

Delft University of Technology

Kinematic and mechanical response of dry woven fabrics in through-thickness compression

Virtual fiber modeling with mesh overlay technique and experimental validation

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4 compression: virtual fiber modeling with mesh overlay technique and

5 experimental validation

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22 Abstract

23 The through-thickness compressive behavior of fabric reinforcements is crucial in liquid composite molding manufacturing processes. Predictive simulations of the compressive 24 25 response are thus necessary to enable a virtual processing workflow. These are complex however, as the compressive behavior of the reinforcement fabrics is non-linear. Altough 26 27 virtual fiber modeling has proven to be a strong kinematical tool, it cannot predict the compressive response due to the lack of bending stiffness in the virtual fibers. Here, we 28 describe a solution that enables predictive compressive simulations through hybrid virtual 29 fibers. It is based on an overlay mesh-element technique, combining both (i) finite 30 elements that determine the in-plane fiber properties as well as (ii) finite elements that 31 determine out-of-plane fiber bending. Using these hybrid virtual fibers, the through-32 thickness compression of a twill woven fabric ply is simulated and experimentally 33 validated using both µCT-based as compliance-based measurements. Excellent 34 agreement between simulation and experiment is obtained for the right set of input 35 36 parameters.

Keywords: A. Fabrics/textiles, B. Mechanical properties, C. Finite element analysis
(FEA), D. X-ray computed tomography, Digital element analysis

39 1. Introduction

Liquid Composite Molding (LCM) processes such as Vacuum Assisted Resin Transfer 40 41 Molding (VARTM), Resin Transfer Molding (RTM), or Injection Compression Molding (ICM), are composite manufacturing processes in which a dry fibrous reinforcement is 42 infused with liquid resin. The permeability of the fibrous reinforcement is key to allow 43 sufficient flow of resin throughout the reinforcement, resulting in high-quality parts that 44 45 are used in industry, for example in the automotive, marine, and civil sectors. The flow is usually dual-scaled, with micro-porous channels between reinforcing fibers, and meso-46 47 porous channels dependent on the reinforcement textile structure each affecting the flow [1]. Moreover, the reinforcement is compressed between two mold halves in LCM 48 49 processes and its microstructure hence changes because of the applied pressure (e.g. tow compaction). Hence, understanding and predicting the through-thickness compression 50 51 behavior of reinforcements is one of the crucial factors in the manufacturing of highquality composite parts through LCM processes. 52

A large body of research is devoted to experimental analysis of the through-thickness 53 compression of fabrics and typically relies on extensive high-resolution X-ray micro-54 computed tomography (μ CT) analysis to determine the structure of reinforcement textiles 55 56 under pressure [1–8]. Although modeling techniques would provide a good alternative, 57 the fibrous nature of textile reinforcements complicates out-of-plane properties such as through-thickness compression due to dominant deformation mechanisms on the 58 59 microscale such as fiber realignment. Although macro- and mesoscale models exist, these are usually fitted with specific constitutive laws based on experimentally determined 60 61 through-thickness input properties [9]. In addition, they do not reveal conclusions about 62 the micro-level which would be relevant for microscale permeability.

Microscale models offer the possibility to analyze the through-thickness compression at the reinforcement level while taking into account the microscale deformation mechanisms. Most notably, the virtual fiber modeling method, initially conceived by Wang *et al.* [10,11], is proving to be one of the most viable options. Often referred to as the digital element method, this method is centered around the modeling of fibers through a chain of truss-like elements (digital elements), simulating the textile material as made up of a relatively small amount of such "virtual" fibers (up to 100-200). The virtual fiber method thus explicitly takes into account the fibrous behavior of the reinforcements atnear-microscale.

72 Recently, multiple researchers have shown that the virtual fiber modeling of textile 73 reinforcements captures relevant deformation mechanisms such as the yarn cross-74 sectional deformation and yarn path waviness and that the predicted microstructures 75 correlate well to those observed by μ CT imaging [12–17]. Yet, the kinematic nature of 76 the simulations limits their ability to link the mechanics with the kinematics, e.g. to 77 simulate pressure-dependent microstructure. This is especially true in the case of out-ofplane loadings such as compression, where the reinforcement response is dominated by 78 79 fiber bending and by creation of new contact surfaces due to fiber realignment. In the case of in-plane loadings, the response is dominated by tensile loading in the fiber directions 80 81 and we have previously reported a successful correlation between kinematics and 82 mechanics for these cases [18,19]. The virtual fiber modeling method has recently been applied to through-thickness compression of woven fabrics with successful validation of 83 the microstructure (e.g. yarn compaction, yarn paths, ...) to experimental data [20,21]. 84 Yet in both cases, the modeling is kinematic only in that the compaction in the models is 85 realized without taking the actual pressure level into account. Although this enables 86 87 meso-scale models to include the correct mechanical behavior by using the predicted microstructures [21], virtual fiber modeling with correct out-of-plane mechanics might 88 provide a more efficient route for virtual mechanical analysis of textile reinforcements. 89

90 The benefits of the virtual fiber modeling approach over the wider used meso-scale91 approach, e.g. [9,22,23], can be summarized as:

(i) The initial yarn shape and configuration in the textile fabric can be simulated, and thus
does not require any a priori shape assumptions (e.g. elliptical, lenticular, ... crosssections) nor dedicated geometry extraction from microscopic images (e.g. microcomputed tomography).

