
 209 of $L = 23.4$ m from the wave board (fig. 1). The perspex
 210 wall is attached to a frame, which is fixed to a stable con-235
 211 crete block (i.e., with dimensions of $0.78 \times 0.80 \times 1.00$ m³
 212 and a weight of approximately 1500 kg) placed in the236
 213 flume. Silicone sealant is applied at the edges of the237
 214 perspex wall to make it watertight. Nonetheless, exact238
 215 Froude scaling is not achieved, due to practical limita-239
 216 tions (e.g., the camera measurement system limits the240
 217 water depth to 500 mm). The Froude scaled ratio is241
 218 (1 : 7.3) compared to the (1 : 6) ratio of the Sloskel ex-242
 219 periments, which will result in a smaller wave (i.e., with243
 220 a smaller gas pocket and lower wave impact height).^{18,20}244

221 The flume is equipped with a control system, a data245
 222 acquisition system, resistance-type wave gauges, a posi-246

tion sensor on the wave board, and temperature sensors
 for both the water and air (TSP01, Thorlabs). The wave
 shape is additionally determined on a global and local
 scale with a camera measurement system. The genera-
 tion of repeatable waves is not trivial and the required
 experimental procedure is further detailed in section IID.
 The wave gauge, position sensor, and trigger signals are
 collected at a frequency of 100 Hz. The three wave gauges
 measure the surface elevation ($\eta = y - h_0$) at respectively
 $x/h_0 = 8.98$, 18.0 , and 26.9 (fig. 1). The position sen-
 sor (GHM2000MD601V2 position sensor, Temposonics)
 records the position of the wave board (fig. 1).

B. Wave generation

We obtain a large gas pocket wave with a technique
 that focuses the wave energy in the temporal domain.¹²
 The wave board (fig. 1) generates wave groups with their
 own group velocity and phase speed, which results in a
 variety of wavelengths as shown later in figure 3. The
 wave energy of these wave groups is focused on a single
 location in the flume, the focal point (x_f). The focal
 point defines the wave shape upon impact, where a shift
 of the focal point results in respectively: an aerated,
 a flip-through, a gas pocket, or a slosk impact.⁷
 The focal point also determines the angle between the

This is the author's peer reviewed, accepted manuscript. However, the online version of record will be different from this version once it has been copyedited and typeset. PLEASE CITE THIS ARTICLE AS DOI:10.1063/1.50016467

247 wavefront and the impact wall, where a parallel front²⁹⁹
 248 (i.e. a wave crest aligned with the impact wall) results in
 249 a high impact pressure.⁴⁴ The normalized focal point of³⁰⁰
 250 $x_f/h_0 = 0.81$ is selected with a trial-and-error approach.³⁰¹
 251 to obtain a large gas pocket with a parallel front, which³⁰²
 252 results in a spray cloud.⁶³⁰³

253 The generation of nominal identical waves with a fo-³⁰⁴
 254 cusing technique is not trivial, as changes in the initial³⁰⁵
 255 conditions, such as the water depth, are amplified by the³⁰⁶
 256 non-linear wave focusing, which results in a different im-³⁰⁷
 257 pact type.^{6,13} The variance in impact type results from³⁰⁸
 258 two sources of variability: the *system variability*, and the³⁰⁹
 259 *hydrodynamic variability*. Minimization of the system³¹⁰
 260 variability is essential to study the hydrodynamic vari-³¹¹
 261 ability (e.g., the growth of free surface instabilities on a³¹²
 262 wave crest). The system variability (i.e., the water depth,³¹³
 263 variation, piston motion variation, and residual motion)³¹⁴
 264 is minimized within the limitations of the experimen-³¹⁵
 265 tal facility. The comparison between measurements over³¹⁶
 266 several days is limited due to inevitable day-to-day vari-³¹⁷
 267 ations present in the current experimental facility.^{13,18}³¹⁸
 268 The day-to-day variations are all variations related to³¹⁹
 269 water depth, water quality (i.e., natural accumulation of³²⁰
 270 particles on the free surface), and water temperature that³²¹
 271 cannot be fully controlled in the current facility. The ini-³²²
 272 tial water depth variations are expected to be the most³²³
 273 significant source of day-to-day variations, as the water³²⁴
 274 depth in this facility could only be set with limited accu-³²⁵
 275 racy (i.e., 0.5 mm). Therefore, a single data set is high-³²⁶
 276 lighted, for which the differences in input parameters are³²⁷
 277 carefully reported in section III A.³²⁸

278 C. Free surface profile measurement³²⁹

279 The wave impact upon a wall displays global and local³³³
 280 behavior.¹⁸ The global wave is Froude scaled, whereas³³⁴
 281 hydrodynamic variability alters the local wave behavior.³³⁵
 282 The difference in length scales of the global and local³³⁶
 283 waves require separate measurement systems, which are³³⁷
 284 introduced in the following section.³³⁸

285 1. Global wave profile³³⁹

286 A high-speed visualization camera determines the³⁴³
 287 global wave shape (fig. 1). This CMOS camera (Im-³⁴⁴
 288 ager HS 4M, LaVision) is equipped with a 35 mm Micro-³⁴⁵
 289 Nikkor objective with an aperture number of $f^\# = 8$.³⁴⁶
 290 Two LED floodlights (ProBeam 170w, Noxon) provide³⁴⁷
 291 background illumination on a diffusion plate, which for³⁴⁸
 292 the selected aperture results in good image contrast be-³⁴⁹
 293 tween the background and laser light (see next section).³⁵⁰
 294 The field of view is approximately 353 mm \times 174 mm at a³⁵¹
 295 magnification of $M_0 = 0.06$. The image resolution is re-³⁵²
 296 duced to 2016 \times 1000 pixels for a higher camera frame rate³⁵³
 297 (f_{aq}) of 2.5 kHz with an exposure time (Δt_e) of 358 μ s,³⁵⁴
 298 which is sufficiently low to avoid motion blur.³⁵⁵

2. Local wave profile

A stereo planar laser-induced fluorescence (stereo-
 PLIF) technique measures the local wave shape at the
 center plane of the wave flume. This system is described
 in detail by van Meerkerk, Poelma, and Westerweel⁶⁰.
 The advantage of the stereo camera system is two-fold.
 For a single camera, free-surface measurements can be
 obstructed by liquid filaments, which is largely avoided
 by using a stereo-camera system. Second, the stereo cam-
 era system enables the use of a self-calibration procedure,
 which improves the measurement accuracy and reduces
 alignment errors.⁶⁰

Two high-speed CMOS cameras (Imager HS 4M, LaV-
 ision) equipped with 55 mm Micro-Nikkor objectives and
 a high-pass filter (OG570, Schott) are placed between
 the impact wall and the concrete block (fig. 1). The
 separation angle (2α) of the stereo camera system is ap-
 proximately 60° , with an aperture number of $f^\# = 16$
 to accommodate the large separation angle. The image
 resolution is reduced to 1392 \times 1400 for a higher camera
 frame rate (f_{aq}) of 2.5 kHz with an exposure time (Δt_e)
 of 363 μ s. The field of view of 150 mm \times 150 mm aligns
 with the tip of the focused wave.

The cameras are calibrated with a two-plane dot-
 pattern target (Type 22, LaVision) with its center at
 $(x, y, z) = (-104, 730, 2)$ mm. The bottom corner of the
 impact wall at the center of the flume defines the origin
 of the coordinate system and the positive x -direction is
 defined from the wave board towards the impact wall, so
 that the wave runs with a positive velocity from $x = -L$
 to the impact wall ($x = 0$) (fig. 1). The calibration pro-
 cedure requires us to initially image a fluorescent plate
 to determine a mapping function at the light sheet loca-
 tion. In the following paragraphs, we often refer to
 the details of the stereo-PLIF technique described in a
 previous manuscript.⁶⁰

A light sheet is created from the beam of a Nd:YLF
 laser (LDY 304 PIV laser, Litron) and focused at the
 center plane of the flume. The light sheet illuminates
 the approaching wave, which contains a fluorescent dye
 at a low concentration (Rhodamine WT, Sigma-Aldrich
 at 120 mg m⁻³). The static surface tension does not
 change at the current fluorescent dye concentration.⁶⁰
 The dynamic surface tension is, in some cases, altered
 by the presence of natural surfactants that settle on
 the free surface over time (i.e., dust and other natural
 contaminants).⁶¹ The dynamic surface tension is not de-
 termined in the current experiments.

The local wave shape is obtained from the image with
 the following processing steps (fig. 2) implemented in
 Matlab 2020. First, a 3 \times 3 median filter reduces the ef-
 fects of noise (fig. 2a). Then, a multi-step edge detection
 procedure is applied, which uses Otsu's method.⁶² The
 boundary contour (fig. 2b) is traced after morphologi-
 cal operations are applied to close holes inside the wave
 shape and to remove small elements outside the wave
 shape.⁶³ After that, the contour coordinates are mapped

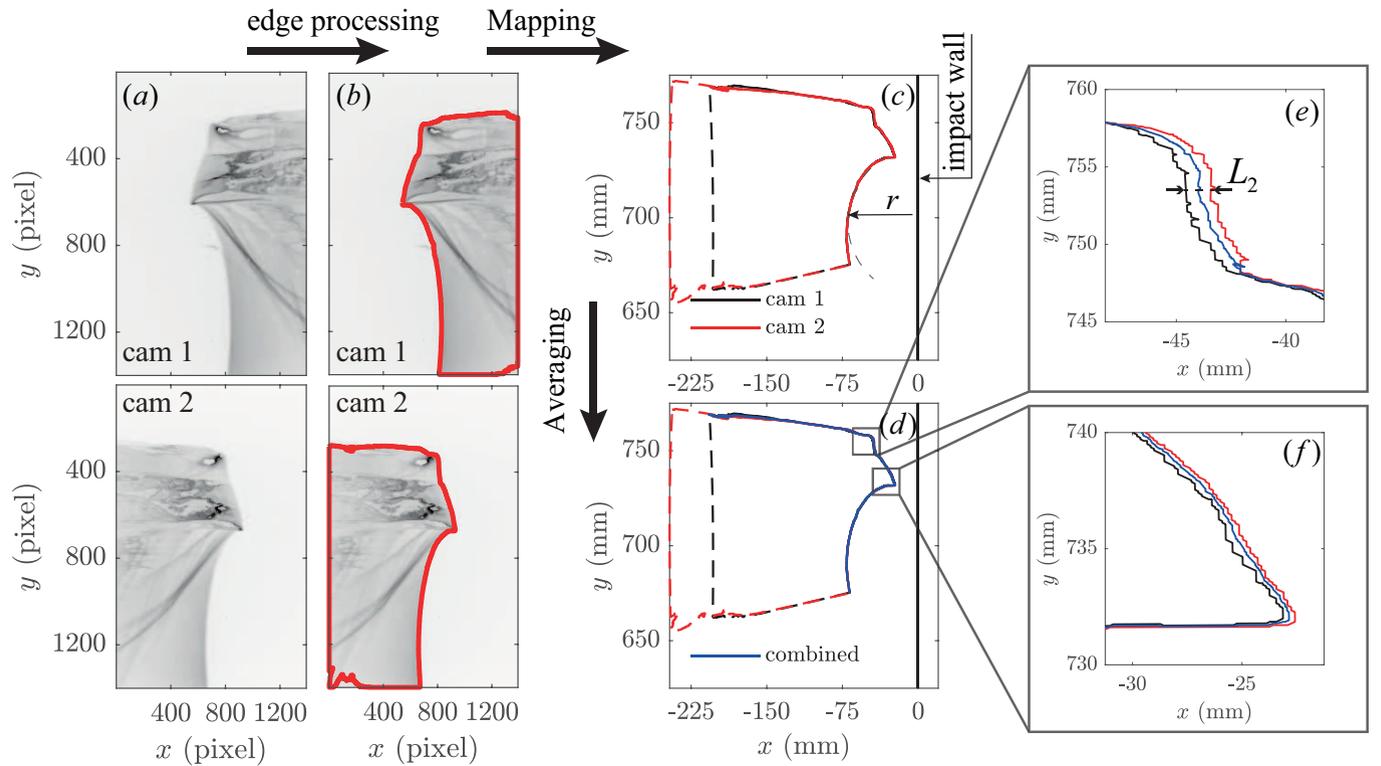


Figure 2. Data processing steps for the stereo-PLIF for the present measurements. (a) The original image pair from cameras 1 and 2 (Fig. 1). (b) The free surface profile after edge processing. (c) The profiles of both cameras mapped to world coordinates. The valid free surface profiles are indicated by a continuous line, whereas the invalid parts of the free surface reconstruction (i.e., the image borders) are indicated by a dashed line. The impact wall is located at $x = 0$ and the wave approaches the wall (i.e., from negative x which is defined to point towards the wave board). (d) The final combined profile based on the k -nearest neighbor search, with insets (e-f) showing the typical variance of the averaged profile with respect to the separate camera profiles as the distance norm of $L_2 \approx 0.5$ and $L_2 \approx 0.2$ mm for panels e and f, respectively.

