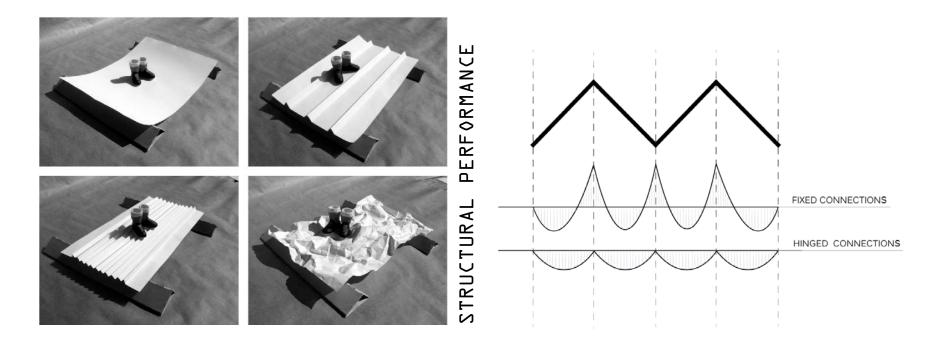
Sustainable Design Graduation Studio\_Alkistis Krousti\_4420705

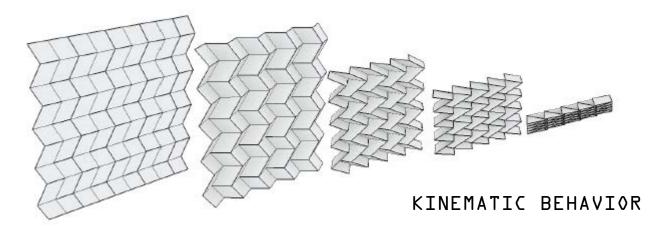
# FOLDED GLASS PLATE STRUCTURES: A DEPLOYABLE ROOF SYSTEM

MENTORS: Ate Snijder, Peter Eigenraam, Michela Turrin





# FOLDED PLATE PROPERTIES



# RESEARCH QUESTION :

To which extent can the kinematic qualities of folded geometries be combined with the structural benefits of glass plates and more specifically, how can these be applied in the case of a deployable glass roof system?

SUB-QUESTIONS

• What are the criteria for selecting a folding pattern that provides both stiffness and deployment potential?

• How do the geometrical parameters of the folding pattern affect the structural properties of the system?

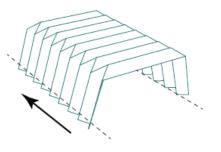
• What kind of mechanism enables the specified deployment movement and what restrictions does this present for the design?

• What types of connections are required between plates and in the structure supports to ensure that the load transfer is done as expected and also to allow the necessary degree of freedom for the deployment?

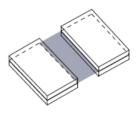
• How can those connections be designed to be as invisible as possible, while providing the required tolerances and envelope properties, such as waterproofing, etc.?

# RESEARCH QUESTION :

To which extent can the kinematic qualities of folded geometries be combined with the structural benefits of glass plates and more specifically, how can these be applied in the case of a deployable glass roof system?







DESIGN CRITERIA

Provide natural lighting and maximum transparency Self-supported glass plate structure (no frame) Deployable on one side (fully adaptable) Feasibility

STRUCTURAL CRITERIA

Controlled element deformation + stress levels (all phases of deployment)

