

# INTERCONNECT SCHEMES FOR STRETCHABLE ARRAY-TYPE MICROSYSTEMS



SEBASTIAN SOSN

# **Interconnect schemes for stretchable array-type microsystems**

PROEFSCHRIFT

ter verkrijging van de graad van doctor  
aan de Technische Universiteit Delft,  
op gezag van de Rector Magnificus Prof. ir. K. C. A. M. Luyben,  
voorzitter van het College voor Promoties,  
in het openbaar te verdedigen

op woensdag 6 april 2011 om 12.30 uur

door

**Sebastian SOSIN**

Master of Science Microsystems Engineering,  
Universitatea Politehnica București,  
geboren te Boekarest, Roemenië.

Dit proefschrift is goedgekeurd door de promotor:

Prof. dr. P. M. Sarro

Samenstelling promotiecommissie:

Rector Magnificus, voorzitter

Prof. dr. P. M. Sarro

Technische Universiteit Delft, promotor

Dr. M. Bartek

Technische Universiteit Delft, copromotor

Prof. dr. R. A. M. Wolters

Universiteit Twente

Prof. dr. ir. R. Dekker

Technische Universiteit Delft

Prof. dr. P. J. French

Technische Universiteit Delft

Prof. dr. ir. L. J. Ernst

Technische Universiteit Delft

Prof. dr. J. N. Burghartz

Universität Stuttgart, Duitsland

Prof. dr. ir. C. I. M. Beenakker

Technische Universiteit Delft, reservelid

The research presented in this thesis was financially supported by the Dutch Bsik program MicroNed and the Delft Centre for Mechatronics and Microsystems (DCMM).

Sebastian Sosin,

Interconnect schemes for stretchable array-type microsystems,

Ph.D. thesis, Delft University of Technology,

with summary in Dutch.

ISBN: 978-90-5335-390-5

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Printed by Ridderprint BV, Ridderkerk, the Netherlands

# Contents

<b>1</b>	<b>Introduction</b>	<b>1</b>
1.1	Motivation	1
1.2	Scope of the thesis	4
1.3	Organization of the thesis	5
1.4	References	6
<b>2</b>	<b>Stretchability of Silicon Based Microsystems</b>	<b>7</b>
2.1	Definition of stretchability	7
2.2	Mechanical properties of materials	8
2.3	Material selection and material data	14
2.4	State of the art	16
2.5	System aspects of stretchable silicon microsystems	25
2.6	Fabrication compatibility, handling and reliability	26
2.7	Conclusions	28
2.8	References	29
<b>3</b>	<b>Stretchable Interconnect Schemes</b>	<b>30</b>
3.1	Introduction	30
3.2	General requirements on stretchable interconnects	30
3.3	Geometry for stretchability	31
3.3.1	Mesh geometry	31
3.3.2	Spiral geometry	32
3.3.3	Meander and horseshoe geometry	32
3.3.4	Parametric structure description	33
3.4	Finite-element simulation	35
3.4.1	Meander interconnects analysis	38
3.4.2	Horseshoe interconnect analysis	39
3.4.3	Mesh interconnect analysis	40
3.4.4	Analysis conclusions	42
3.5	Mechanical characterization and test set-up used	42
3.6	High-aspect-ratio-silicon springs	44
3.6.1	Design and optimization	45
3.6.2	Fabrication	47
3.6.3	Characterization results	50
3.7	Free-standing metal interconnects	54
3.7.1	Geometry selection	55
3.7.2	Fabrication	56
3.7.3	Characterization results	57
3.7.4	Parylene coating for strength modification	60
3.8	Conclusions	64
3.9	References	66



<b>4</b>	<b>PDMS-Embedded Silicon Electronics Arrays</b>	<b>67</b>
4.1	PDMS-embedded electronics concept	67
4.1.1	Fabrication process development	68
4.1.2	Characterization results	72
4.2	Mesh shape optimization	75
4.3	Fabrication of PDMS-embedded Si electronics arrays	76
4.4	Array description	77
4.5	Measurements	78
4.5.1	Tensile testing of fabricated arrays	79
4.5.2	Electrical measurements of fabricated samples	80
4.5.3	Cyclic tensile testing	83
4.6	Failure of fabricated PDMS arrays	85
4.7	Conclusions	88
4.8	References	89
<b>5</b>	<b>Conclusions and Recommendations</b>	<b>90</b>
5.1	Conclusions	90
5.2	Recommendations for future work	92
5.3	References	93
<b>A</b>	<b>Appendix A: High-Aspect-Silicon-Springs Flow Chart</b>	<b>94</b>
<b>B</b>	<b>Appendix B: Free-Standing Copper Interconnects Flow Chart</b>	<b>95</b>
<b>C</b>	<b>Appendix C: PDMS-Embedded Silicon Electronics Arrays Flow Chart</b>	<b>96</b>
	<b>Summary</b>	<b>98</b>
	<b>Samenvatting</b>	<b>100</b>
	<b>List of Publications</b>	<b>102</b>
	<b>Acknowledgements</b>	<b>104</b>
	<b>About the Author</b>	<b>106</b>



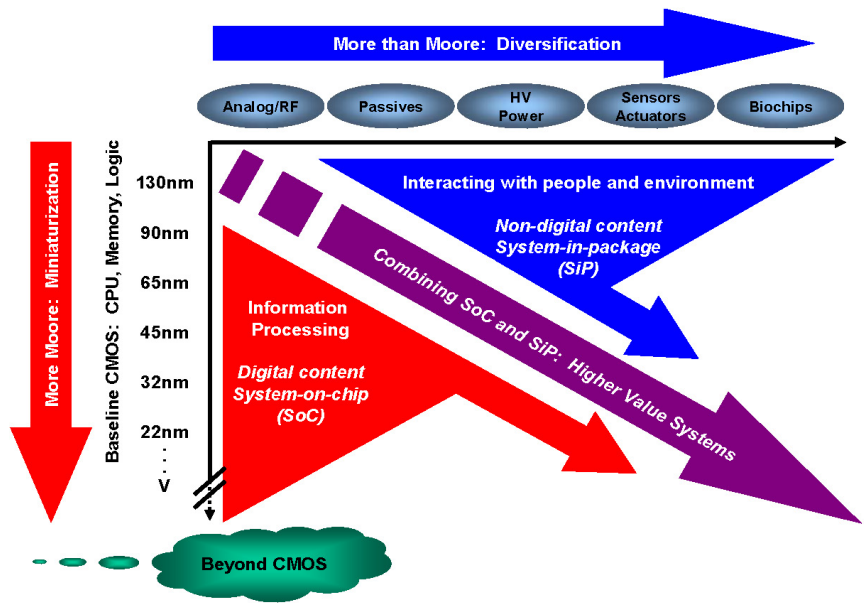


# Chapter 1:

## Introduction

### 1.1. Motivation

From its beginning in the 1950s, microelectronics has been working to make better, faster and more reliable IC's. This was quite accurately predicted by Gordon Moore in 1965 when he stated that the number of transistors that can be placed on an integrated circuit will increase exponentially, doubling approximately every two years.



**Figure 1-1.** The combined need for digital and non-digital functionalities in an integrated system is translated as a dual trend in the International Technology Roadmap for Semiconductors: miniaturization of the digital functions (“More Moore”) and functional diversification (“More-than-Moore”) [1.1].

The microelectronics community is working on finding new solutions to keep Moore's law alive by miniaturization (an approach called “More Moore”). Through transistor scaling, one obtains a better performance-to-cost ratio of products, which drives an exponential growth of the semiconductor market. This in turn allows for further investments in semiconductor technologies which will fuel further scaling

The industry is now faced with the increasing importance of a new trend, “More than Moore” (MtM or better named, “Diversification”, see Figure 1-1), where added value to devices is provided by incorporating functionalities that do not necessarily scale according to Moore's law. This research direction deals with products and technology that are based upon or derived from silicon technologies, but do not simply scale according to Moore's law. Typical examples are RF, power/high-voltage devices, passives, sensors, actuators, MEMS, bio-chips, bio-systems, microfluidics, system in a package, solid-state lighting, etc.

Flexible printed circuit boards are being used to fabricate foldable electronics but the deformation is limited to one axis at a time. In recent years, many groups have started investigating microelectronic systems that can truly deform while maintaining their functionality.

One of the best examples of stretchable electronic systems is the concept of “artificial skin.” It is supposed to have not only all the sensing capabilities of our skin but also similar mechanical behavior (Figure 1-2).



**Figure 1-2.** Artificial electronic skin being developed by Nokia and Cambridge University [1.2].

A flexible display such as an e-paper is an important application with promising demonstrators being presented (Figure 1-3). Having rollable/bendable screens reduces weight and increases portability. Applications are in advertising, media, public notice boards, mobile computing, etc.



(a)



(b)

**Figure 1-3.** (a) The LG and Phillips E-ink display [1.3], and (b) the Polymer Vision (now Wistron) portable e-reader with a 5-inch fold-up display [1.4].

Shapeable and elastic integrated circuits on soft elastomeric substrates are creating new research opportunities for biomedicine, cell research and medical prosthetics and implants. Cell culture mediums that have soft substrates with integrated circuits and stretchable conductors can improve drug testing and in-vivo tissue research. Smart medical prosthetics can restore human sensory and motor functions and medical implants that conform to the surrounding tissue can minimize the patient's discomfort. Medical implants that conform to the surrounding tissues by being elastic and flexible can enhance the patients comfort. Bladder implants for treating incontinency, brain electrodes that treat epilepsy and depression, fall-detection monitors for the elderly or intelligent textiles that monitor health conditions are products that can be made better using stretchable electronics.

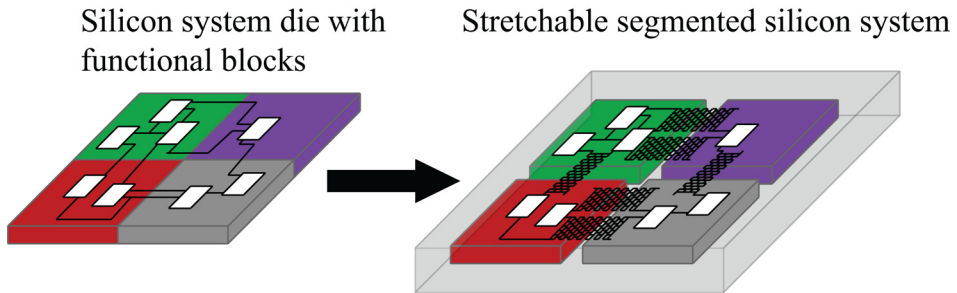
For consumer electronics, ambient intelligence is the keyword for the next decade. This means that a person carries more and more devices, not only on his body but also inside the body. This affects the way devices are designed: they must be light weighted, must take on the shape of the object in which they are integrated, and must even follow complex movements of these objects, hence the need for stretchability. Shapeable electronics also create a more natural environment for living tissue research or drug testing, leading to faster diagnostics or faster drug development.

Smart textiles, biomedical applications, cell research, medical prosthetics, automotive industry, consumer electronics, displays, all are the domains that will welcome flexible, stretchable and elastic electronic systems.

## 1.2. Scope of the thesis

Most of the research done on stretchable systems focuses on hybrid integration of silicon chips on a pre-fabricated carrier, followed by encapsulation using molding of a rubber-like compound. Several groups working on stretchable electronics have reported elongation levels starting from 12-15 % (Lacour, elastically stretchable TFT inverter [1.5]) up to 40-60 % (polyimide-supported copper interconnects [1.6, 7]). Results from the European Project STELLA show demonstrators intended for applications enclosing or attached to the human body (relatively low elongation levels, up to 10 %) like clothing, smart band aid, shoe insole for diabetes monitoring and activity monitor [1.8].

The goal of this thesis work is to develop a wafer-level CMOS-compatible post-processing module that allows transformation of rigid silicon wafers containing MEMS/CMOS devices into stretchable systems using conventional clean room equipment. The fabrication process does not, however, exclude the possibility of hybrid integration of additional chips. This approach based on the limited silicon wafer size is suitable for applications requiring small-area stretchable systems like biomedical implants.



**Figure 1-4.** Illustration of the segmentation method showing (left) monolithic silicon system die with different functional blocks, and (right) segmented system with interconnect network embedded in a protective polymer layer.

The selected fabrication approach (Figure 1-4) modifies the initially rigid silicon substrate by segmenting it into rigid islands connected by an elastic medium. The elastic medium linking the functional islands must provide electrical connections between the segments (using flexible metal interconnects) and mechanical support (using polymers, structured silicon). The focus of this work is on the stretchable interconnect scheme.

The following chapters present the development process of such a stretchable system, starting from testing of individual spring-like structures to the integration of all components into one stretchable silicon-based array.

The interconnect network must be strong enough to withstand tensile loading without sacrificing electrical performance. For this reason metal layers



## INTRODUCTION

are selected instead of conductive polymers. While metal layers have limited elasticity, when patterned for stretchability several geometries (such as mesh, meander or horse shoe) can withstand large strains.

The processing compatibility with already integrated MEMS/CMOS devices requires that the post-processing module has adequately low thermal budget and all of the used fabrication steps will have negligible influence on the performance of already integrated devices.

The developed post-processing module is applicable on wafer scale, reducing handling and assembly times, but if needed, hybrid substrates with chips provided by different technologies could be used without major process modifications.

Biomedical applications are one of the most promising areas and biocompatibility should be taken into account when selecting materials and designing process flow.

### ***1.3. Organization of the thesis***

Chapter 1 presents the motivation behind this work, presenting a short overview of domains where stretchability can increase the performance of existing products.

In Chapter 2 a concise theory of elasticity and material properties used throughout the thesis is introduced. Furthermore, an overview of the state of the art in stretchable electronics research and the fabrication approaches used are given.

Chapter 3 presents the analysis of different free-standing geometries chosen for the stretchable interconnect network. The FE simulations are followed by tensile testing of free-standing High-Aspect-Ratio-Silicon (HARS) springs and copper meshes. Simulation and tensile testing results are used to design a new set of metal interconnects.

Chapter 4 starts with the development of the fabrication process for stretchable silicon electronics arrays. The fabrication process is developed in steps, starting from fabrication of metal interconnects on polydimethylsiloxane (PDMS) membranes and ending with the fabrication of a functional stretchable silicon-based arrays. In the final part of Chapter 4, the results of both mechanical and electrical testing under single and cyclic tensile loading are presented.

Chapter 5 ends the thesis by presenting conclusions and recommendations for future work.

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# Chapter 2:

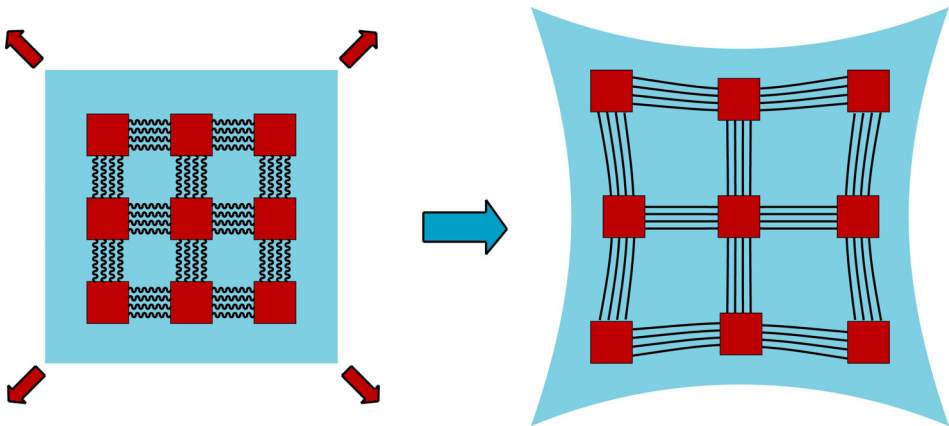
## *Stretchability of Silicon-Based Microsystems*

In the first part of this chapter the theory of material properties used throughout the thesis for analysis of stretchable electronics is presented. The theoretical part is followed by an overview of the state of the art in stretchable electronics with examples of realized stretchable systems. The last part deals with system aspects, fabrication, handling and reliability of stretchable silicon microsystems.

### *2.1. Definition of stretchability*

According to Merriam-Webster's dictionary, the verb "to stretch" is defined as "a: to enlarge or distend especially by force; b: to extend or expand as if by physical force." Stretchable electronics systems must function as designed while being able to conform onto complex shapes, expand, and contract reversibly (within certain limits).

This means that a silicon microsystem has to modify its surface area and shape in order to contract/expand and to preserve its functionality when an external force is applied. A definition valid for all stretchable electronic system research approaches is that a stretchable electronic system is made of a number of rigid or flexible component islands, which are connected by an elastic and electrically conductive medium.



*Figure 2-1. Illustration of the stretchable system concept using rigid islands and stretchable interconnects/substrate.*

2.2. Mechanical properties of materials

The phrase “stretchable silicon array microsystem” can be misleading. Silicon, the main material in almost all the integrated circuits, is intrinsically brittle and rigid, when compared to metals or polymers currently used in microfabrication. Silicon wafers can become flexible by thinning them and useful degrees of flexibility can be achieved for thicknesses below 50  $\mu\text{m}$  but large scale elasticity of bulk silicon is impossible to achieve without using some processing tricks and/or additional materials providing elasticity.

When talking about stretchable systems, terms like stress, strain or Young’s modulus are used to describe the systems and the materials they are made of. Explaining these terms, even if briefly, is necessary for understanding of the following chapters of this thesis [2.1].

In a body, two atoms or molecules are subjected to attraction or repulsion forces that act along a line joining the two particles. These forces hold together the body and are called “internal forces.” Similar to Newton’s universal gravity law, they vary inversely as an exponent of the radial distance that joins the two particles.

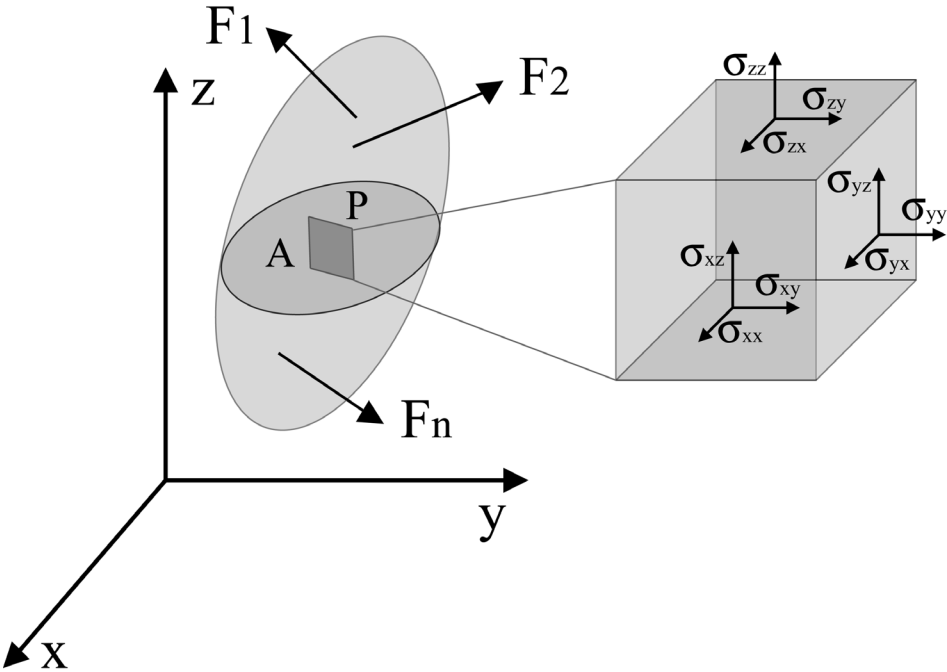


Figure 2-2. Stress in a loaded deformable body.

When the body is subjected to external forces (Figure 2-2), its shape changes thus changing the distance between internal particles. Therefore internal forces also change and if their values exceed a certain limit, the body will break apart.

The change in the intensity of the internal forces is called stress and characterizes the strength of a body.

A single particle P of a body is subjected to many forces that can be replaced by the resulting force that acts on that particle. Depending on the location of the point, the magnitude and direction of the resulting force will vary, meaning that when the body is subjected to external forces, an internal force distribution is generated in the body.

The internal distributed force  $F$  on an imaginary surface  $A$  through the body divided by this surface area is called *stress on a surface*. The force that is normal to the surface is called the *normal stress* ( $\sigma$ ) on a surface while the force parallel to the surface is called the *shear stress* ( $\tau$ ):

$$\sigma_{AVG} = \frac{F_n}{A} \approx \sigma \quad (2.1)$$

$$\tau_{AVG} = \frac{F_s}{A} \approx \tau \quad (2.2)$$

*Normal stress* can be described as internal forces developed as material resistance to the *pulling apart* or *pushing together* of two adjoining planes of the imaginary surface. *Shear stress* can be described as internal forces developed as material resistance to *sliding of two adjoining planes* along the imaginary surface

*Tensile stress* is a normal stress that pulls the surface away from the body while *compressive stress* pushes the surface into the body.

In general, stress is not uniformly distributed over the cross-section of a material body, and consequently the stress at a point in a given region is different from the average stress over the entire area. Therefore, it is necessary to define the stress not over a given area but at a specific point in the body (point “P” in Figure 2-2). According to Cauchy [2.1], the stress at any point in an object assumed to behave as a continuum is completely defined by the nine components  $\sigma_{ij}$  of a second-order tensor known as the Cauchy stress tensor,  $\sigma$ , where  $\sigma_{ii}$  and  $i = j \in (x, y, z)$  represents normal stress components while  $\tau_{ij}$ ,  $i \neq j$ , represents shear stress components (2.3). Shear stress is symmetric, meaning that  $\tau_{ij} = \tau_{ji}$ .

$$[\sigma] = \begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{zz} \end{bmatrix} \quad (2.3)$$

When a body is subjected to an external force, a change in shape appears. The measure of this change is called *strain*. Strain, defined as the ratio between the change in length  $\delta L$  and the original length  $L_0$ , can be easily converted into

percentage. Such representation of the strain makes it easier to describe the stretching effects on an object's dimensions. As an example, if a 100 cm bar is stretched to 110 cm, then the strain can be expressed as  $\varepsilon = 10\%$ .

$$\varepsilon = \frac{L - L_0}{L_0} = \frac{\delta L}{L_0} \quad (2.4)$$

and

$$\varepsilon = \frac{\delta L}{L_0} * 100\% \quad (2.5)$$

The *shear strain* is defined as the change in angle from the right angle as shown in Eq. 2.6, where  $\alpha$  is the final angle.

$$\gamma = \frac{\pi}{2} - \alpha \quad (2.6)$$

A material is said to be elastic if it returns to its original, unloaded dimensions when the load is removed. A particular form of elasticity (which applies to a wide range of engineering materials, at least over a part of their load range) produces deformations that are proportional to the loads producing them. Since loads are proportional to the stresses they produce and deformations are proportional to the strain, this also implies that, while materials are elastic, stress is proportional to strain (Eq. 2.7). *Hooke's law*, in its simplest form, was initially stated in 1676 by British physicist Robert Hooke as a Latin anagram, “*Ut tensio, sic vis*” meaning “*As the extension, so the force.*”

$$\frac{\text{stress}}{\text{strain}} = \frac{\sigma}{\varepsilon} = \text{const.} \quad (2.7)$$

Considering a linear material model, where there is a linear relationship between stress and strain, Hooke's law can be written as:

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ \gamma_{yz} \\ \gamma_{zx} \\ \gamma_{xy} \end{Bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & C_{14} & C_{15} & C_{16} \\ C_{21} & C_{22} & C_{23} & C_{24} & C_{25} & C_{26} \\ C_{31} & C_{32} & C_{33} & C_{34} & C_{35} & C_{36} \\ C_{41} & C_{42} & C_{43} & C_{44} & C_{45} & C_{46} \\ C_{51} & C_{52} & C_{53} & C_{54} & C_{55} & C_{56} \\ C_{61} & C_{62} & C_{63} & C_{64} & C_{65} & C_{66} \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \tau_{yz} \\ \tau_{zx} \\ \tau_{xy} \end{Bmatrix} \quad (2.8)$$

The matrix  $C_{ij}$  in Eq. 2.8 is called the compliance matrix and it is a symmetric matrix,  $C_{ij} = C_{ji}$ , with  $i, j \in \{1..6\}$ . Isotropic materials require only

two independent material constants and Hooke's law for isotropic materials is defined by Eq. 2.9.

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ \gamma_{yz} \\ \gamma_{zx} \\ \gamma_{xy} \end{Bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{12} & 0 & 0 & 0 \\ C_{12} & C_{11} & C_{12} & 0 & 0 & 0 \\ C_{12} & C_{12} & C_{11} & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(C_{11}-C_{12}) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(C_{11}-C_{12}) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(C_{11}-C_{12}) \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \tau_{yz} \\ \tau_{zx} \\ \tau_{xy} \end{Bmatrix} \quad (2.9)$$

Equation 2.9 can be rewritten in equation form using three material constants (based on the original two): the modulus of elasticity  $E$ , Poisson's ratio  $\nu$  and the shear modulus of elasticity  $G$ :

$$C_{11} = \frac{1}{E} \quad (2.10)$$

$$C_{12} = -\frac{\nu}{E} \quad (2.11)$$

$$2(C_{11} - C_{12}) = \frac{1}{G} \quad (2.12)$$

Substituting  $C_{11}$  and  $C_{12}$  in the last relationship, the shear modulus of elasticity is defined:

$$G = \frac{E}{2(1+\nu)} \quad (2.13)$$

Poisson's ratio is a dimensionless measure of the lateral strain that occurs in a body due to longitudinal strain, if the load on the object is restrained within the elastic range:

$$\nu = \frac{\varepsilon_d}{\varepsilon_L} \quad (2.14)$$

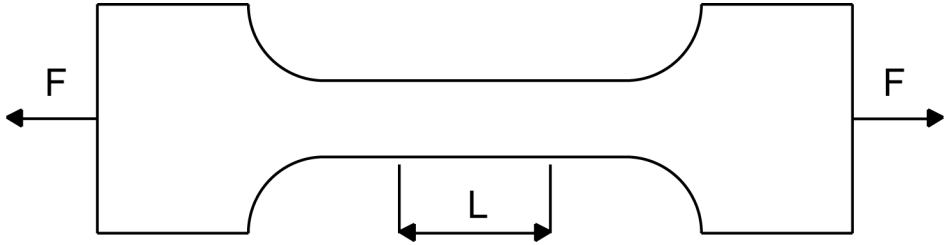
The negative sign is usually ignored, leaving the coefficient simply as a ratio of strain magnitudes but it must be remembered that the longitudinal strain induces a lateral strain of opposite sign.

By rewriting Eq. 2.9 with the previous substitutions, a more simplified version, called *generalized Hooke's law*, valid only for linear, elastic, isotropic materials is given by equation:

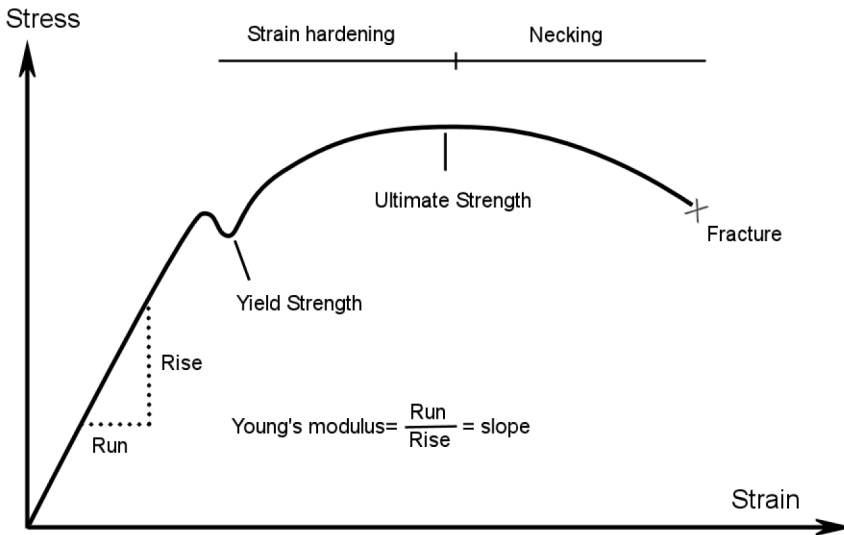


$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \end{Bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu \\ -\nu & 1 & -\nu \\ -\nu & -\nu & 1 \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \end{Bmatrix} \quad (2.15)$$

The strength of a material can be determined with a tensile test using a “dog bone”, illustrated in Figure 2-3. The “dog-bone” structure has uniform cross section and it is subjected to a gradually increasing tensile load until failure occurs. The length change of the gauge length of the bar is recorded and a graph is produced. Such a graph is called the *tensile stress – strain diagram* of the material (see Figure 2-4).



**Figure 2-3.** "Dog-bone" structure used for tensile measurements.



**Figure 2-4.** Typical tensile stress - strain diagram for a ductile material showing various stages of deformation.

Several points can be identified in a tensile stress-strain diagram, depending on the material. If the stress in a body subjected to a load returns to zero once

the load is removed, then the material of the body is said to have been strained within the *elastic limit* or that the material is *perfectly elastic*. If under loading the strain is linearly proportional to the load, the material is said to be strained within the limit of *linear elasticity*. When the load produces a stress that exceeds the elastic limit, the strain does not disappear upon the removal of the load.

*Yield strength*  $\sigma_{YS}$ , (also called elastic limit) is the limit beyond which permanent deformation will occur.

Under tensile stress plastic deformation is characterized by a *strain hardening* region and a necking region and finally, fracture (also called rupture). During strain hardening the material becomes stronger through the movement of atomic dislocations. The necking phase is indicated by a reduction in cross-sectional area of the specimen.

Necking begins after the *ultimate tensile strength* ( $\sigma_U$ ) is reached. Ultimate strength is defined as the maximum stress attained in the stress-strain diagram. During necking, the material can no longer withstand the maximum stress and the strain in the specimen rapidly increases. Plastic deformation ends with the fracture of the material.

As it can be observed in Figure 2-4, the plastic range covers a much wider part of the strain axis than the elastic range. The capacity of a material to allow these large extensions (i.e., the ability to be drawn plastically) is named ductility. Materials with high ductility are named ductile materials.

Materials with low ductility are named brittle materials. There is little or no necking at fracture for brittle materials. Brittle materials such as silicon, concrete, carbon fiber and polymers such as poly(methyl-methacrylate) (PMMA) and polystyrene do not have a yield point, and do not strain-harden (Figure 2-5). Therefore, the yield strength and ultimate strength are the same.