356 using an updated mapping function. A disparity correc-380
 357 tion is additionally applied to improve the reconstructed381
 358 profile's accuracy.⁶⁴ Then, a circle (fig. 2c) is fitted to the382
 359 edge of the gas pocket.⁶⁵ Thereafter, the profiles of both383
 360 cameras are combined by averaging over the k -nearest384
 361 neighbor of camera 1 with respect to camera 2, with a385
 362 limit of $D_l = 2.5$ mm on the point distance (fig. 2d).⁶⁶386
 363 Finally, the combined profile is cropped to remove the387
 364 image boundaries at the minimum y -coordinate of the388
 365 circle fit and the minimum x -coordinate of both camera389
 366 profiles.

367 The measurement accuracy of the stereo-PLIF system
 368 is determined for the initial calibration and a typical wave390
 369 crest (i.e., free surface profile). First, the initial mapping
 370 function is determined with an accuracy of approximately391
 371 0.06 mm (e.g., 0.3 and 0.8 pixels for respectively the x 392
 372 and y -coordinate). The camera perspective results in a393
 373 significant variation of the resolution (S).⁶⁰ The resolu-394
 374 tion over the x (S_x) and y (S_y) coordinate are respec-395
 375 tively 4.9 and 13 pixels mm^{-1} . Second, a systematic er-396
 376 ror is introduced when the free surface profiles from the397
 377 two cameras are combined. This systematic error is de-398
 378 fined as the average Euclidean norm (L_2) between the399
 379 combined and individual profiles. The systematic error Γ_{400}

for a typical free surface profile (fig. 2 e-f) is approx-
 imately $L_2 \approx 0.35$ mm (e.g., approximately 1.7 or 4.6
 pixels based on respectively the x and y -coordinate of
 the initial mapping function). The systematic error is
 larger at the top of the wave crest ($L_2 \approx 0.5$ mm) where
 the light sheet skims over the wave surface (see section
 III C), which results in a increase of the measurement un-
 certainty. The measurement accuracy is mainly defined
 by the systematic error, whereas the error of the initial
 calibration appears to be negligible.

D. Experimental procedure

Nominal identical waves require a repeatable experi-
 mental procedure. The steps in the procedure are de-
 tailed in this section, which describes the residual motion
 reduction, water level control, and measurement proce-
 dure.

The free surface is disturbed by waves at several mo-
 ments during a measurement. The waves are for example
 introduced when the wave board of the flume is zeroed,
 or when the water level is adjusted. The waves that re-
 flect from the impact wall also disturb the free surface.

This is the author's peer reviewed, accepted manuscript. However, the online version of record will be different from this version once it has been copyedited and typeset. PLEASE CITE THIS ARTICLE AS DOI:10.1063/1.50016467

401 This residual motion of the free surface is removed with⁴⁵⁵
 402 the ARC by enabling it for 7 minutes, which based on⁴⁵⁶
 403 previous experiments significantly reduces the free sur-⁴⁵⁷
 404 face fluctuations.⁶⁷ The ARC is disabled after the allot-⁴⁵⁸
 405 ted time, and the water is left untouched for 7 minutes.⁴⁵⁹
 406 However, the longest standing wave (i.e., seiche wave) is⁴⁶⁰
 407 not completely attenuated by the bottom friction, which⁴⁶¹
 408 would require an impractically long downtime between⁴⁶²
 409 measurements.^{13,67} Despite this, the procedure reduces⁴⁶³
 410 the free surface fluctuations within acceptable limits for⁴⁶⁴
 411 the present experiments.⁴⁶⁵

412 The water level is checked with a ruler before the start⁴⁶⁶
 413 of a measurement. Additionally, the water level is moni-⁴⁶⁷
 414 tored with higher precision with the visualization camera⁴⁶⁸
 415 (see section III A). The resolution of the ruler is 0.5 mm,⁴⁶⁹
 416 which defines the minimum threshold for the water depth⁴⁷⁰
 417 change. The water level is adjusted when the thresh-⁴⁷¹
 418 old is exceeded, and thereafter the residual motion is re-⁴⁷²
 419 duced according to the experimental procedure described⁴⁷³
 420 above.⁴⁷⁴

421 The measurement procedure initiates with the start⁴⁷⁵
 422 of the acquisition devices, and wave generation. These⁴⁷⁶
 423 are separate systems where the programming timing unit⁴⁷⁷
 424 (PTU) of the camera system is used as a master clock⁴⁷⁸
 425 during the measurements. The camera and analog acqui-⁴⁷⁹
 426 sition system are both enabled prior to wave generation.⁴⁸⁰

427 The analog acquisition system is manually enabled and⁴⁸¹
 428 collects data from the wave-gauges and piston position⁴⁸²
 429 sensor at a frequency of 100 Hz. Additionally, the trig-⁴⁸³
 430 ger signals from both the wave generation and the camera⁴⁸⁴
 431 acquisition system are recorded. The data of the analog⁴⁸⁵
 432 system is matched to the master clock based on the trig-⁴⁸⁶
 433 ger signal of the wave generation system.⁴⁸⁷

434 The camera acquisition system acquires data at a fre-⁴⁸⁸
 435 quency of 2.5 kHz in a ring buffer, which enables contin-⁴⁸⁹
 436 uous recording. This ring buffer allows a remote signal to⁴⁹⁰
 437 trigger the recording of the camera measurement system.⁴⁹¹
 438 The remote trigger signal is sent from a delay generator⁴⁹²
 439 (digital delay generator DG535, Stanford Research Sys-⁴⁹³
 440 tems), which in turn is triggered by the wave flume.⁴⁹⁴

441 The wave generation system is manually activated to⁴⁹⁵
 442 generate a single focused wave. The wave generation sys-⁴⁹⁶
 443 tem sends a trigger signal to both the camera and analog⁴⁹⁷
 444 acquisition systems. Finally, the acquisition system is⁴⁹⁸
 445 disabled after wave impact and the experimental proce-⁴⁹⁹
 446 dure is repeated.⁵⁰⁰

447 III. RESULTS AND DISCUSSION

448 A. System variability

449 On a global scale the wave is considered repeatable for⁵⁰⁷
 450 the current facility when the system variability is mini-⁵⁰⁸
 451 mal within the practical limitations. In this section the⁵⁰⁹
 452 wave-gauge signal, piston motion, and still-water level are⁵¹⁰
 453 analyzed for 12 selected measurements. These 12 mea-⁵¹¹
 454 surements are part of a set of 32 measurements, obtained⁵¹²

over multiple days. The analyzed measurements were performed on the same day to avoid day-to-day system variability. The water quality (in particular the surface tension) is assumed to be constant, and the water temperature variation ($\Delta T = 0.3^\circ\text{C}$) is considered negligible.

The wave shape and wave height change significantly for small water depth variations (i.e., a water depth variation larger than 0.15% of the initial water depth is significant).^{13,67} An estimate of the water depth variation is determined from samples ($N = 100$) of the still-water level that were recorded prior to each measurement. A line is fitted through the still-water level, which shows a variation in initial water depth of $\Delta h_0 = 0.08$ mm with a bias of 0.15 mm with respect to the linear fit of the still-water level. The water depth variation is lower than 0.15% of the initial water height. Therefore, the influence of the initial water height on the system variability is negligible for the measurements performed on a single day.⁶⁷

The piston motion (x_p) and wave-gauge signal (η_{WG1}) are compared with methods commonly used to quantify the repeatability of focused waves.^{11,12,18} The height (H) and last zero up crossing period (T) of the highest wave are determined for both the piston motion and free surface elevation (fig. 3). For both signals the mean (μ), standard deviation (σ), and coefficient of variation ($c_v = \sigma/\mu$) are reported (tab. I).^{12,18} Additionally, the peak root-mean square error (RMSE) is defined.¹¹ Last, the coefficient of variance for the energy of the piston motion signal ($E_s = \int_{t_0}^{t_1} |x(t)|^2 dt$) is computed.¹⁸

The period of the highest wave is repeatable for both the piston motion and free surface elevation, with an insignificant standard deviation compared to the acquisition frequency (i.e., $\Delta t = 10$ ms). The period of the highest wave is reduced as the wave steepens.

The piston motion is also highly repeatable, with a negligible standard deviation compared to the resolution of the acquisition system (i.e., 0.21 mm is equivalent to 2.1 mV). The variation in the signal power (E_s) is also insignificant (tab. I).

Figure 3 shows the free surface elevation signal for the reported experiments, where the insets highlight the small amplitude (b) and large amplitude (c) free surface waves. Colors represent the different repetitions of the experiment. The numbering is kept consistent within the larger experimental campaign for data re-usability. The standard deviation of the peak height is not negligible compared to the free surface elevation (η_{wg}) and outliers (dashed lines) can easily be identified for the highest wave (fig. 3 c). The outliers are based on the median absolute deviation (MADe). A significant reduction in the standard deviation and coefficient of variation of the wave height are obtained with only the repeatable waves (η_{wg}^*). A possible source of the wave height variation is a remaining free-surface fluctuation (i.e., a seiche wave) at the start of the measurement.⁶⁷ The coefficient of variation of the piston motion is low and does not depend on the repeatable and non-repeatable waves. The combined

Table I. The system variability is based on repeatability estimates of the piston motion (x_p) and the free surface elevation at wave gauge 1 (η_{WG1}). The free surface elevation without outliers (η_{WG1}^*) is also reported.

Case	Number of measurements	E_s (mm ² s)	H (mm)			T (s)		
			μ	σ	c_v (%)	μ	$\sigma \times 10^{-3}$	c_v (%)
x_p	12	0.4	213.1	0.21	0.10	2.1	0.30	0.01
η_{WG1}	12	1.2	227.9	2.1	0.92	1.41	0.74	0.05
η_{WG1}^*	9	1.0	229.1	0.21	0.09	1.41	0.58	0.04

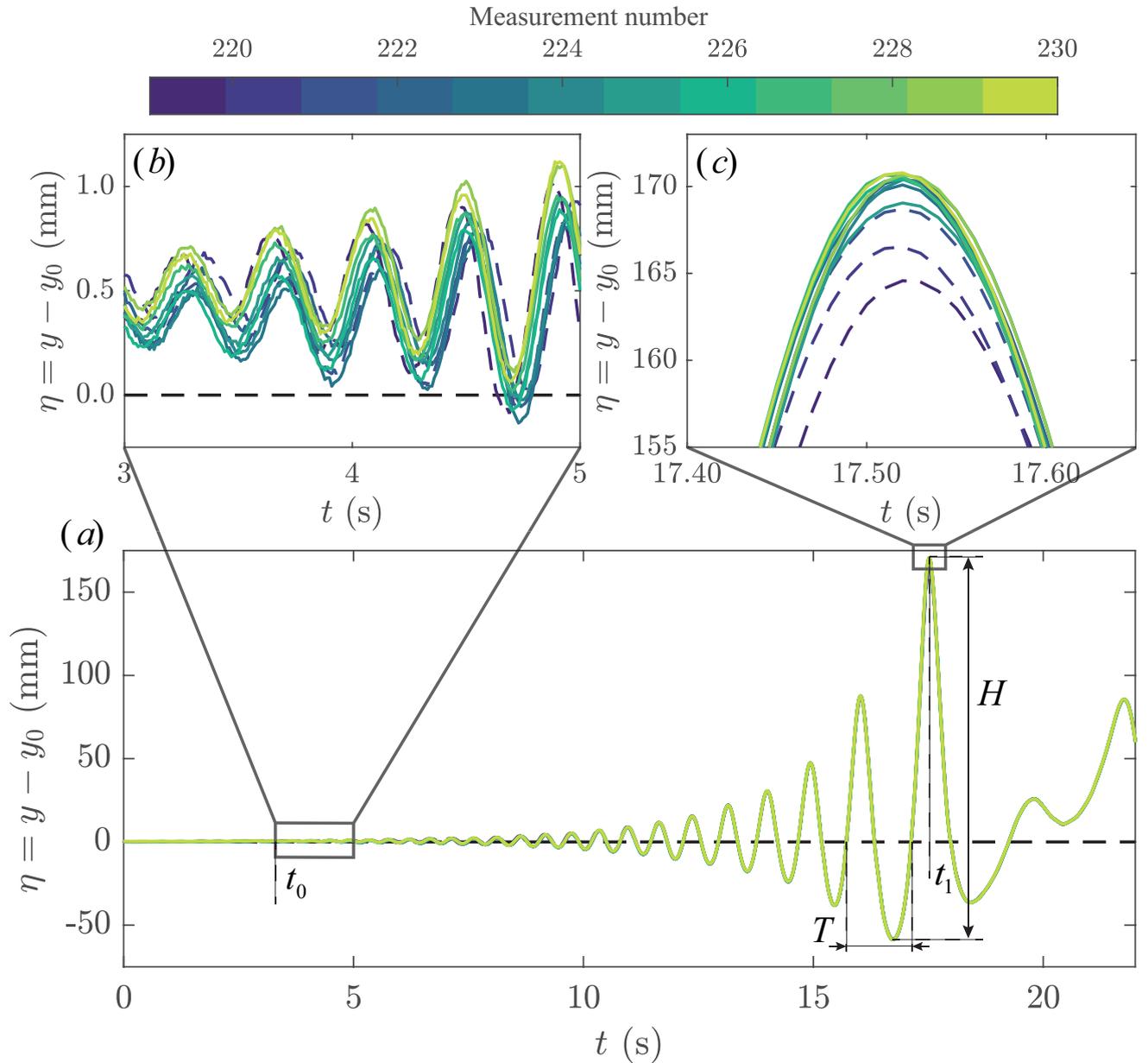


Figure 3. (a) The wave elevation signal at wave gauge 1 (WG1 in fig. 1) is shown for 12 measurements obtained on the same day with an initial water depth of $h_0 = 500$ mm. The still-water level (y_0) of the wave gauges is subtracted from surface elevation signal (y). The amplitude (H) and period (T) of the highest wave are also defined. The continuous lines show repeatable measurements, whereas dashed-lines indicate outliers identified based on the amplitude of the highest wave. The difference between repeatable and non-repeatable (i.e., outlier) waves is highlighted in panels b and c, where a zoom in of the free surface elevation signal is shown for respectively the short and long waves.