96 (ii) The transversal behavior related to the realignment of fibers within the yarn under
97 load is naturally implemented without requiring dedicated material constitutive laws. In
98 essence, this enables to directly take into account yarn specific properties, for example,
99 yarn twist or the use of multiple fiber types within a yarn, instead of recalibrating the
100 constitutive laws. Moreover, there is also an option to use a virtual fiber modeling
101 approach to determine the input properties for mesoscopic and macroscopic models, e.g.

such as developed in Refs. [22,24,25], creating a stronger virtual textile modelingworkflow.

(iii) The discrete nature of the yarn consisting out of many fibers offers more deformation
modes. In comparison, meso-scale models could run into problems under severe yarn
deformations where the continuum approach does not suffice, e.g. where yarns would
flatten out, unravel or split. Such loadings can be encountered for example in process
modeling where the yarns come into contact with surfaces or tools (e.g. tufting needles,
mold surfaces, sharp corners, ...).

These advantages make that the virtual fiber modeling approach has its benefits to coexist alongside meso-scale modeling, where it allows an additional level of detail situated
between the micro- and the meso-scale.

In this paper, we describe a framework to enable the correct inclusion of fiber bending 113 into the virtual fiber modeling approach, to predict the through-thickness compression of 114 a twill woven glass fibre fabric reinforcement. It is based on an overlay mesh-element 115 116 technique, combining both (i) finite elements that determine the in-plane fiber properties 117 as well as (ii) finite elements that determine out-of-plane fiber bending. This method is 118 applied to the through-thickness compression of a single layer of dry fabric reinforcement to exclude any effects by the nesting of multiple layers. To validate the model, two 119 120 different test methods are used to determine the pressure-compaction relationship for a single reinforcement ply with adequate resolution. 121

122 **2.** Simulation details

123

2.1. Generation of an as-woven state

Starting from an idealized unit cell geometry, the as-woven state of the twill fabric is 124 generated using the virtual fiber principle as reported for example in our earlier work for 125 3D woven fabrics within the Abaqus 2019 Finite Element Analysis environment [18] 126 (Figure 1). Briefly, an idealized and "loose" unit cell geometry, based on the values 127 128 reported in **Table 1**, is created. A shrinkage step then creates tensile forces in the yarns 129 similar to those present in an actual weaving step. By tensioning the yarns, the fibers will 130 realign and spread out, creating the typical lenticular yarn cross-sectional shapes. This results in the as-woven state of the fabric. Periodicity of the unit cell is ensured using 131 132 periodic boundary conditions proposed by Green et al. [15] and Thompson et al. [21], consisting of periodicity imposed at individual fiber ends as well as slave yarns thatprovide a contact surface at the unit cell edges.

Here, the virtual fibers consist of chains of linear elastic truss elements (T3D2 elements,
Abaqus/Explicit) with properties representing glass fiber properties. All input properties
that are used are related to actual physical and measurable parameters (see Table 1). Note
that the majority of these parameters can already be found in the datasheets of the fiber
and fabric material, making extended experimental characterization unnecessary.

140 The diameter of the virtual fibers D_{vf} (circular cross-section) is chosen such that the 141 physical cross-sectional area of the virtual and real yarn, A_{vv} and A_{rv} are equal:

$$A_{vy} = n_{vf} \frac{\pi}{4} D_{vf}^2 = A_{ry} = n_{rf} \frac{\pi}{4} D_{rf}^2 \to D_{vf}$$
(1)

where n_{vf} and n_{rf} are the number of fibers in the virtual and in the real yarn, and, D_{rf} is the diameter of the fibers in the real yarn (which can be measured, or calculated from the tex-value of the fibers if the density is known).

145 Since the cross-sectional area is equal, volumetric properties such as the Young's modulus or tensile strength of the real fibers can be assigned to the virtual fibers as well. 146 147 Surface properties, such as frictional constants, however, will be affected by the 148 discretization of the yarns into a relatively low amount of virtual fibers (30 - 130,149 compared to the thousands of fibers in an actual yarn). The yarn-to-yarn friction for glass 150 fiber yarns is reported to vary between 0.2 - 0.4 depending on the tension and normal pressure applied to the yarns, their velocity (static versus dynamic friction), and the 151 sliding directions (parallel or transverse to each other) [26]. Generally, lower friction 152 153 coefficients are found at higher velocities and forces, such as may be encountered during 154 weaving. Therefore, it was decided to impose a friction coefficient of 0.2 in our as-woven 155 simulations. Contacts are handled by the General Contact algorithm in Abaqus between the fiber surfaces. The final as-woven state is reached by imposing a shrinkage factor on 156 157 the yarns in the "loose-state" such that the final crimp of the yarns is equal to the 158 experimentally determined crimp.

The idealized unit cell geometry is created through the Dynamic Fabric Mechanical Analyzer (DFMA, <u>www.fabricmechanics.com</u>) from the Fabric Mechanics group at Kansas State University (Wang *et al.*), who originally published the virtual fiber principle in Refs. [10,11]. Although the software itself is capable of mechanical analysis, the simulations in this work are performed using the finite element analysis package Abaqus as it allows the use of an explicit solver, a mesh overlay technique (see Section 2.2) and more complex boundary and contact conditions. The DFMA software was thus used here only as a pre-processor; the idealized geometry is generated in DFMA and then exported through a dedicated python script to an Abaqus 2019 input file for the as-woven simulation.

The initial loose-state unit cell consists of fibers and yarns in a curved position. As such, truss element based virtual fibers (which have no bending stiffness) are perfectly suited. In fact, implementation of bending stiffness at this step would only result in spurious stresses as the virtual fibers would want to relax back to their initial, and thus curved, positions. In reality, yarns entering a weaving loom are straight instead of curved and are subsequently deformed to form the as-woven state.