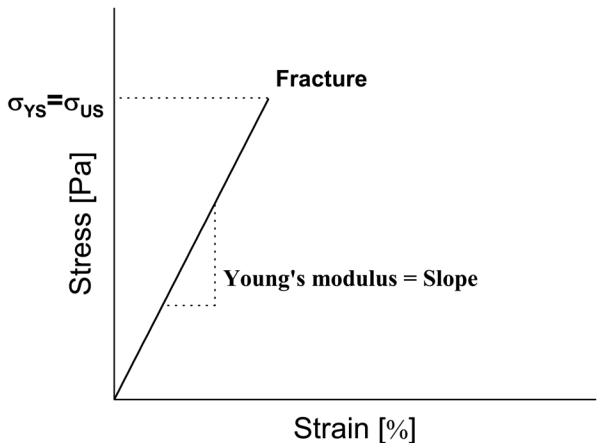


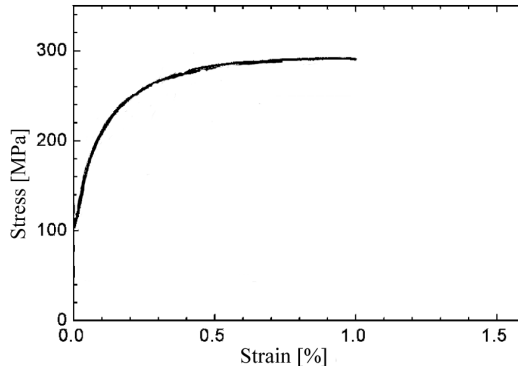
Figure 2-5. Typical stress-strain curve for brittle materials.

### 2.3. Material selection and material data

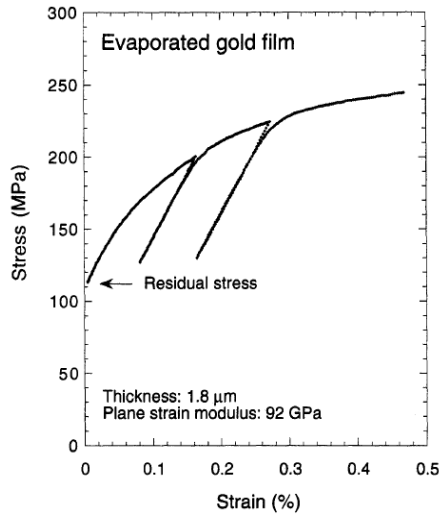
Selection of materials has to take into account several factors: mechanical properties, electrical properties, deposition and patterning methods and IC fabrication compatibility. Materials used for stretchable silicon arrays can be divided into three groups. In the first group, there are materials used for the fabrication of array elements, mainly silicon. Materials used for the stretchable part of an array form the second group. Silicon can also be part of this group, along with metals common in IC fabrication like copper and aluminum. The last group is for materials used as mechanical support and extra protection and it is composed of polymers such as Parylene, PDMS and photosensitive elastomers.

Silicon, a brittle material, if patterned into very thin foils or beams, can be a good candidate for the stretchable part of an array. Although not a good conductor, it can support metal interconnects which have good electrical conductivity.

Electrical conductivity, deposition methods, IC compatibility and mechanical properties are factors that limit the options for metal interconnects. Electrical interconnects between islands are realized using copper or aluminum, most common metals to the IC industry, with well known deposition and patterning techniques and good electrical and mechanical properties. Gold is also used for stretchable interconnects due to its good electrical conductivity and chemical resistance.



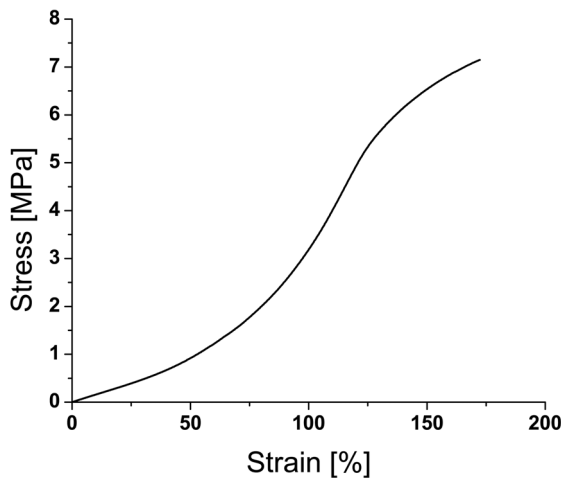
**Figure 2-6.** Stress-strain curve of an electroplated copper film [2.2].



**Figure 2-7.** Typical stress-strain curve for an evaporated gold film [2.3].

For a stretchable silicon array to function properly, protection against a large number of external factors is necessary. Protection against humidity, gases, corrosive chemicals, electrical insulation or just additional mechanical support can be achieved using polymers. Depending on the design, the array can be fully embedded in a polymer foil or just coated with an insulating layer, similar to regular electrical conductors.

Polymers can also be used during fabrication as a temporary mechanical support layer or just protective coating. PDMS can be spin coated, molded and casted in a wide range of thicknesses. It provides good mechanical protection and electrical insulation by fully embedding a system, providing a large extension interval in the same time.



**Figure 2-8.** Strain-stress curve for a PDMS foil at room temperature [2.4].

**Table 2-1.** Material property data of selected materials [2.4, 5].

(Note: The missing data are either not relevant for given combination of the material property and the material type, or a reliable value was not possible find in the available literature. The values listed are for bulk material properties and are used to illustrate the differences between selected materials. Values for thin films depend on deposition method and can exhibit large variations).

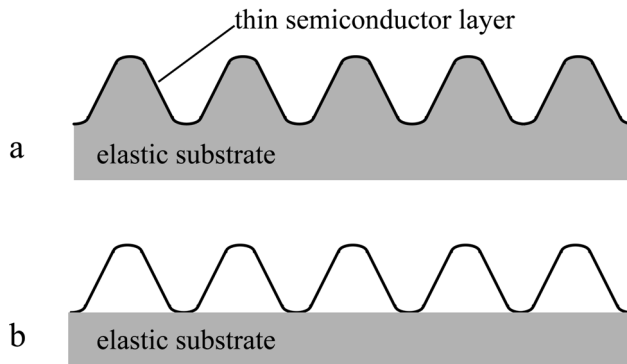
	Silicon	Copper	Aluminum	Gold	PDMS
Young's Modulus [GPa]	112.4	110	68.0	77.2	0.0017
Yield Strength [GPa]	0.120	0.262	20	0.240	
Ultimate Tensile Strength [GPa]	0.120	0.310	50	0.302	
Bulk Modulus [GPa]	98.74	140	76		
Poisson's Ratio	0.280	0.343	0.35	0.42	0.5
Shear Modulus [GPa]	43.9	46.0	26	27.2	
Electrical Resistivity [ $\Omega \cdot \text{cm}$ ]	-	1.70E-6	2.70E-6	2.20E-6	-
Elongation at Break [%]	-	-	-	-	174
Relative Dielectric Constant		-	-	-	2.8

## 2.4. State of the art

The methods of producing stretchable silicon electronics can be divided into two categories [2.6]. Both methods use substrate segmentation to create rigid functional islands connected by a stretchable and electrically conductive medium. The first category gets its stretchability by using very thin semiconductor (brittle) membranes or ribbons that have been compressed using substrate transfer. In the second category the stretchability is achieved by using metal layers that are patterned for stretchability and/or polymers that can accommodate large elongation.

The first approach transforms brittle materials (e.g., single crystalline silicon) into ultra thin membranes, ribbons or wires. With further processing and substrate transfer the material becomes “wavy” and the amplitudes and wavelengths can change according to the applied strain. In plane strains are accommodated through out-of-plane displacements in the wavy structure.

Stretchability is achieved by transferring such a thin network of semiconductor islands and ribbons to a pre-stretched elastomer substrate. When the elastomer substrate returns to its relaxed state, semiconductor ribbons form waves (see Figure 2-9). The first wavy configuration involves continuous intimate mechanical coupling between the ribbons and elastomer. In the case of silicon and PDMS, bonding is done by covalent  $-O-Si-O-$  linkage that forms due to condensation reactions between surface  $-OH$  groups on the PDMS surface and the native oxide of the silicon. When formed with the elastomer substrate in a pre-stretched state, mechanical relaxation creates non-linear buckling or wavy configurations with amplitudes that depend mainly on the elastic properties of PDMS and silicon and thickness of silicon.



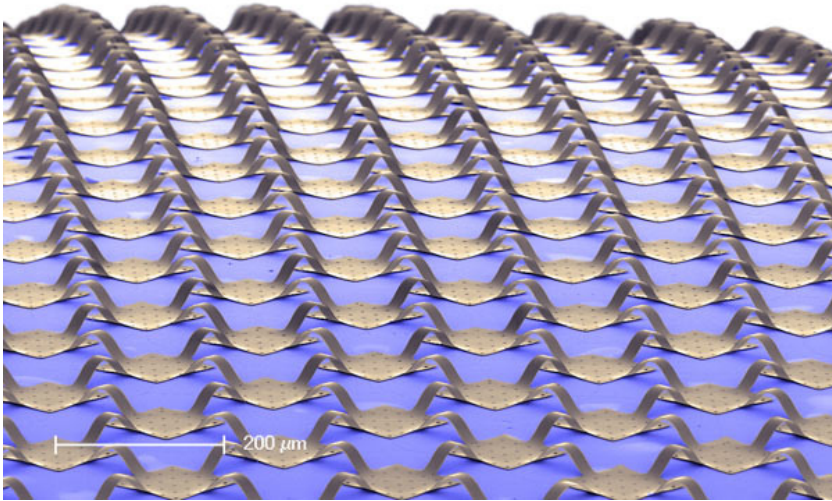
**Figure 2-9.** Schematic illustration of ultra-thin ribbon of "wavy" semiconductor transferred to elastic substrate (a) fully bonded, and (b) partially bonded.

This method has the advantage that the electrical performance and reliability of such a system are similar to those of wafer-scale electronics and has been used to create stretchable conductors, diodes and even fully integrated systems such as an integrated hemispherical electronic eye camera. The method provides practical levels of stretchability up to 15 %, exceeding by  $\sim 15$  times the intrinsic fracture limit of silicon.

The second wavy configuration uses patterned sites of adhesion on the substrate and/or ribbons to create localized positions of bonding. When the pre stretched substrate is released, the non bonded regions delaminate from the substrate. In this case the waviness can be controlled thus controlling the level of stretchability of the device. With this approach it is possible to achieve reversible stretching to strains of 100 % or more.

All the research results on stretchable silicon electronics have one thing in common: silicon has to be patterned, leading to a network with more or less rigid nodes and flexible/stretchable connexions between nodes.

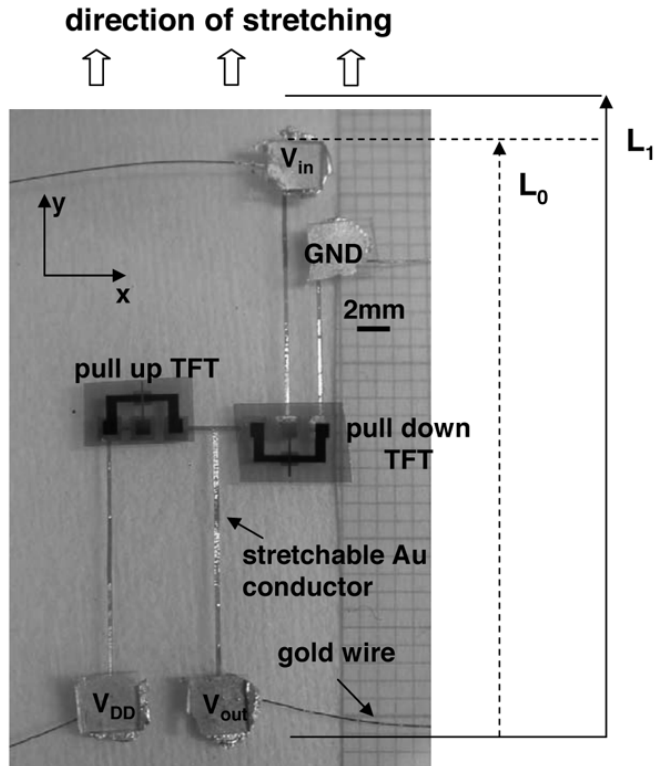
Silicon wafers can become flexible if thinned to thicknesses under  $100\ \mu\text{m}$ . The thinner the silicon, the more bendable it becomes. This way, products that rely on unfoldable thin silicon structures have been developed.



**Figure 2-10.** Stretchable mesh: the square silicon photodetectors, connected by thin ribbons of metal and polymer, are mounted on a hemisphere-shaped rubber surface. The entire device is able to conform to any curvilinear shape due to the flexibility of the ribbons that connect the silicon islands. Credit: Beckman Institute, University of Illinois.

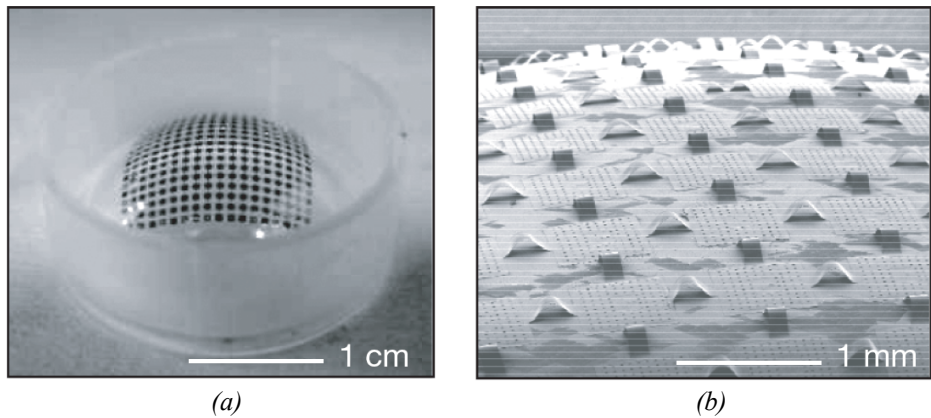
In 2004, Lacour presented stretchable electrical conductors of 25 nm thick gold films with surface waves with  $\sim 8.4 \mu\text{m}$  wavelength and  $\sim 1.2 \mu\text{m}$  amplitude [2.7]. The gold waves showed reproducible and repetitive resistance change and deformation for a cyclic strain variation for 0 % to 15 %. Using the wavy gold interconnects, the same group reported the fabrication of an elastically stretchable TFT circuit (inverter) with identical performances in the relaxed state and after stretching up to 12 % [2.8]. Amorphous silicon TFTs made on polyimide foil are mounted face down on contact pads and interconnected with the stretchable gold conductors. The clear silicone membrane covers the entire field of view and is partly underlaid by millimeter-ruled paper.





**Figure 2-11.** Photograph of the elastomeric inverter. The circuit is shown in its relaxed state [2.8].

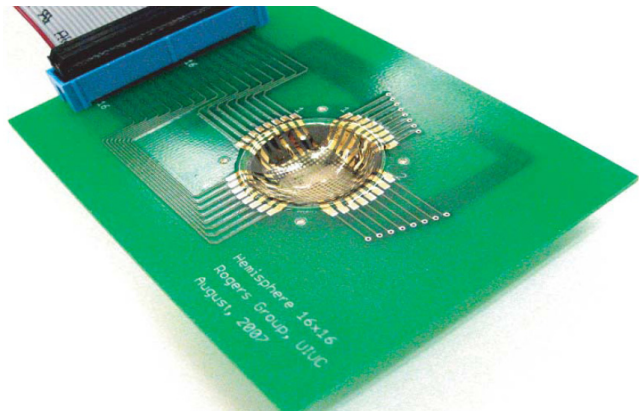
Using a mesh of silicon photodetectors and very thin wavy metal interconnects [2.9], a hemispherical electronic eye camera was fabricated by separating the complex control circuitry from the imager circuitry.



**Figure 2-12.** (a) Photograph of a hemispherical PDMS transfer element with a compressible focal plane array on its surface and (b) scanning electron microscope image of a portion of the array in (a), illustrating the compressible interconnects [2.9].

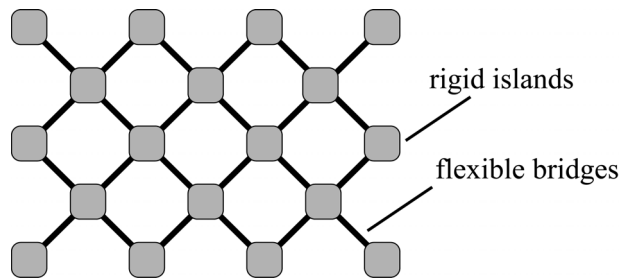
The approach uses wafer-scale optoelectronics formed in unusual, two-dimensionally compressible configurations and elastomer transfer elements capable of transforming the planar layouts in which the systems are initially fabricated into hemispherical geometries for their final implementation.

Although in this case, the stretchability is exploited only to create fixed 3D-curved imagers, the concept is valid for systems that modify their shape during operation. The imager is an array of identical islands with a small number of inter-island interconnects supported by thin bendable semiconductor bridges.



**Figure 2-13.** A PCB-mounted hemispherical electronic eye camera [2.9].

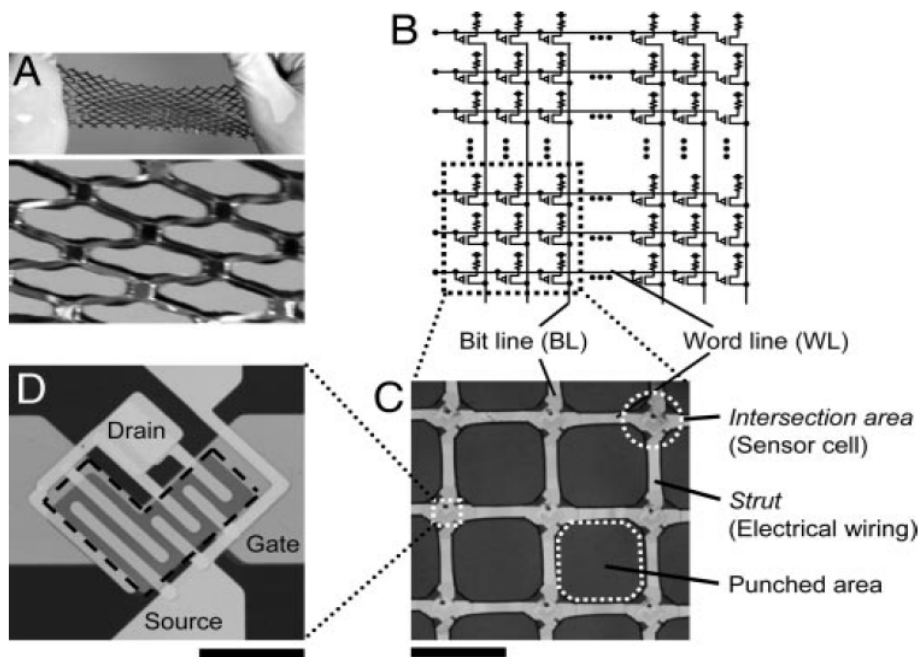
The second approach uses meshes constructed from bendable materials to achieve large and reversible deformation for strains applied on certain axes, while having functional islands in the nodes of the mesh. In this case in plane strains are accommodated by in plane rotations, similar to the movement of a scissors (Figure 2-14). Tensile strains applied at the ends of the structure transform the rectangles of the mesh into diamond like shapes, the mesh becoming longer and narrower.



**Figure 2-14.** Mesh-shaped semiconductor island stretchable system.

Using the mesh topology, conformable, flexible, large-area networks of pressure and thermal sensors with organic transistor active matrices were fabricated by the Quantum-Phase Electronics Centre of the University of Tokyo [2.10]. Arrays of pressure and temperature sensors are built on a film of 75  $\mu\text{m}$

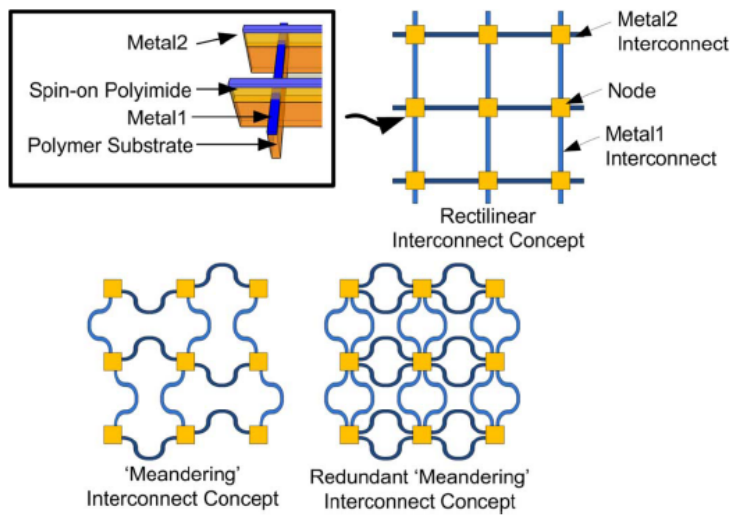
thick polyimide or poly (ethylenenaphtalate) (PEN). Stretchability is obtained by using a laser to pattern the film in a mesh-like pattern having nodes containing active IC's, resulting in a system with 25 % stretchability.



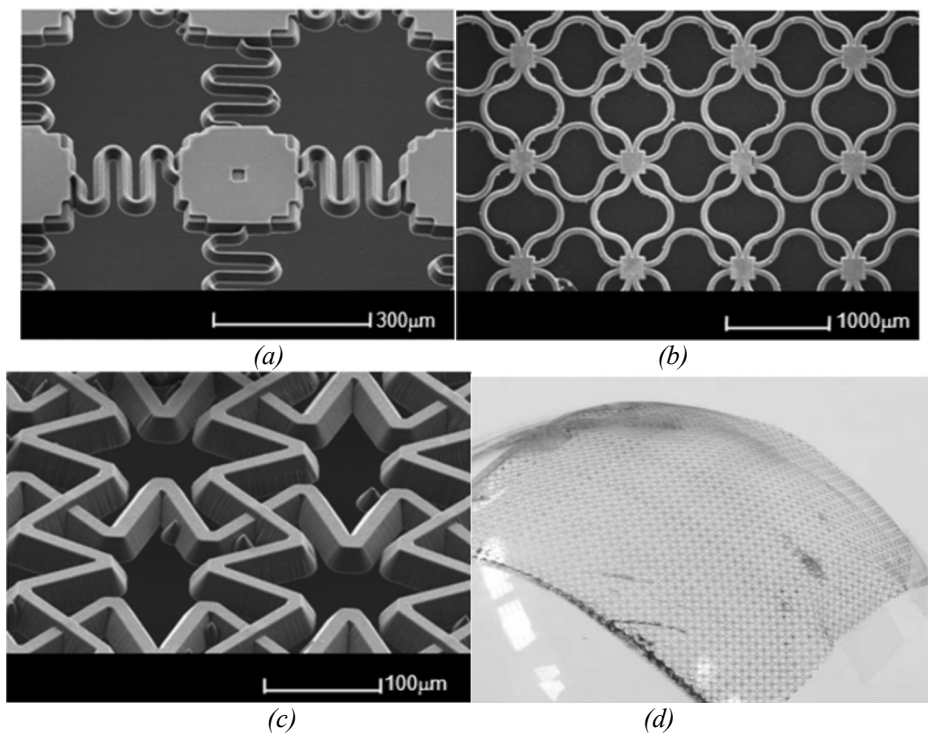
**Figure 2-15.** A conformable network of pressure sensors. (A) A plastic film with organic transistors and pressure-sensitive rubber is processed mechanically to form a unique mesh-shaped structure; (B and C) Circuit diagram of the pressure sensor network (B) together with a picture of the 3x3 sensor cells (C); (D) Microscope image of an organic transistor before shaping the net or integrating it with sensors. Dotted line indicates the semiconductor channel layer. (Scale bar: 1 mm.) [2.10].

Robust polymer meshes are fabricated using metal patterns both as functional interconnect layers and as in situ masks [2.11]. The method uses laser ablation to pattern polymer layers into rectilinear and meandering designs with single or multilayer metal interconnects.

Fabricated structures can be stretched uniaxially up to 50 % while maintaining good electrical conductivity and structural integrity. "Meandering" interconnects are observed to have a maximum stretchability of greater than 50 % with a change in resistivity of only 5 %. Redundant interconnect mesh have a maximum stretchability of almost 30 % (redundant designs will increase the robustness and viability of interconnects without sacrificing stretchability). A 40 mm by 40 mm interconnect mesh prototype using the redundant mesh design was fabricated as a proof of concept for feasibility of the developed process.

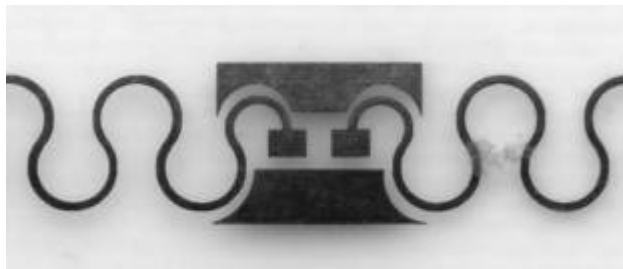


**Figure 2-16.** Interconnect mesh concept for rectilinear, “meandering,” and redundant “meandering” interconnects. The interconnects consist of a polymer substrate and two metal layers separated by a spin-on polyimide dielectric layer [2.11].



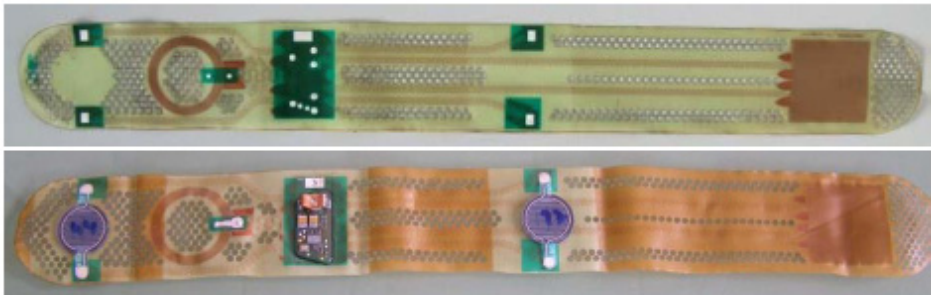
**Figure 2-17.** (a) Meandering interconnect design with 10 line-width; (b) and (c) Redundant interconnect designs with 40 µm line-width; (d) Demonstration of flexibility of 40 mm by 40 mm redundant sensor mesh (conformability to a spherical flask with diameter of 50 mm) [2.11].

Several stretchable electronics products (for sensor applications for medical electronics and textile electronics) have been developed in the frame of the European project STELLA [2.12]. PCB quality copper sheets (17-70  $\mu\text{m}$  thick) are laminated on thermoplastic polyurethane (TPU) foils (100  $\mu\text{m}$  thick). Following the lamination process, the copper foil is patterned into meanders to define stretchable interconnects. Additional “stiffening” structures are fabricated in the copper patterning step. Their role is to locally prevent substrate stretching, being able to protect the interconnections up to very high elongations (500 % tested). Additional components are assembled (manually or using automated pick & place machine) and soldered to the substrate using low temperature reflow (165 °C). The final encapsulation is done using a cast overmolding process. The profile of the encapsulation is such that the transition between the thick encapsulation and the thinner part containing only the stretchable lines is smoothed out.



**Figure 2-18.** Close-up of stiffening structure around a component position [2.12].

One of the products fabricated with this technology is a band aid equipped with pressure sensors and a simple humidity sensor. Pressure sensors are used for monitoring of the pressure on the wound during compression therapy. Data is read out wireless using inductive data transmission with a reach of  $\sim 1$  m from the antenna. Rather large holes in the polyurethane were included for permeability of air and humidity in order to improve user comfort.



**Figure 2-19.** Band aid demonstrator [2.12].

A second product is a three point pressure sensor for integration into a shoe insole. Most of the systems are wires only connecting the three sensors which

are located at the heel, the ball of the first and the fifth toe, respectively. In order to provide as much as possible user comfort by allowing circulation of air and moisture, most of the polyurethane surrounding the wires connecting the three sensors was eliminated. However, in order to be easy to handle the remaining structure was laminated into a highly permeable non-woven fabric.

Textile industry already uses polyurethanes for various applications such as glue to join textiles, as ultra thin layer acting being a semi permeable membrane in waterproof textiles, or as interlining separating different layers of cloths from each other. The connection of polyurethanes is a lamination process operated at similar temperatures as ironing. An example of a stretchable electronic system containing acceleration sensors, processors and LEDs was designed and laminated onto a commercial fabric to be integrated into a shirt, see Figure 2-20. A high robustness in many wear cycles has been field proven with this application.



**Figure 2-20.** *Stretchable electronic system laminated to fabrics [2.12]*

Since the islands can only bend (if thin enough), the stretching part has to be done by the materials between the islands. Conductive and non conductive polymers, silicone rubbers and metals are some of the choices. These materials have to support the islands, provide electrical connections between the islands and stretch. When looking for the right combination of materials, one has to take into account processing compatibility and properties of individual materials, such as conductivity, elasticity, bio-compatibility, reactivity to chemicals and so on.

Although the goal of stretchable electronics research is to develop systems that can uniformly stretch, changing their dimensions by hundreds of percents, current materials properties (both mechanical and electrical) limit the design and fabrication choices.

While the functional blocks of a stretchable electronic systems are fabricated on a rigid substrate with limited flexibility, the materials used for interconnects and protection are more stretchable. This is why the focus of stretchable electronics is on the stretchability of interconnects and protective polymers while leaving the circuitry largely unchanged on rigid substrates.



2.5. *System aspects of stretchable silicon microsystems*

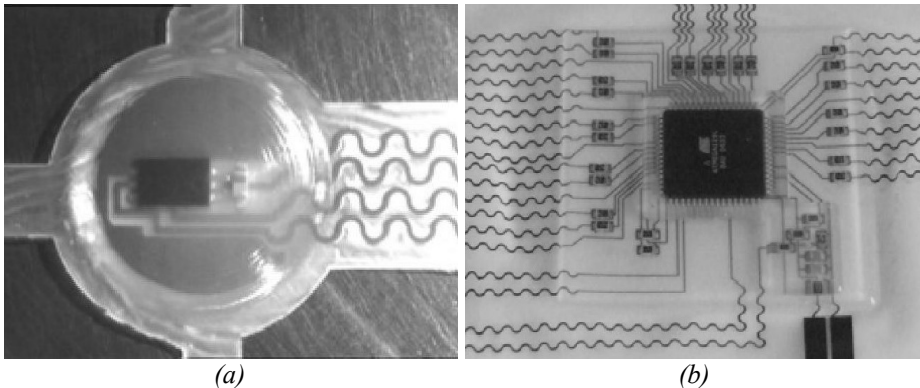
Traditional chips can have hundreds, even thousands of interconnects. When a system is segmented into functional blocks, two types of interconnects are noticed: interconnects between blocks and interconnects on individual blocks. By rethinking the system for this approach and dividing it into small enough blocks but keeping the number of inter-block interconnects to a minimum, more complex systems can be made stretchable. While in some cases, it is necessary to make the whole system stretchable, in some applications only a part of the segmented system will be stretchable this way minimizing the number of needed stretchable interconnects.