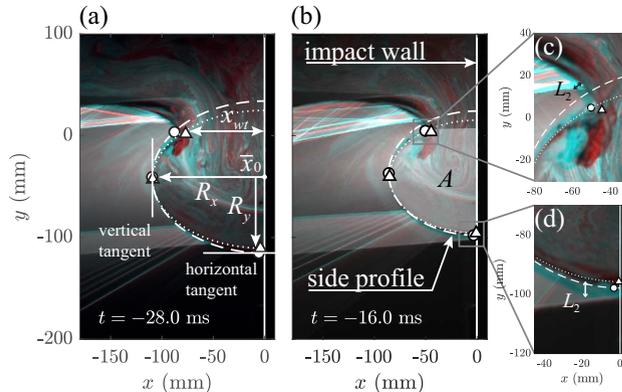
513 repeatability measures indicate insignificant system vari-535
514 ability and, consequently, the global wave is expected to536
515 be well repeated.^{12,13,18}

516 B. Global wave behavior

517 The analysis of the system variability indicates that542
518 the wave generation is repeatable for the 9 selected waves543
519 from a set of 12 measurements (tab. I). Now, the images544
520 obtained with the visualization camera are analyzed to545
521 compare the repeatability of the generated waves. First,546
522 the global wave behavior is visually compared. Then, the547
523 shape of the gas pocket and the location of the wave tip548
524 are determined. Finally, the cross-sectional area of the549
525 gas pocket in the plane of observation is determined and550
526 an estimate of the local gas velocity at the wave crest is551
527 derived.

528 1. Visual comparison

529 The qualitative repeatability is determined with a vi-555
530 sualization camera by comparing differences in image
531 intensity.³⁸ Here the global wave shape, as obtained with556
532 the visualization camera, is compared for two typical557
533 measurements (M225 and M228) shown in panels (a)558
534 and (b) of figure 4. The red (M225) and cyan (M228)559



576 Figure 4. (a)-(b) The back-projected side-view images of two
577 nominal identical waves are superimposed at two time steps,
578 where differences in intensity are indicated in red (M225) and
579 cyan (M228). The colors highlight the variance in wave shape.
580 The striations behind the wave crest result from refraction of
581 the light sheet at the wave crest, and are measure of the sub-
582 pixel variations present on the wave crest. Additionally, the
583 semi-ellipse fit of the gas pocket is shown for M225 (dotted
584 line) and M228 (dash-dotted line). The wave crest (\bar{x}_{wt}), el-
585 lipse center (\bar{x}_0), and the ellipse's semi-major and semi-minor
586 axes (R_x, R_y) are defined in window (a). The cross-sectional
587 area (A) of the gas pocket is defined in panel (b). The panels
588 (c) and (d) show the intensity variation between waves, with
589 respectively an averaged free surface variation of $L_2 \approx 5.0$ mm
and $L_2 \approx 3.8$ mm.

537 highlights show the difference in image intensity between
538 both measurements at two time steps $t = -28.0$, and $t =$
539 -16.0 ms with respect to the time of impact ($t = 0$ ms).
540 The wave crest development for a typical wave (i.e.,
541 M225) can be observed at different time steps in the sup-
542 plemental electronic material.

The free surface is determined at the side-wall of the
wave flume, where the width of a color band (i.e., red and
cyan areas) is a measure of the differences in global wave
shape. The width is estimated at the tip of the wave crest
(fig.10c) and the bottom of the trough (fig.10d). The dif-
ference in global wave shape is on average $L_2 \approx 4.4$ mm
for $t = -16.0$ ms at the indicated regions. Although,
these results must be interpreted with care, as variations
in image intensity arise from multiple sources (e.g., laser-
intensity fluctuations, a wetted or unwetted side wall).
The overall shape of the global wave is quite similar.
However, a more detailed analysis should be performed,
as the variability in impact pressure is also related to
small variations in gas pocket shape.⁷

2. Cross-sectional shape of the gas pocket

Initially, the focused wave resembles a plunging
breaker, which is used to define the gas pocket shape.
The area of the gas pocket is typically reported at the mo-
ment of impact or during the compression cycle, where
the gas pocket cross-sectional area is either fitted with
a semi-ellipse³⁶ or as a semi-circle.^{11,18} The area under-
neath a plunging breaker can also be approximated by
an ellipse with a constant aspect ratio³³, but the accu-
racy of this ellipse fit is a subject of debate for a plunging
breaker.³⁴ Here the gas pocket cross-sectional area is fit-
ted with a semi-ellipse constrained to the impact wall.

The parameters of the ellipse (i.e., the semi-major axis
 R_x , the semi-minor axis R_y , and the center-point \bar{x}_0) are
defined in panel (a) of figure 4. The ellipse semi-axes are
manually determined using the images of the visualiza-
tion camera, where the ellipse axes tend to correspond
to the horizontal and vertical tangent of the gas pocket
(fig. 4a-b). The ellipse center is defined by the x -location
of the wall and additionally the y -location of the vertical
tangent. The location of the tangent (i.e., vertical and
horizontal) is manually estimated. The manual estimate
is improved by detecting the maximum intensity gradient
over a line perpendicular to the tangent.

The semi-ellipse fit overlaps with the cross-sectional
area of the gas pocket of the visualization camera as
shown in panels (a) and (b) of figure 4. However, small
differences are observed near the wave crest and in the
trough of the gas pocket (fig. 4). The manual selec-
tion accuracy over repeated evaluations is approximately
0.3 mm and 2.2 mm (i.e., equivalent to 0.5 and 5.4 %
of the semi-major and semi-minor axes for a typical gas
pocket at the moment of impact) for respectively the R_x
and R_y axes. The uncertainty in the R_y component is
larger due to the reduced image intensity at the horizon-

tal tangent of the gas pocket (fig. 4b). This results in a variation of R_y over repeated experiments as shown in figure 5.

The semi-major and semi-minor axes are approximated by a linear function (fig. 5). The upward motion of the wave trough (i.e., the contact point of the wave and the wall) is defined by the derivative of the semi-minor axis (\dot{R}_y), and is approximately constant at -1.23 m s^{-1} . The wave speed is defined by the derivative of the semi-major axis (\dot{R}_x) and is conjectured to change. The wave speed is initially 2.38 m s^{-1} for $-80 \leq t \leq -40 \text{ ms}$, but it decreases to 2.00 m s^{-1} for $-40 \leq t \leq 0 \text{ ms}$. The averaged wave speed is 2.18 m s^{-1} , which is approximately equal to the shallow water phase speed ($\sqrt{gh_0} \approx 2.21 \text{ m s}^{-1}$). The aspect ratio of the ellipse is nearly constant at $R_x/R_y = 1.6$ for $-60 \leq t \leq -20 \text{ ms}$, which approximates the aspect ratio of $\sqrt{3}$ for plunging breakers.^{33,34,68} The velocity ratio is also relatively constant, which results in a velocity \dot{R}_y of approximately $\sqrt{gh_0}/3$.

The repeatability of the global wave is determined from the ellipse fit. First, the systematic error with respect to the linear fit is defined per measurement, which is on average 0.8 and 3.6 mm for respectively the semi-major (R_x) and semi-minor (R_y) axes. A measure of the wave shape repeatability is the random error, which is on average 1.1 and 1.7 mm for both axis. The higher random error of the semi-minor axis is a result of the detection method. Small variations in gas pocket size⁶²¹ are a source of variability in impact pressure.⁷ However,⁶²² the random error is negligible (i.e., 2.0 and 4.2 % of a

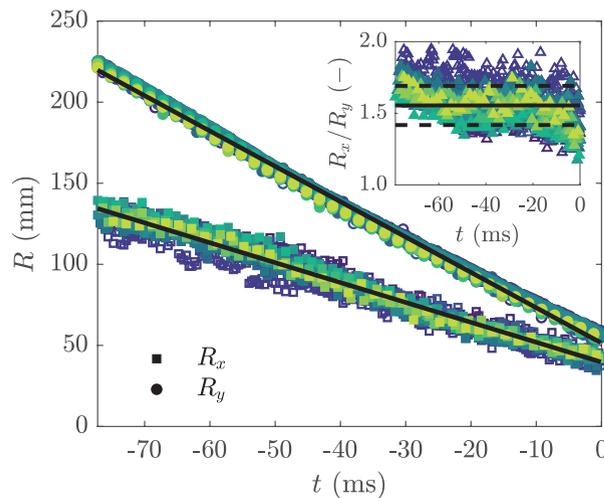


Figure 5. The semi-major (R_x) and semi-minor (R_y) axis of the fitted ellipse for the characterization of the observed gas pocket (see fig. 4) are shown, where the open markers define the non-repeatable waves of figure 3. The semi-minor and semi-major axis are approximated by a linear fit $R_y = -1.23t + 39.7$ and $R_x = -2.18t + 51.4$. The inset shows a nearly constant aspect ratio of $R_x/R_y \approx 1.6 \pm 0.1$ (for $-60 \leq t \leq 20 \text{ ms}$).

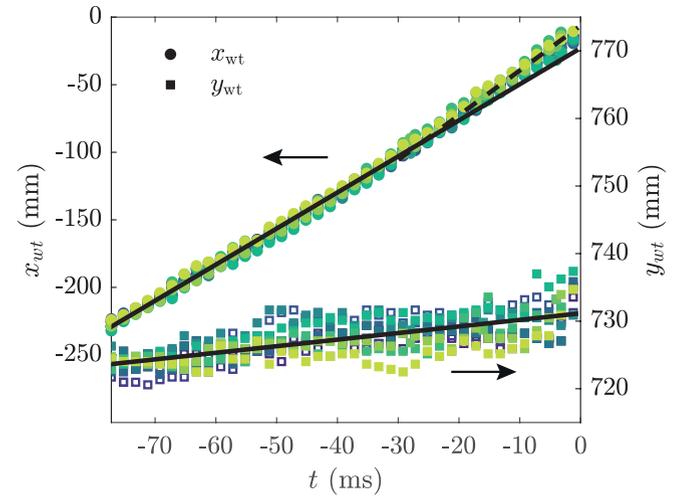


Figure 6. The wave tip coordinates (x_{wt}, y_{wt}) obtained with the manual fitting procedure from the visualization camera. The tip coordinate is approximated by a linear function in both $x_{wt} = 2.67t - 22.97$ and $y_{wt} = 0.10t + 731.2$. The closed markers indicate repeatable waves, based on the surface elevation data, whereas the open markers indicate non-repeatable waves.

typical gas pocket at the moment of impact); as such the global wave shape appears to be repeatable based on the gas pocket size.

3. Wave tip

The development of the plunging wave tip is determined from the images of the visualization camera. The wave tip is formed when the gradient of the free surface profile is large, which results in a pressure gradient in the fluid that accelerates a liquid jet horizontally.³² The wave tip becomes thinner and longer, while following a ballistic trajectory.³² In the present measurements the wave tip does not follow a ballistic trajectory, as the cross flow at the wave tip results in a drag force that counteracts the gravitational force.

The wave tip trajectory is determined with the detection method previously used for the ellipse axes. The tip coordinate is determined for every fifth time step ($\Delta t = 2.0 \text{ ms}$), which is sufficiently small to determine the global wave tip behavior. The wave tip is detected with an accuracy of approximately 0.96 and 0.31 mm for respectively the x and y -coordinate of the wave tip. Figure 6 shows the wave tip trajectory for both the x (x_{wt}) and y -coordinate (y_{wt}).