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Figure 1 – Conceptual overview of the as-woven fabric generation using the virtual fiber principle and periodic boundary conditions.

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179 2.2. Overlay mesh technique and compression simulation

Bending stiffness is imposed on the virtual fibers by overlaying the truss elements with beam elements (B31, Abaqus/Explicit) using the same nodes, see **Figure 2a**. This creates hybrid virtual fibers in which the truss elements will determine the properties in the fiber direction (tensile stiffness), while the beam elements are chosen such that they do not affect those properties (negligible Young's modulus), but have a certain bending stiffness *EI*. The value of that bending stiffness can be set to the required value by changing either the Young's modulus *E* or the beam element radius which defines its second moment of inertia $I \ (= \pi r^4)$. Here we opted to fix the Young's modulus such that $E_{truss} =$ 100 E_{beam} to suppress any effects that might rise from the stiffness of the beam elements. For the range of *EI* considered, the beam radius was similar to that of the truss elements, ensuring that the overall tensile stiffness of the virtual fibers was barely affected by the superimposed beam elements. The virtual fiber bending stiffness was set as follows:

$$n_{vf}E_{beam}I_{vf,beam} = (EI)_{measured}$$
(2)

The through-thickness compression is simulated by two rigid platens (rigid shell elements 192 193 R3D4, Abaqus/Explicit) which move towards each other (displacement-controlled) with the fabric reinforcement in between, see Figure 2b. The simulation is performed under 194 195 quasi-static conditions in Abaqus 2019 Explicit (the internal energy is much higher than 196 the kinetic energy). Contact between the virtual fiber surfaces and their surroundings 197 (other fibers and compression platens) is imposed on the truss elements only and is 198 defined by Abaqus' General Contact algorithm. The beam elements are excluded from 199 any contact definition as their radius is dependent on the required bending stiffness and does not represent the fiber radius, the fiber radius is equal to the truss element radius. 200 During compression of the as-woven fabric, the reaction forces on the platens and the 201 202 distance between them are used to determine the pressure-thickness and pressure-volume 203 fraction curves. The same periodic boundary conditions as before are used.



Figure 2 – (a) The fabric is constructed of hybrid virtual fibers that consist of a chain of beam elements for bending
 stiffness and truss element for tensile stiffness. (b) Overview of the through-thickness compression setup (for clarity
 the top platen is hidden in the overview image).

208 3. Experimental details

204

209 *3.1. Fiber, yarn and fabric properties*

210 Table 1 gives an overview of all the properties of the dry glass fiber twill fabric (Interglas 92140 aero, finish FK 144, 2x2 twill woven, 390 g/m², purchased through R&G 211 212 Faserverbundwerkstoffe GmbH, Waldenbuch, Germany) that serve as input for the simulations. Several properties are measured experimentallyt, instead of relying on 213 214 averaged datasheet values, to achieve higher accuracy of the simulations. Note that the majority of the required input properties are usually given in the datasheet of the fabric 215 216 and fiber material, or are relatively easy to determine/calculate. Only the bending stiffness of the yarns is usually not given and has to be measured for example by Peirce's cantilever 217 method. This method is relatively simple and barely requires any investment in 218 219 machinery.

Table 1 – Input properties used in the simulations obtained from measurements, calculations, or datasheet values.
 Values between brackets are datasheet values.

| Property | Warp | Weft | Property determination |
|----------|------|------|------------------------|
| FIBERS | | | |

| Linear density (dTex) | 1.57 | | Measured according to ISO 1973 (vibroscope) | |
|---|------------|-----------|--|--|
| Average fiber diameter (µm) | 8.9 (9) | | Calculated from linear density and volumetric density. | |
| Volumetric density (kg m ⁻³) | (2550) | | Datasheet value. | |
| E-modulus (cN/dTex GPa) | 292 74.6 | | Measured according to ASTM D3822. | |
| YARNS | | | | |
| Linear density (Tex) | 338 (340) | 269 (272) | Measured according to ISO 7211-5. | |
| Fibers per yarn (-) | 2148 | 1708 | Calculated from linear densities of yarn and fiber. | |
| Bending stiffness (10 ⁻⁷ Nm ²) | 1.49 | 0.99 | Measured according to ASTM D1388. | |
| FABRIC | | | | |
| Areal density (g m ⁻²) | 387 (390) | | Measured. | |
| Thread count (cm ⁻¹) | 6.1 (6.0) | 6.5 (6.7) | Measured according to ISO 7211-2. | |
| Yarn spacing (cm) | 0.164 | 0.154 | Calculated from thread count. | |
| Crimp (%) | 0.55 | 0.75 | Measured according to ISO 7211-3. | |

223 3.2. Through-thickness compression testing

We opted for two different ways to measure the through-thickness compressive response of a single fabric layer due to the difficulty in achieving the required platen-to-platen distance accurately enough: (i) using a compliance-calibration and self-aligning compression setup (setup 1, **Figure 3a**, available at Ecole Polytechnique Fédérale de Lausanne), and, (ii) using in-situ X-ray micro-computed tomography (μCT) with a dedicated compression cell that allows imaging of specimens during mechanical loading (setup 2, **Figure 3b**, available at Ghent University).