General shape stability (all phases of deployment) Glass element redundancy

Damage sensitivity - Fracture mode – Safety factors

DETAILING CRITERIA

Discrete design- Invisible connection

Tolerances

Restriction of gaps- waterproofing

Repair work facilitated

BC maintained by connection detailing

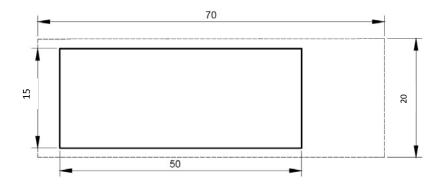


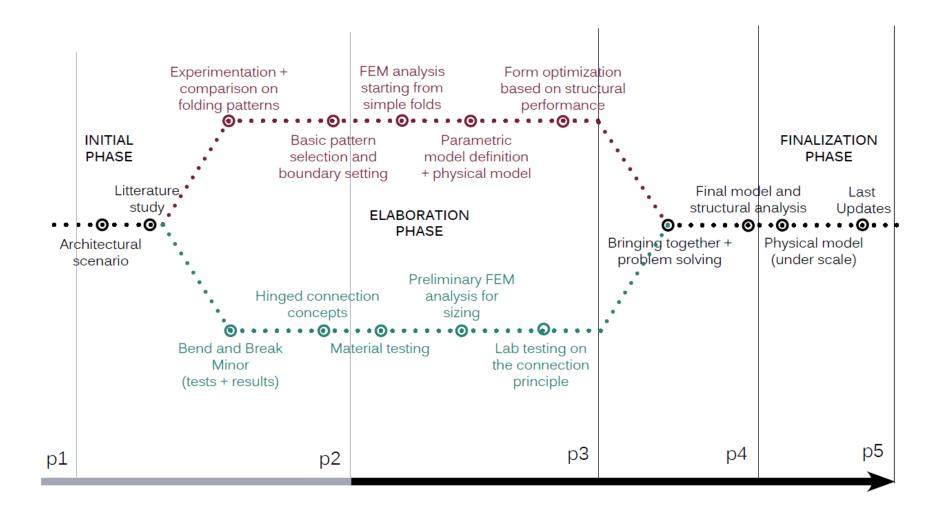




# CASE STUDY

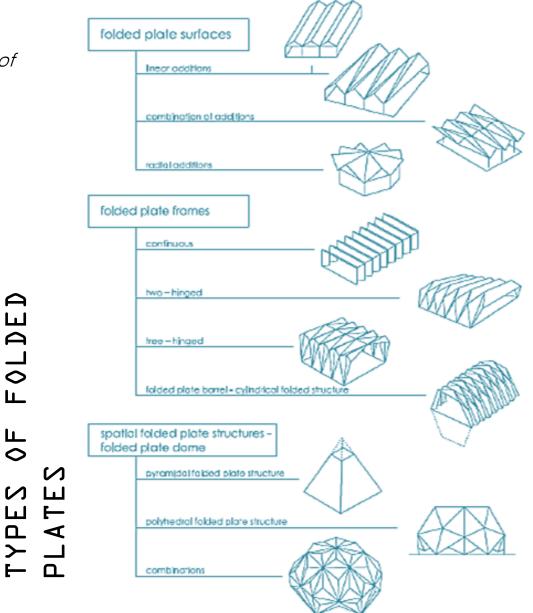
Swimming pool area : 15\* 50m – Total span: 20m