The wave tip trajectory appears to be nearly linear for both coordinates (fig. 6). The residual error of the linear fit is 3.6 and 2.1 mm for respectively the x and y -coordinate, which indicates repeatable wave-tip behavior. Furthermore, there is no clear distinction in wave tip behavior between the previously defined repeatable and

649 non-repeatable waves.

650 The components of the wave tip velocity are $\dot{x}_{wt} =$
 651 2.67 m s^{-1} and $\dot{y}_{wt} = 0.1 \text{ m s}^{-1}$, which results in a ris-
 652 ing wave tip as it approaches the impact wall. The ratio
 653 of wave tip and global wave velocity $\dot{x}_{wt}/\sqrt{gh_0}$ is ap-
 654 proximately 1.22, which is similar to the velocity ratio
 655 of a plunging breaker.⁷ The wave tip trajectory deviates
 656 from the linear fit for $-20 \leq t \leq 0 \text{ ms}$, which indicates
 657 an acceleration of the wave tip during the final stage be-
 658 fore impact. The acceleration of the wave tip is approx-
 659 imately $a \sim 100 \text{ m s}^{-2}$ based on $a \sim (2\Delta x)/\Delta t^2$ with
 660 $\Delta x \approx 15 \text{ mm}$ with respect to the linear fit of x_{wt} and
 661 $\Delta t \approx 18 \text{ ms}$ for $-19.2 \leq t \leq -1.2 \text{ ms}$ (fig. 6).

662 4. Gas pocket cross-sectional area

663 Small variations in the gas pocket shape can result
 664 in impact pressure variability.⁷ The gas pocket cross-
 665 sectional area is determined to define the global wave
 666 shape repeatability and estimate the local gas velocity in
 667 front of the wave crest. The variability in impact pres-
 668 sure due to the variation in gas pocket size is expected to
 669 be minimal, as the ellipse axis and wave-tip coordinate
 670 already indicate a repeatable global wave behavior. The
 671 gas pocket cross-sectional area is defined as the ellipse
 672 segment *underneath* the wave crest tip:

$$673 \quad A = \frac{1}{2} (\pi R_x R_y - A_s), \quad (1)_{700}$$

674 with A_s the area of the elliptical segment *above* the wave
 675 crest tip. The area of the elliptical segment is defined as
 676 follows:

$$677 \quad A_s = R_x R_y \left[\arccos \left(1 - \frac{h}{R_y} \right) - \right. \\ 678 \quad \left. \left(1 - \frac{h}{R_y} \right) \sqrt{2 \frac{h}{R_y} - \frac{h^2}{R_y^2}} \right], \quad (2)_{707}$$

679 with $h = R_y - (y_{wt} - y_0)$ the sector height of the ellip-
 680 tical segment (fig. 4). Figure 7 shows the calculated gas
 681 pocket cross-sectional $A - A(0)$ area, where the value at
 682 impact (A_0) is subtracted. The gas pocket cross-sectional
 683 area at impact is approximately $4.1 \times 10^3 \text{ mm}^2$ with a
 684 standard deviation of 6.5%. The power of the best fit
 685 function to the gas pocket cross-sectional area is 1.52
 686 which is approximately 3/2, as shown in the log-log in-
 687 set of figure 7. Furthermore, the non-repeatable waves,
 688 based on the free surface elevation, are indistinguishable
 689 from the results for the repeatable waves.

690 The gas velocity at the wave crest increases as the
 691 wave approaches the wall. The incompressible gas veloc-
 692 ity at the wave crest (V_g) follows from a control-volume
 693 attached to the ellipse

$$694 \quad V_g = \frac{1}{\Delta x} \dot{A} \sim |t|^{-0.48} \sim |t|^{-1/2} \quad (3)_{724}$$

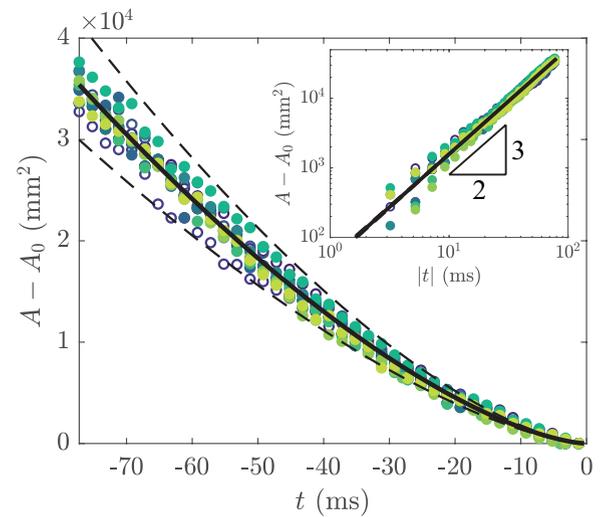


Figure 7. The gas pocket cross-sectional area is the area en-
 closed by the wave tip and the ellipse (fig. 4b). The gas
 pocket cross-sectional area is approximated by a power-law
 $A(t) - A(0) = 1.74|t|^{1.52}$, which is shown in the log-log inset.

where $\Delta x = \dot{x}_{wt} t \sim 1.2\sqrt{gh_0}t$ is the distance between
 the wave crest and the wall, and $\dot{A} = 2.64|t|^{0.52} \sim$
 $1.2\sqrt{gh_0}|t|^{1/2}$ is the temporal derivative of the cross-
 sectional area of the gas pocket. The gas can be consid-
 ered as incompressible for a Mach number ($M = V_g/c$)
 lower than 0.3. The gas in the cavity is incompressible
 for $V_g = |t|^{-1/2} = 0.3c$ or up to $|t| = (0.3c)^{-2} \approx 0.09 \text{ ms}$
 where c is speed of sound (343 m s^{-1} at standard
 conditions). The gas velocity at the wave crest ranges from
 $3.5 \leq V_g \leq 15.8 \text{ m s}^{-1}$ for $-80 \leq t \leq -0.4 \text{ ms}$. The
 global wave does not appear to decelerate through com-
 pression of the gas pocket.

694 C. Local wave behavior

The variation in impact pressure of nominal identical
 waves is caused by the development of free surface in-
 stabilities on the wave crest.^{26,45,54} Here, a stereo-PLIF
 system is used to accurately measure the free surface of
 the wave crest and to determine both the development of
 instabilities and the wave tip deflection. The wave crest
 is determined with a smaller field-of-view than the visu-
 alization camera, which results in a higher resolution and
 accuracy of the free surface measurements. The system
 enables free surface measurements in the center plane
 of the wave flume where side-wall effects (i.e., friction⁵⁸
 and wetting⁶⁷) do not directly influence the measurement
 of the wave shape. First, the visualization camera and
 stereo-PLIF system are compared. Then, the temporal
 development of a local wave crest is discussed both in the
 context of measurement accuracy and wave tip behavior.
 Thereafter, the free surface profile is compared over sev-
 eral time steps. Finally, the wave tip and the variability

726 due to variations in wave shape are discussed.

727 1. Global and local measurements

728 The stereo-PLIF data of two typical measurements
 729 (e.g., M225 and M228) are compared with the images of
 730 the visualization camera that are obtained simultane-
 731 ously (fig. 8). The stereo-PLIF results are superimposed
 732 (continuous line) on the combined side-view images of
 733 the visualization camera by matching the origin of both
 734 coordinate systems. The ellipse fit (dashed and dotted
 735 line) is also included, which shows a qualitative agree-
 736 ment with the stereo-PLIF results.

737 The wave-crest profile at the center plane (stereo-PLIF
 738 data) is similar to that at the side wall (visualization
 739 data). The large field-of-view of the visualization cam-
 740 era combined with the relatively small focal length lens
 741 results in a perspective view of the wave crest, which
 742 emphasizes the spanwise differences of the wave crest (see
 743 the supplemental electronic material). For example, a
 744 liquid filament is suspended from the wave crest at the
 745 side-wall, whereas the filament is absent on the rest of
 746 the wave crest (i.e., the spanwise direction). The side-
 747 wall effects, such as friction⁵⁸ and wetting⁶⁷, limit the
 748 use of side-view measurements for quantitative repeata-
 749 bility studies of the wave tip behavior.

750 The application of a stereo-PLIF system in the wave
 751 flume is not without problems. For example, the liquid
 752 exerts a large pressure on the wall when it impacts, which⁷⁸¹
 753 results in vibrations in the camera system. The vibra-
 754 tions can introduce a misalignment in the camera system
 755 and a self-calibration procedure is needed to correct for
 756 the misalignment.

757 Additionally, loss of information occurs when a free
 758 surface undulation casts a shadow. This effect is observed
 759 at the top of the wave crest where the light-sheet skims
 760 over the free surface and obstructs the backward side of
 761 the wave (fig. 8). The wave tip also blocks the inside of
 762 the gas pocket as it plunges over the top. A straight line
 763 results at the blocked segment, that connects the wave
 764 tip and the backward face of the gas pocket (fig. 8).
 765 The wave tip is accurately determined by the light-sheet
 766 cut-off, whereas the accuracy decreases at the wave top.

767 The stereo-PLIF system enables a quantitative com-
 768 parison of repeated measurements, whereas the side-view
 769 camera only enabled a qualitative comparison. A zoom
 770 of the free surface profile shows the difference between
 771 two selected measurements M225 and M228 (fig. 4 panels
 772 c and d). The averaged difference between the free surface
 773 profiles as determined by the stereo-PLIF measurements
 774 is $L_2 = 2.45 \pm 1.49$ mm over the entire field-of-view. The
 775 difference was previously determined to be $L_2 = 4.4$ mm
 776 for $t = -16.0$ ms based on the visualization camera. The⁷⁸²
 777 quantitative difference determined with the stereo-PLIF⁷⁸³
 778 measurements is lower, even for a later time step. The⁷⁸⁴
 779 stereo-PLIF and visualization measurements show that⁷⁸⁵
 780 the wave is repeatable on a global scale. 786

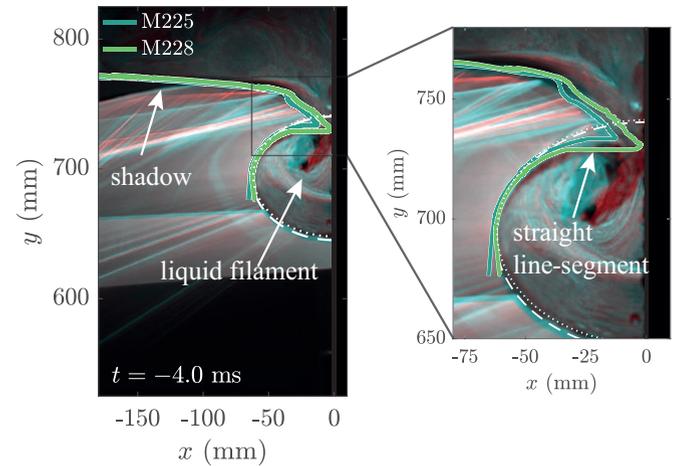


Figure 8. The side-view images of two nominal identical waves are superimposed for $t = -4.0$ ms and combined with the free surface profile from the stereo-PLIF measurement (continuous line). The ellipse fit from the visualization camera is also included. The refraction of the light sheet at the wave crest results in striations. These striations present a sub-pixel measure of the wave crest variability. However, they are neglected when comparing the visualization and stereo-PLIF measurements. (b) A zoom on the wave crest shows the difference between both waves and the formation of liquid filaments at the side-wall.

787 2. Temporal development

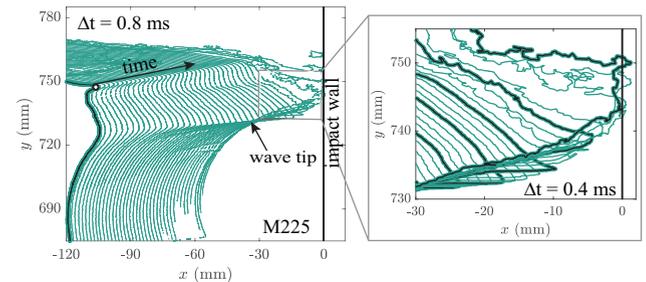


Figure 9. The free surface stereo-PLIF data for experiment M225 is consistent over multiple time steps ($-29.6 \leq t \leq 2.8$ ms) at a reduced temporal resolution ($\Delta t = 0.8$ ms). The marker shows the location of the *liquid jet*, that is initially ejected outside the field-of-view of the stereo-PLIF measurement (see supplemental electronic material). The zoom shows the free surface stereo-PLIF data at its actual temporal resolution for $-7.2 \leq t \leq 2.8$ ms with an increased line width for every fourth time step.