Setup 1 consists of two steel platens, the lower one a square surface of 100 mm x 100 mm and connected to a universal testing machine (Walter + Bai LFM-125kN equipped with a 10kN load cell) with a fixed support [27]. The upper platen has a circular surface with a diameter of 45 mm and contains a ball-pivot system for self-alignment that maintains parallelism between the lower and upper platens. A single layer of fabric cut to dimensions of 60 mm x 60 mm was compressed between the platens. Each compression measurement on this setup is preceded by a measurement without a specimen. The calibration curve obtained in the measurement without a specimen is used for compliance correction of the following compression measurement. Compliance correction is performed, for all recorded force values in the compression test, by subtracting the corresponding thickness in the compliance test from the thickness in the compression test [28].

243 Setup 2 consists of two machined poly-ether-ether-ketone (PEEK) compression platens 244 that are attached to an in-situ mechanical stage (CT5000, Deben, Suffol, United Kingdom) in combination with an internal load cell of 1 kN. A step-and-shoot procedure 245 246 is followed to obtain µCT datasets of the fabric layer and the spacing between the 247 compression platens. At different pressure levels, each µCT scan is made in 248 approximately 10 minutes using the Environmental Micro-CT system (EMCT) at Ghent 249 University [29]. During this time, the position of the compression platens remains 250 stationary, and a slight relaxation of the pressure by 1.5% of the set pressure value was 251 observed for the two highest pressure levels. No movement artefacts were observed in the 252 μ CT scan, indicating that the relaxation in the sample and PEEK compression platens 253 during scanning were smaller than the image resolution, and thus did not affect imaging. For each scan, a total of 1440 projection images are acquired during a rotation of the 254 scanner setup around the object from 0° to 360°. The X-ray source is operated with a 255 source accelerating voltage of 80 kV, with a power of 8 W. Tomographic reconstruction 256 257 is performed using Octopus Reconstruction [30]. The load is recorded by the CT5000 258 testing stage and combined with the displacement data measured on the µCT images.

The fabric under consideration has a thickness of approximately 0.5 mm in a non-259 260 compressed state and is compressed to approximately 0.25 mm. Hence, to have an 261 adequate pressure-thickness or pressure-volume fraction curve, the distance between the 262 compression platens needs to be accurately monitored at the micrometer scale. Studies 263 often analyze the compressive properties by compression of multiple layers of 264 reinforcement, e.g. [31-33]. This is advantageous for the platen-to-platen measurement accuracy since it results in a larger spacing between the compression platens, which in 265 266 turn allows for the use of less sensitive displacement measurements such as the crosshead 267 displacement. Nevertheless, the resulting compression curve is not representative for a 268 single ply of reinforcement due to nesting of the layers and thus cannot be used to validate 269 the model here.

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Figure 3 – (a) Setup 1: self-aligning compression setup, (b) Setup 2: in-situ compression setup for μ CT measurements.

In addition to the compressive experiments, four composite specimens were produced by 273 274 compression moulding of a single fabric reinforcement layer between two flat plates 275 under a pressure of 0.0, 0.5, 0.12 and 0.25 MPa applied by stacking weights on top of the plate. These specimens were then cured at room temperature (matrix resin is EPIKOTE 276 277 MSG RIMR 135 and EPIKURE MGS RIMH 137 from Momentive Performance Materials, Hemiksem, Belgium) and subsequently µCT images were obtained to 278 279 determine the thickness of the specimens.

- 280
- 281

4. Results and Discussion

4.1. Generation of the as-woven state through virtual fiber modeling

282 Figure 4 represents the as-woven states for yarns made up of 61 and 91 virtual fibers. 283 These numbers were based on earlier work. For example, Green et al. [15] studied the effect of the number of virtual fibers per yarn, going from 19 to 91, and indicated that at 284 285 least two or three virtual fibers across the thickness of the yarns in their deformed state 286 where required for sufficient accuracy. In their case, a 61 fiber model represented the best 287 compromise between analysis time and accuracy, and only very minor improvements were noted using 91 fibers. Similarly, the results in Figure 4 indicate that 61 virtual fibers 288 289 suffice here as well.

290 The predicted fabric geometry (fabric thickness, yarn widths, cross-sectional shapes) 291 corresponds well to that of the actual fabric. The as-woven state is simulated using the Abaqus/Explicit solver with dynamics similar to the actual weaving process: a step time 292 of 0.06 s corresponding to a weaving speed of 1000 picks/min, a typical value for 293

projectile, rapier and air-jet looms. Three different length-to-diameter ratios for the truss 294 elements and beam overlay elements were investigated, i.e. L/D equals 0.5, 1, or 3 (see 295 296 Section 4.3.1). The General Contact algorithm in Abaqus 2019 performed well for each 297 L/D ratio considered here and contact/collision detection remained robust even for much larger L/D ratios. Nevertheless, large L/D ratios are not feasible as they would affect the 298 "flexibility" of a virtual fiber (in the digital element method, material properties are 299 induced by discretization into elements, in contrast to finite element modeling where they 300 301 are defined by the material constitutive laws).



Figure 4 – Comparison of real fabric from μCT experiments (left) and simulated fabric (micro-)geometry (right)
 shows good visual agreement between both.

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4.2. Experimental determination of the through-thickness compressive response of a single fabric layer

The compressive response of the fabric measured according to the different methods/setups explained in Section 3.2 is given in **Figure 5** (fiber volume fraction was determined based on the fiber density, fabric areal density and platen-to-platen distance). Overall, the curves show an exponentially increasing pressure for higher compression levels as expected. As a comparison, the thickness of the molded composite specimens are added as well (red datapoints).