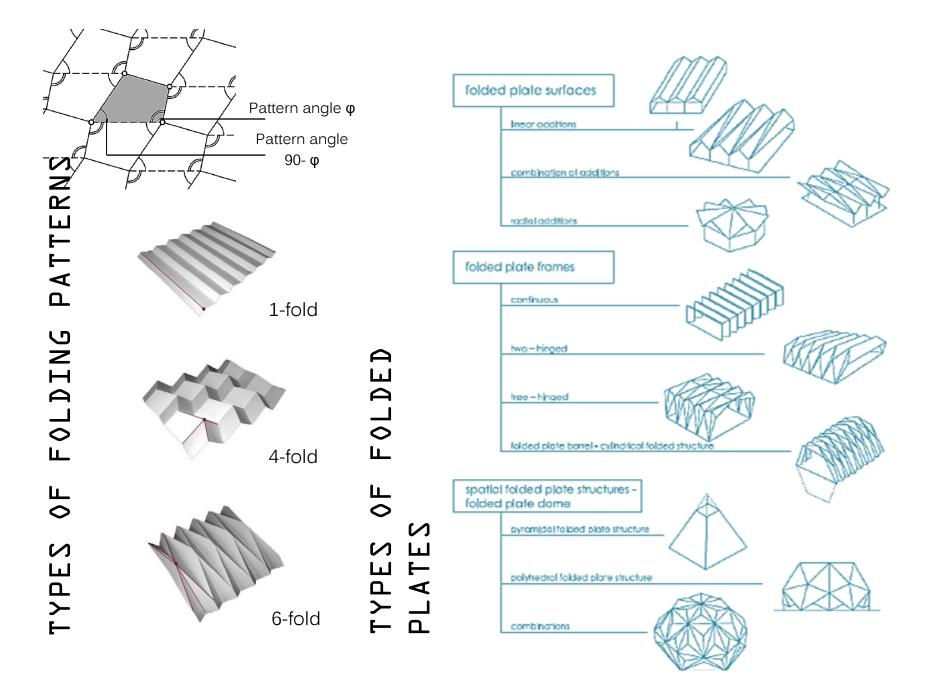


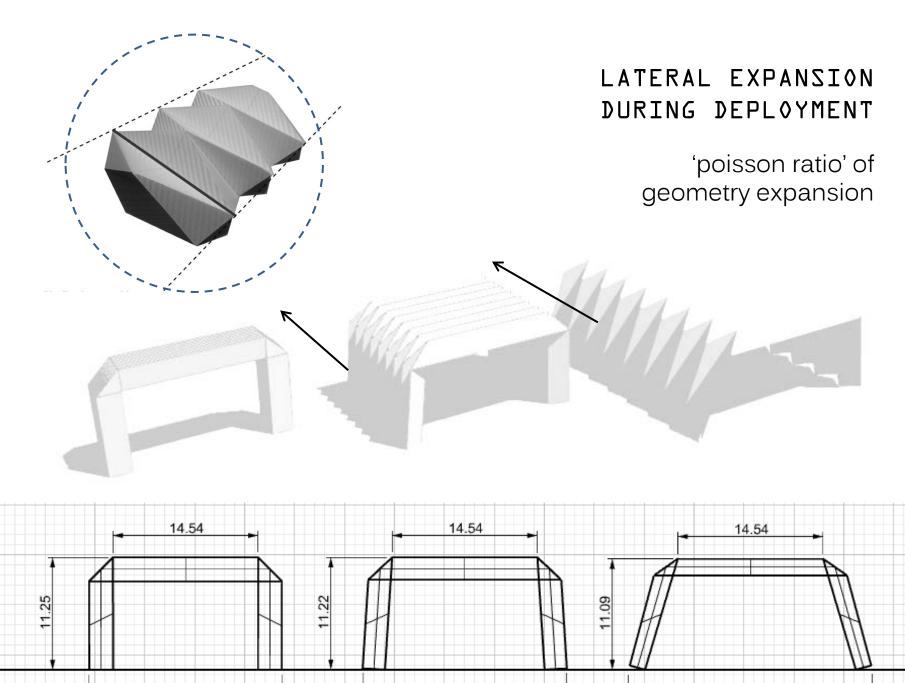


# PART 1: FOLDING PATTERN SELECTION

A folded plate structure is a threedimensional structure formed out of thin plate elements arranged in a manner to form a load bearing system. [J.Born,1954.]

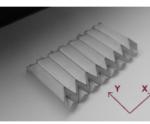


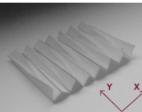


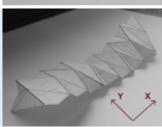


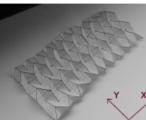
19.27 \_\_\_\_\_ 20.44 \_\_\_\_\_

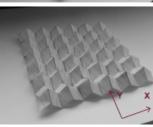
24.40











# FOLDING PATTERN COMPARISON

1. Simple 1-fold with hybrid endings

## TYPE: planar quadrilaterl pattern

quadrilaterl pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

This is the hybrid of a simple 1-fold pattern. It can effectively span over one direction, but is prone to bending. The extra folding around the ends helps the load distribution, avoiding large stress concentration around supports.

### 2. Triangular 1-fold

TYPE: planar

triangular pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

This is a triangular version of the above. Due to the triangulations, this structure copes with bending better. It has been used in architectural applications, such as the new Terminal of Pulkovo International Airport.

### 3. Diamond G-fold (single)

### TYPE: planar

triangular pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

This is one of the few triangular meshes with zero lateral expansion during deployment. This makes it ideal for rail systems. The traingulation of the surface also increases structural height, reduces the bending stresses.

### 4. Miura-Ori 4-fold

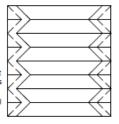
TYPE: planar quadrilateral pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

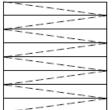
This is one of the most typical examples of folding patterns for architectural applications. Such a level of tesselation of a flat surface cannot achieve stiffness in the case of hinged connections. Unstable structure.

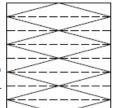
### 5. Eggbox 4-fold

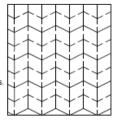
TYPE: planar quadrilateral pattern non- developable - no DOF rigid motion span direction: Y and X

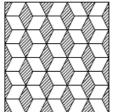
This is one of the very few non-developable patterns, since it involves cutting it doesn't directly fall under the definition of origami. Much like the Miura-Ori, it looses stiffness in both span directions, when hinged connections are used.