Figure 9 shows the temporal development of the free surface for a typical case (M225) at two different time steps, which show the local ($\Delta t = 0.8$ ms) and detailed ($\Delta t = 0.4$ ms) free surface behavior. The *local wave behavior* shows the displacement of a small amplitude *liq-*

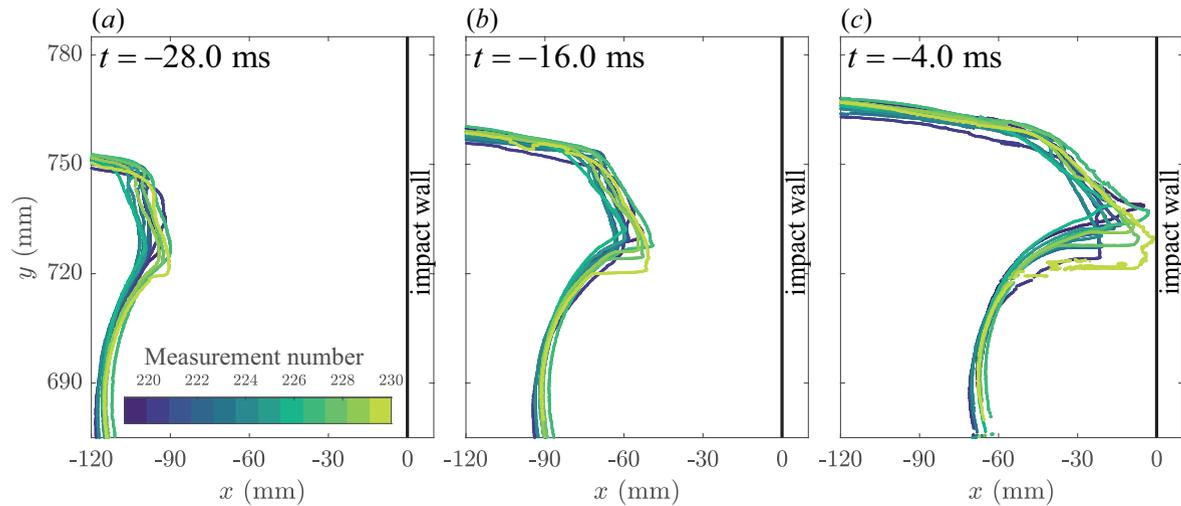


Figure 10. The stereo-PLIF data of all 12 measurements for three time steps, additionally a movie of the wave crest development is available as electronic supplemental material. (a)-(b) Initially all 12 measurements tend to overlap. (c) The overlap between the different measurements reduces significantly as the waves approach the wall. The variation in free surface profile concentrates near the wave tip, which is influenced by an increase in gas velocity.

PLEASE CITE THIS ARTICLE AS DOI:10.1063/1.50016467

787 *uid jet*, which is initially ejected from the wave crest (e.g.,⁸¹⁶
 788 outside of the field-of-view of the stereo-PLIF measure-⁸¹⁷
 789 ments) as shown in the supplemental electronic material.⁸¹⁸
 790 The disturbance (i.e., the *liquid jet*) is displaced to the⁸¹⁹
 791 back of the wave crest by the gas flow over the wave⁸²⁰
 792 crest. The growth and displacement of the disturbance⁸²¹
 793 is continuous over time, which is indicative of the tempo-⁸²²
 794 ral consistency of the stereo-PLIF data (e.g., the initial⁸²³
 795 disturbance is physically there).⁸²⁴

796 The details of the wave crest moments before impact⁸²⁵
 797 are displayed in panel b (fig. 9). Initially, a liquid jet⁸²⁶
 798 is ejected from the wave crest as the gradient of the free⁸²⁷
 799 surface profile increases, which results in a large pressure⁸²⁸
 800 gradient in the fluid.^{32,68} In this measurement a liquid jet⁸²⁹
 801 jet is ejected at two times, which results in the initial⁸³⁰
 802 disturbance (i.e., defined by the marker) and the wave⁸³¹
 803 tip. The wave tip of a plunging breaker follows a ballistic⁸³²
 804 trajectory, but here the wave tip is displaced upwards⁸³³
 805 by the air flow from the gas pocket. The gas velocity⁸³⁴
 806 at the wave crest increases as the wave approaches the⁸³⁵
 807 wall, which results in a wave tip that is stretched and⁸³⁶
 808 deflected.⁵¹ The formation of spray (i.e., droplets) and⁸³⁷
 809 ligaments results in a higher noise level in the stereo-⁸³⁸
 810 PLIF data, which is observed in the last few time steps
 811 of panel b.

812 3. Local repeatability

813 In the previous analysis of the system variability sev-
 814 eral repeatable and non-repeatable waves were identified.
 815 The stereo-PLIF data for both the repeatable and non-

repeatable waves is presented (fig. 10). The waves ini-
 tially ($t = -28.0$ ms) overlap and the variation increases
 as the waves approach the wall. The variability concen-
 trates in the vicinity of the wave tip for all waves. Ini-
 tially, the formation of instabilities is not observed, both
 in the processed free surface profile and in the original
 shadowgraph of the stereo-PLIF images. However, at
 later stages (fig.10c), the wave tip but is deflected differ-
 ently. This is hypothesized to be caused by an interaction
 of the gas flow and interface around the wave crest.

The variability of the free surface profiles is quanti-
 fied. First, the difference in wave crest height ($-120 \leq$
 $x \leq -115$ mm) is determined from stereo-PLIF data at
 $t = -28.0$ ms (fig. 10 a). The standard deviation in
 the height of all waves is approximately 1 mm, whereas
 the nominal identical waves show a standard deviation
 of approximately 0.9 mm. The difference between both
 sets (i.e., repeatable and non-repeatable waves) is negli-
 gible, which is also confirmed by the initial visual overlap
 of all waves (fig. 9a). However, the variation in free
 surface profile is more significant at the wave crest (i.e.,
 $730 \leq y \leq 740$ mm) for $t = -28.0$ ms with a standard
 deviation of approximately 3.5 mm.

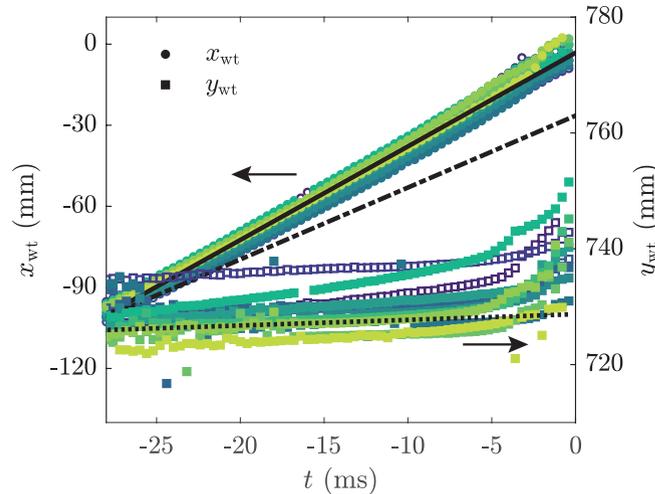


Figure 11. The wave tip coordinates for $-28 \leq t \leq 0$ ms ob-888
tained from the stereo-PLIF data. The wave tip coordinates889
are, initially, fitted with a linear function $x_{wt} = 3.31t - 6.36$ 890
(continuous line) and $y_{wt} = 0.10t + 731.2$ (dotted line). The891
wave tip has accelerated in the x -direction compared to the892
data from the visualization camera (dash-dotted line). The893
 y_{wt} coordinate deviates from the linear fit of the visualization894
camera for $t > -5$ ms, which shows an acceleration of the895
wave tip coordinate in the y -direction.896

839 A parametric representation of the free surface pro-888
files is determined with an arc-length method. The889
curve is parameterized with a fixed number of elements890
($N = 2500$), which results in a spacing of approximately891
0.15 mm. A Euclidean distance metric (L_2) is computed892
from the difference between parametric curves and their893
respective averaged free surface profile. The distance894
metric increases from approximately $L_2 = 1.5$ mm at895
 $t = -28.0$ ms, to $L_2 = 5.1$ mm at $t = -16.0$ ms, and896
to $L_2 = 8.0$ mm at $t = -4.0$ ms. The Euclidean norm897
(L_2) confirms the buildup of variability in wave shape as898
the wave approaches the wall. The variation is most ob-899
vious at the wave tip, whereas the global wave (i.e., the900
wave top and the wave trough) remain similar, which is901
additionally supported by the movie in the supplemental902
material.903

839 The wave tip variation is further investigated to de-913
termine its possible effect on the pressure variability,914
where the extreme position of the wave tip is defined915
as the maximum x -location of the stereo-PLIF profile916
(fig. 11). First, the wave tip velocity in the x -direction917
 $\dot{x}_{wt} = 3.31$ m s $^{-1}$ is higher than previously determined918
from the visualization camera, $\dot{x}_{wt} = 2.66$ m s $^{-1}$. A919
deviation from the linear fit was already observed for920
 $-20 \leq t \leq 0$ ms, which indicated an acceleration of the921
wave tip. However, the wave tip was, for $-20 \leq t \leq 0$ ms,922
observed by either the perspective of the visualization923
camera or the formation of a liquid filament at the side924
wall. The wave tip velocity in the x -direction is sig-925
nificantly higher $\dot{x}_{wt} \sim 1.5\sqrt{gh_0}$ for $-20 \leq t \leq 0$ ms,926
which is higher than the wave tip velocity of a plung-

870 ing breaker ($\dot{x}_{wt} \sim 1.2\sqrt{gh_0}$).⁷ However, the wave tip913
velocity in the x -direction is comparable to that of a914
plunging breaker that impinges on the free surface in915
front the wave tip.⁶⁸ In the y -direction the wave tip916
trajectory is altered by the gas flow escaping from the gas917
pocket, which is obvious from the acceleration in the y -918
direction for $-5 \leq t \leq 0$ ms. The wave tip trajectory in919
the y -direction is not comparable to that of a plunging920
breaking, which typically shows a ballistic trajectory.³²921
The wave velocity at the center plane can increase due922
to wave focusing of a concave wave crest^{4,11} or Bernoulli923
suction⁵⁷, where the air pressure drops due to an increase924
in velocity at the wave crest.

870 A small amplitude wave grows on the wave crest for913
every wave impact, which is either caused by the large914
gradient of the free surface profile or by the Bernoulli suc-915
tion (i.e., which is equivalent to the growth of a Kelvin-916
Helmholtz instability).^{32,49,57} However, the growth of917
a Kelvin-Helmholtz instability is in this study not ex-918
pected, as there are no small scale disturbances ob-919
served on both the reconstructed free surface profiles920
and the shadowgraphs of the original stereo-PLIF im-921
ages. Nonetheless, the small amplitude wave is defined922
as the wave tip, that is the maximum x -coordinate of the923
wave crest.924

870 The wave tip is observed to grow as it approaches the913
wall, which results in a thinner and longer wave tip.³²914
The length change of the wave tip is linear, up to ap-915
proximately $t = -5$ ms, with respect to the global wave916
tip velocity $\dot{x}_{wt} \sim 1.2\sqrt{gh_0}$. In this time the tip stretches917
approximately $L \sim (1.5 - 1.2)\sqrt{gh_0}\Delta t \sim 15$ mm, which918
is, based on visual inspection, a good estimate of the tip919
length. The stretched wave tip resembles a liquid sheet.920

870 Villermaux and Clanet⁵⁹ studied the break up of a913
liquid sheet formed by the impact of a jet on a circular914
disk. The liquid sheet expands into the surrounding915
air, which results in a shear force that destabilizes916
the sheet by an initial Kelvin-Helmholtz instability. The917
waves that result from the Kelvin-Helmholtz instability918
induce an additional motion at the tip of the liquid sheet.919
This finite motion at the tip of the liquid sheet provides920
the acceleration required for a secondary Rayleigh-Taylor921
instability.922

870 A similar type of mechanism is observed to trigger the913
development of a span-wise instability on the tip of a914
plunging breaking wave. The wave tip is stretched into915
a thin liquid sheet, which is destabilized by an initial916
Kelvin-Helmholtz instability. This is observed as a finite917
amplitude wave that forms on the wave tip for $t > -5$ ms918
(fig. 12 panels a and b). The finite amplitude wave,919
combined with the acceleration of the wave tip by the gas920
flow, results in an acceleration that triggers a Rayleigh-921
Taylor instability (fig.12c). The wavelength of the span-922
wise instability (i.e., liquid filaments or fingers) is defined923
as924

$$\lambda_{\perp} \sim (\gamma/\rho_a \dot{x}_{wt}^2)(\rho_a/\rho_l)^{1/3} \quad (4)$$