The compliance calibration procedure using crosshead displacement data seems to result in an underestimation of the platen-to-platen distance (fabric thickness), especially in the low compression regime (here 0.0 - 0.2 MPa). This could have several causes, e.g. (i) the 317 platen-to-platen misalignment is not identical during recording of the calibration curve 318 and the specimen curves, or, (ii) the set-up compliance is not perfectly reproducible due 319 to spurious misalignments in the fixtures or testing machine. Since there is no physical 320 measurement of the actual platen-to-platen distance, it remains difficult to asses whether 321 the response in the low compression regime can be considered correct.

322 Using in-situ µCT during compressive testing did allow to determine the physical platen-323 to-platen distance at each pressure level from the μ CT images (Figure 6). There is good 324 agreement with the results from the compression molded specimens, especially in the low pressure regime. At the specimen size required for sufficient unit cells to be loaded under 325 326 compression, however, the resolution of the μ CT-scans is approximately 15 μ m. 327 Moreover, the use of the in-situ compression test set-up and the limited platen-to-platen 328 distance (platens made from PEEK polymer) inevitably lowered the quality of the µCT 329 images. This means that there is an uncertainty of 1 - 2 voxels on the determination of the borders of the compression platens, which results in the relatively large error bars 330 compared to the compliance calibration method in Figure 5. In comparison to the 331 compliance calibration method, the compression set-up used in the µCT scanner is not 332 self-aligning and maintains a fixed platen misalignment of approximately 0.25° in both 333 warp and weft direction (determined on the μ CT images). 334

335 Overall, the experimental compressive response of one reinforcement ply remains 336 difficult to assess with adequate certainty. Therefor, the range described by both the 337 compliance calibration and the in-situ μ CT method is considered as the validation 338 window for the simulations.





Figure 5 - (a) Pressure versus platen spacing and (b) pressure versus fiber volume fraction obtained by the three different setups. These results clearly illustrate the difficulty associated with measuring the through-thickness compressive response of a single fabric layer with adequate certainty.

Warp yarn cross-section (YZ plane)

Weft yarn cross-section (XZ plane)



344

Figure 6 – μ CT images of the yarn cross-sections during compression.

345

346 4.3. Simulating the through-thickness compressive response

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4.3.1. Mesh convergence

Although the simulation of the as-woven state is not affected much by the considered L/D-ratios, this is different for the compression simulations using the beam element overlay (**Figure 7a**). During compression, virtual fibres are bent around other off-axis virtual fibers. Hence, the element length needs to be similar to the virtual fiber diameter to conform to the geometry of the neighbouring off-axis fibers. The (bending) strain energy of the superimposed beam elements was indeed higher for L/D ratios of 1 and 0.5 compared to a ratio of 3, indicating a higher degree of bending happening in the virtual fibres for L/D ratios of 1 and 0.5. This explains why a stiffer compaction response is recorded for these simulations. Here an L/D ratio ≤ 1 sufficed to achieve convergence of the simulations using 61 virtual fibres, and L/D = 1 is selected for further simulations (reduced computational resources compared to L/D = 0.5).

359 Beside the element length, the number of virtual fibers can also be seen as a mesh convergence parameter (e.g. dividing the yarn into a very small amount of virtual fibers 360 361 would not lead to the same results). The results (Figure 7b) show that the difference for 362 61 and 91 fibers is small, taking into account that both simulations are run with the same 363 frictional constant albeit that the yarns have a differing internal frictional surface (more 364 detailed explanation in Section 4.3.3). Hence, we can assume that the compression 365 simulations have also converged at 61 virtual fibers. This is in close agreement with the required number of virtual fibers reported by other research groups [34,35]. Note that a 366 367 compression simulation for 61 virtual fibers with L/D = 1 took about 6 hours of calculation time on a regular PC (Intel® Core™ i7-3770 3.40 GHz, 4 cores, 16.0 GB 368 369 RAM, SSD hard disk). For the 91 virtual fibres, a difference in compression response is obtained above 0.2 MPa for L/D = 0.5 and L/D = 1. Closer inspection of the results 370 showed that for L/D = 1 a higher amount of contact penetrations occurred at high 371 372 compaction level than for L/D = 0.5, resulting in an (artificially) lower stiffness. This is explained in more detail in Section 4.3.5. 373



Figure 7 – Compressive response for (a) different L/D ratios and (b) different amounts of virtual fibers per yarn n_{vf} show that the compressive response converges for $L/D \le 1$ and $n_{vf} \ge 61$.

377 The correspondence between the predicted and the experimentally determined compaction response is good, especially at low compaction levels (up to 0.28 mm platen 378 379 spacing). This indicates that the compaction process is well captured by the virtual fibers, even though there are size and number differences between the virtual and real fibers 380 which could lead to a difference in compaction (smaller fibers can compact more densely 381 than large fibers). In addition, simulations using parallel platens (misalignment of 0°) or 382 383 using platens misaligned similarly to the experimental set-up (misalignment of 0.25° in 384 both x- and y-direction) resulted in almost identical results (not shown here). Therefor, platen misalignment is not further considered in the simulations. 385

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4.3.2. Bending stiffness

The need to add virtual fiber bending stiffness to the simulation is illustrated in Figure 8 387 by the difference in the compression reaction force of the fabric when bending stiffness 388 389 is and is not present. Simulations performed without any bending stiffness mainly show a kinematic response where initially the reaction force is predominantly determined by 390 391 the ease with which the (virtual) fibers can be rearranged in the structure by the 392 compression platens (densification). The reaction force on the platens only increases at 393 high compression levels when fiber realignment becomes obstructed and the fibers become transversally loaded. On the other hand, when including fiber bending stiffness, 394

the kinematics remain fairly similar, but an additional compressive reaction force is noted,
especially in the low pressure range. At this stage, the platens deform the virtual fibers by
bending – as well as realigning – them. The compressive response shows better agreement
with the experimentally determined curve, both in terms of its shape and its position.