Polymers can be stretched by hundreds of percents before they break but metals have much more limited stretchability. Current research on stretchable metal interconnect schemes shows that metal interconnects stretchability can be increased several times by replacing narrow straight metal lines with more complex shapes such as horseshoe, meanders or meshes.

Using these shapes for metal interconnects can increase stretchability up to several hundred percents but with serious increase of area used by interconnects. One simple line can become stretchable if it is replaced with a meander but the area occupied is several times larger. This limits the number of interconnects thus limiting the complexity of a stretchable microsystem.

Scaling down the interconnect size by using finer lithography can reduce the area but also some other issues have to be taken into account. Work done in the STELLA project shows how discrete components are assembled to a stretchable printed circuit board (SCB). Two techniques are used to minimize the stress on components interconnects: local stiffener structures and polyurethane glob tops with gradually increasing thickness [2.13].

The transition between stretchable (meandering Cu) and rigid parts of the system (once the components are assembled) has to be accounted for. Stiffening of the substrate around components is done to protect the component interconnections against stress due to stretching (Figure 2-18).



**Figure 2-21.** (a) Encapsulated component with a smooth glob top encapsulation, and (b) fabricated system using polyurethane encapsulation [2.13].



The profile of the glob top encapsulation is such that the transition between the thick, rather stiff encapsulant and the thinner part containing only the stretchable copper lines is smoothed out (Figure 2-21a).

By using a combination of these stress reducing techniques, complex systems can be created and lamination into textiles has been demonstrated (Figure 2-21b).

## ***2.6. Fabrication compatibility, handling and reliability***

Although there is a large variety of materials that can be used, IC fabrication has strict requirements on chemicals purity, air quality, contaminants and safety. When selecting materials for interconnects (usually metals) or for protection (usually polymers), all the IC fabrication contamination and safety requirements have to be met and processing compatibility of all materials has to be guaranteed.

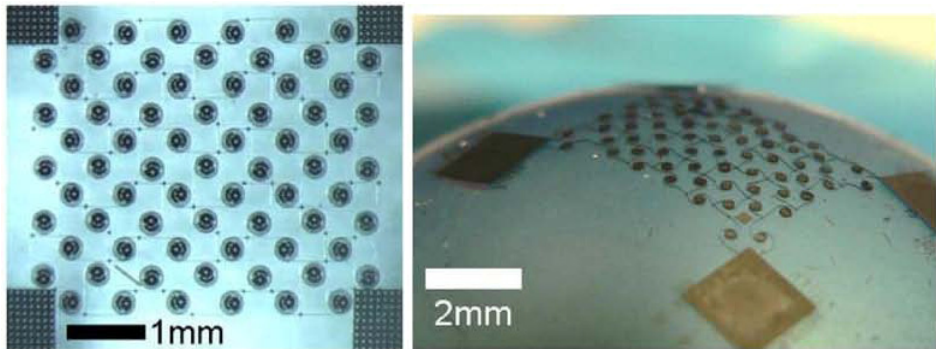
Once the materials are selected and fabrication process compatibility and contamination issues solved, traditional processing and handling steps can prove to be too aggressive for stretchable systems. A simple example: 20  $\mu\text{m}$  thin chips connected with thin metal lines embedded in a soft polymer layer can be tricky to handle during or even after fabrication.

Thin wafers and chips need different handling procedures than traditional thick ones. Wafer dicing uses high pressure water and adhesive foils that can be destructive for some flexible systems.

While regular electronic systems are encapsulated in rigid packages that can be easily handled and connected to circuit boards, new handling and connecting methods have to be developed for stretchable electronics.

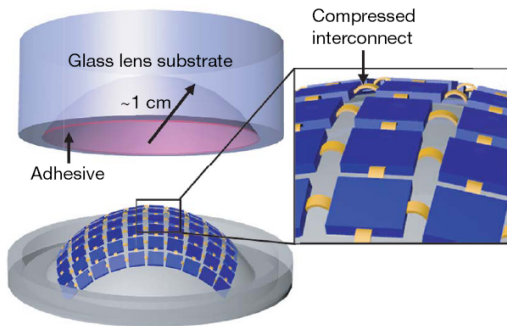
Although the goal is to fabricate systems that can sustain large deformation without losing their functionality, substrate transfer is a common technique to provide the to-be-stretchable systems with enough stiffness needed for traditional IC fabrication and assembly processes. For fabricating diamond like carbon (DLC) islands for stretchable electronics, PDMS foils are mounted on plastic foils for handling during processing and released at the end. A temporary substrate is also employed in the fabrication process for stretchable printed circuit boards, part of STELLA project.

Large area electronics arrays of islands connected with narrow spirals using monolithic silicon use large blocks of silicon needed to handle individual arrays [2.14].



**Figure 2-22.** Silicon island array using narrow spirals as interconnect (left), showing large blocks needed for handling (right)[2.14].

Spherical camera lenses are first fabricated on rigid substrates using established processing techniques. Transfer to a spherical lens is done in several steps. First, a hemispherical curved PDMS carrier is stretched to a planar shape and then the silicon island array is transferred using van der Waals forces to bond the silicon to the pre-stretched PDMS carrier. In the second stage, the pre stretched PDMS carrier is allowed to return to its relaxed hemispherical shape. In the last step, the silicon array is bonded using glue to the hemispherical lens.



**Figure 2-23.** Part of hemispherical camera lens fabrication sequence, illustrating transfer to curved surface stages [2.9].

In current research there are two generic approaches used for fabricating stretchable systems, each having its benefits and disadvantages. In the first approach, the substrate is attached to a flexible/stretchable layer, patterned and then stretched. The method is wasteful as in order to create stretchable regions between rigid islands, most of the wafer is removed, leaving only islands attached to the flexible layer. Another problem appears when one island (die) is defective and the whole system becomes unusable. Same problem (even more wasteful) appears if several wafers are bonded and good devices will be introduced in already defective systems. Depending on the fabrication process, the wafer can be processed until the end using standard clean room

equipment/techniques, without the need to develop new methods for handling/processing.

In the second approach, the substrate is patterned, stretched and then, if necessary, attached to the flexible medium [2.11, 14]. The method suffers from the same yield problem as the previous (one bad die ruining the whole system) and a possible additional problem caused by handling and stretching fragile patterned structures.

A variation of the first approach is hybrid integration (using a dummy substrate or a functional substrate). It can solve the handling problems by using established pick-and-place technology to assemble any die configuration on a target substrate (removed or patterned later), increasing the yield by using known good dies. Most research examples previously presented use hybrid integration on a temporary carrier [2.12, 13].

## ***2.7. Conclusions***

This chapter presented a theoretical overview of mechanical notions used throughout the thesis to describe the behavior of stretchable electronic systems. Current methods of creating stretchable systems have been described and each fabrication method has been illustrated with relevant state of the art devices. The performances of current devices were presented along with design, fabrication and handling issues specific for stretchable systems. Next chapters will present in detail the selected interconnect geometries, development of wafer-level fabrication process of stretchable silicon electronics embedded in PDMS and testing of the fabricated arrays.

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# Chapter 3:

## *Stretchable Interconnect Schemes*

In this chapter the interconnect schemes and geometries selected for the fabrication of stretchable interconnects are presented. First the general requirements on a stretchable interconnect are defined and suitable interconnect geometries are selected. These are then analyzed using finite-element simulation tools and the obtained results are used for further optimization. In the final part of this chapter, the fabrication and measurement results obtained for free-standing high-aspect-ratio silicon springs and free-standing stretchable metal interconnects are presented.

### *3.1. Introduction*

A stretchable silicon microsystem fabricated using a segmented substrate approach will get its stretchability from two sources: the interconnect between segments and the protective polymer. The stretchability of the interconnect depends on two factors: the material and the geometry. The choice of materials is limited to metals or conductive polymers. There are advantages and disadvantages of each material as metals have high electrical conductivity but are elastic for only a rather small tensile strain region, while conductive polymers have comparably poor electrical performances but much more favorable elastic properties.

### *3.2. General requirements on stretchable interconnects*

The first question one can ask about a stretchable interconnect scheme is “How much can it stretch?”. This can be described as the most obvious requirement of a stretchable interconnect and of a stretchable electronic system. The strain, as presented in Chapter 2, is defined as the ratio between the change in length  $\delta L$  and the original length  $L$ .

Depending on the final product requirements, the stretching can be reversible or not. Non-reversible stretching can be useful if the system is designed to cover a large area but it is fabricated in a folded state and finally unfolded to its final size. Reversible stretching is more difficult to achieve for large strain values and fatigue becomes an important failure factor.

Besides the ability to accommodate strains, there are more factors that influence the performance of a stretchable electronic system. All stretchable electronic approaches use lateral segmentation of the substrate, i.e., rigid functional nodes connected by an elastic and electrically conductive medium. The nodes or islands must be supported by the surrounding elastic and conductive network. Depending on the fabrication methods and materials,

mechanical support can be provided by interconnects only (if strong enough), or achieved by application of soft and elastic polymers. The latter being a more frequent choice. In addition to the mechanical support, the polymers can also provide mechanical and/or chemical protection.

In a traditional monolithic chip, the role of interconnects is to carry signals between the different components with minimum losses. When compared to traditional monolithic chips, stretchable systems use a larger area as the chip is divided into functional blocks linked with a stretchable and conductive medium. In a stretchable system, besides carrying electrical signals between individual blocks, interconnects (together with the surrounding supporting medium) must stretch to withstand strain. The interconnect geometry determines its ability to stretch but also determines the electrical resistance. The change of electrical resistance with changing strain must be taken into account when selecting a certain shape and size for the interconnect network as it can seriously affect the system performance.

Depending on the application, a stretchable system might need to stretch more in one direction but nevertheless 2D stretchability is desired. 1D stretching is a (relatively) simple case as all the interconnect designs presented can be stretched mainly in one direction (longitudinal) but can also withstand small transversal deformation.

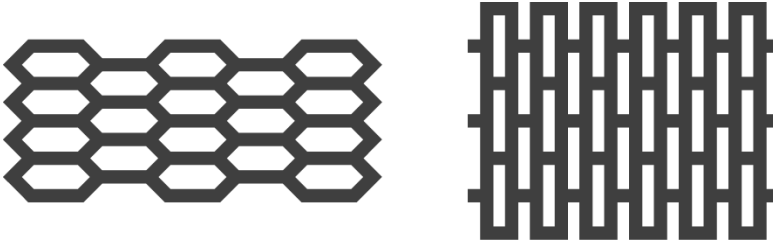
For 2D stretching, there are two cases that can be considered. A simple solution is to use interconnects with 1D stretching ability, distributed along the main stretching axes, obtaining in this way a system that can globally stretch in 2D. Such a system will have limited 2D stretchability as interconnects still have a preferred axis of deformation. Another approach is to use an interconnect shape that can stretch in 2D because of its design. A mesh geometry, similar to woven textiles, is an example of such design.

### ***3.3. Geometry for stretchability***

Elastic deformation in metals is limited to very small strains. Patterning metals into various shapes gives them the ability to deform elastically, thus recovering their original shape without sustaining damage. Patterned metal tracks can also be supported by silicon geometries patterned for stretchability. The geometries discussed next can be used either way, only as thin metal tracks or as silicon structures supporting the thin metallization.

#### ***3.3.1. Mesh geometry***

One such pattern is the mesh, consisting of a network of metal beams in a repeated pattern. Although there can be different geometries for a mesh, such as honeycomb, rectangular or diamond like, the effect of tensile strains applied at the ends of the mesh is the same: the elements of the mesh change their shape, in plane strains are accommodated by in plane rotations, similar to the movement of a scissors as the mesh becomes longer and narrower.

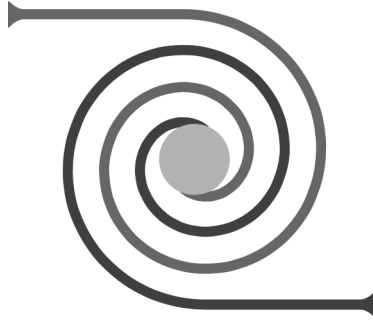


**Figure 3-1.** *Different mesh designs: honeycomb (left) and rectangular (right).*

Such interconnect has some advantages (it is strong and if some parts are damaged, electrical conductivity is maintained), but its size will greatly limit the number of interconnects that can be realized on a given area. The mesh shape is more suitable for unsupported interconnects as the brittle supporting silicon will greatly reduce the maximum elongation achievable when using only metals.

### **3.3.2. Spiral geometry**

Two spirals, with opposite rotation directions and connected in the middle (Figure 3-2), can unfold when the ends are pulled apart. Similar to the mesh, islands can be connected with spiral springs or can be fabricated in the centre of the two spirals, acting also as connector between the spirals.



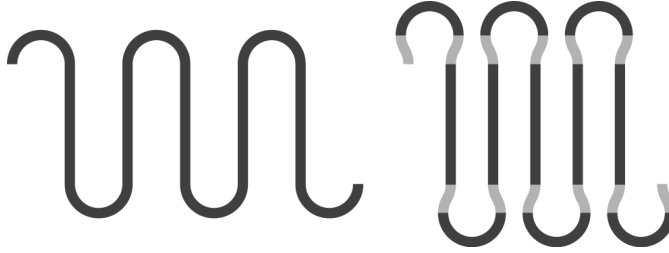
**Figure 3-2.** *Spiral design, showing opposite spirals.*

The spiral geometry must rotate to unfold and its ability to unwind will be greatly limited if used as patterned metal embedded in a protective medium. As supporting silicon structure, the width of the arms must be small ( $<1 \mu\text{m}$ ) to allow the arms to unfold to almost a straight line [3.1].

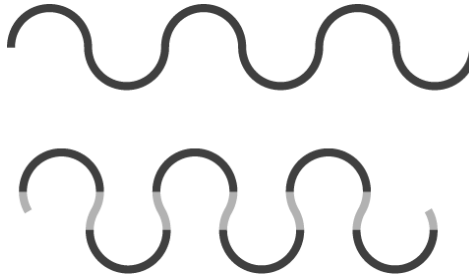
### **3.3.3. Meander and horseshoe geometry**

Other patterns that have been investigated are meander and horseshoe. The meander pattern is made of a series of half circles (round caps) connected with vertical straight segments. The transition between the rounded cap and the straight segment can also have a certain curvature that reduces the stress concentration in that area, allowing for a higher deformation. If made from a ductile material, the meander can be stretched to a straight line.

The horseshoe interconnect is very similar to the meander design. It is made of half circle segments with optional smaller segments of opposite curvature that reduce the stress concentration, allowing larger extensions.



**Figure 3-3.** Meander designs showing simple design (left) and design with additional curved segments (light grey) (right).



**Figure 3-4.** Horseshoe designs showing simple design (top) and design with additional curved segments (light grey) (bottom).

These geometries can be easily tuned to support large longitudinal deformation by adjusting the curvature angle of the transition segment. Small size allows large number of interconnects to be used in a system. The meander version uses more area and is suitable to be used as silicon supporting structure as the vertical beams allow larger deformation for a certain strain level when compared to the horseshoe geometry.

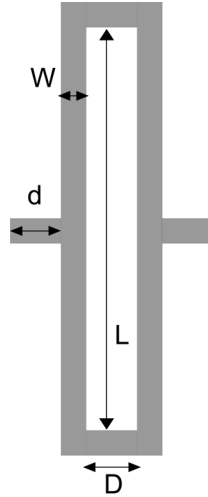
#### 3.3.4. Parametric structure description

Each of the structures presented is fabricated by repeating a simple element (base element) in 1D or 2D. The base element is defined by a set of geometric parameters, independent of material and processing parameters.

For the rectangular mesh, the base element is defined by four parameters:

- the width of the metal line,  $W$ ;
- the width of one element,  $D$ ;
- the length of the vertical lines,  $L$ ;
- distance between two elements,  $d$ .



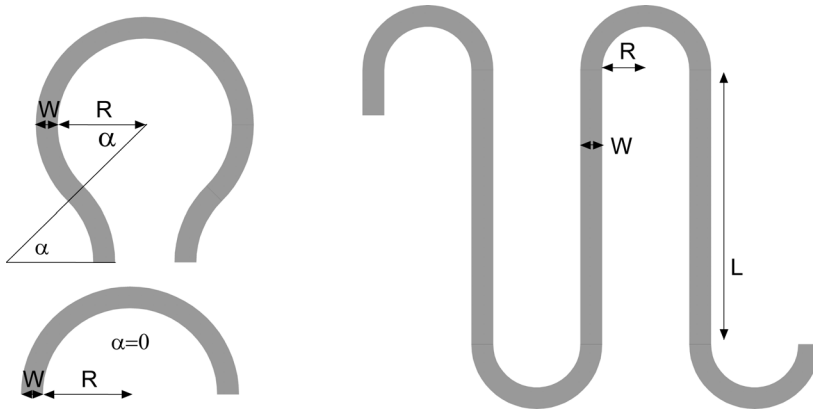


**Figure 3-5.** Mesh base element with parameters.

Starting from this simple design, more complex meshes can be created that have lines with variable width and curved transitions instead of sharp corners.

Meander interconnects are determined by four parameters:

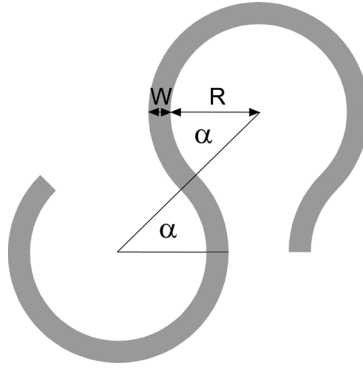
- the width of the metal line,  $W$ ;
- the radius of the curved element,  $R$ ;
- the length of the vertical lines,  $L$ ;
- the angle of the additional curved element,  $\alpha$ .



**Figure 3-6.** Meander curved part (left) and meander (right) with parameters.

The horseshoe design is a simplified version of the meander, without the vertical lines. Taking this into account, the horseshoe is determined by the three parameters:

- the width of the metal line,  $W$ ;
- the radius of the curved element,  $R$ ;
- the angle of the additional curved element,  $\alpha$ .



**Figure 3-7.** *Horseshoe interconnect geometry with design parameters.*

### **3.4. Finite-element simulations**

When constructing an interconnect scheme for a stretchable system, factors such as overall dimension of the system, number and size of interconnects or biocompatibility limit the number of design and fabrication alternatives. Area available for interconnects, total length (increased electrical resistance), maximum elongation and stress (leading to mechanical failure) are the primary factors that have to be considered while selecting a certain interconnect.

Chapter 2 presented an overview of the selected materials and their mechanical and electrical properties. The equations presented are useful when discussing homogeneous materials but current state-of-the-art systems are all heterogeneous. The behavior of heterogeneous systems requires numerical analysis such as finite element simulations.

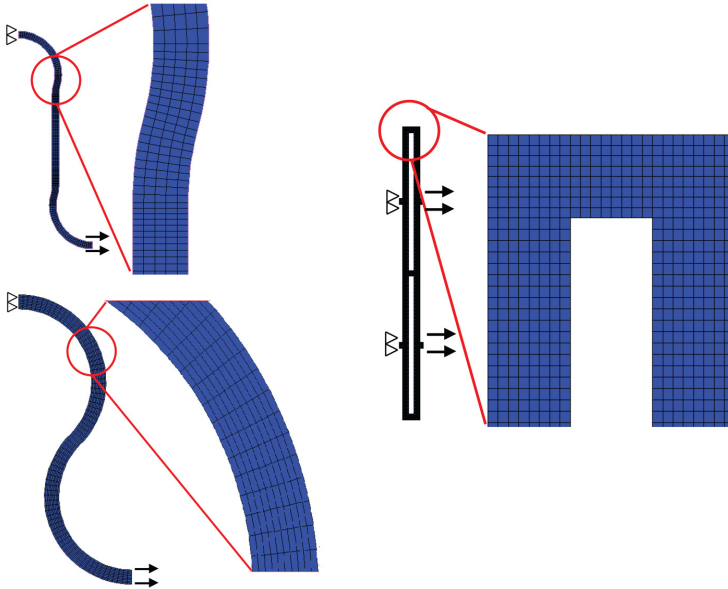
Tensile load simulations were performed to understand the failure causes and to optimize the system for increased elongation and lifetime. Because of the better electrical and mechanical performance but also chemical resistance, copper interconnections are chosen. Several geometries of spring-like structures are numerically tested. For the simulations adequate material parameters are necessary, which were discussed in Chapter 2. The large ratio of width (or length) to thickness offered a challenge to reliable FE simulation. Therefore multilevel FEM simulations were performed. A global-local model has the advantage of subdividing large models into multiple, moderate-size models and thus separating fixed model parts from parts of the model that may undergo design changes [3.2].

When exploring the suitability of the new designs to survive large extensions, the limiting factor for failure is assumed to be the ultimate equivalent strain in the copper.

As all interconnects are made up from repetition of basic shapes and based on symmetry considerations, FE simulations can be restricted to the basic shapes only (Figure 3-5, Figure 3-6 and Figure 3-7). The models used are illustrated in Figure 3-8. The boundary conditions include fixed x-direction

displacement of nodes at the left end and prescribed x-direction displacements on all boundary nodes at the right end. The rigid body movements are additionally suppressed [3.3].

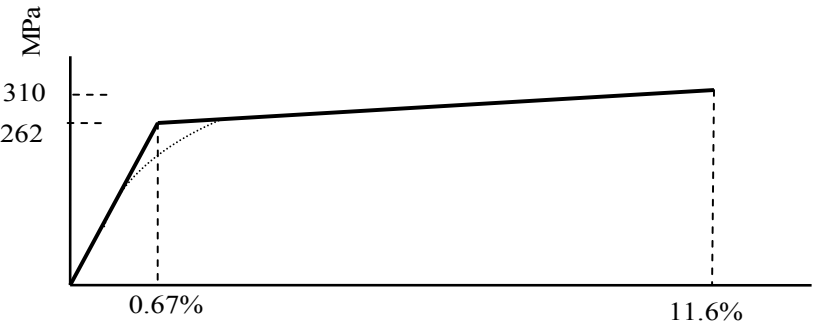
The elastic plastic model for copper has been used [3.4] and material properties are listed in Table 3-2. Experiments were not performed to establish the real elasto-plastic properties of the copper interconnect. Instead an elasto-plastic behavior is assumed in the present simulations on the basis of measurement results as published by Dao for a copper line of a thicknesses only a few microns [3.5]. The actual stress-strain curve as given by Dao is shown as the dashed line in Figure 3-9. The Young's modulus and Poisson's ratio (in the elastic region) are 128 GPa and 0.36, respectively. As the elasto-plastic behavior is strongly dependent on the grain size, it is obvious that Dao's stress strain curve can only be considered as a rough indication for the real material behavior of our interconnects. Therefore, the stress strain curve used is a simplified version using a linear hardening slope and steady plastic behavior after reaching the ultimate strength (see Figure 3-9). The ultimate strain according to Dao's measurements is 11.6 %. In the FE simulation results, reaching this ultimate strain value is used as a failure criterion, although when this just occurs locally, the whole structure is not really failed and electrical conduction will hardly be affected.



**Figure 3-8.** FE models of free-standing interconnect base parts: (a) meander, (b) horseshoe, and (c) mesh.

The behavior of a sample under tensile stress can be described by using three intervals defined by the yield and ultimate strengths (YS and US). For stress values in the interval  $[0, YS]$ , the sample gets no damage, only elastic

deformation (100 % reversible deformation). In the second interval [YS, US] samples will not get damaged if extended but will not be able to completely return to the original shape (combination of elastic and plastic deformation). The last interval is [US,  $\infty$ ] where samples will suffer only plastic deformation (0 % reversible deformation), will get damaged and finally break.



**Figure 3-9.** Stress-strain curve of copper film, dotted line is measurement results according to [3.5]. Solid line is constructed bi-linear behavior as used in the FE simulations.

**Table 3-1.** Interconnect parameters.

<i>Meander and Horseshoe</i>				
<i>Parameter</i>	<i>Values (μm)</i>			
Angle <b>α</b>	0	15	30	45
Width <b>W</b>	5		10	20
Radius <b>R</b>	10		50	100
Length <b>L</b> (meander)	250			
<i>Mesh</i>				
	<i>Values (μm)</i>			
Width <b>W</b>	5		10	20
Distances <b>D, d</b>	5		10	20
Length <b>L</b>	100		300	500

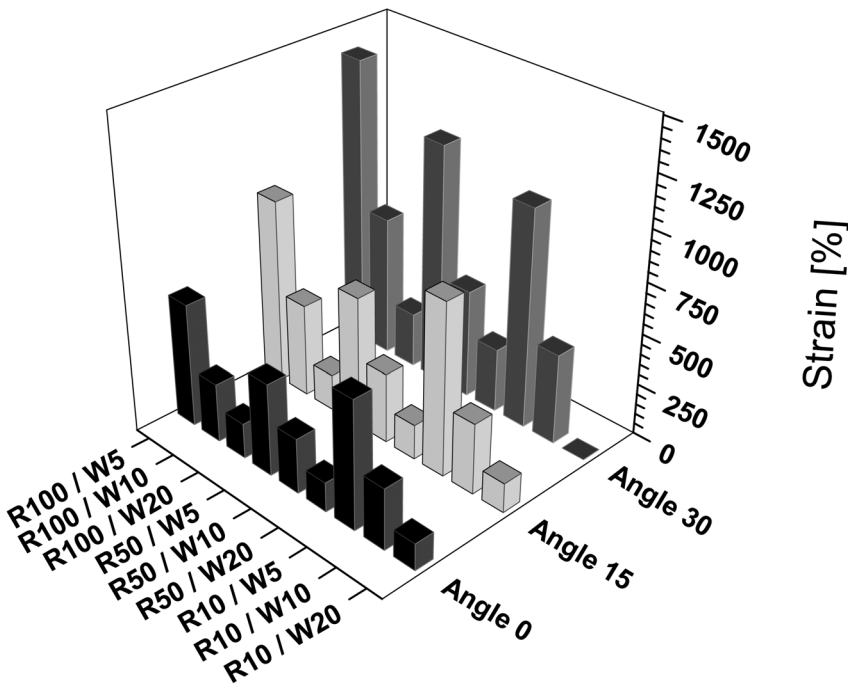
**Table 3-2.** Material properties of copper used in simulations.

<i>Young's modulus</i>	128 GPa
<i>Poisson's ratio</i>	0.36
<i>Yield strength</i>	262 MPa
<i>Ultimate strength</i>	310 MPa

### 3.4.1. Meander interconnects analysis

For meander base elements, defined by  $(W, L, R, \alpha)$  (see Figure 3-6), simulations are performed for elongation steps increasing with 5 % up to a maximum elongation of 1500 %. The maximum equivalent strain on the interconnects under the applied elongation steps is determined and by comparing it with the failure strain (ultimate strain) it is possible to find the sample elongation (or mean strain level) at (assumed) onset of failure.

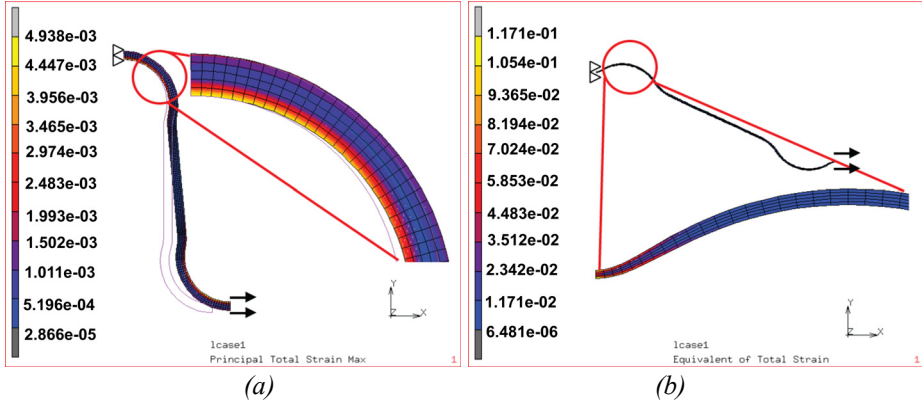
The mean strain at onset of failure (defined here as reaching the ultimate strain of the copper) is presented in Figure 3-10 for various model parameter sets (width  $W$ , radius  $R$  and angle  $\alpha$ ). It is found that the behavior improves with larger radius  $R$  and smaller width  $W$ . Also the increase of the angle  $\alpha$  has a positive influence. The best result (being the one with the highest mean strain) is found for  $W=5 \mu\text{m}$ ,  $R=100 \mu\text{m}$  and  $\alpha=30^\circ$ .



**Figure 3-10.** The mean strain at onset of failure of the meander shape interconnect for various model parameter sets.

Figure 3-11 shows the equivalent strain distribution at an elongation of 176 % (maximum strain for PDMS) for the meander interconnect with the best results ( $W=5 \mu\text{m}$ ,  $R=100 \mu\text{m}$  and  $\alpha=30^\circ$ ) and at the maximum strain before reaching ultimate strength. The maximum equivalent strains of the free-standing meander interconnects are found within the area of the wave peak (at the inner

side). A combination of tensile force and bending moment in the interconnect line is responsible.

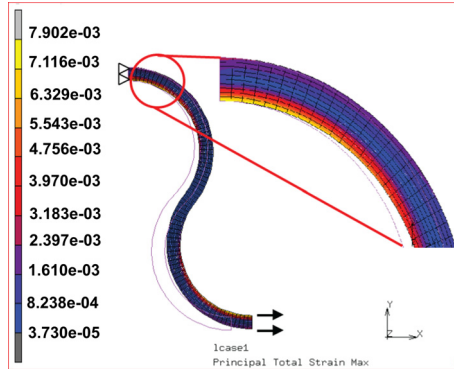


**Figure 3-11.** Equivalent stress distributions for meander interconnect ( $W, R, \alpha$ ) = (5, 100, 30) at (a) 176 % and (b) 1375 % strain.

Since the strain in a metal beam is inversely proportional to its width, if the same curvature is used, the elongation can be increased by using narrower metal lines [3.6].

### 3.4.2. Horseshoe interconnect analysis

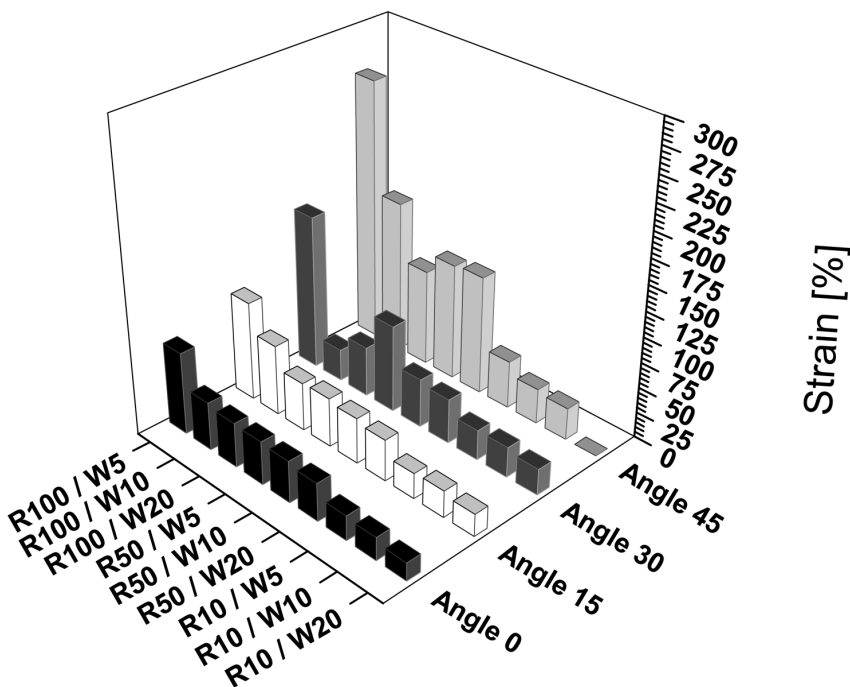
For horseshoe interconnect base elements, simulations are performed for elongation steps increasing with 5 % up to a maximum elongation of 300 %. The maximum equivalent strain of the free-standing horseshoe interconnect is found within the area of the wave peak (at the inner side, see Figure 3-12).