870 where $\dot{x}_{wt} = 1.5\sqrt{gh_0}$ is the wave tip velocity, γ is the

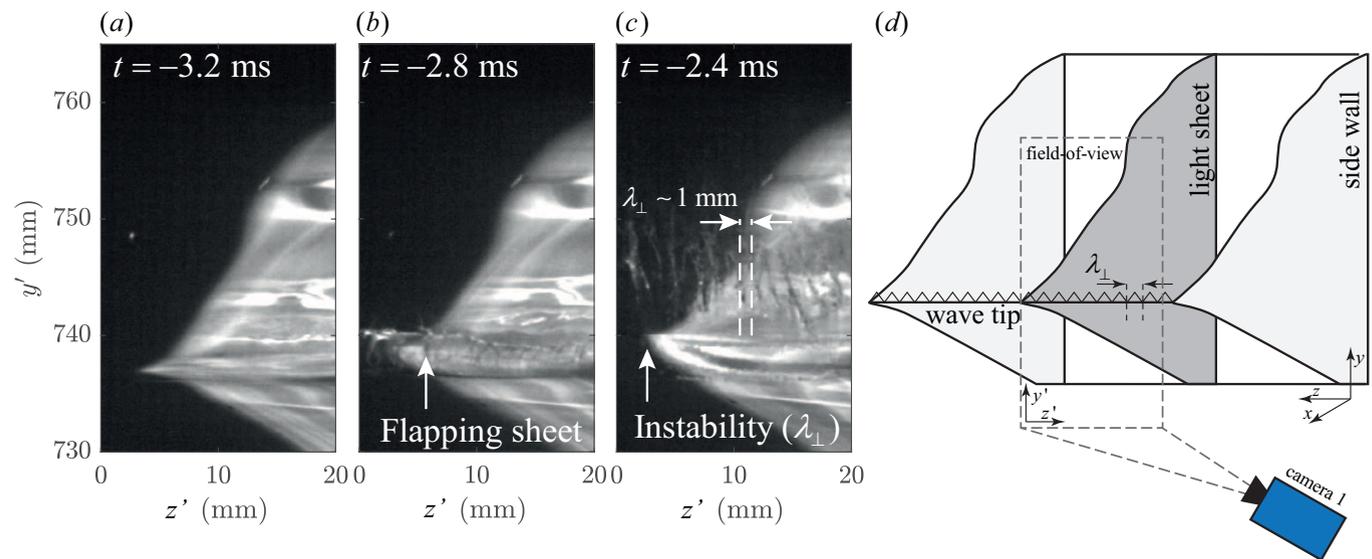


Figure 12. The images of camera 1 are back-projected to a plane parallel to the impact wall (a)-(c). (a) A typical wave (M221) approaches the wall and the tip elongates. (b) The elongated wave tip is destabilized by a shear instability. (c) A flapping liquid sheet develops with a spanwise wavelength (λ_{\perp}) defined by the Rayleigh-Taylor instability. (d) The images are acquired with camera 1 of the stereo-PLIF system (fig.1). The camera images the wave from the front at an angle with respect to the light sheet. Note the difference in coordinate system compared to that defined in figure 1.

927 surface tension of the air-water interface (72.3 mN m^{-1}),⁹⁵⁹
 928 ρ_a is the gas density (1.23 kg m^{-3}), and ρ_l is the liq-⁹⁶⁰
 929 uid density (998 kg m^{-3}) at standard atmospheric con-⁹⁶¹
 930 ditions (1 bar, 20°C). The spanwise wavelength of ap-⁹⁶²
 931 proximately $\lambda_{\perp} \sim 1 \text{ mm}$ agrees well with the visually⁹⁶³
 932 observed finger spacing (fig. 12c).⁹⁶⁴

933 In previous work the impact pressure variability was⁹⁶⁵
 934 shown to depend on the density ratio (ρ_a/ρ_l) and the⁹⁶⁶
 935 surface tension. A higher density ratio results in more⁹⁶⁷
 936 well-developed (i.e., larger) liquid filaments.^{39,40} Further-⁹⁶⁸
 937 more, the free surface at the wave crest fragments earlier⁹⁶⁹
 938 for lower values of the surface tension.³⁸ The increase in⁹⁷⁰
 939 liquid filaments at higher density ratios and the spray⁹⁷¹
 940 formation at lower surface tension values are both cap-⁹⁷²
 941 tured by the span-wise wavelength of the Rayleigh-Taylor⁹⁷³
 942 instability in Eq. (4). A mechanism for the development⁹⁷⁴
 943 of instabilities is presented, where a flapping liquid sheet⁹⁷⁵
 944 develops into liquid-filaments.⁴⁵ Furthermore, the liquid-⁹⁷⁶
 945 filaments are accelerated by the gas flow from the gas⁹⁷⁷
 946 pocket and eventually break-up in small droplets due to⁹⁷⁸
 947 a capillary instability of the liquid filament.⁶⁹⁹⁷⁹

948 The variability in wave impact pressure is linked to⁹⁸⁰
 949 the variation in wave impact location. However, the for-⁹⁸¹
 950 mation of liquid filaments decreases the accuracy of the⁹⁸²
 951 wave tip detection prior to impact (i.e., close to the wall).⁹⁸³
 952 The variation in wave impact location is, therefore, de-⁹⁸⁴
 953 termined just prior to the formation of a flapping liquid-
 954 sheet. The impact location is determined over a small
 955 time interval ($\Delta t = 2 \text{ ms}$) to improve the reliability of
 956 the measured coordinate. Figure 13 displays the varia-
 957 tion in vertical wave tip location for $-6.0 \leq t \leq -4.0 \text{ ms}$,
 958 which is an indication of the variation in wave impact lo-

cation.

The variation in vertical wave tip location is significant on a global scale with a standard deviation of 4 mm (i.e., 0.5 % of the typical wave height). The membrane surface ($d \sim 1 - 5.5 \text{ mm}$)^{19,58} of a typical pressure sensor is small compared to the variation in vertical wave tip location. Even for large ($d \sim 9.5 \text{ mm}$) pressure sensor membranes the integrating effect of the surface area is not sufficient to remove all pressure variability.¹⁷ Furthermore, the physical spacing of the pressure sensor, which is typically on the order of 20 mm^{19,37}, limits the possibility of detecting these small wave tip variations. The variation in vertical wave tip location is similar for the other, not reported, measurements. However, the measurements cannot be combined due to the significant day-to-day variations.

Additionally very close to the wall ($x/h_0 \lesssim 0.18$), the wave tip accelerates to about $1.5\sqrt{gh_0}$ compared to the global wave velocity of $1.2\sqrt{gh_0}$. The pressure sensor membrane is hit with either the wave tip velocity or the global wave velocity, which can result in a pressure difference of approximately 25%. The variation in pressure is similar to previous reported values for nominal identical waves.^{18,37} The variation in wave tip velocity due to either wave focusing or Bernoulli suction is a source of variability in impact pressure.

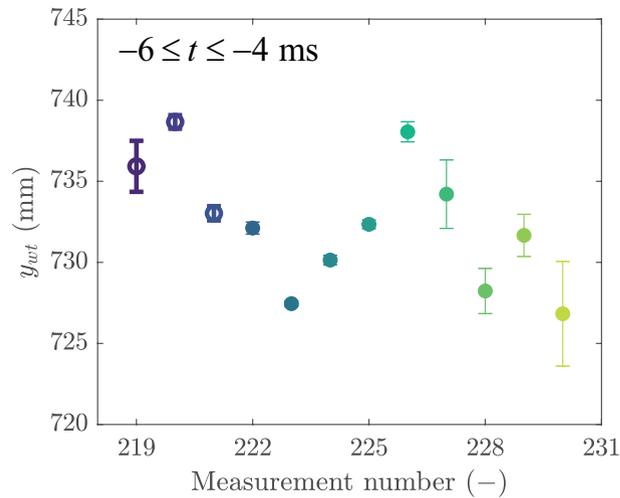


Figure 13. The variation in vertical wave tip location (y_{wt}) for $-6.0 \leq t \leq -4.0$ ms. The open markers are non-repeatable waves (i.e., outliers based on the surface elevation measurements) and closed markers are repeatable waves.

The source of impact pressure variation is a combination of *system* and *hydrodynamic* variability, but even for well-repeated waves (i.e., with insignificant *system variability*) a significant wave tip variability is observed. The variability in vertical wave tip location over repeated waves on a single day is shown to be significant compared to typical pressure membrane diameters (i.e. $d_p \approx 1 - 5.5$ mm). Furthermore, this variation is observed over several other days with a similar order of magnitude. The *hydrodynamic* variability is, even when the waves are well repeated, a source of pressure variability. The shear-driven flapping motion of the liquid sheet results in significant variability in impact location, which also triggers a Rayleigh-Taylor type of instability along the spanwise direction of the wave. The presented mechanism is probably one of the many types of instabilities that can occur on the wave crest, but for the reported gas pocket impact it occurs over a significant range of wave shapes. The reported measurements can be used for physical and numerical model validation.

IV. CONCLUSION

Repeated focused wave impacts on a vertical wall are reported. The generation of repeatable focused waves is not trivial. A limited number (i.e., $N = 12$) of the total set of 32 measurements is reported, as the day-to-day variations limit the detailed comparison. Therefore, the experimental variability (i.e., system variability) is reported in detail, which indicates that the wave generation is well-repeated over a single day. Several repeatable waves are identified ($N = 9$) based on the surface elevation measurements. These repeatable waves are studied and compared to the remaining non-repeatable

waves ($N = 3$).

The global wave behavior prior to impact is studied with a visualization camera. The cross-sectional area of a large gas pocket impact is approximated by a semi-ellipse constrained to the impact wall. The aspect ratio of the ellipse is relatively constant $R_x/R_y = 1.6$ ($\sim \sqrt{3}$), which is comparable to that of a plunging breaker.³³ Initially, the global wave behavior is also comparable to that of a plunging breaker, as both have a similar wave velocity ($\sqrt{gh_0}$) and wave tip velocity ($1.2\sqrt{gh_0}$). However, the trajectory of the wave tip does not resemble that of a plunging breaker. The drag at the wave crest, due to the escaping gas velocity, partially counteracts the gravitational force. Furthermore, the wave tip accelerates to a velocity of $1.5\sqrt{gh_0}$ as it approaches the wall ($x/h_0 \lesssim 0.18$).

Moments before impact the wave tip is deflected by the strong gas flow at the wave crest. The wave tip resembles a liquid sheet, that is destabilized by an initial Kelvin-Helmholtz instability. A flapping liquid sheet develops and the acceleration of the tip triggers a Rayleigh-Taylor instability. The spanwise wavelength of the Rayleigh-Taylor instability is well approximated by $\lambda_{\perp} \sim (\gamma/\rho_a \dot{x}_{wt}^2)(\rho_a/\rho_l)^{1/3}$. The Rayleigh-Taylor instability is one of the free surface instabilities that can be a source of wave impact pressure variability. Furthermore, the flapping liquid-sheet is an indication of an instability that results in pressure variability with varying density ratio (ρ_a/ρ_l) and surface tension (γ). The other, not reported, measurements show a similar wave crest development with a flapping liquid-sheet that triggers a Rayleigh-Taylor instability.

In previous work the variability in impact pressure is often attributed to Kelvin-Helmholtz type instabilities at the wave crest.⁴ The current work shows that the variability in impact location is initially drag induced, with a standard deviation in impact location of approximately 0.5% compared to a wave height of 732.4 mm. The variation in impact location is large compared to typical contemporary pressure sensor sizes. A shear-driven flapping liquid sheet develops moments before impact, which delays the impact time and triggers a Rayleigh-Taylor instability that forms equally spaced liquid filaments. However, the variability in impact height already exists before the formation of the liquid filaments. The liquid filaments can impact the pressure sensor, although, it is more likely that the wave tip will directly impact the pressure sensor. The acceleration of the wave tip compared to the wave crest and global wave presents a more likely explanation of the variance in impact pressure.

SUPPLEMENTARY MATERIAL

See supplementary material for a movie of a typical (i.e., M225) wave impact and the detailed structure of the wave tip for the 12 reported measurements.

1071 **ACKNOWLEDGMENTS**

1072 This work is part of the public-private research pro-
1073 gram Sloshing of Liquefied Natural Gas (SLING) project
1074 P14-10. The support by the Netherlands Organization
1075 for Scientific Research (NWO) Domain Applied and En-
1076 gineering Sciences, and project partners is gratefully ac-
1077 knowledged. The authors are grateful to the lab techni-
1078 cians at the Environmental Fluid Mechanics Laboratory
1079 of the Delft University of Technology for their assistance
1080 with the experimental setup and their support during the
1081 experiments.

1082 **DATA AVAILABILITY**

1083 The data that support the findings of this study are
1084 available from the corresponding author upon reasonable
1085 request.