400 Figure 8 – Compressive response of fabric simulated with and without bending stiffness of the virtual fibers show
 401 that while kinematically similar (yarn cross-sections are visualized on the right-hand side), the mechanical response
 402 (left-hand side) is better predicted when bending stiffness is considered.

399

403 The question arises which value for the bending stiffness should be included in the 404 simulations. Experimentally, the bending stiffness of yarns is usually determined through 405 Peirce's cantilever method (Figure 9a, method 1). Herein, the length of yarn required to have a deflection of 41.5° under the yarn's weight is recorded, which is then converted to 406 a bending stiffness in Nm. Although historically important and still widely used, the 407 method is limited since the bending stiffness is assumed to be constant and measured at 408 409 relatively high curvature. As yarns are a fibrous material, their cross-section easily 410 changes by realignment of the fibers. It is well-known that the yarns can flatten out at 411 high curvatures [36] which likely affects the results.

Theoretically, one can make an estimated guess of the yarn bending stiffness by using beam theory to determine the bending stiffness of a single fiber within that yarn and multiplying that value with the number of fibers present in the yarn (Figure 9a, method 2). This corresponds to the assumption that the yarn is made up of fibers all lying next to each other on the yarn's heart line without any interaction between them. Hence such a value underestimates the actual yarn bending stiffness. If the position of each of the fibers

within the yarn envelope is known, a more realistic approximation of the yarn bending 418 stiffness could be made using the parallel axis theorem. The distance between the fiber 419 bending axis and the yarn bending axis then causes an additional increase in the calculated 420 yarn bending stiffness (Figure 9a, method 3). An idealized approximation of the fiber 421 422 positions can be made a priori by assuming that they all lie evenly spaced in layers with a height equal to the fiber diameter in an ellipsoidal configuration that represents the 423 cross-section of the yarn. This assumption would only require the yarn width and height 424 425 and the number of fibers per yarn and should give a ballpark number for the yarn bending stiffness (not considering fiber-to-fiber interaction). 426





428Figure 9 - (a) Calculation of the yarn bending stiffness $(EI)_{yarn}$ through three different methods, and (b) the429resulting values of $(EI)_{yarn}$. These results show that the experimentally determined $(EI)_{yarn}$ is at the lower430bound of the theoretically calculated range. This indicates that the bending stiffness of the yarns is predominantly431determined by the bending stiffness of the individual fibers and much less by their position within the yarn's cross-432section, which in turn indicates that fiber realignment is a major deformation mechanism during yarn bending.

Figure 9b shows the values for the bending stiffness that are determined according to the 433 methods described in the previous two paragraphs (experimentally, theoretically without 434 the parallel axis theorem, and theoretically with the parallel axis theorem at two 435 compression levels). It is worth noting that the experimentally determined bending 436 stiffness (EI)_{measured} falls within the range determined by the idealized theoretical 437 calculations which span about three orders of magnitude. Furthermore, $(EI)_{measured}$ is 438 closer to the predicted bending stiffness without applying the parallel axis theorem for 439 the initial yarn shape (height and width measured from μ CT images). This is further 440 indication – at least at a curvature responsible for the yarn making a 41.5° angle in 441

442 Peirce's cantilever experiment – that the measured bending stiffness is strongly
443 influenced by "flattening" of the yarn. This shows that fiber realignment, and the
444 accompanying cross-section change, is a dominant deformation mechanism during the
445 bending of yarns.

Note that Equation 2, used to determine the value of the bending stiffness imposed on each individual virtual fiber, does not take into account the position of the virtual fibres within the yarn and thus does not use the parallel axis theorem. However, as the number of virtual fibers is rather small, the difference in *EI* determined according to 'method 2' and 'method 3' is only around one order of magnitude for the virtual yarns, compared to three orders of magnitude for the real yarns.

452 Despite these complications, Equation 2 is applied and $(EI)_{yarn} = (EI)_{measured}$ is used as a baseline in the simulations to induce a bending stiffness in the virtual fibers, as 453 bending stiffness measurements according to Peirce's method are readily available in 454 literature. Figure 10 shows the effect of increasing the bending stiffness by two orders of 455 456 magnitude on the compressive response of the simulated fabric. Firstly, one can see that 457 the simulations correlate well with the experimentally determined compressive behavior, 458 especially when taking into account the uncertainty on the experimental bending stiffness measurements. For the baseline bending stiffness, the thickness of the fabric at a certain 459 460 pressure is slightly underestimated. The overall shape of the simulated compressive curve 461 is however close to the experimental compressive, which indicates that the simulation 462 does take the correct deformation mechanisms (e.g. yarn flattening, fiber realignment, ...) into account. The agreement between simulation and experiment improves (slightly) for 463 464 a bending stiffness 2 or 5 times the base bending stiffness. At even higher bending stiffness of 10 and 100 times (EI)_{measured}, the compressive response at low compression 465 466 levels is overestimated, indicating that values in this range are too high. Hence, these 467 simulations show that using Peirce's cantilever method results in an adequate value for 468 the varn bending stiffness, perhaps slightly underestimating it. Nevertheless, it might be 469 worthwhile to consider other test methods in the future that allow recording of the moment-curvature relationship to obtain better input parameters, e.g. by using apparatus 470 471 similar to the Kawabata Evaluation System for Fabrics or the set-up described in Ref. [37]. Another possibility would be to explicitly model Peirce's cantilever experiment 472 using virtual fibers to inversely determine the correct $(EI)_{varn}$, but the large 473

- displacements required under quasi-static conditions make this approach unfeasible with
- the current set-up.