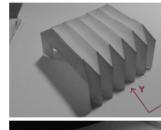


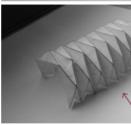


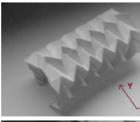


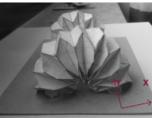












### 6. Simple 4-fold

TYPE: frame / tunnel quadrilateral pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

This is the simplest form of origami-frame structure. Folds across the span direction increase stiffness, replacing beams.



TYPE: planar/ spatial (dep. on folding angle) triangular pattern developable - 2 DOF rigid motion span direction: X, Y / retractable on Y

This is a rather complicated pattern, unsuitable for structures, as it provides very low stifness in both span directions. Performs better if applied on a curved surface but sitil unstable in combination with hinged connections.

### 8. Diamond 6-fold

TYPE: frame / tunnel triangular pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

This is a simple pattern, in which, the pattern angle determines the curvature of the final surface assumed, and the element size the coarseness of the folding. Although its shape and folding provides stifness, behavior with hinged connections is uncertain.

9. Miura-Ori (uneven)

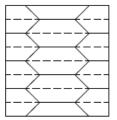
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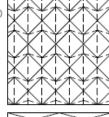
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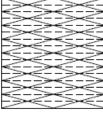
10. Hybrid Dome

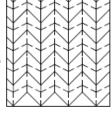
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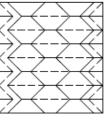
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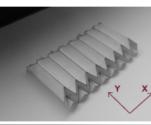


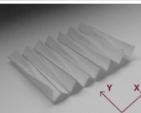


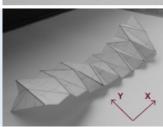


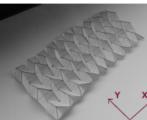


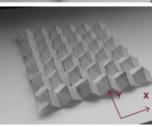












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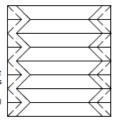
quadrilateral pattern developable - 1 DOF rigid motion span direction: Y / retractable on X

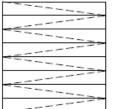
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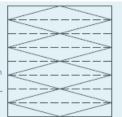
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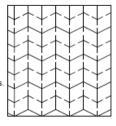
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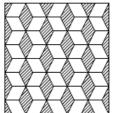
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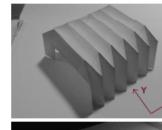


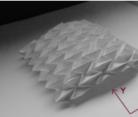


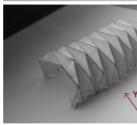


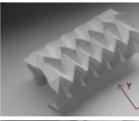














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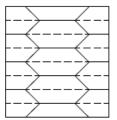
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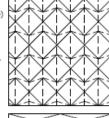
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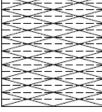
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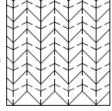
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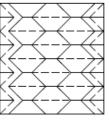
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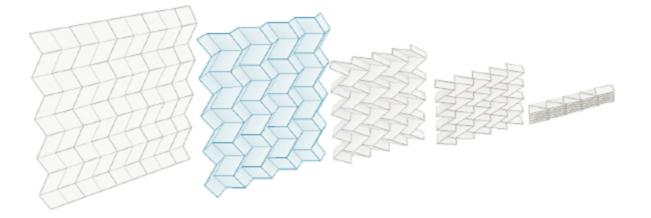