**Figure 3-12.** Equivalent strain distribution for the horseshoe interconnect ( $W, R, \alpha$ ) = (10, 50, 45) under prescribed mean deformation (elongation) of 20 %.

A combination of tensile force and bending moment in the interconnect line is responsible for this behavior. It is found that the behavior improves with larger radius  $R$  and smaller width  $W$ . Increasing the radius leads to a small

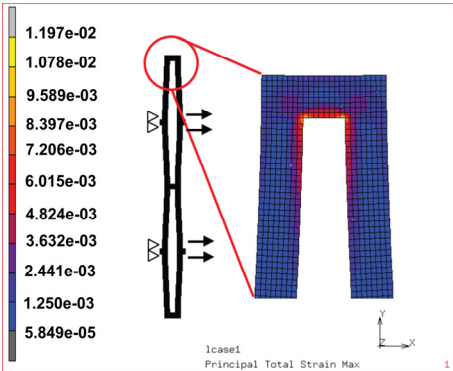
increase of the elongation (several percent) but the overall growth of the structure is much larger, leading to an inefficient use of available area for interconnects. Also the increase of the angle  $\alpha$  has a positive influence. The maximum mean strain of the sample can reach 250 % for the parameter set  $W=5\text{ }\mu\text{m}$ ,  $R=100\text{ }\mu\text{m}$  and  $\alpha=45^\circ$ .



**Figure 3-13.** The mean strain at onset of failure for horseshoe interconnects, for various model parameter sets (width  $W$ , radius  $R$  and angle  $\alpha$ ).

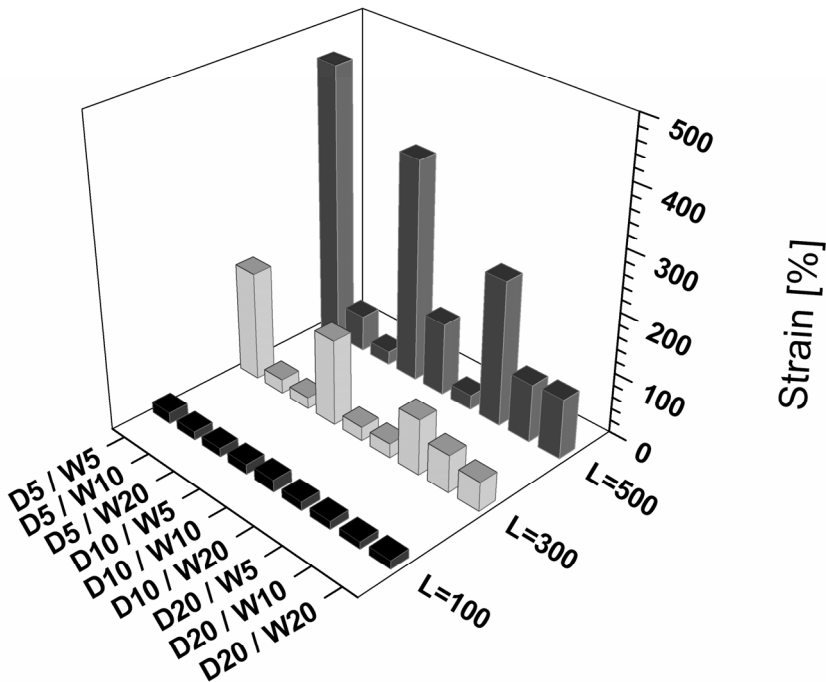
**3.4.3. Mesh interconnect analysis**

Mesh type interconnects simulations are performed for elongation steps increasing with 5 % up to a maximum elongation of 500 %. The local maximum equivalent strain of the free-standing mesh interconnect is found at the inner side of the vertices (Figure 3-14). A combination of tensile force and bending moment is responsible for this maximum equivalent strain.



**Figure 3-14.** Equivalent strain distribution for the mesh interconnect with  $(W, D, L) = (20, 20, 500)$  under prescribed mean deformation (elongation) of 20 %.

For the mesh shaped interconnects, the mean strain at onset of failure for various model parameter sets (width  $W$ , length  $L$  and distance  $D$ ) is presented in Figure 3-15. It is found that the behavior improves with larger length  $R$  and smaller width  $W$ . Also the increase of the distance  $D$  has a positive influence. The maximum mean strain of the sample can reach 450 % for the meshed interconnect with beam length  $L=500 \mu\text{m}$ , width  $W=5 \mu\text{m}$  and distance  $D=5 \mu\text{m}$ .



**Figure 3-15.** The mean strain at onset of failure of the mesh shape interconnects, for various model parameter sets (beam length  $L$ , width  $W$  and distance  $D$ ).



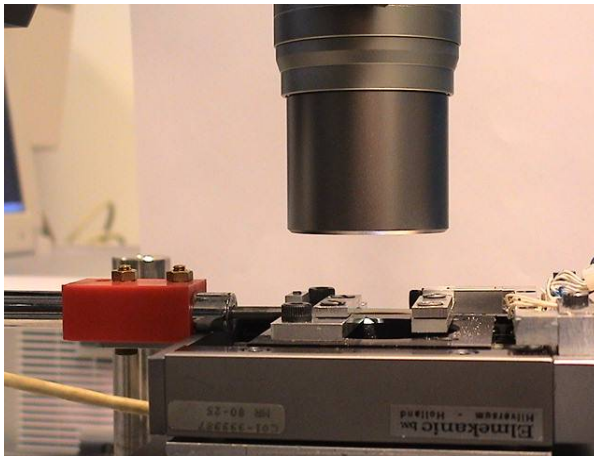
### 3.4.4. Analysis conclusions

Three types of the interconnects, meander-shaped, horseshoe shaped and meshed shaped, were designed with various parameter sets. Simulations for the basic parts were performed to evaluate the influence of the geometric parameters on the flexibility and stretchability. The free-standing interconnect shapes and their geometric parameters have significant influence on the stretchability.

From the three types of interconnects being considered, the meander-shaped design appears to be most favourable. Because of the enormous flexibility of the meander-shaped interconnect design, the maximum mean strain of the substrate (PDMS embedding the copper interconnects) is limited by the maximum mean strain that other parts can withstand. The maximum mean strain of the rubber sheets is limited to about 176 %, or less. Therefore the behaviour of the interconnect up to about 176 % mean strain is considered as quite interesting. For the most favourable case ( $W=5\text{ }\mu\text{m}$ ,  $R=100\text{ }\mu\text{m}$  and  $\alpha=30^\circ$ ), the established equivalent strain for this mean elongation is found to be  $6.439 \times 10^{-3}$  only. This is below the elastic strain limit of the copper. As a result, the meander interconnect ( $W=5\text{ }\mu\text{m}$ ,  $R=100\text{ }\mu\text{m}$  and  $\alpha=30^\circ$ ) will behave fully elastic and thus will not be damaged, even not under cyclic elongation.

### 3.5. Mechanical characterization and test set-up used

Mechanical testing of stretchable samples is done using a tensile testing setup, custom designed and built for easy visual inspection of the sample during tensile testing. The setup has small form factor and samples can be easily clamped.



**Figure 3-16.** Photograph of the testing setup and microscope lens.

The tensile test setup includes clumper, loading part, force sensor, displacement sensor and data recorder. Force and displacement signals are

converted by the force sensor and displacement gauge, amplified by the HBM KWS amplifier, changed into digital signal by A/D converter and finally recorded by a LabView virtual interface. The software can record the data when a key is pressed (manual mode, allowing investigation of samples with the microscope) or in automated mode (one measurement each second).

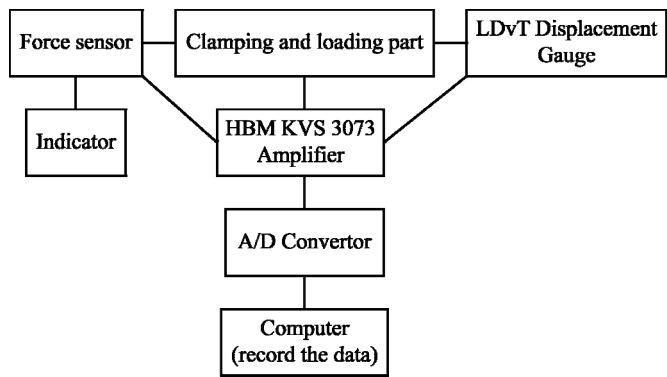


Figure 3-17. Diagram showing tensile setup components.

Clamping of the stretchable samples is done using two aluminum plates attached to the tensile setup with screws and the displacement is controlled by a micro screw attached to the clammer. A diagram of the tensile setup clamping parts is shown in Figure 3-18.

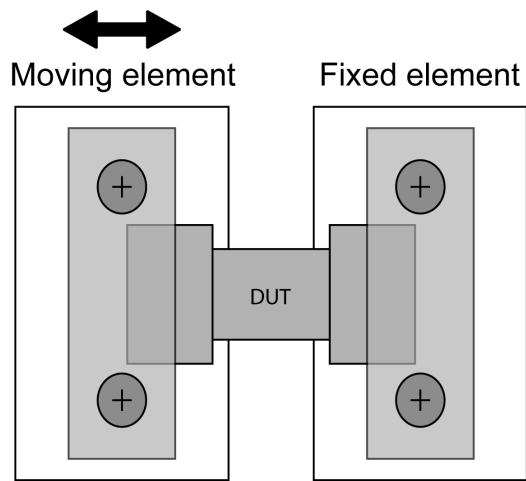
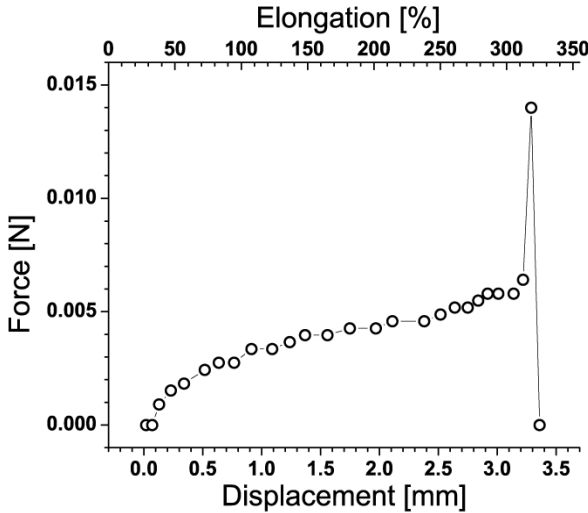


Figure 3-18. Tensile setup sample clamping diagram.

After the raw data file is processed, the measurement data is displayed as a graph showing displacement versus tensile load. Figure 3-19 shows the typical response of a rectangular copper mesh interconnect (elongation, strain vs. force).



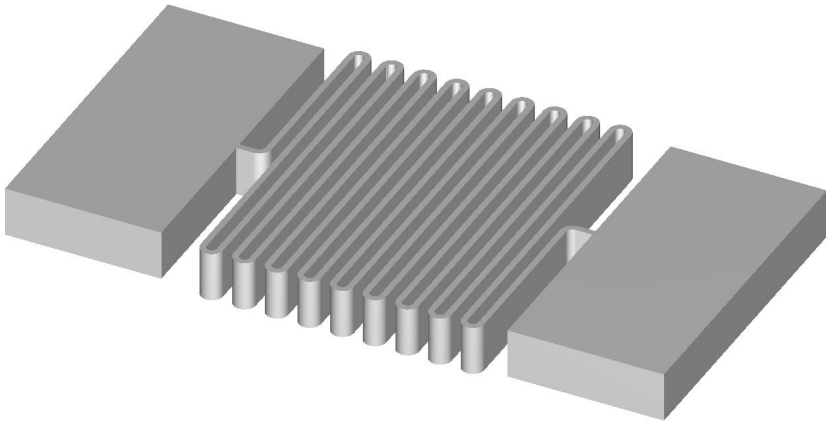
**Figure 3-19.** Typical elongation vs. force response of a free-standing copper rectangular mesh.

The fragility of the silicon springs and free-standing metal interconnects made handling and clamping difficult and many samples were destroyed before getting tested. While clamp slipping was avoided by using fine sand paper (between sample and clamps) and high strength adhesive tape, very precise alignment of the sample with the elongation axis was difficult, resulting in small variations of the maximum elongation due to diagonal pulling and higher stress concentration on anchoring points.

To get more accurate data, measurements on a large set of samples and Weibull statistics would be required to come to more reliable results.

**3.6. High-aspect-ratio silicon springs**

Although silicon is a brittle material and fractures easily when a load is applied, it can become flexible if thinned, either as a horizontal membrane or as narrow vertical walls. High-Aspect-Ratio-Silicon (HARS) has already been used for devices such as micro grippers, various actuators, accelerometers and oscillators. Considering the segmented microsystem approach, silicon islands can be linked using silicon springs. Silicon islands and spring like narrow vertical silicon walls can be easily fabricated by using deep reactive ion etching (DRIE). Figure 3-20 illustrates this concept. Depending on fabrication capabilities, multiple metal lines can be fabricated on the silicon meanders, providing multi wire connections between islands.

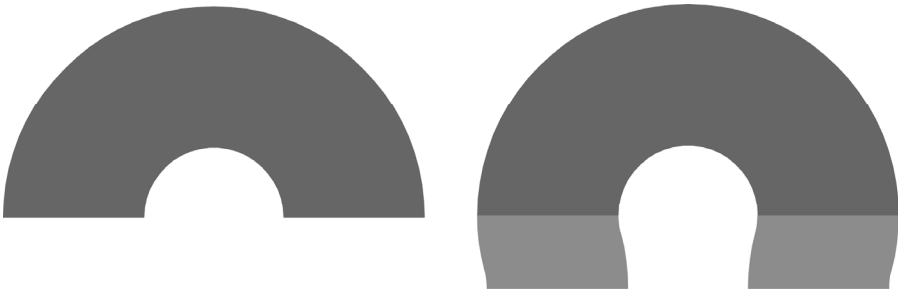


**Figure 3-20.** 3D rendering of HARS thin silicon flexible meander connecting two islands.

**3.6.1. Design and optimization**

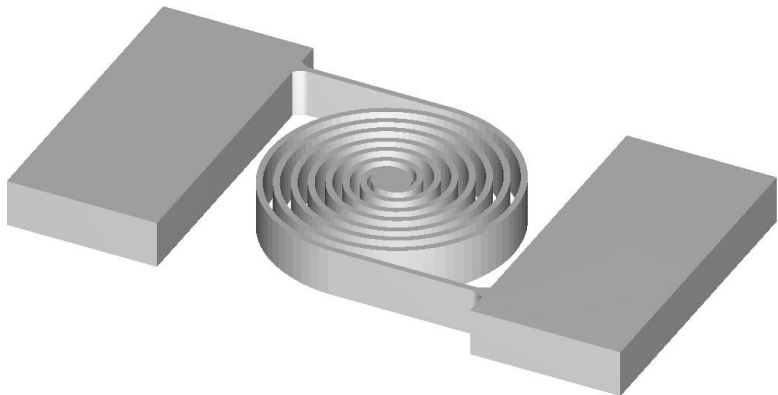
HARS springs were designed and fabricated, using two designs: meander and spiral. The meander design was presented before (Figure 3-3). It consists of narrow beam connected with half circle segments. The strains are accommodated by in plane rotations, similar to the movement of a scissors. The shape of the round connectors has significant effect on the maximum elongation a meander interconnect can sustain (Figure 3-21).

Two different designs have been tested, depending on the angle of the additional curved segment, one with a 180° connector and another using a 15° compensating curvature.



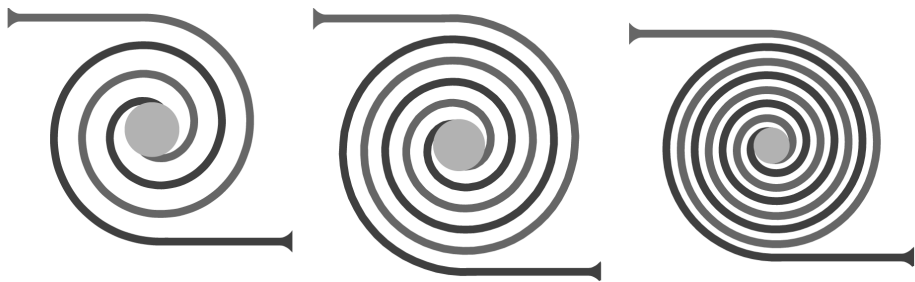
**Figure 3-21.** Layout capture of 180° connector (left) and additional 15° curvature (right).

The second design consists of two opposite spirals connected in the centre (Figure 3-22).



**Figure 3-22.** Illustration of thin silicon flexible spiral connecting two islands.

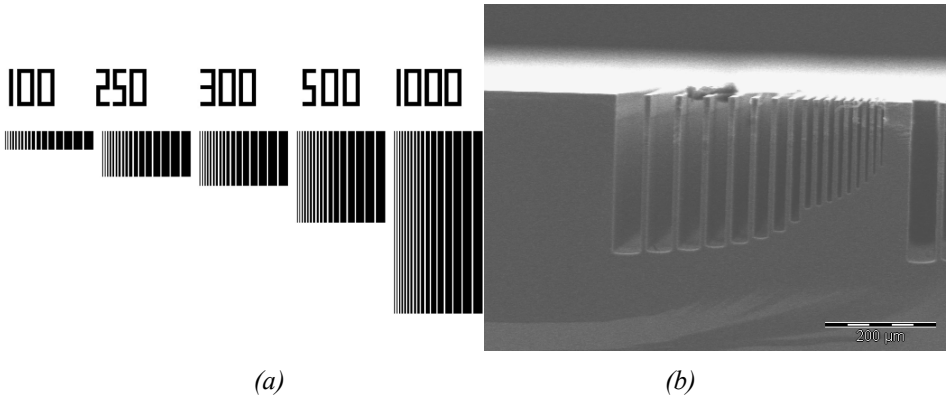
Three spiral designs using 2, 3 and 4 turns were investigated (Figure 3-23). When a load is applied to one of the spiral ends, the spirals unfold and the centre rotates. There are two approaches for fabricating stretchable systems using spirals. One way is to connect silicon islands using spirals. A second approach is to fabricate the island in the centre of the spiral and use the spiral arms to create the interconnect network between centers.



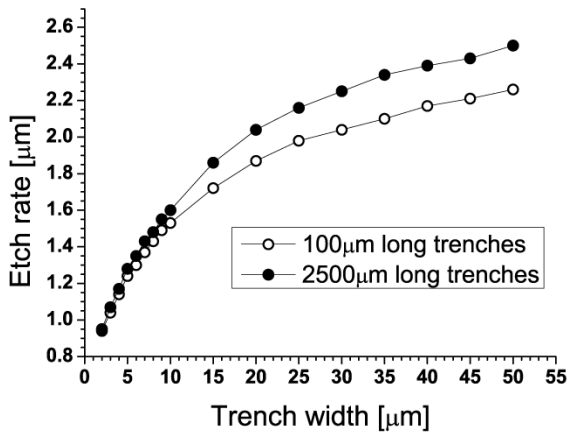
**Figure 3-23.** Spiral designs having 3, 5 and 6 turns.

3.6.2. Fabrication

Silicon DRIE shows etch rate dependency of the etch trench width. This dependency can be easily seen by etching a set of trenches with increasing width. For this, a set of trenches with constant length and increasing width from 2  $\mu\text{m}$  up to 50  $\mu\text{m}$  was used.



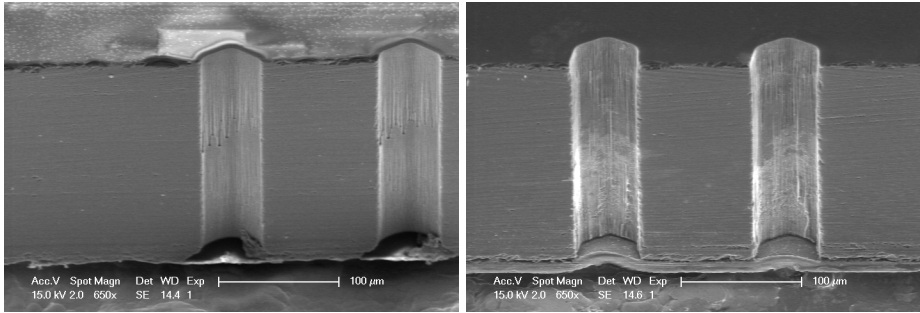
**Figure 3-24.** Test structures with increasing trench width for etch rate measurements: (a) layout; (b) SEM cross-sectional photograph showing significant dependency of the DRIE plasma etch rate on the etched trench width.



**Figure 3-25.** Dependency of the DRIE plasma etch rate on the etched trench width for 100  $\mu\text{m}$  and 2500  $\mu\text{m}$  long trenches.

The etch stop layer has also a significant role on the shape of the bottom of the etched trench. If trenches with large variation in aspect ratio are used, then larger trenches are etched faster and the etch stop layer is exposed until the narrower trenches are etched. As etch stop layer, silicon dioxide has lower selectivity than aluminum and for long etch times it can be etched through, leading to loss of vacuum and helium leaking into the etch chamber. The difference between silicon dioxide and aluminum used as etch stop layers can

be observed in Figure 3-26. While the trenches etched with a silicon dioxide etch stop layer have significant notching at the bottom, the ones etched with an aluminum etch stop layer have straight walls.

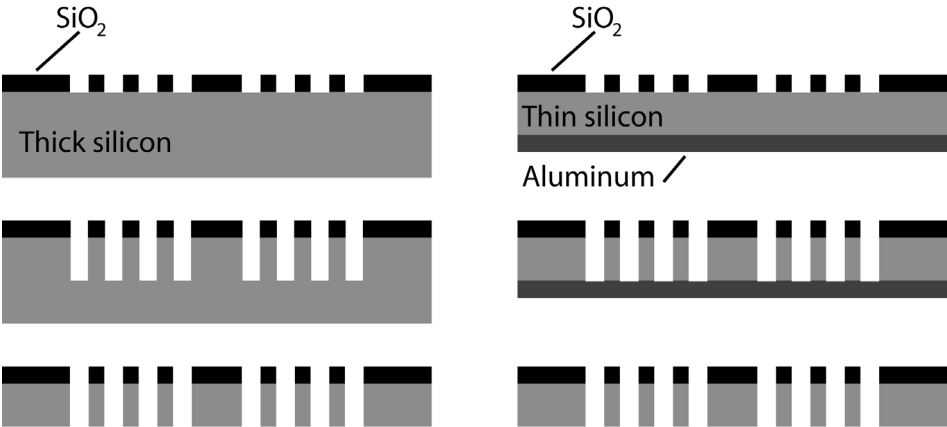


**Figure 3-26.** SEM photograph showing notching effect on the bottom of the trench for  $\text{SiO}_2$  etch stop layer (left); and trench with Al etch stop layer (right).

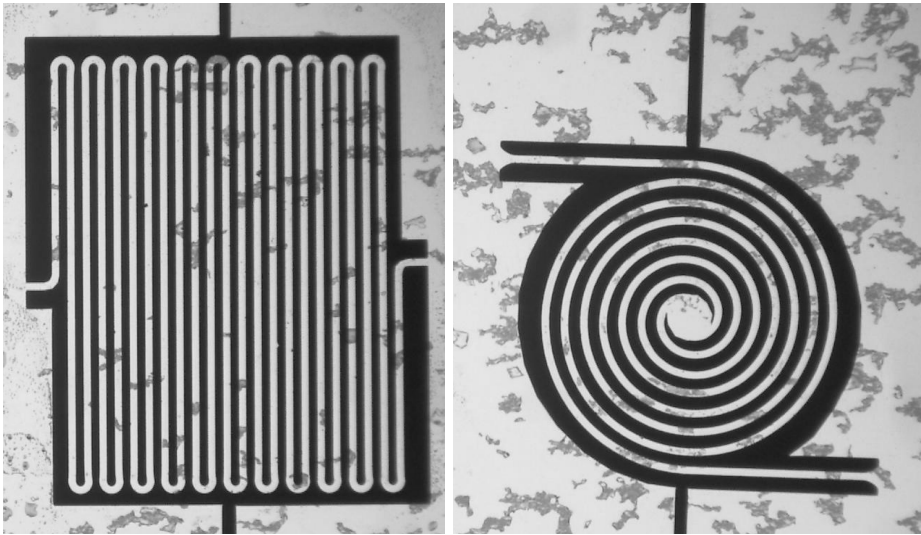
The silicon springs have been fabricated using two processes. The first fabrication process starts with 525  $\mu\text{m}$  thick 4" silicon wafers. On the wafer front side (FS) a thick  $\text{SiO}_2$  (6  $\mu\text{m}$ ) etch masking layer is deposited and patterned. After mask opening, the springs are etched to a depth of approximately 300  $\mu\text{m}$ , deeper than the desired thickness of 250  $\mu\text{m}$ . In the following step lapping and polishing of the wafer from the back side (BS) down to the required thickness of 250  $\mu\text{m}$  is performed. Before individual springs are released by fracturing the narrow beams that keep them attached to the wafer, the wafer is thoroughly cleaned to remove any debris left from lapping and polishing. The process is illustrated using in Figure 3-27 on the left side. This process has several major disadvantages as the lapping and polishing processes are extremely stressful for the wafers and uniformity is rather poor, resulting in springs with different thicknesses.

For these reasons a new process has been developed. It starts with already thinned wafers that have much better uniformity than the ones thinned using lapping. On the wafer front side, the same thick  $\text{SiO}_2$  (6  $\mu\text{m}$ ) etch masking layer is deposited and patterned on the wafer FS while on the BS an aluminum etch stop layer (4  $\mu\text{m}$ ) is sputtered. After the wafer is etched through, the aluminum layer is removed and individual springs are released from the wafer. The process is presented in Figure 3-27 on the right side.

STRETCHABLE INTERCONNECT SCHEMES



**Figure 3-27.** Cross sections of the etch first, thin last process (left) and thin first, etch last process (right)



**Figure 3-28.** Fabricated silicon meander spring (left), and a 4-turn spiral (right).

Meander springs and spiral springs with 20  $\mu\text{m}$  wide beams and 250  $\mu\text{m}$  high were fabricated from double side polished 250  $\mu\text{m}$  thick silicon wafers. Each silicon spring was fabricated attached to two large silicon blocks (2x3  $\text{mm}^2$ ) for easier handling and clamping in the tensile-load setup.



3.6.3. Characterization results

Tensile tests were performed using the automated measurement mode (one measurement per second) while the micrometer screw that provided the deformation was manually actuated. Clamping of the silicon spring samples was done using two aluminum plates attached to the tensile setup with screws. A diagram of the tensile setup clamping parts is shown in Figure 3-29.

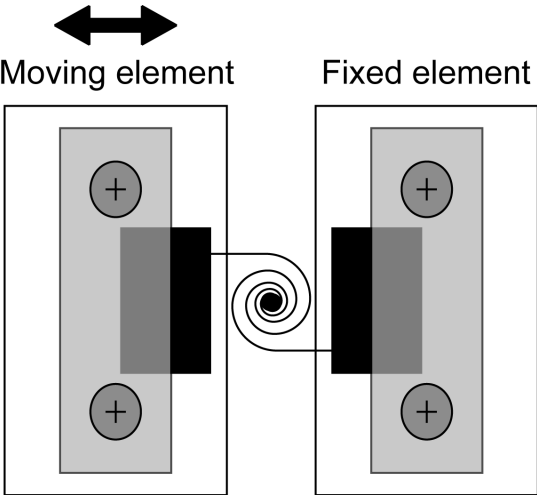


Figure 3-29. Tensile setup sample clamping diagram.

For the basic meander design, the maximum elongation that was achieved was 75 %.

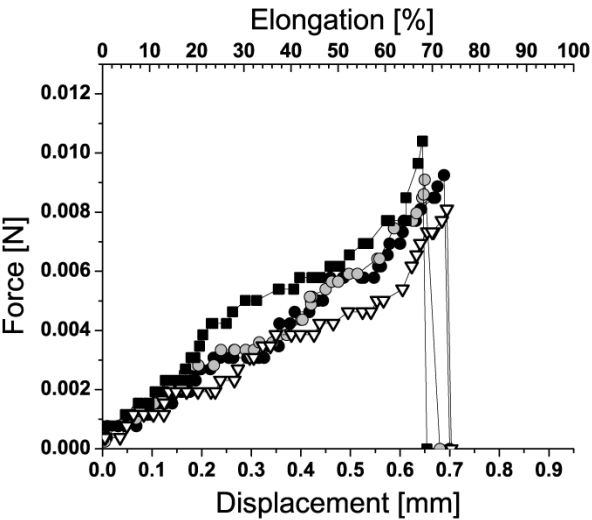
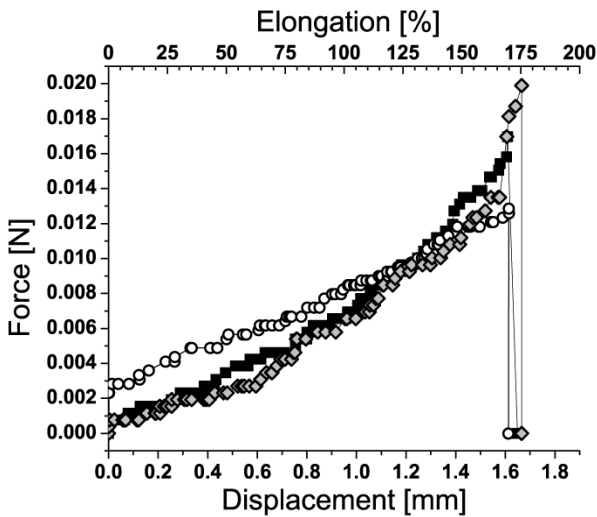


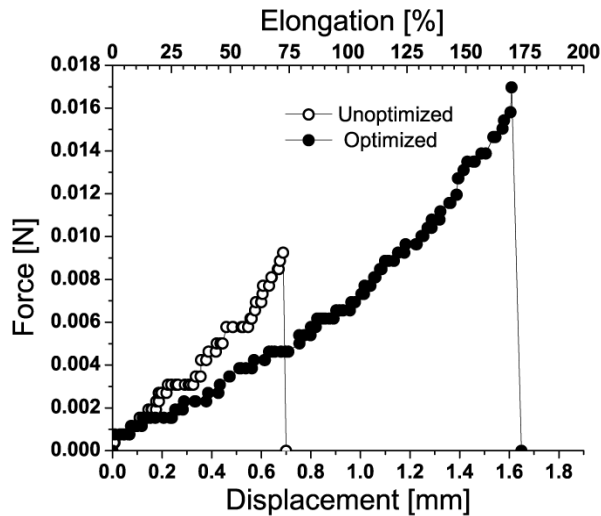
Figure 3-30. Measurement of silicon meander springs, unoptimized design.