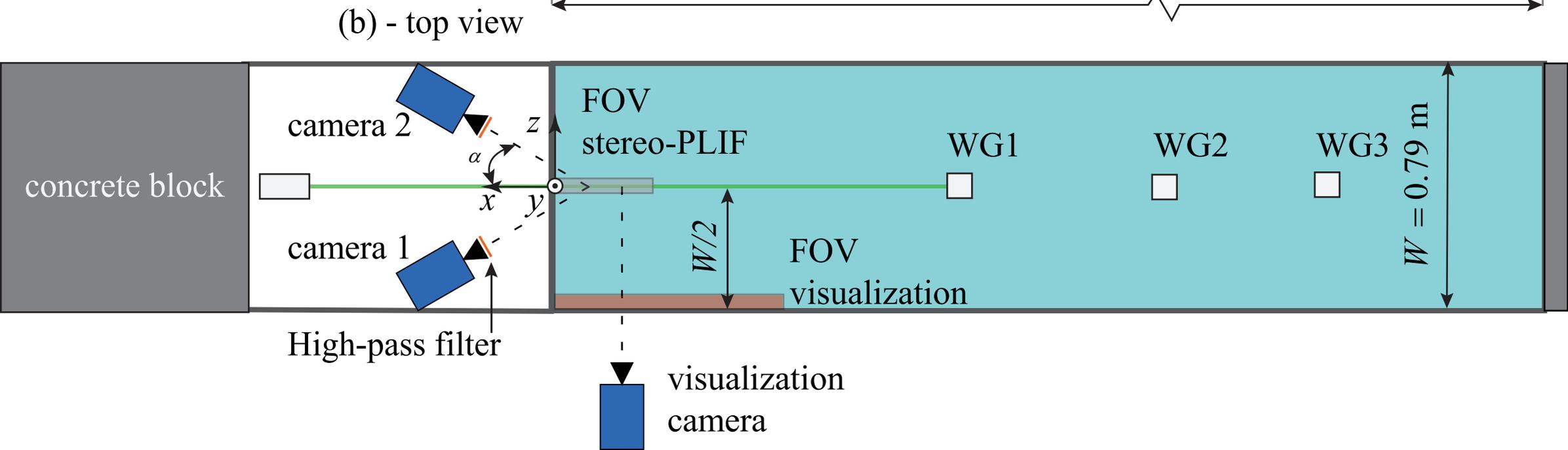
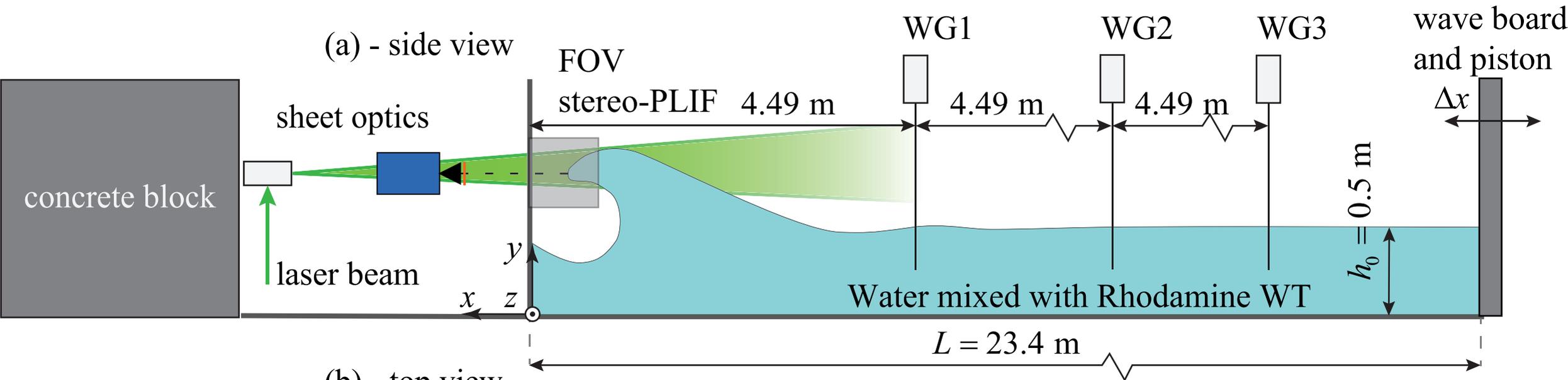
1086 **REFERENCES**

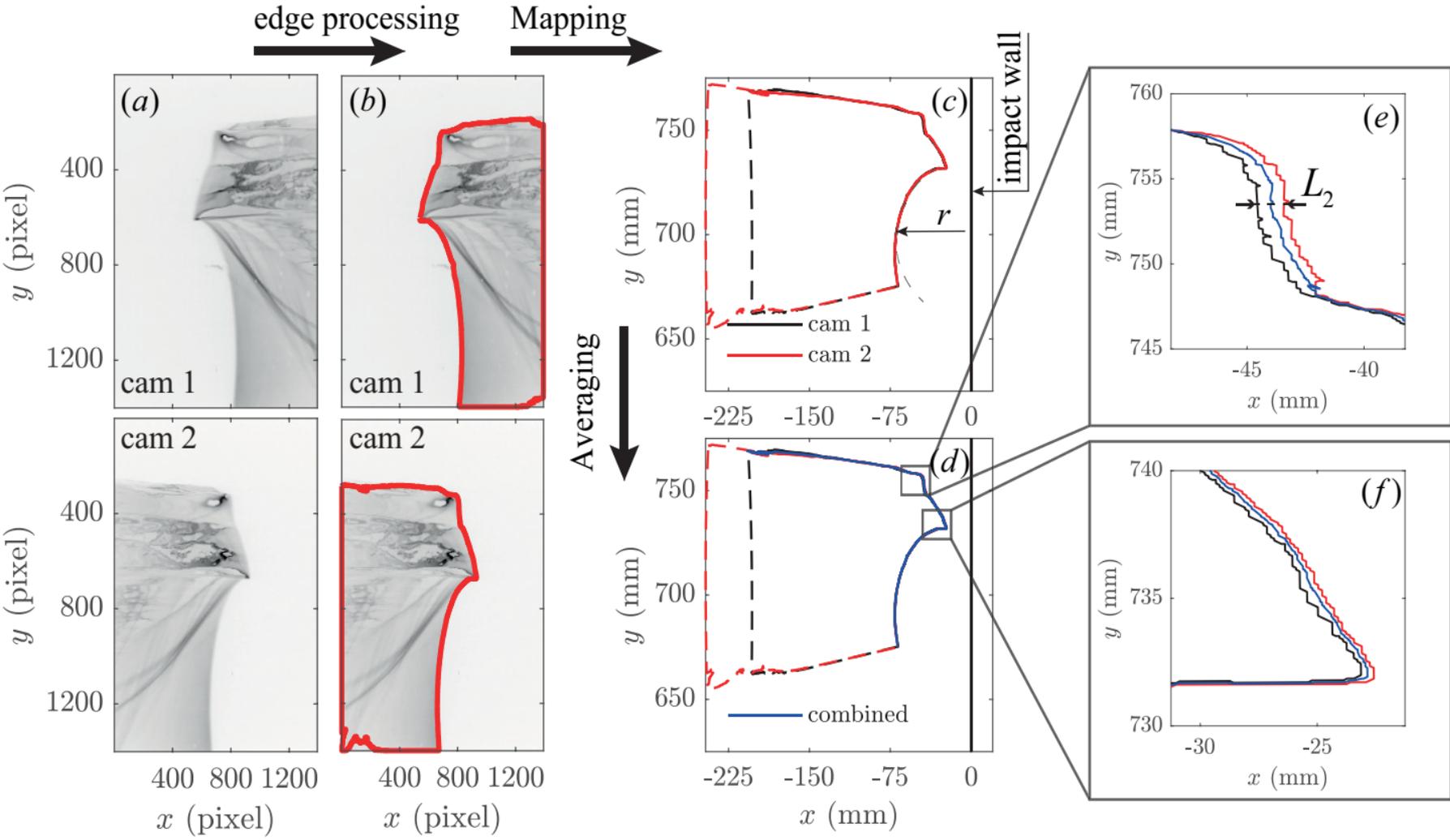
- 1087 ¹R. Aronietis, C. Sys, E. van Hassel, and T. Vanel
1088 slander, “Forecasting port-level demand for LNG
1089 as a ship fuel: the case of the port of antwerp,”
1090 *J. Ship Trade* **1** (2016), 10.1186/s41072-016-0007-1.
1091 ²L. Delorme, A. Colagrossi, A. Souto-Iglesias, R. Zamora
1092 Rodríguez, and E. Botia-Vera, “A set of canonical problems in
1093 sloshing, part I: Pressure field in forced roll—comparison between
1094 experimental results and SPH,” *Ocean Eng.* **36**, 168–178 (2009).
1095 ³B. Buchner and T. Bunnik, “Extreme Wave Effects on Deepwater
1096 Floating Structures,” in *Offshore Technology Conference* (2007).
1097 ⁴D. H. Peregrine, “Water-wave impact on walls,”
1098 *Annu. Rev. Fluid Mech.* **35**, 23–43 (2003).
1099 ⁵M. Ancellin, L. Brosset, and J.-M. Ghidaglia, “Nu-
1100 merical simulation of wave impacts with interfa-
1101 cial phase change: An isothermal averaged model,”
1102 *Eur. J. Mech. B. Fluids* **72**, 631–644 (2018).
1103 ⁶R. A. Bagnold, “Interim report on wave-pressure research,”
1104 *Inst. Civ. Eng.*, 202–226 (1939).
1105 ⁷M. Hattori, A. Arami, and T. Yui, “Wave impact pressure on
1106 vertical walls under breaking waves of various types,” *Coast. Eng.*
1107 **22**, 79–114 (1994).
1108 ⁸P. A. Blackmore and P. J. Hewson, “Experiments on full-scale
1109 wave impact pressures,” *Coastal Eng.* **8**, 331–346 (1984).
1110 ⁹G. N. Bullock, A. R. Crawford, P. J. Hewson, M. J. A. Walkden,
1111 and P. A. D. Bird, “The influence of air and scale on wave impact
1112 pressures,” *Coastal Eng.* **42**, 291–312 (2001).
1113 ¹⁰G. Cuomo, W. Allsop, T. Bruce, and J. Pearson,
1114 “Breaking wave loads at vertical seawalls and breakwaters,”
1115 *Coastal Eng.* **57**, 424–439 (2010).
1116 ¹¹A. Wang, C. M. Ikeda-Gilbert, J. H. Duncan, D. P. Lathrop,
1117 M. J. Cooker, and A. M. Fullerton, “The impact of a deep
1118 water plunging breaker on a wall with its bottom edge close to
1119 the mean water surface,” *J. Fluid Mech.* **843**, 680–721 (2018).
1120 ¹²B. Hoffland, M. Kaminski, and G. Wolters, “Large scale wave
1121 impacts on a vertical wall,” *Coast. Eng. Proc.*, 15 (2010).
1122 ¹³O. Kimmoun, A. Ratouis, and L. Brosset, “Sloshing and scaling
1123 experimental study in a wave canal at two different scales,” in
1124 *Int. Ocean Polar Eng. Conf., 20th, 20-25 June, Beijing, China*
1125 (2010).
1126 ¹⁴M. J. Cooker and D. H. Peregrine, “Pressure-impulse theory for
1127 liquid impact problems,” *J. Fluid Mech.* **297**, 193–214 (1995).
1128 ¹⁵E. de Almeida and B. Hoffland, “Validation of pressure-
1129 impulse theory for standing wave impact loading on
1130 vertical hydraulic structures with short overhangs,”
1131 *Coastal Eng.* **159**, 103702 (2020).
1132 ¹⁶A. Ghadirian and H. Bredmose, “Pressure impulse the-
1133 ory for a slamming wave on a vertical circular cylinder,”
1134 *J. Fluid Mech.* **867** (2019), 10.1017/jfm.2019.151.
1135 ¹⁷Y. K. Song, K.-A. Chang, Y. Ryu, and S. H. Kwon, “Experi-
1136 mental study on flow kinematics and impact pressure in liquid
1137 sloshing,” *Exp. Fluids* **54** (2013), 10.1007/s00348-013-1592-5.
1138 ¹⁸H. Bogaert, S. Léonard, L. Brosset, and M. L. Kaminsk, “Slosh-
1139 ing and scaling: results from the sloshel project,” in *Int. Ocean*
1140 *Polar Eng. Conf., 20th, 20-25 June, Beijing, China* (2010).
1141 ¹⁹W. Lafeber, H. Bogaert, and L. Brosset, “Comparison of wave
1142 impact tests at large and full scale: results from the Sloshel
1143 project,” in *Int. Ocean Polar Eng. Conf., 22nd, 17–22 June,*
1144 *Rhodes, Greece* (2012).
1145 ²⁰W. Lafeber, H. Bogaert, and L. Brosset, “Elementary loading
1146 processes (ELP) involved in breaking wave impacts: findings from
1147 the Sloshel project,” in *Int. Ocean Polar Eng. Conf., 22nd, 17–*
1148 *22 June, Rhodes, Greece* (2012).
1149 ²¹Z.-J. Wei, O. M. Faltinsen, C. Lugni, and Q.-J. Yue, “Sloshing-
1150 induced slamming in screen-equipped rectangular tanks in
1151 shallow-water conditions,” *Phys. Fluids* **27**, 032104 (2015).
1152 ²²Y. Wei, T. Abadie, A. Henry, and F. Dias, “Wave interaction
1153 with an oscillating wave surge converter. part II: Slamming,”
1154 *Ocean Eng.* **113**, 319–334 (2016).
1155 ²³A. Henry, T. Abadie, J. Nicholson, A. McKinley, O. Kim-
1156 moun, and F. Dias, “The vertical distribution and evolution
1157 of slam pressure on an oscillating wave surge converter,” in
1158 *Int. Conf. Offshore Mech. Arctic Eng.* (2015).
1159 ²⁴E. Renzi, Y. Wei, and F. Dias, “The pressure impulse
1160 of wave slamming on an oscillating wave energy converter,”
1161 *J. Fluids Struct.* **82**, 258–271 (2018).
1162 ²⁵R. A. Ibrahim, “Assessment of breaking waves and liquid sloshing
1163 impact,” *Nonlinear Dyn.* **100**, 1837–1925 (2020).
1164 ²⁶F. Dias and J. M. Ghidaglia, “Slamming: recent
1165 progress in the evaluation of impact pressures,”
1166 *Annu. Rev. Fluid Mech.* **50**, 243–273 (2018).
1167 ²⁷E. S. Chan and W. K. Melville, “Deep-water plunging wave pres-
1168 sures on a vertical plane wall,” *Proc. R. Soc. Lond. A-Math. Phys.*
1169 **417**, 95–131 (1988).
1170 ²⁸C. Lugni, M. Brocchini, and O. M. Faltinsen,
1171 “Wave impact loads: The role of the flip-through,”
1172 *Phys. Fluids* **18**, 122101 (2006).
1173 ²⁹C. Lugni, A. Bardazzi, O. M. Faltinsen, and G. Graziani,
1174 “Hydroelastic slamming response in the evolution of
1175 a flip-through event during shallow-liquid sloshing,”
1176 *Phys. Fluids* **26**, 032108 (2014).
1177 ³⁰A. A. Korobkin, T. I. Khabakhpasheva, and S. Malenica, “Max-
1178 imum stress of stiff elastic plate in uniform flow and due to jet
1179 impact,” *Phys. Fluids* **29**, 072105 (2017).
1180 ³¹T. Vinje and P. Brevig, “Numerical simulation of breaking
1181 waves,” *Adv. Water Resour.* **4**, 77–82 (1981).
1182 ³²M. S. Longuet-Higgins, “On the disintegration of the jet in a
1183 plunging breaker,” *J. Phys. Oceanogr.* **25**, 2458–2462 (1995).
1184 ³³A. L. New, “A class of elliptical free-surface flows,” *J. Fluid Mech.*
1185 **130**, 219–239 (1983).
1186 ³⁴P. Bonmarin, “Geometric properties of deep-water breaking
1187 waves,” *J. Fluid Mech.* **209**, 405–433 (1989).
1188 ³⁵B. C. Abrahamsen and O. M. Faltinsen, “The effect of air leakage
1189 and heat exchange on the decay of entrapped air pocket slamming
1190 oscillations,” *Phys. Fluids* **23**, 102107 (2011).
1191 ³⁶C. Lugni, M. Miozzi, M. Brocchini, and O. M. Faltinsen, “Evo-
1192 lution of the air cavity during a depressurized wave impact. I.
1193 The kinematic flow field,” *Phys. Fluids* **22**, 056101 (2010).
1194 ³⁷C. Lugni, M. Brocchini, and O. M. Faltinsen, “Evolution of the
1195 air cavity during a depressurized wave impact. II. The dynamic
1196 field,” *Phys. Fluids* **22**, 056102 (2010).