477 Figure 10 – Effect of the bending stiffness of the virtual fibers on the compressive response of the simulated fabric.
 478 A better agreement between the simulated and experimental responses is obtained for a bending stiffness in the
 479 range of 2 – 5 times the (EI)_{measured} (from Peirce's cantilever method).

480 *4.3.3.* Friction coefficient

Contrary to the as-woven simulation, there are (almost) no in-plane tensile forces active during the through-thickness compression, and the deformation happens under quasistatic conditions. Therefore, it is likely that the higher-bound friction coefficient of 0.3 - 0.4 is more appropriate. Note that there is only one frictional constant allowed in the *General Contact* algorithm of the Abaqus 2019 solver, which determines the interaction between all surfaces, i.e. fiber-to-fiber as well as fiber-to-platen.

Moreover, similar to the bending stiffness, the friction coefficient input value likely 487 depends on the discretization in digital fibers. Since the friction coefficient is typically 488 determined from yarn-to-yarn friction experiments, the amount of contacting surface area 489 490 is determined by the surface area of the fibers at the yarns' outside perimeter. In addition, as the fibers can realign themselves, intrayarn fiber-to-fiber friction also occurs, while 491 fibers that were present at the "inside" of the yarn might migrate towards the yarn 492 boundary depending on the deformation (e.g. at high compression level). Hence, the 493 yarn's complete surface area $A_{\mu,varn}$, determined by the cylindrical surface of each fiber, 494 comes into play during through-thickness compression. It is clear that this value is highly 495 dependent on the number of fibers as well as their diameter, according to the following 496 497 equation (per unit length):

$$A_{\mu,yarn} = n_{fiber} A_{\mu,fiber} = \pi \, n_{fiber} D_{fiber} \tag{3}$$

499 The ratio of the contact surface for the real yarn $A_{\mu,ry}$ to that of the virtual yarn $A_{\mu,vy}$ is 500 determined as:

$$\frac{A_{\mu,ry}}{A_{\mu,vy}} = \frac{n_{rf}D_{rf}}{n_{vf}D_{vf}} = \frac{n_{rf}}{n_{vf}} \left(\frac{n_{rf}}{n_{vf}}\right)^{-\frac{1}{2}} = \left(\frac{n_{rf}}{n_{vf}}\right)^{\frac{1}{2}}$$
(4)

and is found to be dependent only on the ratio of real to virtual fibers per yarn. For the 501 502 number of virtual fibers used here, i.e. 61 and 91, we thus estimate that the internal surface 503 area inside the real yarn is about 5 times higher than that inside the virtual yarn. This 504 shows that using the experimentally determined frictional coefficient as an input value in 505 the simulation likely results in an underestimation of the contact forces. However, one 506 cannot simply multiply this coefficient with a factor of 5, as the kinematics of the actual 507 fibers will also be different from those of the virtual fibers. For example, while the great amount of small fibers in the real yarn results in a lot of potential contact surface, they 508 509 will also have more freedom to realign themselves and fill in empty gaps between other 510 fibers resulting in overall lower contact forces.

Figure 11 shows the compressive response for simulations with $EI_{varn} = EI_{measured}$ for 511 a range of frictional constants from 0.2 - 0.4 which are still physically relevant and 512 correspond to the measured frictional constants, up to values of 0.8 which correspond to 513 514 artificially increased frictional constants. This shows that higher frictional constants 515 indeed lead to a better correlation between the simulation and the experiment as could be 516 expected. Nevertheless, in general, even a value of 0.35, which corresponds to the 517 measured value at low varn pressure and low sliding speed, still produces acceptable results (falls within the experimently detetermined range) and is thus a good input 518 519 parameter.



520

521Figure 11 – (a) Ratio of contact surface for the real yarn $A_{\mu,ry}$ to that of the virtual yarn $A_{\mu,vy}$ shows that at 61 – 91522virtual fibers per yarn, the virtual contact surface is about one fifth the size of the real contact surface. (b) The effect523of the frictional constant on the compressive response shows that higher frictional constants result in better524agreement with the experimentally determined response.

525 *4.3.4. Hysteresis – losses upon compression-unloading*

526 The implementation of both bending stiffness, as well as friction into the virtual fiber 527 simulations, means that they can be predictive for (frictional) losses upon both compressive loading and unloading. This is illustrated in Figure 12a and Table 2 by a 528 range of simulations with different parameters (the losses are determined by the area 529 between the loading-unloading curve). The experimental losses were determined from 530 the loading-unloading curves obtained through the compliance-calibration procedure 531 (setup #1). In both simulation and experiment, the fabric is loaded monotonically to 532 0.5 MPa and immediately unloaded at the same speed while the load-displacement curve 533 534 is recorded. The results in Table 2 show that the hybrid virtual fibers are indeed capable 535 of simulating hysteresis losses during compression-unloading of a single fabric and the agreement between the experimental and simulated values is good. Moreover, the 536 hysteresis losses seem relatively independent from input parameters such as the bending 537 stiffness and the frictional constant. Figure 12b shows the (cumulative) frictional 538 dissipated energy (obtained from the history output data in Abaqus) in function of the 539 simulation progress (time). In each of the hysteresis simulations, the trend in frictional 540

dissipated energy was similar, with the strongest increase in dissipated energy happening
at the end of the loading step (compaction). Furthermore, after the sharp drop in pressure
during unloading, the frictional dissipated energy keeps increasing, albeit at a lower rate.
This indicates fiber movement which corresponds to a recovery of the (bending) stresses
and spring back of the fibers.