In the case of the optimized design, the maximum elongation is 175 %, approximately 100 % more than for the initial design. Although the optimized design has 21 beams (compared to 17 of the basic design), the optimized design clearly performs much better than the basic one. The large difference in maximum elongation (Figure 3-32) is caused mostly by the 15° curved segment that delays the formation of stress concentration points. The ability of the optimized design to accommodate larger strains can be seen in Figure 3-33 and even if the optimized design has more beams, one pair of beams can deform more in the case of the optimized version.

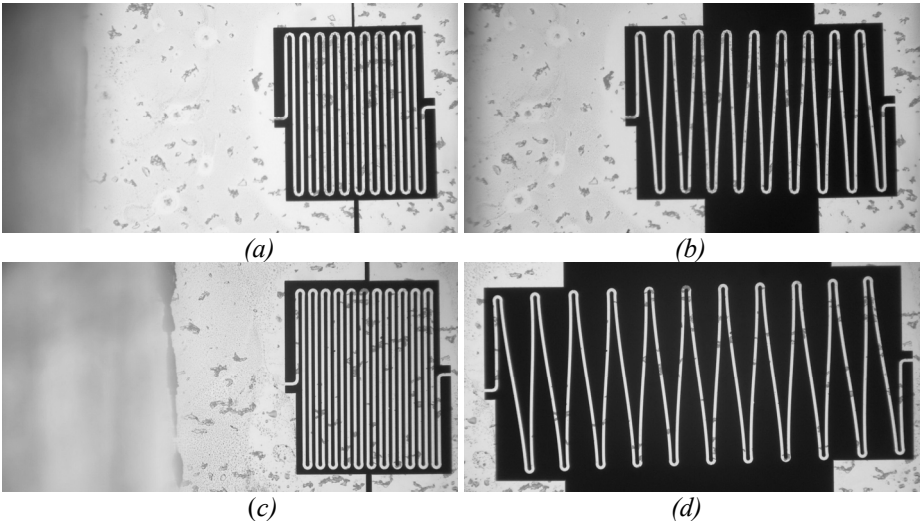
All the deformation that is supported by the springs is reversible as silicon is a brittle material and it shows little or no plastic deformation. For ductile materials, the influence of the curved transition might be more difficult to determine as cracks act as a stress relieve mechanism.



*Figure 3-31. Measurement of silicon meander springs, optimized design.*



*Figure 3-32. Comparison between unoptimized and optimized silicon meander spring designs.*



**Figure 3-33.** Pictures taken during testing, showing large difference in maximum elongation between basic design (a, b); and the optimized design (c, d).

The spiral designs did not achieve large deformations. This is because the curvature radius close to the centre is small and the silicon beams cannot unfold to straight segments due to their thickness and small curvature radius [3.7, 8].

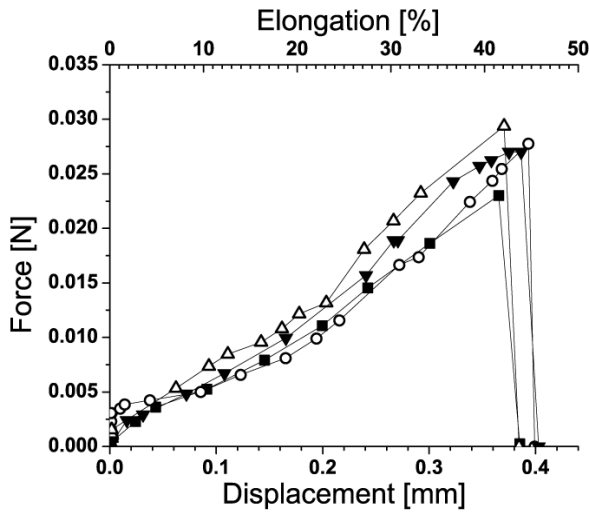


Figure 3-34. Tensile testing results for the 3-turn spiral design.

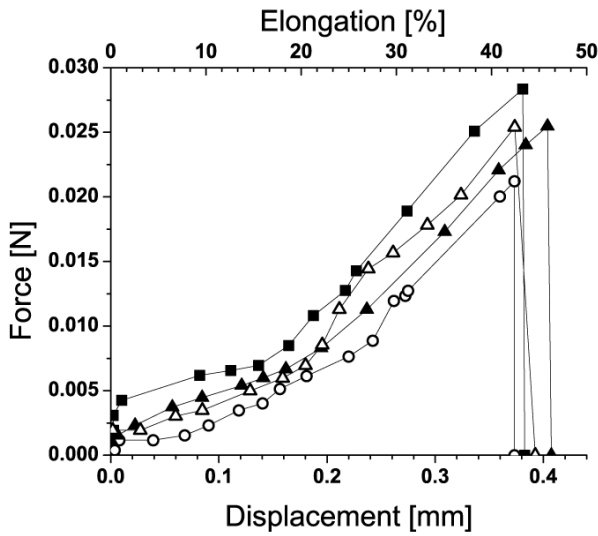


Figure 3-35. Tensile testing results for the 5-turn spiral design.

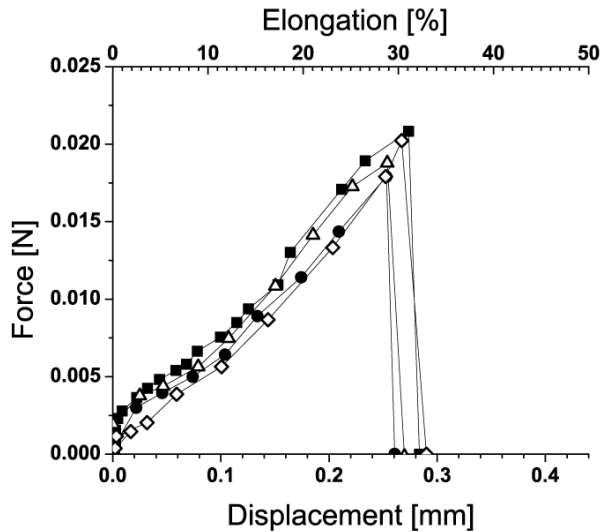


Figure 3-36. Tensile testing results for the 6-turn spiral design.

3.7. Free-standing metal interconnects

Free-standing metal interconnects are an alternative to conductive paths supported by HARS springs. Although silicon can be turned into flexible beams, it is a brittle material and cracks propagate extremely fast compared to metals where plastic deformation can delay crack propagation. Mesh patterns that were unsuitable for silicon springs and patterns developed for HARS springs are used to add stretchability to metal interconnects.

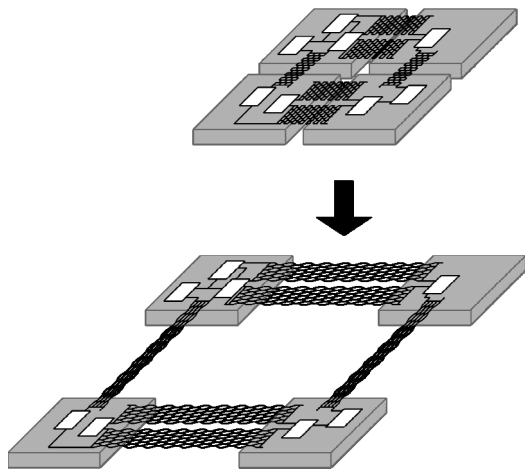


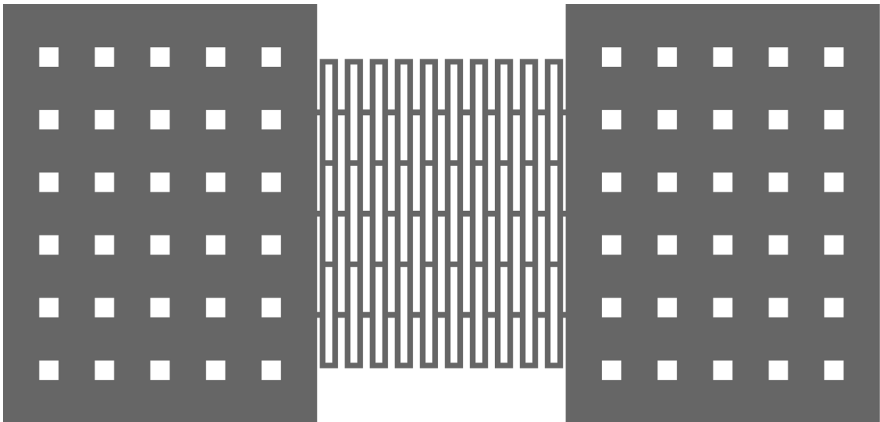
Figure 3-37. Concept drawing of a stretchable microsystem. An array of closely fabricated functional nodes in silicon islands are stretched out using mesh interconnects.

Fabrication of free-standing interconnects allows to fully understand how such interconnects behave, before integrating the interconnect scheme in a stretchable array (Figure 3-37). Such an array is made of functional silicon islands linked with a network of stretchable metal interconnects.

3.7.1. *Geometry selection*

While for HARS springs the height of a stretchable element was around 250  $\mu\text{m}$ , for metal interconnects the height is of 5  $\mu\text{m}$  maximum, depending on the metal and deposition process.

Interconnects designs used are mesh, meander and horseshoe. The basic mesh design generates a rectangular array of base element, each with the same set of ( $W, D, L, d$ ) parameters as presented earlier. Figure 3-38 shows the layout for a mesh interconnect, having large rectangular clamping parts, needed for tensile measurements and handling. Three variations of the mesh design were selected and their parameters are presented in Table 3-3.



**Figure 3-38.** Illustration of mesh interconnect showing large clamping parts and center mesh structure.

The length of an unstretched interconnect is around 1 mm, with small variations from one type to the other, while the clamping parts are 1.5 mm wide and 2 mm long.

**Table 3-3.** Selected mesh parameters.

<i>Parameter (<math>\mu\text{m}</math>)</i>	<i>W</i>	<i>D</i>	<i>L</i>
<i>Mesh 1</i>	10	20	300
<i>Mesh 2</i>	20	40	300
<i>Mesh 3</i>	20	40	500

Meander and horseshoe designs were also selected and fabricated but these designs are much more fragile than the mesh design and handling such free-

standing samples proved extremely difficult and in most of the cases the samples were too damaged to be tested. Analysis of free-standing meander and horse shoe designs will be performed only using finite element simulations.

### 3.7.2. Fabrication

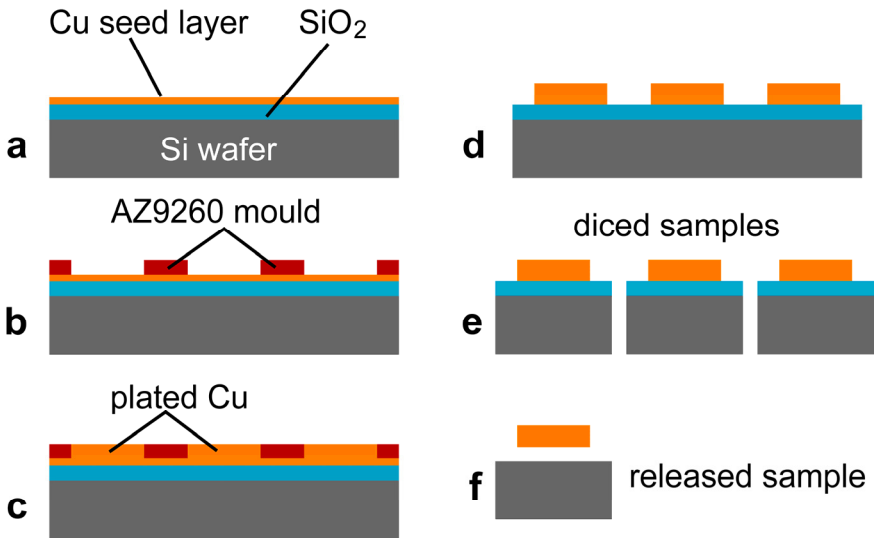
The samples used for tensile testing are fabricated using copper plating in a resist mould (Figure 3-39). Copper seed layer (10 nm Ti / 50 nm TiN / 300 nm Cu) is sputtered on top of a 4  $\mu\text{m}$  thick PECVD silicon oxide used as sacrificial layer, allowing later release of individual samples (Figure 3-39a).

For the resist mould we have used a 5-6  $\mu\text{m}$  thick layer of AZ4562, post-baked for 30 minutes at 110  $^{\circ}\text{C}$  (Figure 3-39b). An oxygen plasma flash treatment is performed to ensure that the resist surface is hydrophilic and the plating solution will come in contact with the seed layer. A DC plating of copper for 10 minutes creates 5  $\mu\text{m}$  thick structures with low surface roughness (Figure 3-39c). After the plating is finished, the resist mould is removed with acetone. The copper seed layer is removed in two steps. First, the copper layer is etched away in solution made of 2.5 g  $\text{Na}_2\text{S}_2\text{O}_8$  + 0.65 ml  $\text{H}_2\text{SO}_4$  (98 %) + 250 ml  $\text{H}_2\text{O}$ . For the Ti/TiN layers, the following solution is used: 25 ml  $\text{NH}_4\text{OH}$  (28 %) + 100 ml  $\text{H}_2\text{O}_2$  (31 %) + 100 ml  $\text{H}_2\text{O}$  (Figure 3-39d). Large rectangular clamping parts needed for handling have 100  $\mu\text{m}$  square holes to minimize sacrificial etch time.

Before the sacrificial layer is removed, the wafer is diced into dies containing one interconnect. To protect the interconnects during the dicing step, the wafer is spin coated with a layer of AZ9260. After dicing (Figure 3-39e), the interconnect dies are cleaned individually with acetone and the sacrificial layer is removed using HF (40 %). The large clamping parts make the handling of each interconnect easy during release and measurements.

*Table 3-4. AZ 9260 recipe used for copper plating mould.*

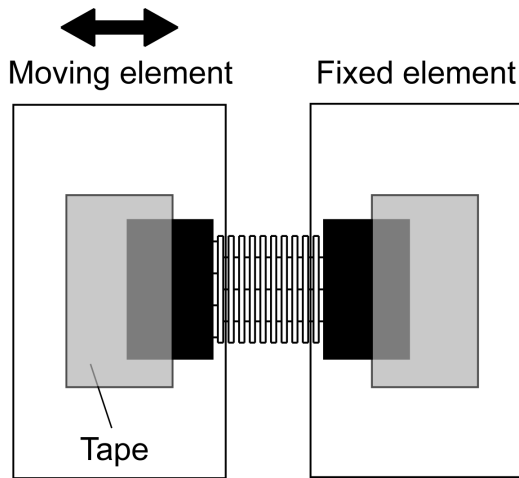
<i>Step</i>	<i>Parameter</i>	<i>Time [min]</i>
<i>Spin coating</i>	5000 rpm	0.5
<i>Soft bake</i>	110 $^{\circ}\text{C}$	5
<i>Exposure</i>	soft contact	1
<i>Development</i>	AZ 400K, 1:4 diluted	5
<i>Post exposure bake</i>	110 $^{\circ}\text{C}$	30
<i>O<sub>2</sub> plasma</i>	200 W	1



*Figure 3-39. Free-standing interconnects fabrication process sequence.*

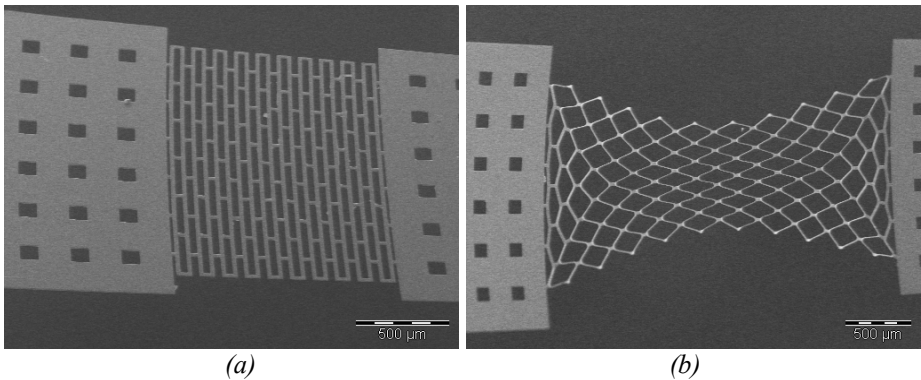
### 3.7.3. Characterization results

Individual samples were attached to the measurement setup using high-strength transparent adhesive tape. The tape is strong enough to prevent any slipping of the part during stretching and it is also easy to apply minimizing damage to interconnects before testing. Measurements were performed by applying deformation by using a micrometer screw, actual displacement being measured by a Linear Variable Differential Transformer (LVDT) displacement gauge. The reaction of interconnects to the applied displacement is recorded by a force sensor. Measured values for displacement and force are recorded using a LabView virtual interface and stored in a text file.



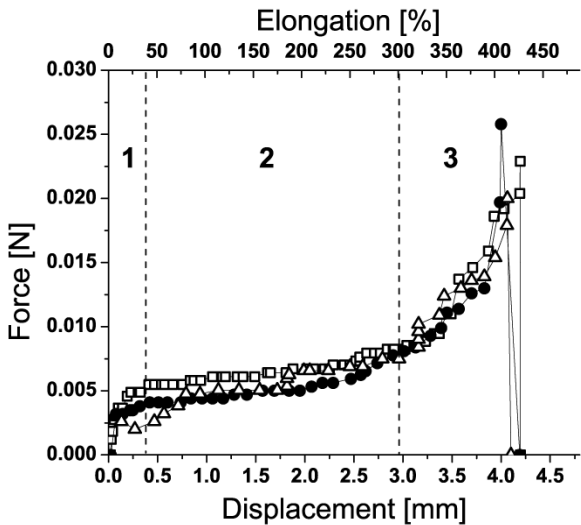
*Figure 3-40. Tensile setup clamping diagram for free-standing mesh testing.*





**Figure 3-41.** SEM picture of mesh interconnect including large clamping square parts used for tensile measurements (left and right of the picture, also showing etch holes) (a) before testing; and (b) partially stretched.

Figure 3-42 shows the measurement data from three identical mesh interconnects. The three areas illustrated on the graph are used to describe the sample behavior under tensile stress. In the first area (marked “1” on the graph), the beams start moving from vertical position in order to form diamond like shapes as can be seen in Figure 3-41. In the next part of the graph (marked “2”), the diamonds change their aspect ratio becoming more elongated. Elongation in the first two areas is a combination of plastic and elastic deformation, elastic behavior being more pronounced at in the first zone. Finally the beams form very narrow diamond patterns and fracture initiates, the sample showing only plastic deformation (marked “3”).



**Figure 3-42.** Elongation for a set of three identical mesh arrays  
( $W, D, L$ ) = (10, 20, 300)  $\mu\text{m}$ .

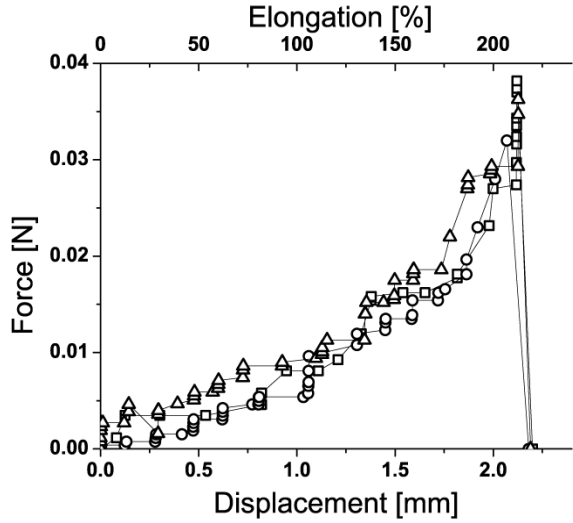


Figure 3-43. Elongation for a set of three identical mesh arrays  $(W, D, L) = (20, 40, 300) \mu\text{m}$ .

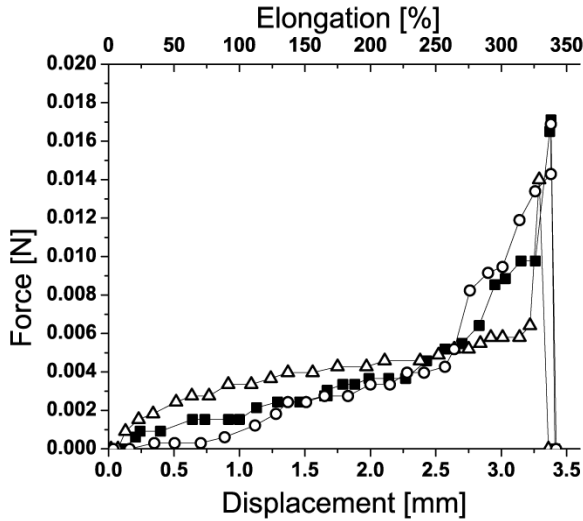
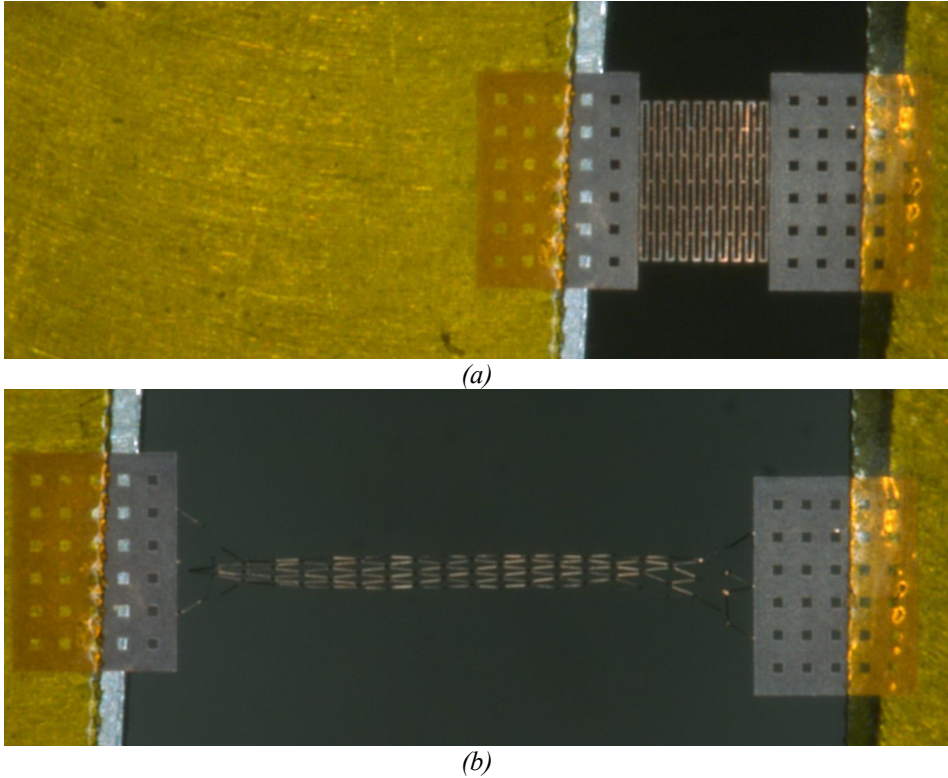


Figure 3-44. Elongation for a set of three identical mesh arrays  $(W, D, L) = (20, 40, 500) \mu\text{m}$ .

During the tensile tests the samples can deform up to 400 % of the initial length of 1.2 mm, but this deformation is permanent, the elastic interval being much smaller (Figure 3-42). Although such a mesh can be stretched up to 400 % before it is fully broken, fracture of the beams starts before reaching this limit but because of multiple beam design the fracture of one beam does not destroy the whole structure.



**Figure 3-45.** Optical microscope picture showing tape clamping and elongation behavior the mesh geometry, (a) before stretching; and (b) after fracture.

#### 3.7.4. Parylene coating for strength modification

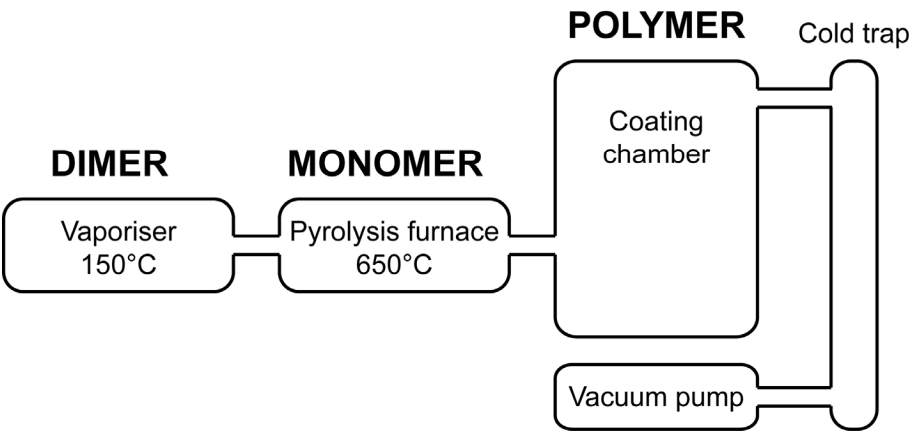
The strength of the interconnects can be modified by coating them by a polymer layer. Polymers can add mechanical strength but also protect the interconnects from the environment.

Parylene was chosen for its mechanical and chemical properties. The change in strength of copper meshes coated with Parylene N was investigated in two experiments, having different Parylene thicknesses: 8  $\mu\text{m}$  and 5  $\mu\text{m}$ .

The deposition of Parylene is done at room temperature, the result being a conformal pinhole-free deposition. Parylene deposition starts with creation of a dimeric gas by vaporization of dimeric powder at around 150  $^{\circ}\text{C}$ . In the next step gas molecules go in a pyrolysis furnace where are cleaved into monomer form at 650  $^{\circ}\text{C}$ . The monomer gas is delivered to the deposition chamber where it polymerizes spontaneously on substrate surfaces at room temperature to form the Parylene film (Figure 3-46).

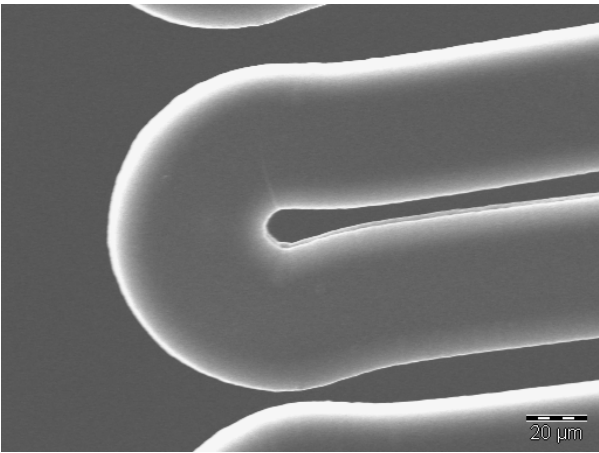
Because of the room temperature deposition the film does not induce any mechanical stresses on fragile components due to differential coefficients of expansion/contraction of materials. Parylene films have a relatively low dielectric constant, provide an inert barrier against moisture, chemicals, bio-

fluids and bio-gases and non-absorption of visible light makes them suitable for optical and medical uses. Pinhole free layers can be achieved with layers of 0.2  $\mu\text{m}$  thickness. Conformal deposition is shown using SEM imaging (Figure 3-47).

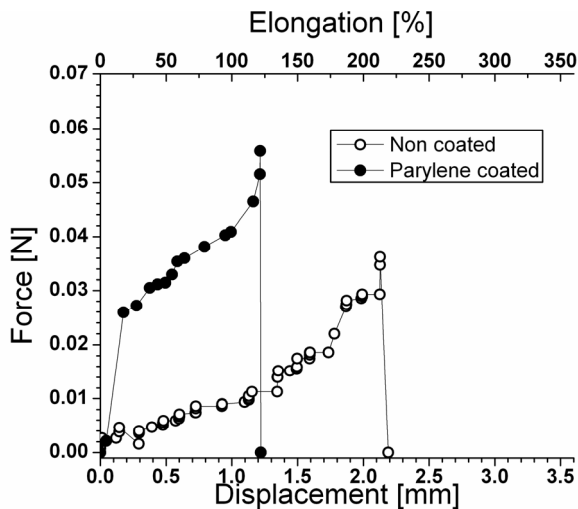


**Figure 3-46.** Schematic of Parylene deposition process.

In the first Parylene coating experiment, the Parylene thickness was about 8-9  $\mu\text{m}$ . A comparison between uncoated and coated mesh interconnect length of 1.22 mm is shown in Figure 3-48. For the  $(W, D, L) = (20, 40, 300)$   $\mu\text{m}$  mesh maximum elongation up to 300 % for the uncoated mesh and 150 % for the coated version can be achieved.

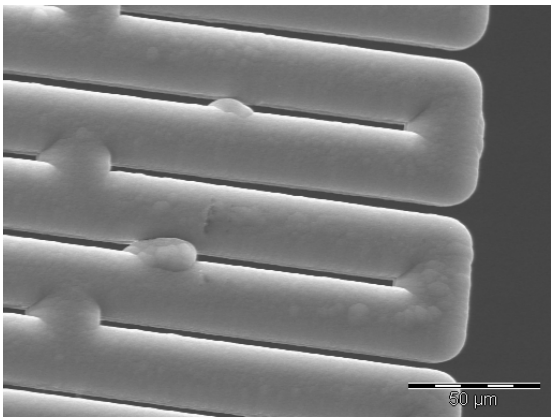


**Figure 3-47.** SEM picture of a meander coated with 8  $\mu\text{m}$  of Parylene N.



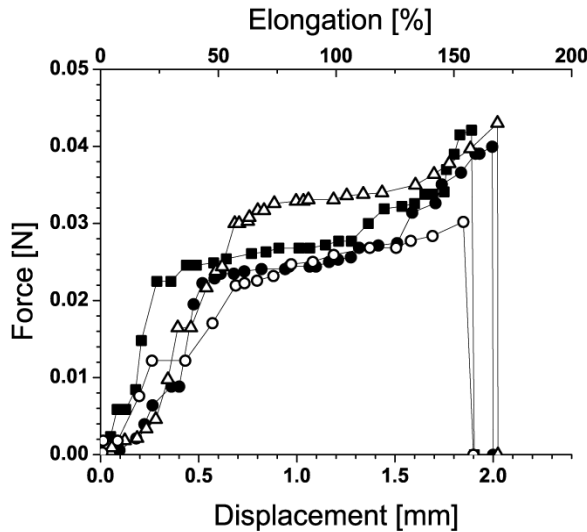
**Figure 3-48.** Results of tensile testing of  $(W,D,L)=(20, 40, 300)$   $\mu\text{m}$  mesh uncoated and coated with 8  $\mu\text{m}$  of Parylene N.