- 1197 ³⁸M. Frihat, L. Brosset, and J. M. Ghidaglia, “Experimental study₂₆₇
1198 of surface tension effects on sloshing impact loads,” in *Proc*₁₂₆₈
1199 *Int. Workshop Water Waves Float. Bodies, 32nd, 23–26 Apr.*₁₂₆₉
1200 *Dalian, China* (2017).₁₂₇₀
- 1201 ³⁹M. R. Karimi, L. Brosset, J.-M. Ghidaglia, and₂₇₁
1202 M. L. Kaminski, “Effect of ullage gas on sloshing₁₂₇₂
1203 part I: Global effects of gas–liquid density ratio,₁₂₇₃
1204 *Eur. J. Mech. B. Fluids* **53**, 213–228 (2015).₁₂₇₄
- 1205 ⁴⁰M. R. Karimi, L. Brosset, J.-M. Ghidaglia, and₂₇₅
1206 M. L. Kaminski, “Effect of ullage gas on sloshing₁₂₇₆
1207 part II: Local effects of gas–liquid density ratio,₁₂₇₇
1208 *Eur. J. Mech. B. Fluids* **57**, 82–100 (2016).₁₂₇₈
- 1209 ⁴¹M. Ancellin, L. Brosset, and J.-M. Ghidaglia, “Nu₁₂₇₉
1210 mercial simulation of wave impacts with interfacial₁₂₈₀
1211 phase change: An interface reconstruction scheme,₁₂₈₁
1212 *Eur. J. Mech. B. Fluids* **76**, 352–364 (2019).₁₂₈₂
- 1213 ⁴²M. Ancellin, L. Brosset, and J.-M. Ghidaglia, “Numerical study₁₂₈₃
1214 of phase change influence on wave impact loads in LNG tanks₁₂₈₄
1215 on floating structures,” in *Int. Conf. Offshore Mech. Arctic Eng*₁₂₈₅
1216 (2018).₁₂₈₆
- 1217 ⁴³H. Bredmose, G. N. Bullock, and A. J. Hogg, “Violent₁₂₈₇
1218 breaking wave impacts. part 3. effects of scale and aeration,₁₂₈₈
1219 *J. Fluid Mech.* **765**, 82–113 (2015).₁₂₈₉
- 1220 ⁴⁴G. N. Bullock, C. Obhrai, D. H. Peregrine, and H. Bred₁₂₉₀
1221 mose, “Violent breaking wave impacts. part 1: results from₁₂₉₁
1222 large-scale regular wave tests on vertical and sloping walls,₁₂₉₂
1223 *Coast. Eng.* **54**, 602–617 (2007).₁₂₉₃
- 1224 ⁴⁵P. Lubin, O. Kimmoun, F. Véron, and S. Glock₁₂₉₄
1225 ner, “Discussion on instabilities in breaking waves₁₂₉₅
1226 Vortices, air-entrainment and droplet generation,₁₂₉₆
1227 *Eur. J. Mech. B. Fluids* **73**, 144–156 (2019).₁₂₉₇
- 1228 ⁴⁶S. Zhang, D. K. P. Yue, and K. Tanizawa, “Simulation of plung₁₂₉₈
1229 ing wave impact on a vertical wall,” *J. Fluid Mech.* **327**, 221–254₁₂₉₉
1230 (1996).₁₃₀₀
- 1231 ⁴⁷B. Song and C. Zhang, “Water column impact on a rigid wall₁₃₀₁
1232 with air cavity effects,” *Phys. Fluids* **31**, 042112 (2019).₁₃₀₂
- 1233 ⁴⁸B. Song and C. Zhang, “Boundary element study of
1234 wave impact on a vertical wall with air entrapment,”
1235 *Eng. Anal. Boundary Elem.* **90**, 26–38 (2018).
- 1236 ⁴⁹S. Ross and P. D. Hicks, “A comparison of pre-impact
1237 gas cushioning and wagner theory for liquid-solid impacts,”
1238 *Phys. Fluids* **31**, 042101 (2019).
- 1239 ⁵⁰S. Etienne, Y.-M. Scolan, and L. Brosset, “Numerical study of
1240 density ratio influence on global wave shapes before impact,” in
1241 *Int. Conf. Offshore Mech. Arctic Eng.* (2018).
- 1242 ⁵¹P. M. Guilcher, Y. Jus, N. Couty, L. Brosset, Y.-M. Scolan, and
1243 D. Le Touzé, “2D simulations of breaking wave impacts on a flat
1244 rigid wall—part 1: influence of the wave shape,” in *Int. Ocean Po-*
1245 *lar Eng. Conf., 24th, 15–20 June, Busan, Korea* (International
1246 Society of Offshore and Polar Engineers, 2014).
- 1247 ⁵²P. M. Guilcher, Y. Jus, and L. Brosset, “2D Simulations of
1248 Breaking Wave Impacts on a Flat Rigid Wall—Part 2: Influence
1249 of Scale,” in *Int. Ocean Polar Eng. Conf., 28th, 10–15 June,*
1250 *Sapporo, Japan* (2018).
- 1251 ⁵³S. Fortin, S. Etienne, C. Béguin, D. Pelletier, L. Brosset, *et al.*,
1252 “Numerical study of the influence of weber and reynolds num-
1253 bers on the development of kelvin-helmholtz instability,” *Int. J.*
1254 *Offshore Polar Eng.* **30**, 129–140 (2020).
- 1255 ⁵⁴M. Frihat, M. R. Karimi, L. Brosset, J.-M. Ghidaglia, *et al.*,
1256 “Variability of impact pressures induced by sloshing investigated
1257 through the concept of ‘singularization’,” in *Int. Ocean Polar*
1258 *Eng. Conf., 26th, 26 June–2 July, Rhodes, Greece* (2016).
- 1259 ⁵⁵R. Firoozkoobi, B. C. Abrahamsen, and O. M. Faltin-
1260 sen, “Study of an entrapped air pocket due to slosh-
1261 ing using experiments and numerical simulations,” in
1262 *Int. Conf. Offshore. Mech. Arctic Eng.* (2017).
- 1263 ⁵⁶P. D. Hicks, “LNG-solid impacts with gas cushioning and phase
1264 change,” *J. Fluids Struct.* **80**, 22–36 (2018).
- 1265 ⁵⁷I. R. Peters, D. van der Meer, and J. M. Gordillo, “Splash
1266 wave and crown breakup after disc impact on a liquid surface,”
J. Fluid Mech. **724**, 553–580 (2013).
- ⁵⁸A. Souto-Iglesias, G. Bulian, and E. Botia-Vera, “A set of
canonical problems in sloshing. Part 2: Influence of tank width
on impact pressure statistics in regular forced angular motion,”
Ocean Eng. **105**, 136–159 (2015).
- ⁵⁹E. Villermaux and C. Clanet, “Life of a flapping liquid sheet,”
J. Fluid Mech. **462**, 341–363 (2002).
- ⁶⁰M. van Meerkerk, C. Poelma, and J. Westerweel, “Scanning
stereo-PLIF method for free surface measurements in large 3D
domains,” *Exp. Fluids* **61** (2020), 10.1007/s00348-019-2846-7.
- ⁶¹J. H. Duncan, H. Qiao, V. Philomin, and A. Wenz,
“Gentle spilling breakers: crest profile evolution,”
J. Fluid Mech. **379**, 191–222 (1999).
- ⁶²N. Otsu, “A threshold selection method from gray-level his-
tograms,” *IEEE Trans. Syst., Man, Cybern.* **9**, 62–66 (1979).
- ⁶³R. C. Gonzalez, R. E. Woods, and S. L. Eddins,
Digital image processing using matlab (Gatesmark Publishing,
2009).
- ⁶⁴R. J. Adrian and J. Westerweel, *Particle image velocimetry*, 30
(Cambridge University Press, 2011).
- ⁶⁵V. Pratt, “Direct least-squares fitting of algebraic surfaces,”
ACM SIGGRAPH Computer Graphics **21**, 145–152 (1987).
- ⁶⁶J. H. Friedman, J. L. Bentley, and R. A. Finkel, “An algo-
rithm for finding best matches in logarithmic expected time,”
ACM Transactions on Mathematical Software (TOMS) **3**, 209–
226 (1977).
- ⁶⁷V. Novaković, J. J. Costas, S. Schreier, O. Kimmoun, A. Fernan-
des, R. Ezeta, M. Birvalski, and H. Bogaert, “Study of global
wave repeatability in the new Multiphase Wave Lab (MWL),”
in *Submitted to Int. Ocean Polar Eng. Conf., 30th, 12–16 Oct.,*
Shanghai, China (2020).
- ⁶⁸G. Chen, C. Kharif, S. Zaleski, and J. Li, “Two-
dimensional navier–stokes simulation of breaking waves,”
Physics of Fluids **11**, 121–133 (1999).
- ⁶⁹P. Marmottant and E. Villermaux, “On spray formation,” *J.*
Fluid Mech. **498**, 73–111 (2004).

This is the author's peer reviewed, accepted manuscript. However, the online version of record will be different from this version once it has been copyedited and typeset.

PLEASE CITE THIS ARTICLE AS DOI:10.1063/1.50016467





Measurement number

220

222

224

226

228

230