546

547Figure 12 – (a) Experimental and simulated (61 fibers, L/D=1, $EI=(EI)_{meas}$, $\mu=0.2$, and 61 fibers, L/D=1,548 $EI=(EI)_{meas}$, $\mu=0.35$) loading-unloading compressive response of a single fabric layer. Overall, the curve shape is549well predicted by the simulations with an immediate and large drop in pressure at the start of unloading. (b) The550frictional dissipated energy obtained from the history output data of Abaqus in function of simulation progress551shows that the main frictional dissipation occurred at high compaction pressures during the loading step . After the552immediate pressure drop during unloading, the frictional dissipated energy still increases slightly, indicating some553fiber movement during unloading (spring back).

554

Table 2 – Hysteresis losses during loading-unloading of the fabric layer. The experimental value is determined from
 the curves obtained by the compliance-calibration procedure (setup #1). The simulated values are higher, but
 correspond well to the experimental value.

| Experiment | | | | | Hysteresis losses (J/m ²) |
|------------|---------------------|-----|---------------------|------|--|
| Simulation | # virtual fibers | L/D | EI | μ | 6.72 ± 0.24 |
| | 61 | 1 | EI _{meas} | 0.2 | 7.62 |
| | 61 | 1 | EI _{meas} | 0.35 | 8.04 |
| | 61 | 1 | EI _{meas} | 0.6 | 7.15 |
| | 61 | 1 | EI _{meas} | 0.8 | 7.06 |
| | 61 | 1 | 2EI _{meas} | 0.35 | 7.92 |

559 *4.3.5.* Contact problems arising at high pressure levels

560 The General Contact algorithm implemented in Abaqus/Explicit is very robust at detecting colliding/contacting surfaces for truss elements. Yet it is a kinematically 561 562 enforced algorithm, meaning that in each time step, the surfaces are allowed to move 563 "into" each other at a ratio of k (penalty stiffness), resulting in an opposing contact force 564 which is then iterated and resolved to have proper surface-to-surface contact. The 565 downside to such an algorithm – in this case – is that numerical softening (and damping) 566 is required. The softening prevents extreme static loadings. This might be a reason why 567 the simulated compressive response is always slightly lacking in stiffness at high compression (lower slope compared to the experimental data, see e.g. Figure 10). More 568 importantly, with the current implementation using the standard numerical settings, where 569 570 the penalty stiffness is automatically determined by the software based on the material 571 properties, the amount of compressive pressure is limited to approximately 1 MPa. At 572 pressures above this value, the penetration of contacting surfaces becomes very prominent 573 and affects the predictive capability of the simulation method as the fabric thickness is underestimated, see Figure 13. Nevertheless, pressures relevant for vacuum infusion and 574 autoclave production from 0-8 bar remain viable. 575



Pressure pprox 0.5 MPa

576

Figure 13 – Contact overclosures/penetrations (softening) between neighboring fibers occur in large amounts at
 pressures above 1 MPa.

579 **5.** Conclusions

580 The results clearly show the potential for the virtual fiber modeling method including fiber bending stiffness as a general textile modeling framework. Taking our previous 581 582 work on in-plane properties into account [18,19], the addition of out-of-plane property simulation capabilities indicates that a large range of textile-relevant loadings can be 583 584 considered. This approach, fully implemented in the commercial finite element software package Abaqus using the standard element, material and contact libraries, allows 585 586 research groups with FEA experience but without dedicated virtual fiber tools to 587 implement this type of modeling. Good agreement is obtained between the experimental 588 and the numerical determined pressure-compression behavior. The macroscopic 589 compressive response of the fabric is well predicted for the right set of input parameters. 590 Moreover, the microscale deformation mechanisms are captured as well, resulting in good 591 agreement with µCT-scans and prediction of hysteresis-losses due to fiber realignment 592 and friction.

593 On the other hand, the results also show that improvements are still required. For example, 594 an L/D element ratio of 1 was used in this work, but larger ratios could improve computation time. Yet, here, L/D ratios of 3 result in non-converged solutions. One 595 possible solution could be the use of non-constant element lengths along the fiber to 596 597 reduce the number of elements while maintaining convergence. On the input property side, the friction coefficient and bending stiffness that should be implemented remain 598 599 difficult to assess. Although the results have shown that the simulation output is rather 600 robust for a wide range of input properties, it would be best to determine which values 601 are most appropriate. Future studies could, for example, consider a numerical calibration 602 of the material parameters by using a 1-to-1 simulation-experimental approach, where the 603 -previously experimental - friction and bending stiffness experiments are explicitly 604 modeled using virtual fibers. In addition, more complex material behavior such as non-605 linear bending and non-linear friction can be included to increase the prediction accuracy. 606 To extend the method to high-pressure LCM processes, a robust contact algorithm with 607 minimal contact penetration is required.

Overall, the proposed methods allow a predictive analysis of fabric compression behavior
and are thus well-suited to analyze and optimize fabric structures for example in LCM
manufacturing processes.

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