The Parylene coating increases the stiffness and the strength of a sample but reduces the maximum deformation that can be achieved by around 50 % for mesh interconnects. Parylene is elastic for deformations up to 10 % and an 8-9  $\mu\text{m}$  layer has significant influence of the 5  $\mu\text{m}$  thick interconnects.



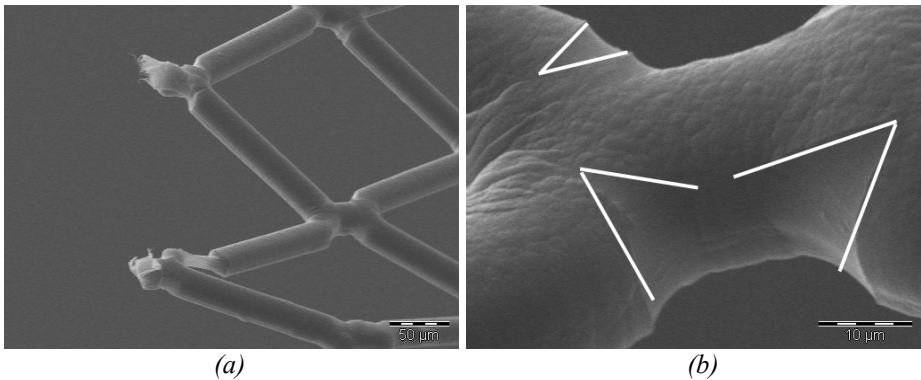
**Figure 3-49.** SEM image showing unwanted bridging due to particles on a sample coated with 8  $\mu\text{m}$  of Parylene N.

For the second experiment Parylene thickness was decreased to around 5  $\mu\text{m}$  in order to avoid bridging due to particles left on the samples (see Figure 3-49) but also to preserve the original large maximum elongation a interconnect can withstand.



**Figure 3-50.** Results of tensile testing of four identical  $(W, D, L) = (20, 40, 300) \mu\text{m}$  mesh coated with  $5 \mu\text{m}$  of Parylene N.

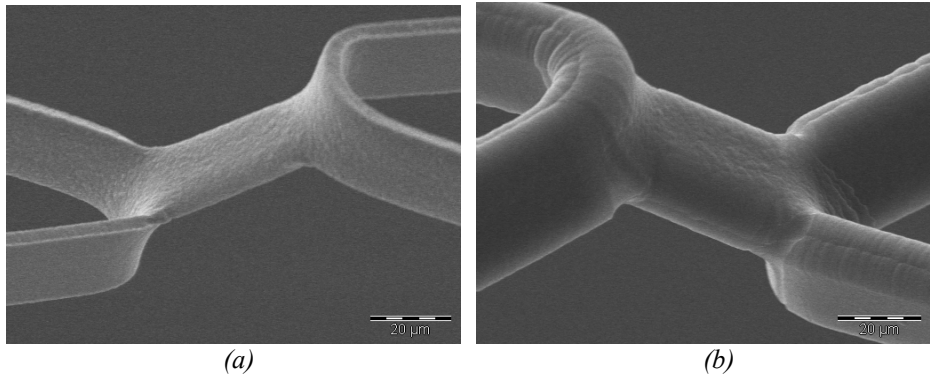
Tensile measurements showed similar force levels as the samples coated with a  $8 \mu\text{m}$  layer of Parylene but with much less reduced maximum elongation (see Figure 3-43 for the uncoated and Figure 3-48 and Figure 3-50 for the coated samples). Parylene coatings up to  $5 \mu\text{m}$  only make the interconnects stronger, larger forces being needed to get the same deformation as for uncoated samples but without reducing the maximum elongation when compared with the uncoated version.



**Figure 3-51.** Copper mesh showing fracture under Parylene coating layer: large view (a); and close up of fractures highlighted using white lines (b).

SEM investigation of stretched samples revealed different behavior of samples with wide metal tracks compared to sample with narrower metal lines. All samples had the same thickness ( $5 \mu\text{m}$ ) but the width of the metal lines was  $10 \mu\text{m}$  or  $20 \mu\text{m}$ . Interconnects having  $20 \mu\text{m}$  wide metal lines showed out of

plane deformation at nodes (Figure 3-52) while interconnects with only 10  $\mu\text{m}$  wide metal line deformed in plane (Figure 3-51).



**Figure 3-52.** SEM picture of uncoated sample (a) and sample coated with 5  $\mu\text{m}$  of Parylene N.

### 3.8. Conclusions

An overview of geometries used for stretchability was presented at the beginning of the chapter. Several variations of mesh, meander and horseshoe were investigated using finite element simulations and tensile testing of free-standing samples.

Using curved segments to control stress concentration points, HARS meanders can be optimized to have a maximum elongation of around 170 % (compared with 70 % for the unoptimized version). Although not conductive, HARS springs can support, depending on technology limitations and dimensions, several metal interconnects. Silicon springs can have very large reversible deformations but are more prone to failure as cracks propagate much faster in a brittle material.

Variations of free-standing copper interconnects were analyzed using finite element simulations. An overview of how geometry parameters influence maximum elongation was presented for mesh, meander and horse shoe geometries. Best performing geometries were meander ( $W=5\ \mu\text{m}$ ,  $R=100\ \mu\text{m}$ ,  $\alpha=30^\circ$ ), horseshoe ( $W=5\ \mu\text{m}$ ,  $R=100\ \mu\text{m}$ ,  $\alpha=45^\circ$ ) and mesh ( $L=500\ \mu\text{m}$ ,  $W=5\ \mu\text{m}$ ,  $D=5\ \mu\text{m}$ ). As free-standing interconnects they can reach elongations of up to 400 %. PDMS embedding is considered for protection of silicon electronics arrays and the maximum elongation such a system can withstand is limited by the PDMS to around 170 %. Interconnect variations with elongations much larger than PDMS can perform only in their elastic regime, increasing this way the life time of the stretchable system.

Before being integrated into stretchable systems, free-standing copper interconnects were fabricated and tested under a tensile load. Large deformations (close to 400 %) are achievable (although not reversible). Parylene coating of interconnects can be used to tune the strength and

elongation of interconnects and also to increase protection against chemicals or to isolate close metal tracks. For mesh geometries, anchor position and interconnect shape can greatly influence the maximum elongation an interconnect can sustain. Before being integrated into a stretchable system, further optimization of mesh geometry is needed to control the deformation during stretching.

By using Parylene, the strength and stretchability of interconnects can be easily tuned by changing the interconnect metal, the thickness of the metal or of the Parylene layer.



### 3.9. References

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# Chapter 4:

## *PDMS-Embedded Silicon Electronics Arrays*

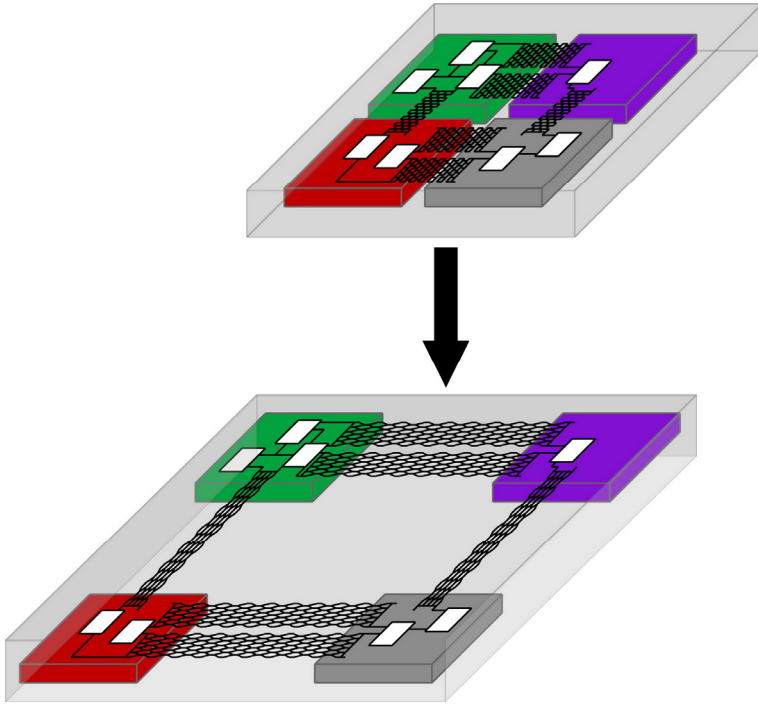
In this chapter all the development steps and optimization needed for integration of the individual elements (interconnects, protective polymers, silicon islands) into a complete stretchable electronic system are presented. The focus is on the resulting processing module for the PDMS-embedded stretchable silicon electronics arrays. In addition, the tensile-load testing measurement results are presented together with a brief investigation of the failure mechanisms and their causes.

### *4.1. PDMS-embedded electronics concept*

As previously discussed in Chapter 3, the use of polymers is essential for stretchable silicon system design. The concept of PDMS embedded electronics uses 1D and 2D silicon island arrays linked with a network of stretchable metal interconnects, all embedded in PDMS. An illustration of this concept is shown in Figure 4-1, using metal interconnects providing low electrical resistivity and PDMS for mechanical support and protection.

Although metal is not stretchable (except for very small strains), metal interconnects can behave like springs when patterned into certain shapes. Metal interconnects patterned for stretchability form the stretchable network linking all the systems nodes of the stretchable system.

As previously discussed, the use of polymers is essential for stretchable silicon system design. Polydimethylsiloxane (PDMS) is a soft and compliant polymer that can add extra mechanical, chemical and electrical protection to the system and to the interconnects which are the most sensitive part. PDMS is optically clear, and, in general, is considered to be inert, non-toxic and non-flammable. The resistance of PDMS to many clean room chemicals (acid or alkali solutions) makes it an important candidate as protection material for stretchable electronics. The concept of PDMS embedded electronics uses 1D and 2D silicon island arrays linked with a network of stretchable metal interconnects, all embedded in PDMS. An illustration of this concept is shown in Figure 4-1.



**Figure 4-1.** Concept of a stretchable system embedded in PDMS.

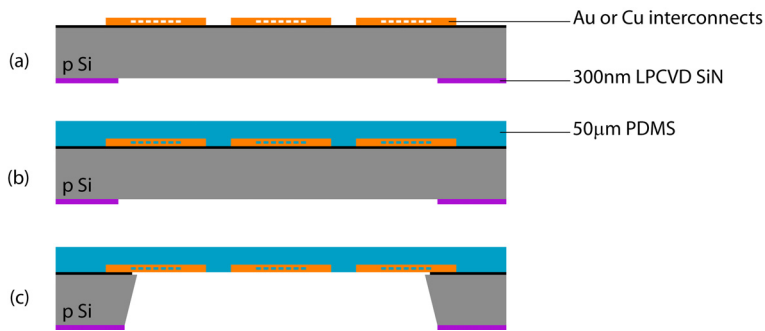
#### **4.1.1. Fabrication process development**

Starting from the results of the free-standing interconnects, a wafer-level process was developed for fabricating a stretchable silicon system embedded in a PDMS elastomer layer. The first step in developing the process was to test metal interconnects on a PDMS membrane. For this initial experiment, silicon islands were not included.

The process starts with 525  $\mu\text{m}$  thick silicon wafers with thermal oxide on the front side and 300 nm silicon nitride on the back side. A 5 nm thick Cr adhesion layer followed by a 250 nm or 500 nm thick Au layer are evaporated on the wafer front side.

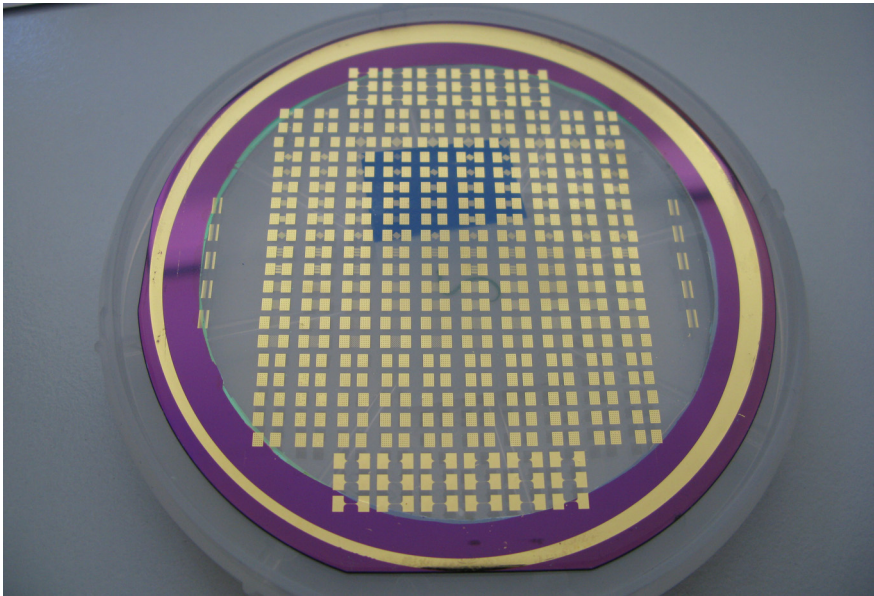
A variation of the process uses 5  $\mu\text{m}$  copper plated on a 10 nm Ti / 50 nm TiN / 300 nm Cu seed layer using a 6  $\mu\text{m}$  thick AZ 4562 resist mould.

After etching the gold layer in a mesh pattern, a layer of 50  $\mu\text{m}$  of PDMS is spin coated. The PDMS layer will later become the membrane supporting the stretchable interconnects. On the wafer backside, silicon nitride is patterned as a 10 mm wide ring along the edge of the wafer. This ring will act as a mask for the release of the membrane using KOH etching. The schematic fabrication sequence is shown in Figure 4-2.



**Figure 4-2.** Cross section for the fabrication process of gold interconnects on a PDMS membrane.

Using the process described in Figure 4-2, a 50 µm thick PDMS membrane with gold interconnects on one side (see Figure 4-3) was fabricated. Interconnects were tested with a multimeter and were conductive even after repeated stretching of the membrane.

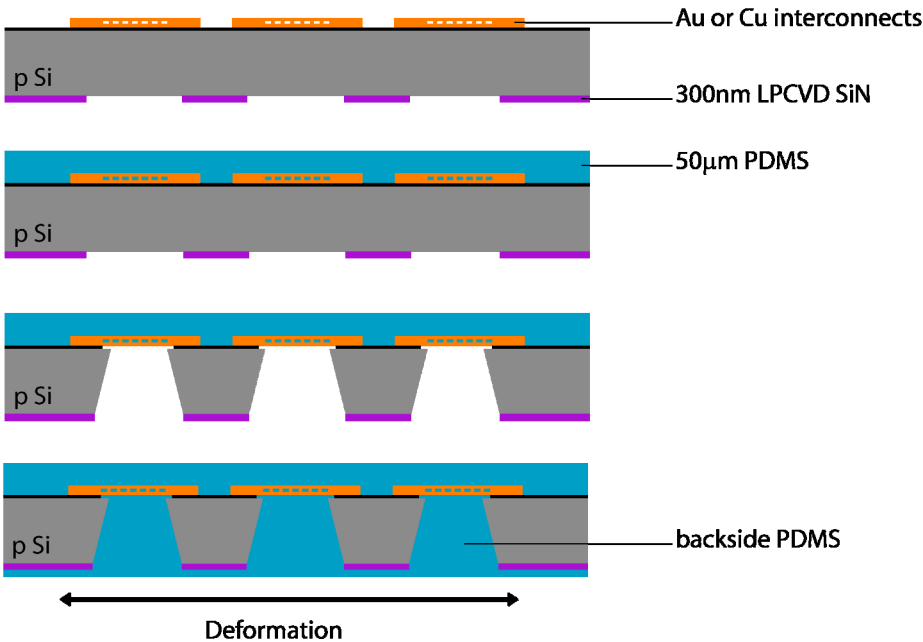


**Figure 4-3.** Fabricated membrane with Au interconnects.

After the first experiment, the fabrication process was modified to include 1D and 2D arrays of silicon islands with contact opening in the PDMS layer to allow electrical measurements during testing.

The process starts with 250 µm thin silicon wafers to reduce the KOH etching time. First, a KOH etch masking layer of LPCVD SiN is patterned on the backside of the wafers. After the backside silicon nitride mask is patterned, the stretchable interconnects are fabricated using evaporated gold or plated

copper. Gold (500 nm thick) is evaporated on the wafer front side and patterned to define the interconnects. For plated copper interconnects, a 5  $\mu\text{m}$  thick copper layer is plated on a 10 nm Ti / 50 nm TiN / 300 nm Cu seed layer using a 5.5-6  $\mu\text{m}$  thick AZ 9260 resist mould.

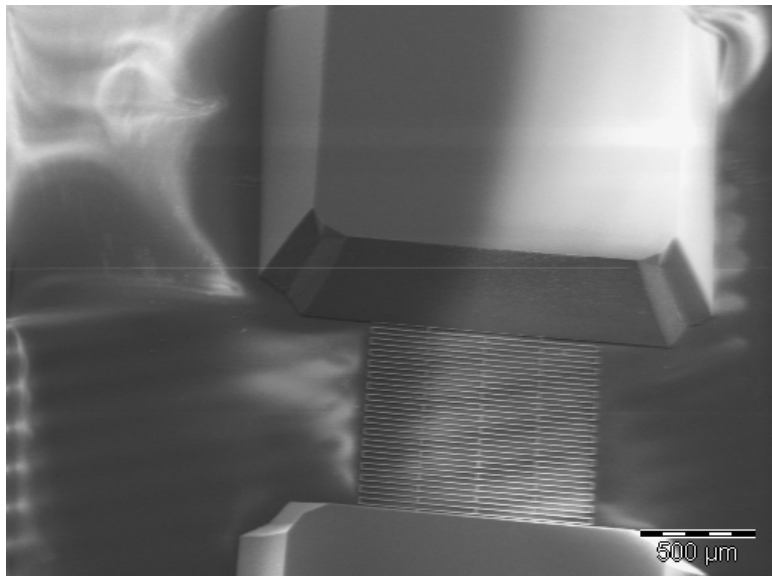


**Figure 4-4.** Fabrication process for silicon island arrays using stretchable interconnects embedded in PDMS.

Next, a 50  $\mu\text{m}$  thick AZ9260 photoresist layer is spin coated on the wafers. After coating, the resist is degassed in vacuum for 10 minutes to prevent de-wetting and agglomeration of the thick resist layer. When the sample was placed immediately after spin coating onto a hot-plate at a temperature of 110  $^{\circ}\text{C}$ , the thick photoresist tended to retract from the edges of the wafer and agglomerate unevenly toward the centre as the sudden escape of solvents from the polymer would cause the initiation of de-wetting. After soft bake and development, tall resist pillars are left on top of the measurement pads. No hard baking of the resist is done to preserve the pillar profile (Figure 4-4a). PDMS is then spin coated to produce a layer of around 50  $\mu\text{m}$ , comparable in thickness with the resist pillars. Vacuum degassing removes any air bubble trapped in the elastomer layer and curing is done at room temperature for 24 hours (Figure 4-4b). Through-wafer KOH etching is performed to separate the islands. For this step, a supporting wafer is used on the wafer front side to prevent the stretching or fracture of the PDMS layer towards the end of the etching process (Figure 4-4c).

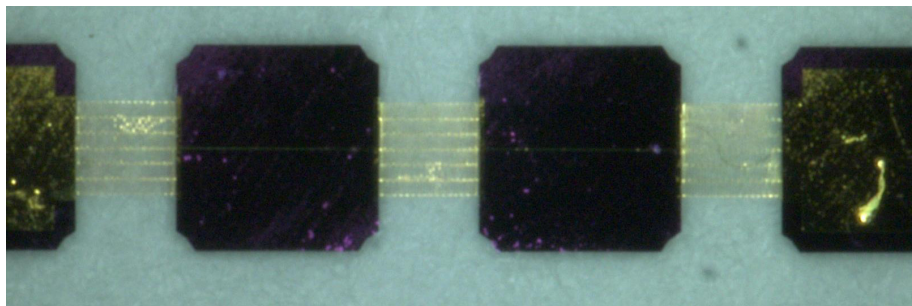
The second layer of PDMS is poured on the etched wafer back side and any air bubbles are removed using another degassing step (Figure 4-4d). The last step consists of PDMS etching in  $O_2/SF_6$  plasma to expose the photoresist pillars. After the pillars are exposed they are dissolved in acetone thus exposing the pads for electrical measurements (Figure 4-4e). In the last step, wire-bonding is performed to provide electrical contact to the bond pads.

Figure 4-5 shows a close-up of fabricated arrays on PDMS membrane, showing silicon islands, mesh interconnects and the silicone membrane, just before the deposition of the final silicone layer.



**Figure 4-5.** SEM image showing island and metal mesh interconnect on silicone membrane.

Using the updated process, silicon arrays consisting of a network of gold interconnects linking four islands of the array were fabricated. Figure 4-6 shows a fabricated four island array with mesh interconnects embedded in a transparent PDMS membrane.



**Figure 4-6.** Photographs of fabricated 4-segment array embedded in PDMS.

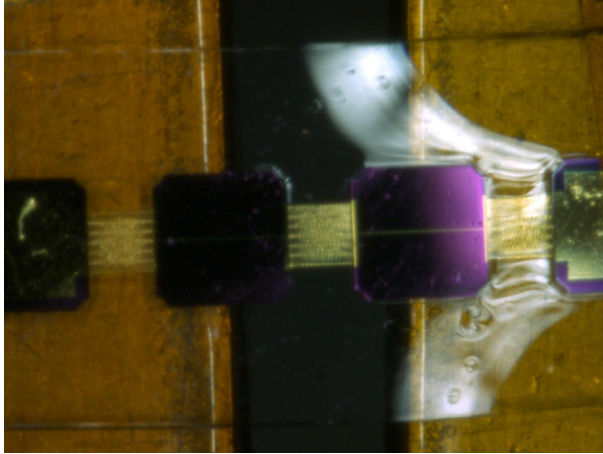
#### 4.1.2. Characterization results

Figure 4-7 shows the 4-island array embedded in silicone mounted in the tensile testing setup. The mesh interconnects and transparent PDMS can easily be seen in the picture.

Similar to the free-standing meshes, the hourglass shape can be observed during stretching but in this case deformation is limited by silicone encapsulation. Another cause of the hourglass deformation is the lateral compression of PDMS when longitudinal strain is applied. This contraction affects all interconnect design not only the mesh.

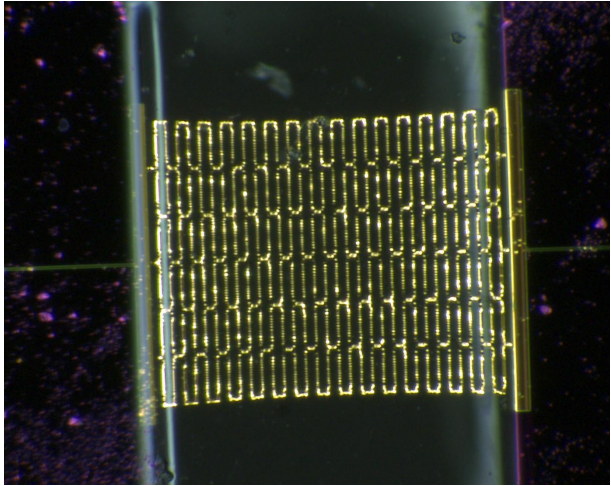
Tensile stress measurements were performed on arrays of four  $2 \times 2 \text{ mm}^2$  silicon islands connected using mesh interconnects embedded in Sylgard 184.

Visual inspection using high magnification optical microscope was performed during measurements to observe crack initiation. Cracks appeared at the transition points between island and mesh, similar with the free-standing mesh interconnects measurements. While no drop in force can be detected on the displacement vs. force graphs recorded during measurements, cracks in the gold layer were observed forming first at the rigid-stretchable transition between the rigid island and the stretchable PDMS. The formation of cracks was the failure criterion as fabricated arrays could not be measured electrically.



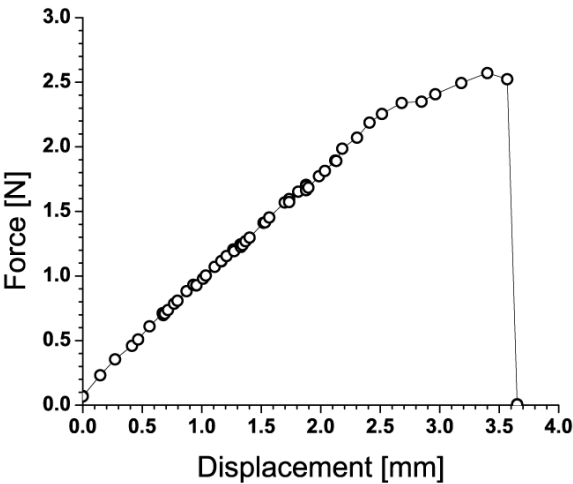
**Figure 4-7.** Four-island array mounted in the tensile-load measurement setup.





**Figure 4-8.** Mesh interconnect between two islands showing typical hourglass-like deformation under tensile stress.

Figure 4-9 and Figure 4-10 present the tensile test results for two variations of the mesh design,  $(W, L, D) = (10, 20, 300)$  and  $(10, 40, 300)$   $\mu\text{m}$ . As predicted by simulations and free-standing interconnect tests, an increase in maximum elongation can be observed for the mesh with larger spacing between elements.

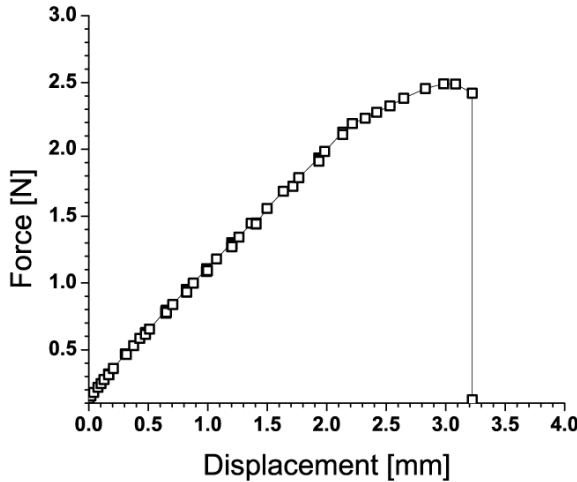


**Figure 4-9.** Elongation for  $(W, L, D) = (10, 20, 300)$   $\mu\text{m}$  mesh interconnected array.

Compared with free-standing meshes overall elongation is much smaller since it is limited by the surrounding silicone but handling and overall strength of the four island array is increased. For a system with overall interconnect length of 3 mm a maximum elongation of 2 mm was observed; meaning that



66 % maximum elongation can be achieved before total fracture for the 500 nm evaporated gold interconnects. Since the electrical conductivity of the fabricated samples was not monitored, the strain values measured before might be smaller if electrical failure was to be measured.



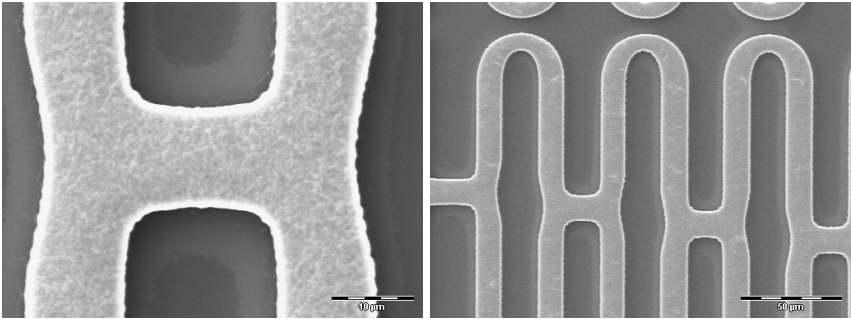
**Figure 4-10.** Elongation for  $(W, L, D) = (10, 40, 300) \mu\text{m}$  mesh interconnected array.

#### 4.2. Mesh shape optimization

Initial experiments with free-standing rectangular meshes showed that during stretching the initial design deforms to an hourglass shape and this leads to stress concentration points that lead to crack formation and early failure. The hourglass shape is caused by the positioning of the anchoring points, the initial shape of the interconnect and the lateral compression of the surrounding PDMS.

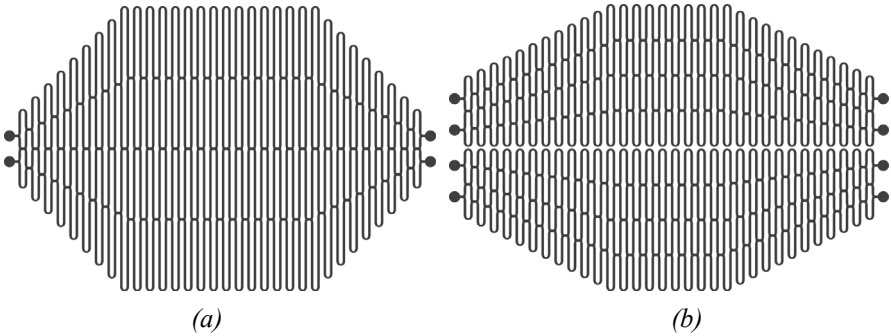
This deformation leads to early fracture because of stress concentration. Optimization of the mesh interconnects has to solve two issues: shape modification during stretching and early crack initiation at sharp corners. By modifying the position of the anchoring points and the shape of the mesh, after stretching the mesh will have a more uniform, rectangular shape.

Maximum elongation and implicit fracture initiation is determined by the mesh shape and by the shape of mesh nodes. FEM and experiments showed that cracks initiate at mesh nodes having sharp corners. These sharp corners reduce maximum elongation of a mesh and cause early failure. That is why all sharp corners have been replaced by curved surfaces and the terminations of periphery elements have been replaced with semi circular shapes (Figure 4-11).



**Figure 4-11.** Illustration of fabricated rounded mesh nodes, (left) close-up of 10  $\mu\text{m}$  wide lines, and (right) 10  $\mu\text{m}$  rounded transitions and corners

Figure 4-12 shows two variations of the modified mesh, the first having one conductive path and the second being split in two conductive paths.



**Figure 4-12.** Modified mesh designs with (a) one conductive path, and (b) two paths.

### 4.3. Fabrication of PDMS-embedded Si electronics arrays

The previous fabrication process [4.1] was modified to include 1D arrays of four silicon islands with contact opening in the PDMS layer to allow electrical measurements during testing. The arrays have no electronics built on the islands. Stretchable interconnects provide electrical conductivity between islands and straight copper line on the surface of the islands connect the stretchable segments.

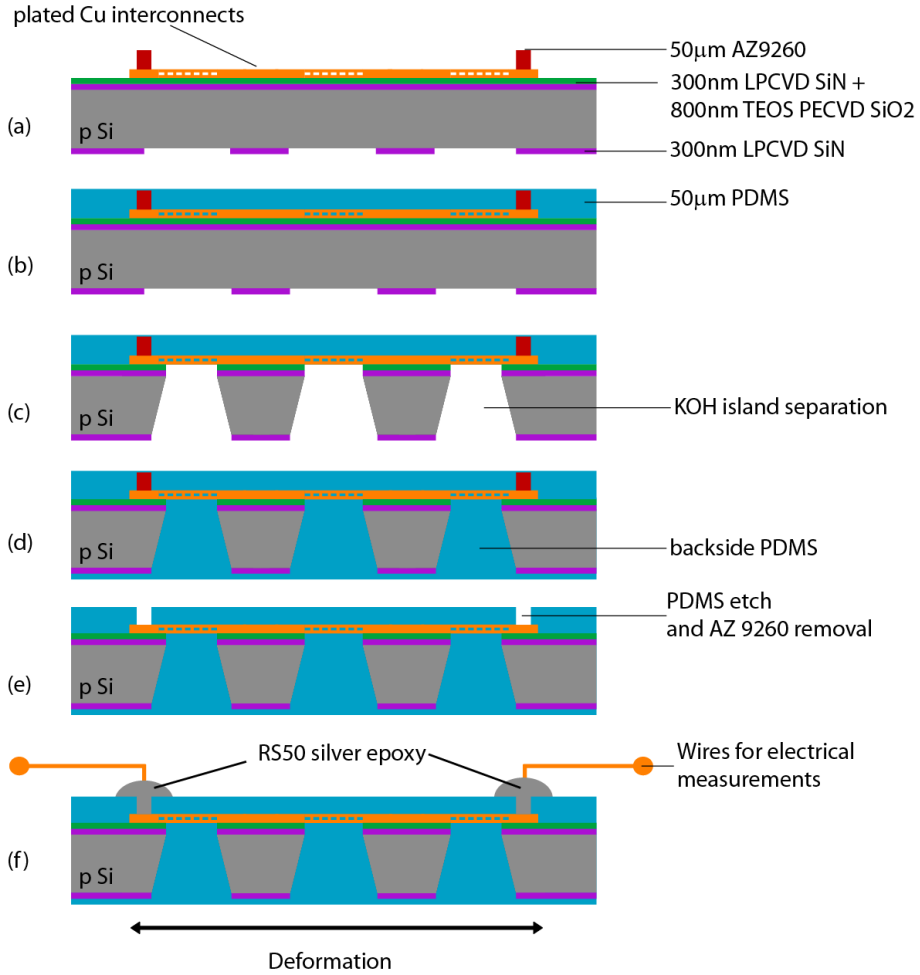
The process starts with 250  $\mu\text{m}$  thick silicon wafers to reduce the KOH etching time. First, a KOH etch masking layer of LPCVD SiN is patterned on the backside of the wafers and left unpatterned on the frontside. A 800 nm layer of PECVD TEOS silicon oxide is deposited on the wafer front side. Due to the non-uniform KOH etching, already released islands have to stay in the bath for an additional 15 minutes to insure that all islands are released. To prevent peeling of the islands during KOH etching, the island-PDMS interface is protected by the etch-stop layer (300 nm LPCVD SiN / 800 nm PECVD TEOS SiO<sub>2</sub>).

After the backside silicon nitride mask is patterned, the stretchable interconnects are fabricated using copper plating. The plated copper interconnects are a 5  $\mu\text{m}$  thick copper layer plated on a 10 nm Ti / 50 nm TiN / 300 nm Cu seed layer using a 5.5-6  $\mu\text{m}$  thick AZ 9260 resist mould.

Next, a 50  $\mu\text{m}$  thick AZ9260 photoresist layer is spin coated onto the wafers. After coating, the resist is degassed in vacuum for 10 minutes to prevent dewetting and agglomeration of the thick resist layer. After soft bake and development, tall resist pillars are left on top of the measurement pads. No hard baking of the resist is done to preserve the pillar profile (Figure 4-13a). PDMS is then spin coated to produce a layer of around 50  $\mu\text{m}$ , comparable in thickness with the resist pillars. Vacuum degassing is used to remove any air bubble trapped in the elastomer layer that is then cured at room temperature for 24 hours (Figure 4-13b). Through wafer KOH etching is done to separate the islands. For this step, a supporting wafer is used on the wafer front side to prevent the stretching or fracture of the PDMS layer towards the end of the etching process (Figure 4-13c). After the KOH etch is completed, the etch-stop layer is removed by dry etching.

The second layer of PDMS is poured on the etched wafer backside and another degassing step is repeated (Figure 4-13d). The last step consists of PDMS etching in  $\text{O}_2/\text{SF}_6$  plasma to expose the photoresist pillars. After the pillars are exposed they are dissolved in acetone and the pads for electrical measurements are exposed (Figure 4-13e). In the last step, RS 50 conductive silver epoxy is used to fill the contact openings to connect the arrays to the measurement equipment.

## PDMS-EMBEDDED SILICON ELECTRONICS ARRAYS



**Figure 4-13.** Final version of fabrication process for silicon island arrays using stretchable interconnects embedded in PDMS.

### 4.4. Array description

Each array type consists of four  $3 \times 3 \text{ mm}^2$  islands using two or four parallel stretchable conductive paths. The first two arrays (T1 and T2) have four conductive paths (5  $\mu\text{m}$  and 10  $\mu\text{m}$  wide copper tracks), using meshes similar to Figure 4-14a while the next two arrays have two conductive paths (T3 with 10  $\mu\text{m}$  wide tracks and T4 with 5  $\mu\text{m}$  wide copper tracks), using meshes similar to Figure 4-14b. The meander design T5 uses 300  $\mu\text{m}$  long elements, 10  $\mu\text{m}$  wide metal tracks, 100  $\mu\text{m}$  radius and 45 degrees transition and horseshoe design T6 uses 10  $\mu\text{m}$  wide metal tracks, 100  $\mu\text{m}$  radius, and 45 degrees transition (Table 4-1).

Table 4-1. Array type description.

Type	T1	T2	T3	T4	T5	T6
Nr. paths	4		2		4	4
Width	5 $\mu\text{m}$	10 $\mu\text{m}$	10 $\mu\text{m}$	5 $\mu\text{m}$	10 $\mu\text{m}$	10 $\mu\text{m}$
Interconnect geometry						

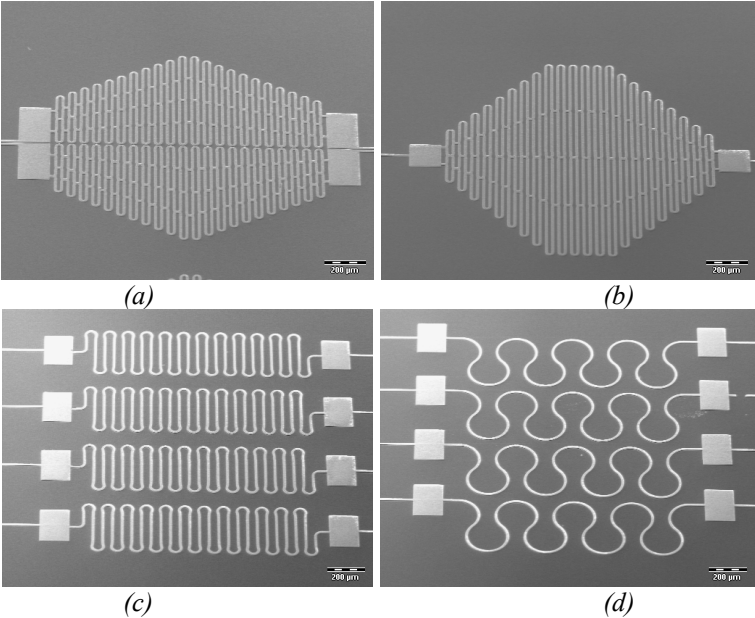
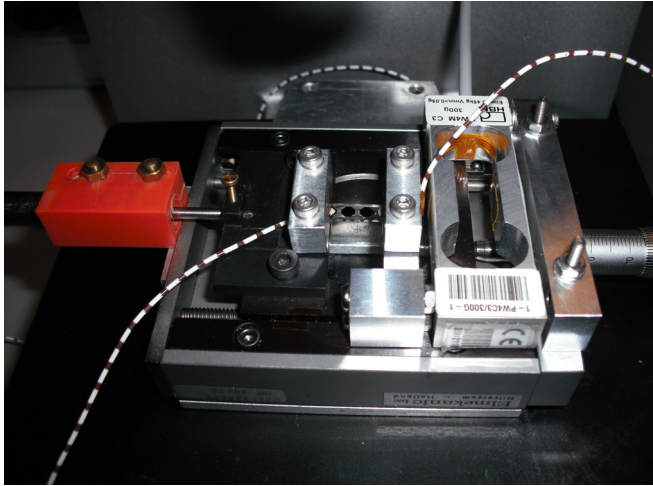


Figure 4-14. SEM images of fabricated interconnect geometries: (a) T1/T2 meshes; (b) T3/T4 meshes; (c) T5 meander, and (d) T6 horseshoe.

4.5. Measurements

Measurements were performed using a tensile test setup designed and built in order to observe the deformation under microscope during loading. The tensile test setup includes clamber, loading part, force sensor, displacement sensor and data recorder. Force and displacement signals are converted by the force sensor and the displacement gauge, amplified by HBM KWS amplifier, changed into digital signal by A/D converter, recorded by custom data recorder program.



*Figure 4-15. Photograph of the tensile test setup.*

Each of four island array has pads for electrical measurements and after each handling step the resistivity was measured. There were no resistivity changes or failure of arrays due to (rough) handling or cutting. In order to determine electrical failure during tensile testing, a drop of silver-based conductive epoxy is dropped on top of the arrays copper pads and when clamped in the tensile setup, thin electrical wires connect the array to a multimeter.

A sample is considered broken if during stretching either one out of the two following failure criteria is met: a/ loss of electrical conductivity; or b/ fracture of the PDMS encapsulation. For large extensions, in most cases the PDMS failed before electrical conductivity was lost. For the horseshoe and meander interconnect, copper lines unfolded from/were pulled off the fractured PDMS and behaved as free-standing interconnects, stretching to an almost straight line before fracturing.

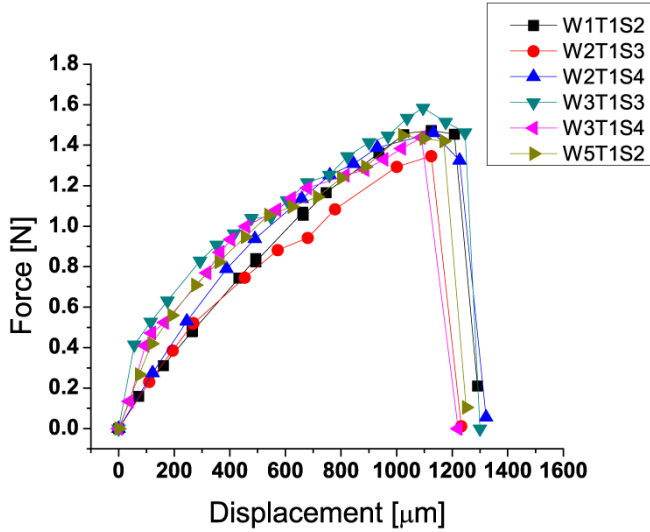
#### ***4.5.1. Tensile testing of fabricated arrays***

The tensile-load measurements provide an elongation vs. force graph. The deformation is recorded for the whole array but because of the non-uniformity of the array, only the three identical areas between islands can deform. Since no slipping of the samples was observed, we can safely assume that the maximum overall elongation of a four island array is equal to three times the elongation of one stretchable area.

All array types showed very similar strain-stress (in our case displacement-force) behavior. For free-standing mesh interconnects, the force needed to fully stretch them is around 0.025 N ([4.2]) while the force needed to fully extend one PDMS-embedded array is around 1.6 N. This difference makes impossible to distinguish different types of arrays based on the displacement-force curve. The samples were cut by hand using surgical blades and this resulted in small

differences in array width which yield small variations in force and maximum displacement achievable.

Typical behavior of fabricated arrays is shown in Figure 4-16. The maximum achievable elongation before failure is around 1200  $\mu\text{m}$  at a force of 1.6 N, resulting in an average maximum strain of one stretchable zone of 400  $\mu\text{m}$ . All stretchable areas have a similar length of 1200  $\mu\text{m}$  resulting in a maximum strain of  $\epsilon \approx 32.5\%$ .



**Figure 4-16.** Tensile test results for T1 arrays with four conductive mesh paths. Legend shows identical samples from different wafers.

#### 4.5.2. Electrical measurements of fabricated samples

Measured electrical resistance values were verified by manually calculating (estimating) the resistance of one conductive path. Each conductive path is made of three elements connected in series: pads, fixed straight metal lines (on the surface of the islands) and three stretchable interconnects. The length of meander and horse shoe designs is easily extracted from the layout but for the mesh design, some approximations are done. Each mesh is considered to be made of four identical conductive meandering paths in parallel. The length of such path is extracted from the layout.

Having all geometrical parameters of the interconnects (length, width and height), the electrical resistance of each path was estimated using the formula:

$$R = \rho \frac{\text{Length}}{\text{Area}},$$

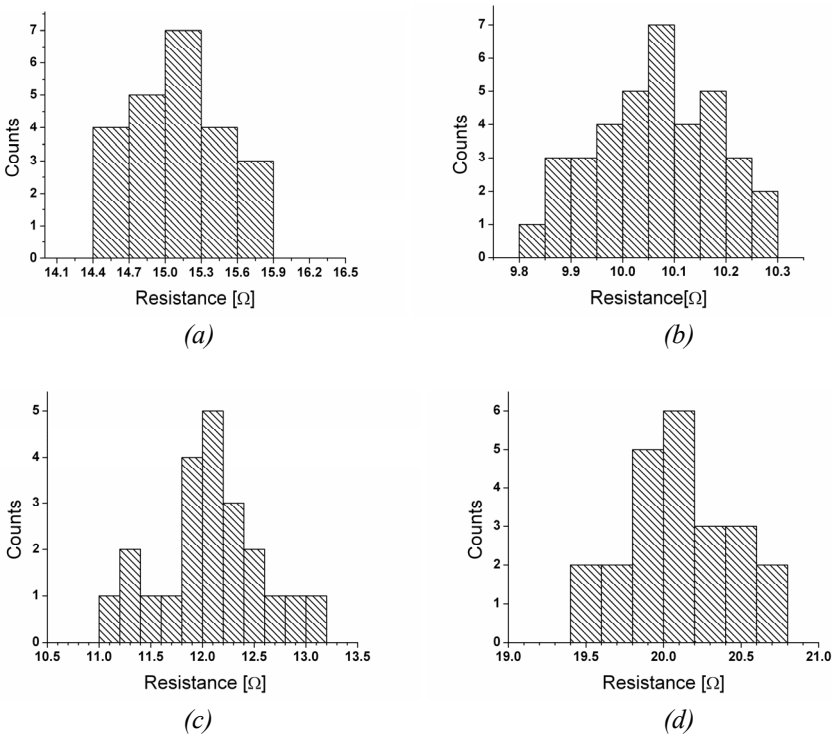
where the resistivity of plated copper was assumed to be  $2.6 \mu\Omega \cdot \text{cm}$  [4.3]. Estimated values and measured values are listed in Table 4-2.

Table 4-2. Estimated and measured electrical resistance values for one conductive path.

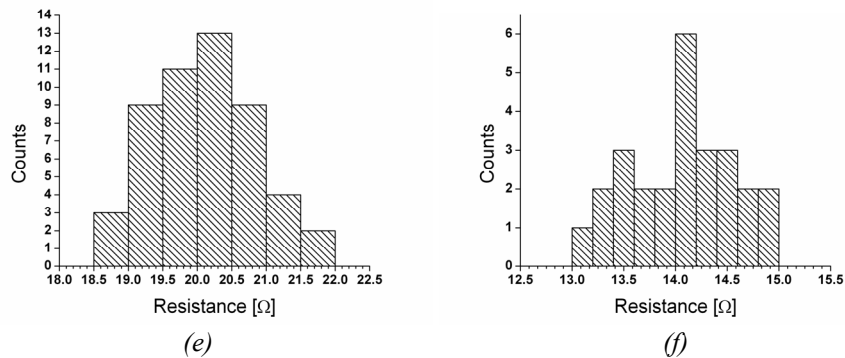
Interconnect type	Measured R for one path [ $\Omega$ ]	Estimated R for one path [ $\Omega$ ]
T1	15	14.5
T2	10	7.65
T3	12	9.89
T4	20	22.5
T5	20	19.1
T6	14	11.7

Resistivity change was monitored during stretching to determine conductivity loss but also to observe how the interconnects electrical resistance is affected by stretching.

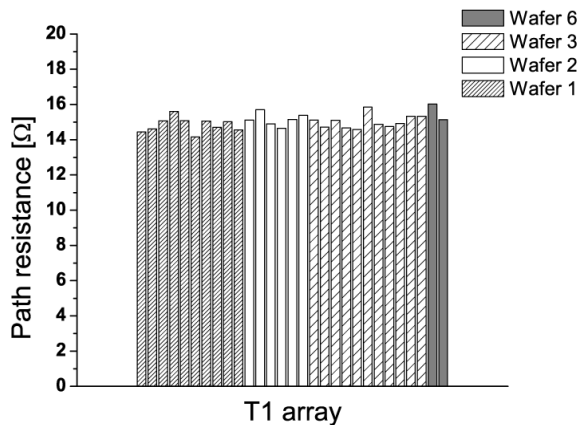
Electrical resistance values of conductive paths from identical samples (before stretching) from different wafers show small variations from wafer to wafer, comparable to variations across the same wafer (Figure 4-18).







**Figure 4-17.** Electrical resistance distribution across all wafers, showing normal distribution pattern for (a) T1, (b) T2, (c) T3, (d) T4, (e) T5, and (f) T6.



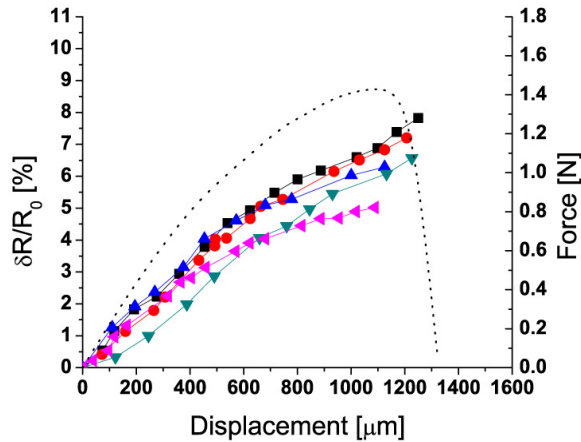
**Figure 4-18.** Resistivity measurements on different wafers for T1 arrays.

By applying silver conductive epoxy on the pad openings, all conductive path of the array are connected in parallel. Since the silver epoxy is applied manually, the electrical resistance of contact varies from sample to sample but, during stretching, the change in electrical resistance is caused only by the stretchable interconnects as the contacts and wires are fixed and unaffected by stretching.

**Table 4-3.** Relative electric resistance change at maximum elongation.

Array type	T1	T2	T3	T4	T5	T6
Average $\delta R/R_0$	6.58	7.97	7.85	7.32	5.42	6.88

The relative change in resistivity is similar for individual samples of the same type. For T1 interconnects the average relative resistance variation is  $\approx 6.6\%$ . Similar values were determined for all arrays. Figure 4-19 shows  $\delta R/R_0$  and force change with increasing displacement until failure.

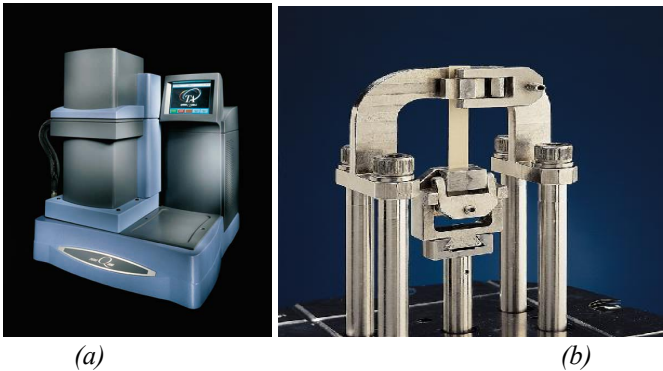


**Figure 4-19.** Relative resistivity change and force (dotted line) with displacement for T1 arrays.

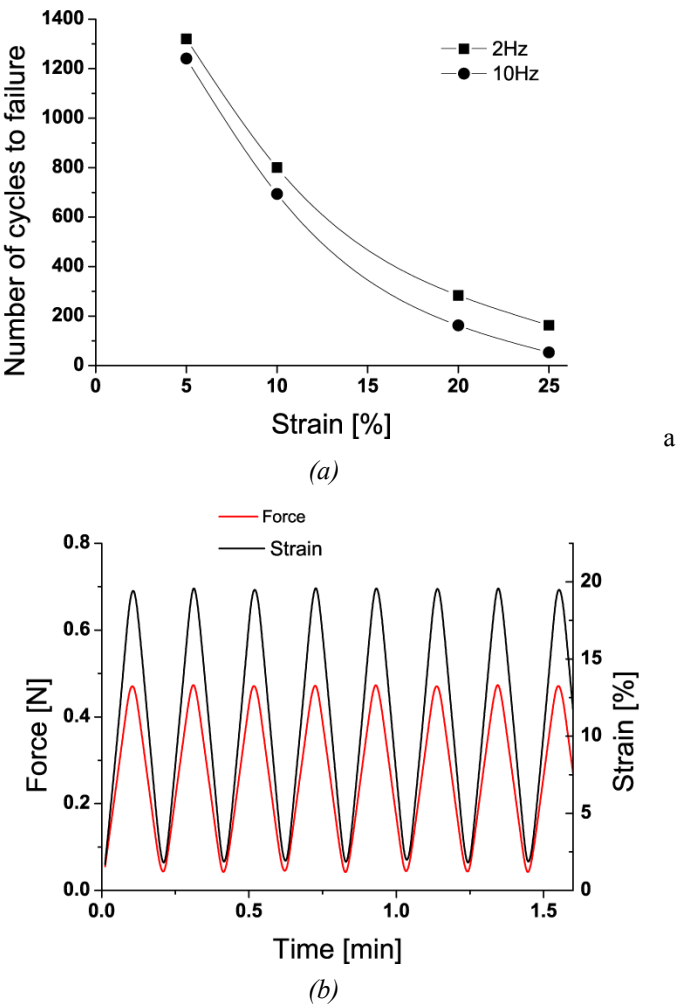
The variation of  $dR/R_0$  was not fully analyzed but initial measurements showed that it is partially reversible. The increase of resistance is caused by cracks appearing in the copper lines that can extend if strain is increased or can “close” when strain is removed.

**4.5.3. Cyclic tensile testing**

Two tests have been performed to investigate changes that might occur after repetitive stretching of the arrays. Clamps of the Q800 DMA were electrically isolated with adhesive tape and thin wires connected the arrays to a multimeter.

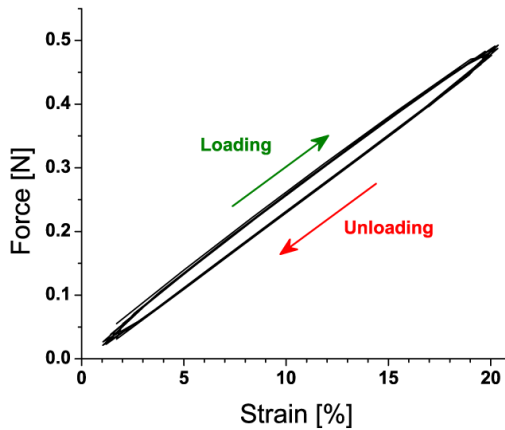


**Figure 4-20.** (a) DMA Q800 from TA Instruments used for cyclic test, and (b) close up of the “tension” clamp used.



**Figure 4-21.** (a) Number of cycles to failure for increasing maximum strain, and (b) strain variation in time for a constant force variation.

In the second test, the force was increased from 0 N to 0.5 N and decreased back to 0 N with a rate of 5 N/min. Based on previous measurements, for a force  $F = 0.5$  N, the maximum expected strain is  $\epsilon_{\max} \approx 20$  %. This time the frequency was given by the rate with which the force was increased or decreased (5 N/min). The frequency of the cycles was 0.0833 Hz. Sample failed after 280 cycles (Figure 4-21b). The difference between loading and unloading behavior of the sample is shown in Figure 4-22.



**Figure 4-22.** Elastic hysteresis of stretchable arrays.

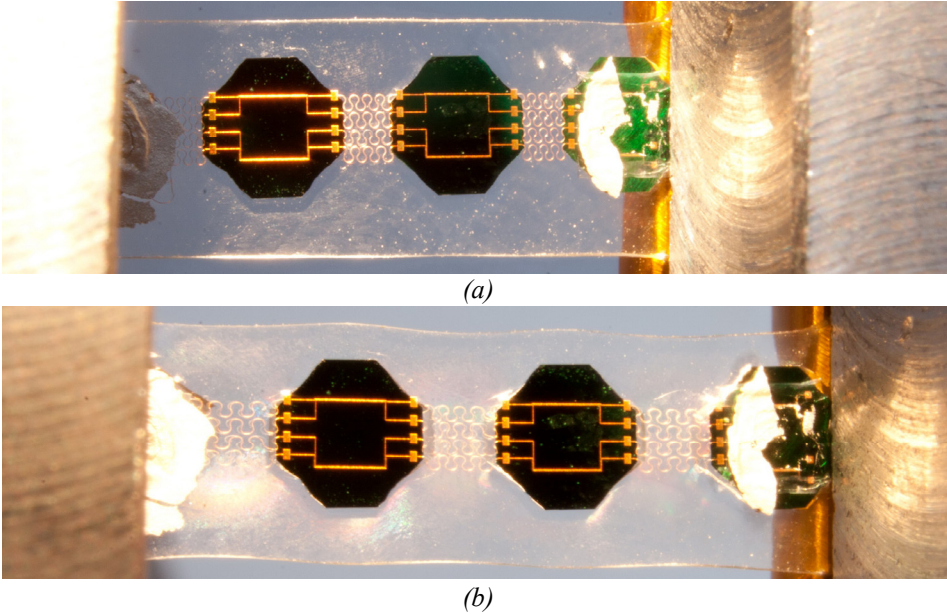
Single stretching tests showed that for large strain values it is the PDMS encapsulation that fails but for cyclic stretching at low strain values it is the interconnects that fail.

#### **4.6. Failure modes of fabricated PDMS arrays**

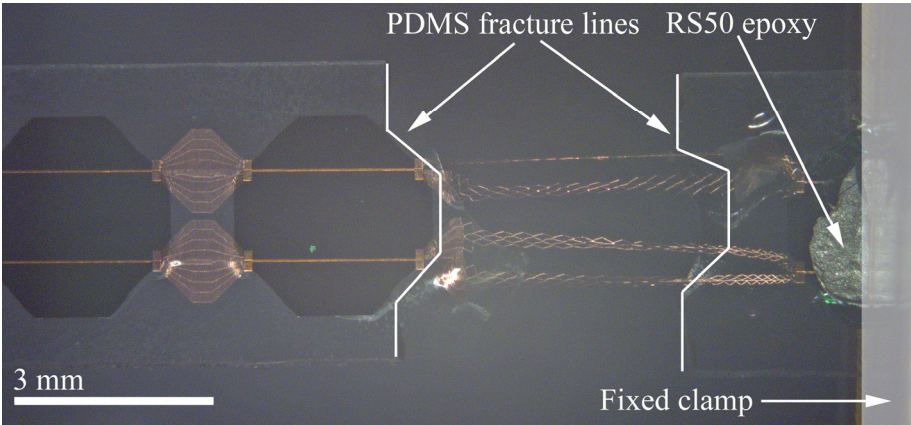
Failure of the PDMS encapsulation is caused by two factors. First, the top PDMS layer ( $\sim 50\ \mu\text{m}$ ) can peel off due to unwanted residues at the interface between top and bottom PDMS layers. These residues are particles left after cleaning and remains of the SiN/SiO<sub>2</sub> etch-stop layer. This defect does not cause the array to break or loose electrical conductivity but can lead to possible short circuits and contamination by loss of encapsulation.

The cracks in the bottom layer of PDMS were the main cause for the loss of conductivity. The shape of the silicon islands and lack of an appropriate adhesion promoter were the main causes for the rapid crack formation in the bottom PDMS layer. The backside lithography mask was designed for  $260\ \mu\text{m}$  thick wafers but because of wafer availability, it was used with  $525\ \mu\text{m}$  thick wafers. As the etch progressed, the corner compensation structures used to define square islands were etched away and the final shape of the island was faceted. The unwanted facets created additional stress concentration lines at the interface between Si and PDMS that reduced the maximum elongation.

The early failure of the encapsulation can be delayed by using dedicated adhesion promoters and surface modification (O<sub>2</sub> plasma treatment) to increase the adhesion of the two PDMS layers. Better control of the island shape can also delay the initiation of cracks by removing stress concentration areas.

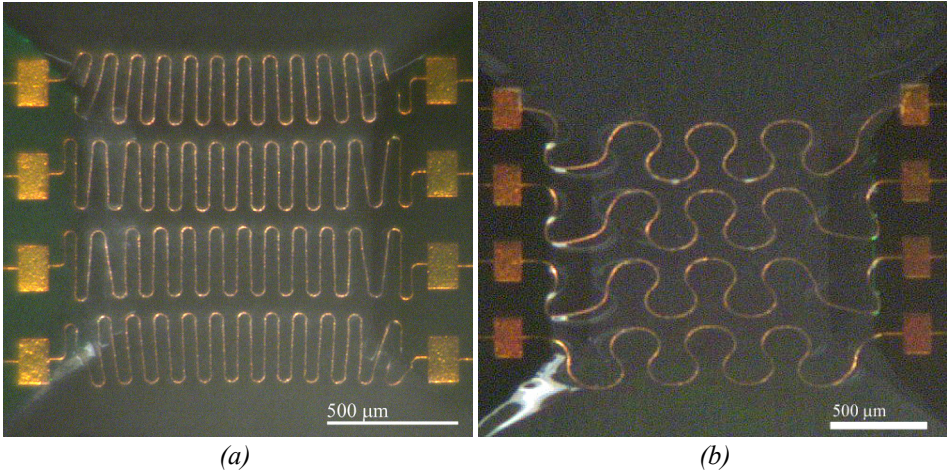


**Figure 4-23.** Photographs of horse shoe sample (a) before stretching, and (b) at  $\approx 25\%$  strain.



**Figure 4-24.** Photograph of failed stretched T1 sample (still conducting), showing PDMS fracture pattern and island with silver epoxy under fixed clamp.

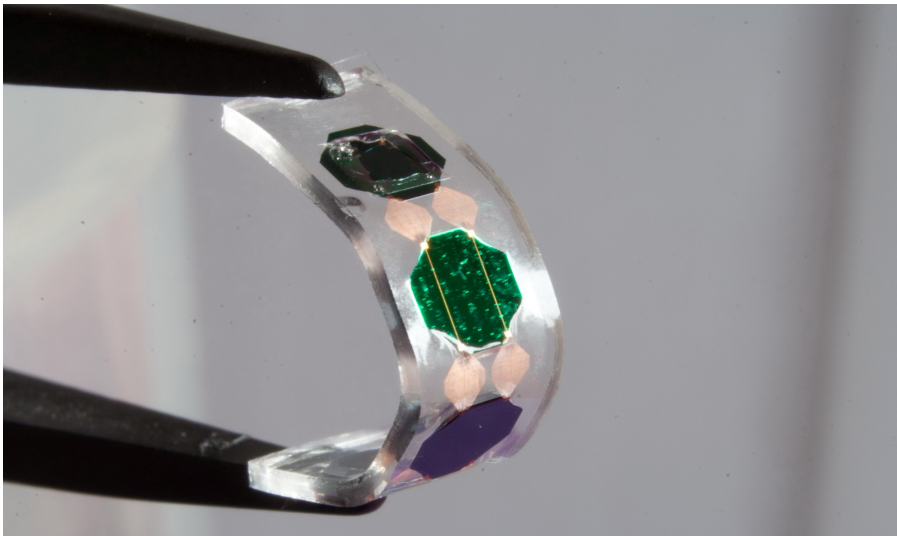




**Figure 4-25.** Photographs of (a) meander interconnect T5, and (b) horse shoe interconnect T6.

Hourglass deformation can be seen also in the case of meander and horseshoe interconnects. In this case the cause of the deformation is the lateral compression of the PDMS and only the outer paths are affected (Figure 4-25a). Delamination of the top PDMS layer can be seen in Figure 4-25b where the imprints of a horse shoe interconnects can be seen next to the interconnect.

Island size and shape are important for the arrays performance. Islands created by anisotropic etching have sharp edges that can slice through PDMS if samples are bent too much. The size of the island also influences the bending radius.



**Figure 4-26.** Optical photograph of bent array, showing faceted profile due to island size (3 mm).

Figure 4-26 illustrates how large islands influence the curvature profile of a bent array. The sharp top edges affect not only the encapsulation but also the transition area of the interconnect from a rigid to an elastic substrate.

#### 4.7. Conclusions

In this chapter, the evolution of the fabrication process for creating stretchable silicon electronic arrays embedded in PDMS is presented. Tensile testing of the fabricated arrays showed that strains up to 32 % are achievable but lower levels are sufficient for biomedical applications and applications related to the human body (such as medical implants, intelligent textiles), where typical strain levels consist of cyclic elongations of up to 5 % [4.4]. PDMS hardening during processing and delamination are the main causes for the maximum strain level achieved.

At maximum elongation, resistance increase varies from 6 % up to 8 %, depending on interconnect configuration. Similar results are reported in literature for horseshoe interconnects, with a resistance increase of maximum 5 % at 75 % of the maximum elongation [4.5].

During single tensile testing, it was the PDMS encapsulation that failed before the interconnects. Bad adhesion, dirt particles and unfavorable silicon island shape were responsible for the encapsulation failure. The island shape was defined using KOH etching, leaving very sharp top edges. These sharp edges and large island size ( $3 \times 3 \text{ mm}^2$ ) contributed to early failure and a reduction in bending radius.

The metal interconnects failed before the PDMS encapsulation during cyclic tensile testing. Cyclic testing showed that the number of cycles until failure decreases with increasing strain levels, samples being conductive for 1300 cycles for  $\epsilon_{\text{max}} = 5 \%$  and 150 cycles for  $\epsilon_{\text{max}} = 25 \%$ . The selected interconnect geometries showed small electrical resistance variation ( $(\delta R/R_0)_{\text{max}} = 8 \%$ ) for a strain level  $\epsilon_{\text{max}} = 30 \%$ . Tensile cycling work on horseshoe interconnects [4.6-8] presents similar behavior for strain levels from 10 % up to 20 %, showing significant decrease in the maximum number of cycles with increasing cycle frequency. When compare to literature results, there is a significant difference in the number of maximum cycles a sample can withstand and the main causes for this are bad adhesion, non-optimized structures and poorly controlled island shape.

#### 4.8. References

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# Chapter 5:

## *Conclusions and Recommendations*

### *5.1. Conclusions*

This thesis presented the development of a wafer-level post-processing module used for the fabrication of stretchable silicon electronic arrays. The fabrication process uses lateral substrate segmentation to create rigid and stretchable regions on the wafer, connecting the rigid blocks with a network of metal interconnects patterned for stretchability.

An overview and analysis of available stretchable geometries is presented in the first part of the thesis. High-aspect-ratio silicon meander springs, with 20  $\mu\text{m}$  wide beams, were fabricated and characterised using a custom-build tensile-load measurement setup. The width of the beams allows future addition of metal tracks to carry electrical signals between silicon islands. Two designs were investigated: basic beam design that had a curved connection ( $180^\circ$ ) between the parallel beams and an optimised version having a rounded connection with additional curved elements. The additional curved elements in the optimised connection prevent formation of stress concentration points at the curved-straight transition. For the basic design, the maximum elongation achieved was around 70 % but for the optimised version, a maximum elongation of 170 % was recorded, clearly showing the benefits of the optimised curved transition. Fracture in these structures occurred suddenly due to the brittle nature of silicon making the integration of such stretchable elements difficult.

Wafer-size PDMS membranes and free-standing plated copper interconnects were analysed separately before integrating them into the final stretchable system. PDMS membranes showed good chemical resistance, making it possible to use PDMS both as a mechanical support layer and a KOH etch-stop layer. While KOH does not attack PDMS, prolonged etching can reduce the adhesion at the interface between PDMS and silicon (or oxides, nitrides, etc.). For this reason a barrier layer (300 nm LPCVD SiN / 800 nm PECVD TEOS SiO<sub>2</sub>) was used to make sure the adhesion is preserved throughout the last minutes of the etching step.

While three various geometries (mesh, meander and horseshoe) were studied, only the more robust mesh design could be safely handled and tested. The optimised curved transition was used for all meander and horseshoe interconnects. Permanent elongations of 300-400 % are easily achievable using the mesh design, reversible deformation levels being much smaller. FE simulations of all three geometries showed that for some combinations of

## CONCLUSIONS AND RECOMMENDATIONS

geometric parameters, large deformation levels could be obtained using all three shapes.

Parylene coating can protect the structures from mechanical damage and chemical agents but can also be used to fine-tune the strength and maximum elongation of copper interconnects. Copper interconnects, 5  $\mu\text{m}$  thick, were coated with a 5  $\mu\text{m}$  or 8  $\mu\text{m}$  thick Parylene sealing layer. The thicker Parylene layer reduces the maximum elongation by 50 % and at the same time generates an increase of around 100 % in the interconnect strength. The thin coating preserves the original elongation (adding extra protection without an increase in strength). SEM analysis of fractured coated samples showed that Parylene continues to coat the copper lines even after their partial failure, covering the cracks in the metal.

During mechanical testing it was observed that sharp corners and anchoring point placement strongly influence the maximum elongation of the interconnects, cracks starting from the sharp corners. By removing of all the sharp corners, stress concentration points are removed, thus increasing the maximum elongation and the life of such an interconnect by delaying the crack initiation. The position of the anchoring points influences the shape of mesh interconnects during deformation, leading to an hourglass shape that can generate additional stress concentration areas.

Four-segment silicon 1-D chains were fabricated, having 2 or 4 parallel conductive paths made of variations of the mesh, meander and horseshoe interconnects. The total elongation of such a system is achieved by the three stretchable areas between the four rigid segments. PDMS embedding will limit the maximum elongation of a stretchable area to 170 %, the maximum elongation measured for PDMS. Measured maximum elongation was similar for all interconnect geometries, at a level of 30 %.

Two primary failure modes were identified. The first failure mode observed is the failure of the PDMS encapsulation, appearing at large strains due to bad adhesion, dirt particles and island shape with sharp edges caused by KOH etching. Dirt, island shape, bad adhesion and rigid-flexible transitions greatly reduce the maximum elongation at levels below those of free-standing interconnects or PDMS. The fact that there is no serious difference in maximum elongation between arrays having different types of interconnects can be explained by the early failure of the PDMS encapsulation and the large difference in forces needed to extend one interconnect (0.04-0.06 N) or one PDMS array (1.2-1.6 N).

Interconnect failure is the second failure mode and it is observed during cyclic tensile testing at strain levels lower than the maximum strain ( $N_{\text{max}} = 1300$  cycles at 5 Hz, at  $\epsilon_{\text{max}} = 5\%$ ), before the failure of the PDMS encapsulation.

## 5.2. Recommendations and future work

Initial work on metal deposition on PDMS did not yield good results as waves and cracks appeared on the deposited layer. Patterning of metal layers on PDMS is also difficult due to the different coefficients of thermal expansion of PDMS, metals and photoresist. Some sort of reaction was observed between cured PDMS and photoresist (AZ 9260), the image of the initial puddle of resist being visible after photoresist removal. For these reasons it was more convenient to first deposit and pattern the metals on traditional rigid substrates and then deposit the PDMS.

Wet etch of PDMS can be done but is time consuming, needs a metal mask layer (that creates permanent waves in PDMS) and uses large amounts of chemicals. The development of a fast dry etching recipe for PDMS would be beneficial for further growth of stretchable electronics.

KOH etch for lateral segmentation of the substrate is done at high temperature (80-90 °C) and ideally should stop on PDMS. The high temperature and long etch time may influence the elasticity of the etch-stop PDMS layer by curing the layer. This will create a difference between the top and bottom PDMS layers. Without using an etch stop layer (300 nm LPCVD SiN + 800 nm TEOS SiO<sub>2</sub>) the adhesion of the silicon islands to the supporting PDMS membrane is short lived as KOH creeps between Si and PDMS and releases the islands.

A dry etching recipe is recommended for the lateral segmentation of the substrate as it can improve the island shape, reduce etch times and preserve adhesion.

Adhesion of PDMS to the substrate can be also improved by using special formulated PDMS containing adhesion promoters.

Fine tuning of the interconnect strength can be done by coating the metal shape with Parylene. In Chapter 3, subsection 3.7.4, the strength of interconnects is tuned by coating with Parylene. For a 5 µm thick plated copper interconnect, the force needed to extend it is doubled if coated with a Parylene layer of 5 µm. If the thickness of the metal layer is greatly reduced (nanometer range), a similar force could be obtained only from a 10 µm thick sandwich of Parylene (5 µm on each side) with a thin metal layer in between. Experiments involving liquid metal alloys confined in diamond mesh channels fabricated in PDMS showed good results (up to 60 % elongation). This way, the difference of maximum elongation between polymer and metal interconnect is eliminated [5.1, 2].

Starting from this, one can imagine a configuration where stress from the encapsulating polymer is not transferred to the metal interconnects, but the interconnects (coated or not) can freely move in polymer channels without being bonded to the walls.

**5.3. References**

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# Appendix A:

## *High Aspect Silicon Springs Flow Chart*

### ***Version 01:***

#### **Frontside processing:**

1. Substrate: Si wafer, thickness= $525\pm 15\text{ }\mu\text{m}$
2. Plasma enhanced SiO<sub>2</sub> deposition (6.0  $\mu\text{m}$ ): Novellus (recipe: 6musio)
3. Photoresist coating: Co – XI – SPR3017M – 3000 nm – no EBR
4. Alignment and exposure: EV240 contact aligner
5. Development: Dev - Double Puddle 3
6. Hard mask opening: Drytek 384T, plasma etcher with program VAROXIDE
7. Cleaning procedure: TEPLA program 1 + HNO<sub>3</sub> 100 % and 65 %
8. DRIE: Adixen, recipe: scribeline, 3 hours
9. Wafer thinning to 250  $\mu\text{m}$
10. Cleaning
11. HARS springs separation

### ***Version 02:***

#### **Frontside processing:**

1. Substrate: DSP thin Si wafer, thickness= $250\pm 25\text{ }\mu\text{m}$
2. Plasma enhanced SiO<sub>2</sub> deposition (6.0  $\mu\text{m}$ ): Novellus (recipe: 6musio)
3. Photoresist coating: Co – XI – SPR3017M – 3000 nm – no EBR
4. Alignment and exposure: EV240 contact aligner
5. Development: Dev - Double Puddle 3
6. Hard mask opening: Drytek 384T, plasma etcher with program VAROXIDE
7. Cleaning procedure: TEPLA program 1 + HNO<sub>3</sub> 100 % and 65 %

#### **Backside processing:**

8. Aluminum deposition: Sigma, recipe: 4075 nm al@350C

#### **Frontside processing:**

9. DRIE: Adixen, recipe: scribeline, 3 hours (make sure wafer is etched through)
10. Wet etch aluminum: PES solution
11. HARS springs separation

# Appendix B:

## *Free-Standing Copper Interconnects Flow Chart*

### **Frontside processing:**

1. Substrate: Si wafer, thickness= $525\pm 15\text{ }\mu\text{m}$
2. PECVD SiO<sub>2</sub> deposition (6.0  $\mu\text{m}$ ): Novellus (recipe: 6musio)
3. Deposition of Ti/TiN/Cu Copper seed layer
4. Coating an baking: CO-AZ9260-SYR-10micron-no EBR (allow 20 minutes time for rehydration before exposure)
5. Alignment and exposure: EV240 contact aligner, 60 seconds
6. Development: AZ400K: Water (1:2)
7. Resist bake-out: Memmert for 30 min @ 110°C
8. Europlasma # 7, O<sub>2</sub> plasma flash 1 min in barrel reactor (SALab)
9. 5 $\mu\text{m}$  Copper plating (SALab) (1.22849E+008 square microns)
10. Resist removal
11. Ti/TiN/Cu seed layer removal
12. Dicing
13. Individual interconnect release in HF (40 %)

# Appendix C:

## *PDMS Embedded Silicon Electronics Arrays Flow Chart*

### **Frontside processing:**

1. Substrate: DSP thin Si wafer, thickness= $250\pm 25\text{ }\mu\text{m}$
2. Deposition of LPCVD low stress silicon nitride (300 nm)
3. Deposition of PECVD TEOS 800 nm

### **Backside processing:**

4. Pattern LPCVD nitride: Drytek Triode 384T

### **Frontside processing:**

5. Deposition of Ti/TiN/Cu Copper seed layer
6. Coating and baking: CO-AZ9260-SYR-10micron-no EBR (allow 20 minutes time for rehydration before exposure)
7. Alignment and exposure: EV240 contact aligner, 60 seconds
8. Development: AZ400K: Water (1:2)
9. Resist bake-out: Memmert for 30 min @  $110^{\circ}\text{C}$
10. Europlasma # 7,  $\text{O}_2$  plasma flash 1 min in barrel reactor (SALab)
11.  $5\mu\text{m}$  Copper plating (SALab)
12. Resist removal
13. Ti/TiN/Cu seed layer removal
14. Coating and baking:  $45\text{ }\mu\text{m}$  AZ9260 (allow 60 minutes time for rehydration before baking and before exposure)
15. Exposure: MA6, 450 seconds (allow 60 minutes time for rehydration before development)
16. Development: AZ400K: Water (1:2)
17. Spin coat:  $50\text{ }\mu\text{m}$  PDMS
18. Curing:  $120^{\circ}\text{C}$ , 15 minutes
19. Protect wafer front side with blue dicing tape
20. KOH etch, 33 %,  $90^{\circ}\text{C}$  for 3 hours (wafer mounted with support wafer)
21. DI water cleaning, drying
22. Removal of LPCVD nitride and PECVD TEOS: Alcatel GIR 300
23. DI water cleaning, drying

### **Backside processing:**

24. Backside filling with PDMS
25. Overnight curing at room temperature

**Frontside processing:**

26. PDMS etch back: Alcatel GIR 300 (1 h or until photoresist is exposed)
27. Dicing



# Summary

In recent years, the term “More-than-Moore” (MtM or “Diversification”) has been introduced to describe the evolution of systems with non-conventional (non-digital) functionality. These are devices that exploit the possibility of cleanroom processing not only to realize highly miniaturised electronic functions but to integrate sensors and actuators interacting with the environment. Such microsystems can sense their environment, detect movement or change their shape and size. Many areas, such as consumer electronics or biomedical devices benefit from the added functionality, to mention just a couple. Microsystems designed to operate in proximity or directly attached to or as a part of the human body (“smart” clothes or implantable devices) should be made less intruding and better matching their target environment. In these types of applications, the traditional, rigid electronics must become stretchable withstanding cyclic deformations. In this thesis, stretchable silicon electronics concept based on a segmented rigid substrate, deformable metal interconnects and a wafer-level processing is investigated.

Chapter 1 presents the motivation behind stretchable electronic systems and presents the proposed fabrication approach. Chapter 2 starts with a theoretical introduction to material properties, needed to explain common terms used to characterise materials and in the end, stretchable systems. In the last part of Chapter 2 an overview of the state-of-the-art stretchable systems is given, followed by discussions concerning fabrication, handling, material selection or economic issues.

Chapter 3 describes the geometries used for stretchable interconnects, presenting the set of parameters needed to define each shape. Variations of presented shapes (meander, horseshoe and mesh) are analysed using finite element simulations to identify weak spots and find parameter combinations that give the largest elongation without failure. The initial experiment investigates High-Aspect-Ratio-Silicon (HARS) silicon springs meant to electrically and mechanically connect silicon islands. Basic and optimised meander shapes are tested in a custom-built tensile setup, showing 70 % and 170 % maximum elongation, respectively, clearly showing the benefits of curved smooth transitions between elements. Tensile testing of individual copper-plated free-standing interconnects revealed that for the mesh shape (meander and horseshoe structures were too fragile for unprotected manipulation), large elongations (not fully reversible), up to 400 % can be achieved. Similar levels are predicted using simulations for all interconnect shapes.

Parylene is used to coat interconnects for protection but also for fine tuning of the strength and maximum elongation.

Chapter 4 focuses on the integration of individual elements (free-standing copper interconnects, PDMS membranes) into stretchable 1D arrays of PDMS embedded silicon islands. The integration of different elements is done gradually to prove the feasibility of each step. First, wafer-scale PDMS membranes are fabricated by spin coating 50 $\mu$ m of PDMS on silicon wafers followed by KOH etching, leaving only a silicon ring on the wafer edge for mechanical support. In the next step, PDMS is spin coated on silicon wafers with metal interconnects. After the silicon is removed, the interconnects remain attached to the PDMS membrane. Interconnects remained conductive after manual deformation.

Based on the previous results, 1D arrays are fabricated and tested. Each array consists of four silicon islands (3x3 mm<sup>2</sup>) linked by stretchable metal interconnects (1.2 mm long variations of mesh, meander, horseshoe), all embedded for support and protection in PDMS. Optimised mesh shapes are used to delay crack initiation (rounded corners and curved transitions between vertical beams) and to control the change of shape during deformation.

Tensile testing of fabricated arrays reveals an average 30 % elongation before failure for all interconnect geometries used. Cyclic tensile testing shows that the arrays can withstand repeated strain levels lower than the maximum strain ( $N_{\max} = 1300$  cycles at 5 Hz, at  $\epsilon_{\max} = 5\%$ ), before the failure of the PDMS encapsulation. Failure of the PDMS encapsulation usually appears at large strains while at small repeated strains the electrical interconnects are the structures that fail.

Chapter 5 presents conclusions on mechanical and electrical testing of fabricated devices, failure modes of such devices (with possible solutions). The last part gives recommendations for future improvements of such systems.

*Sebastian Sosin*

# Samenvatting

In de afgelopen jaren is de term ‘Meer dan Moore’ (MtM of ‘diversificatie’) geïntroduceerd om de evolute te beschrijven van systemen met onconventionele (niet-digitale) functionaliteit. Het gaat om apparaten waarbij gebruik gemaakt wordt van de mogelijkheden van bewerkingen in een cleanroom, niet alleen om de elektronische schakelingen zo klein mogelijk te maken, maar ook om sensoren en actuatoren te integreren die interactie met de omgeving hebben. Dergelijke microsystemen kunnen de omgeving waarnemen, bewegingen bespeuren of van vorm en grootte veranderen. Veel gebieden, zoals consumentenelektronica of biomedische apparatuur, om er maar een paar te noemen, halen voordeel uit de extra functionaliteit. Microsystemen die zijn ontworpen om in de nabijheid van, direct op of zelfs in het lichaam te werken (‘slimme’ kleding of implantaten), moeten zo min mogelijk verstorend gemaakt worden en beter in overeenstemming met de omgeving waarin ze moeten functioneren. In zulke toepassingen moet de traditionele, onbuigzame elektronica rekbaar gemaakt worden en cyclische vervorming kunnen weerstaan. In dit proefschrift wordt het onderzoek beschreven naar rekbaar elektronica op siliciumbasis, uitgaand van een onbuigzame onderlaag die is opgedeeld in segmenten, vervormbare metalen verbindingen en vervaardiging op wafer-niveau.

In hoofdstuk 1 wordt de motivering gepresenteerd van het onderzoek naar rekbaar elektronica en de voorgestelde methode voor de vervaardiging daarvan. Hoofdstuk 2 begint met een theoretische inleiding over materiaaleigenschappen die nodig is om een aantal veelvoorkomende termen uit te leggen die worden gebruikt om materialen en, uiteindelijk, rekbaar systemen te beschrijven. In het laatste deel van hoofdstuk 2 wordt een overzicht gegeven van de *state-of-the-art* waar het rekbaar systemen betreft, gevolgd door een discussie over fabricage technieken, omgang met en keuze van materialen en economische overwegingen.

Hoofdstuk 3 beschrijft de geometrieën die gebruikt worden voor de rekbaar verbindingen, waarbij de parameters worden geïntroduceerd die nodig zijn om de verschillende vormen te definiëren. Variaties van de verschillende vormen (‘meanderend’, ‘hoefijzer’ en ‘net’) worden geanalyseerd door gebruik te maken van eindige-elementen simulaties om zodoende de zwakke plekken te ontdekken en combinaties van parameters te vinden die de langste uitrekking geven zonder dat het apparaat breekt. Het initiële experiment onderzoekt veerconstructies van silicium gemaakt in ‘High-Aspect-Ratio-Silicon’ (HARS) die siliciumeilanden zowel elektronisch als mechanisch moeten verbinden. Eenvoudige en geoptimaliseerde meanderconstructies zijn getest in een speciaal daarvoor gebouwde trek-opstelling, waarbij bleek dat ze, respectievelijk, maximaal 70 % en 170 % konden worden opgerekt, waaruit duidelijk het

voordeel van gebogen, geleidelijke overgangen tussen onderdelen blijkt. Het testen van de trekvastheid van losse, vrijstaande verbindingen die met koper zijn bedekt, laat zien dat met de ‘net’-vorm (‘meanderend’ en ‘hoefijzer’ bleken te breekbaar om onbeschermde te kunnen manipuleren) grote elongatie (niet volledig omkeerbaar) tot wel 400 % kunnen worden behaald. Vergelijkbare waarden worden door de simulaties voorspeld voor alle vormen van de verbindingen. De verbindingen zijn met Parylene gecoat, niet alleen om ze te beschermen, maar ook om de sterkte en maximale uitrekking nauwkeurig te kunnen beïnvloeden.

Hoofdstuk 4 richt zich op het samenvoegen van de losse elementen (vrijstaande verbindingen van koper, membranen van PDMS) tot rekbare 1-dimensionale rijen van siliciumeilanden, verzonken in PDMS. Het samenvoegen van de verschillende elementen is stap voor stap uitgevoerd om de haalbaarheid van elke afzonderlijke stap te bewijzen. Ten eerste worden PDMS membranen gemaakt door een laag van 50  $\mu\text{m}$  PDMS te spin-coaten op een silicium wafer, waarna geëët wordt met KOH zodat alleen een ring van silicium aan de rand van de wafer overblijft voor mechanische ondersteuning. In de volgende stap wordt PDMS ge-spincoat op silicium wafers met metalen verbindingen. Nadat het silicium is verwijderd, blijven de verbindingen verbonden met het PDMS membraan. De verbindingen bleven na handmatige vervorming elektrisch geleidend.

Op basis van de voorgaande resultaten zijn 1-dimensionale rijen gemaakt en getest. Elke rij bestaat uit vier silicium eilanden ( $3 \times 3 \text{ mm}^2$ ) verbonden door rekbare metalen verbindingen (1,2 mm lang in de variaties ‘net’, ‘meanderend’ en ‘hoefijzer’), verzonken in PDMS ter ondersteuning en als bescherming. Geoptimaliseerde ‘net’ vormen zijn gebruikt om het begin van een breuk uit te stellen (ronde hoeken en gebogen overgangen tussen verticale balken) en om tijdens vervorming de vormverandering te controleren.

Uit de test van de trekvastheid van de gemaakte rijen blijkt dat een gemiddelde uitrekking van 30 % wordt bereikt bij alle gebruikte vormen van de verbindingen, voordat er een breuk optreedt. Uit cyclisch testen van de trekvastheid blijkt dat de rijen herhaaldelijke rek kunnen weerstaan lager dan de maximale rek ( $N_{\text{max}} = 1300$  herhalingen bij 5 Hz,  $\epsilon_{\text{max}} = 5 \%$ ), voordat de PDMS encapsulatie breekt. De PDMS encapsulatie breekt normaal gesproken bij grote rek, terwijl bij kleine, herhaaldelijke rek juist de elektrische verbindingen breken.

In hoofdstuk 5 worden conclusies gepresenteerd over de mechanische en elektrische testen van de gemaakte apparaatjes, de manieren van breuk van zulke apparaatjes (met mogelijke oplossingen). In het laatste deel worden aanbevelingen gedaan voor toekomstige verbeteringen van dergelijke apparaatjes.

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1. Moldovan, C., Iosub, R., Tomescu, A., Bercu, M., and Sosin, S., *Micromachined chemoresistive sensor for CO detection*, Sensors for Security Workshop, 2004, Ispra, Italy.
2. Sosin, S., Moldovan, C., and Iosub, R., *Design and simulations for a calorimetric micro-sensor for methane detection*, International Semiconductor Conference (CAS), 2004, Sinaia, Romania, pp. 381-384.
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# Acknowledgments

It has been more than 5 years since I have started the PhD journey. When you will read this, it will be finally over. Looking back, I have to admit it was a nice experience, with lots of ups and downs. It took me through all emotions possible. I would like to think that it made me better, even smarter.

Marian Bartek, you gave me the chance to start my PhD and I thank you for this. I know that my “telegraphic” writing style gave you some trouble but hopefully I have improved (a little) STOP.

Lina Sarro, without your help, this thesis wouldn’t have existed. Thank you for all the support in dealing with DIMES issues.

Ronal Dekker, I will miss the train discussions, even if not many. Your help with Parylene deposition, PDMS adhesion and so on was much appreciated. If talking about Parylene, I have to thank Pascal de Graaf. You spent a lot of time with my wafers and I feel ashamed for not being able to continue the Parylene version of my stretchable samples.

It has been too long since we had a beer, Jacopo. I met you in my first or second day in Delft, at the usual watering hole (the coffee corner) and since then we had a blast. The green fairy nights (thank you, Ottakar for some kick ass absinth), the camping trip, the conferences ... good old times.

Alex Polyakov, a great colleague, although not a smoker, you spent hours in the smoking room with us, just fooling around and joking to fight the Dutch weather and food.

Sasha, a day after I arrived in Delft with my 60kg of luggage (that was stupid), you and your boyfriend took me out for a coffee and a brief introduction session on the Dutch system. Since then we have enjoyed many hours of chatting and even if we weren’t always on the same frequency (I still think A4 ends), you were a good colleague.

I cannot forget Yetao Zhu, my first office mate, although it is still hard to call that room an office. You survived my long internet calls with my mother and my friends and that takes patience and good headphones. Not to forget, my badminton skills are better because of you.

Iwan Kurniawan, you were my partner in PhD suffering and helping you in the MEMS lab offered me some nice breaks. I wish we could have climbed more.

Huang Cong, Han Yan, Jun Tian, Theodoros Zoumpoulidis, Saoer Sinaga, Hsien Chang Wu, Gabriel Macias, Huseyin Sagkol (thank you for not being too terrified by the trip to Bucharest), I thank you for the long lunches, amusing talks, countless hours of pool, the dinners and the hot pot, to mention just a few.

Not to forget, the new kids in Delft: Bruno Morana, Benjamin Mimoun (I would love to see a France vs. Romania rugby match with you), Daniel Vidal, Teresa Adrega, Ana da Silva and Francesco Vitale. I met you in my most

difficult period in Delft and I cannot thank you enough for taking my mind off my troubles.

I would like thank Lingen Wang, Kaspar Jansen and Jos van Driel for their support with simulation work and measurement setups. Mohamed Saadaoui, Nishant Lawand, Thomas Moh, your help with processing was invaluable and I am grateful for your support in the lab.

Special thanks go to Atef Ahnouk, Mario Laros, Hugo Schellevis, Charles de Boer, Jan Groeneweg, Ruud Klerks, Wim van der Vlist and Jan Cornelis Wolff for their assistance in the lab.

Life in Delft would have been sad without a very joyful group of people. I have to start with Ignacio (Nacho), my Mexican neighbour. Not only we had fun but you've created the famous Romanian corner, Anca, Oana and myself. Evandro, many thanks to you and the rest of the gang for some wonderful dinners/parties.

My old high school and university friends deserve special thanks for all the late nights and great memories. I know that wherever we'll be, we can always count on each other for some good times.

This thesis wouldn't have been possible without my mother encouraging me to embark on this adventure, without asking what exactly I am doing 2400 km away from home.

Oana, I thank you for listening to me, for your advice and your support and just for being next to me for the last five years. I am pretty sure you don't like listening to me explaining processing cross sections or how PDMS doesn't stick anymore but you did it anyway and I appreciate it.

Surely I must have forgotten a few names and I apologise for that. Thank you all for your help and I hope you will enjoy the thesis.

*Sebastian*



# About the Author

Sebastian Sosin was born in Bucharest, Romania, on 16<sup>th</sup> of August 1979. He graduated in 2003 with a BSc in microsystem engineering from the University “Politehnica” Bucharest, Faculty of Electronics and Telecommunications after carrying out his diploma thesis at the National Institute of Physics and Nuclear Engineering-“Horia Hulubei” in Bucharest, Romania. In 2004 he graduated with a MSc in microsystem engineering from the same university. His master thesis was carried out at the National Institute for Research and Development in Microtechnologies in Bucharest, Romania. Since May 2005 he has been working as a PhD candidate at the Delft Institute of Microsystems and Nanoelectronics (DIMES) at the Delft University of Technology, in the Netherlands. His main research activities focus on wafer level fabrication of stretchable silicon electronics.