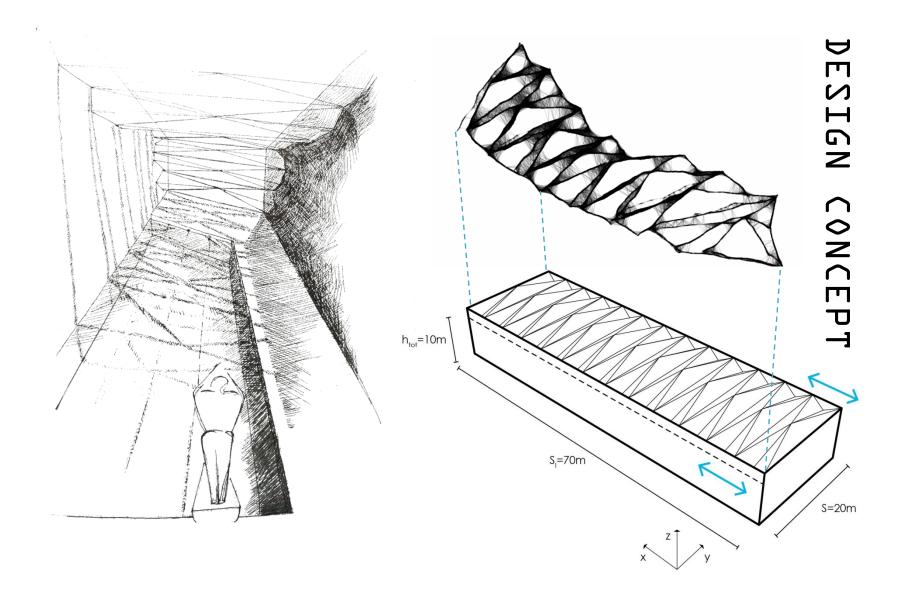


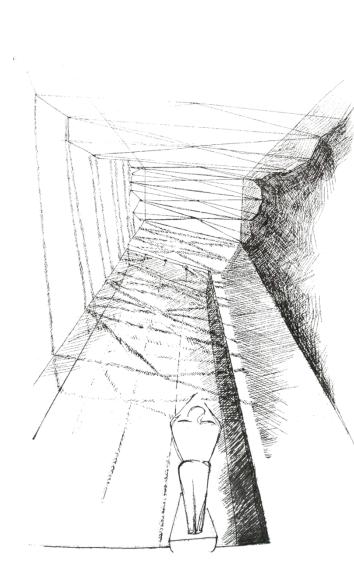


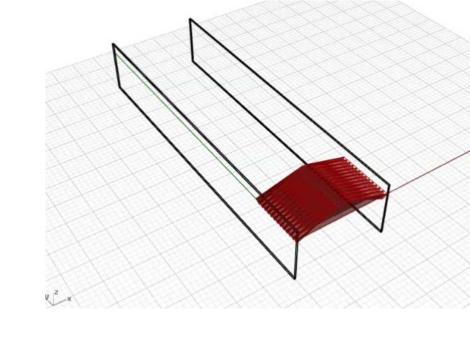


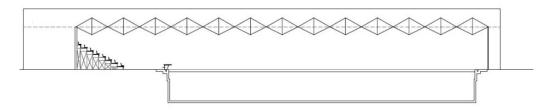


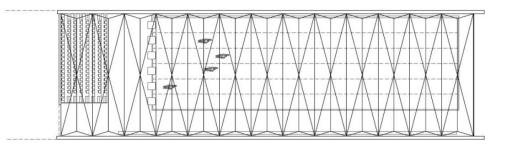
# PART 2: DESIGN DEVELOPMENT



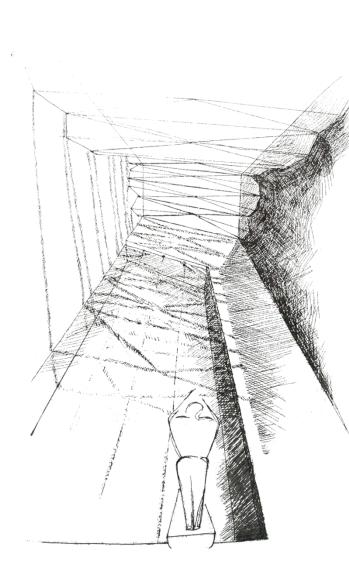


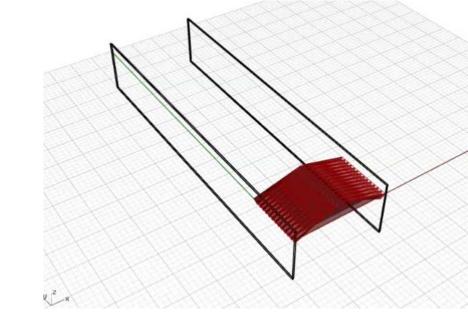


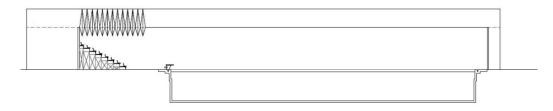


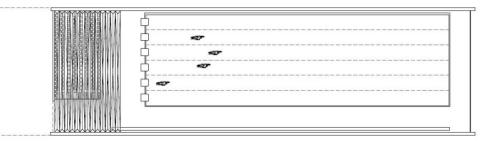


FULLY COVERED STATE

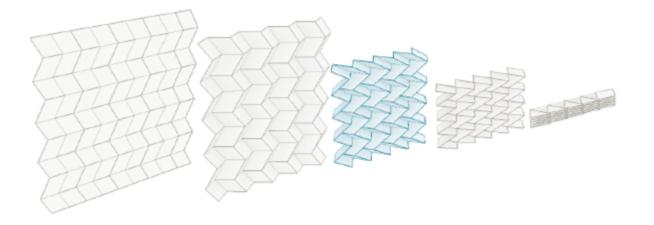




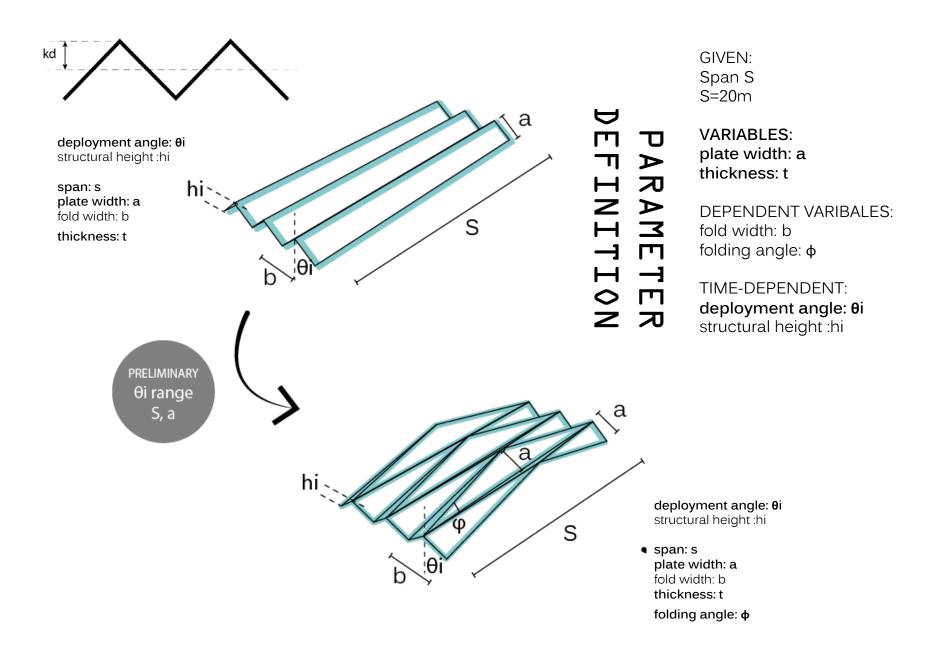




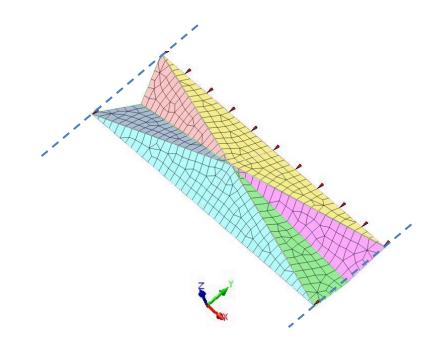
# FULLY OPENED STATE



# PART 3: STRUCTURAL ANALYSIS



		SAFETY FACTOR	TOTAL	for hand calculation
	[KN/m2]		[KN/m2]	[KN/m]
Self weight	1.0591182	1.2	1.27094184	7.62565104
Live load (snow)	0.7	1.6	1.12	0.935243237
Wind suction	-0.08	2	-0.16	-0.133606177
Snow + Wind	surface	e load:	0.96	0.801637061
Wind load (H)	0.2	2	0.4	
	0.2	2	0.4	
	0.13	2	0.26	
SI	LS q= 6.87243313 [KN/m]	]	ULS q=	8.427288101 [KN/m]
S	LS q= 6872.4331 [N/m]		ULS q=	8427.288101 [N/m]



# BOUNDARY CONDITIONS

PINNED SUPPORTS: one side: translation on x / z other side : translation on z

SUPPORTS ALONG LONG EDGES : translation on y

HEAT STRENGTHENED GLASS	PROPERTIES		7 7
	Value	Unit	
Density	2400	kg/m	ך ו
Young's modulus	67	GPa	
	6,70E+07	KN/m <sup>2</sup>	
	6,70E+10	N/m <sup>2</sup>	1 [
Tensile strength	50	Мра	
	5,00E+07	N/m <sup>2</sup>	
with safety factor (global): 1.2	4,17E+07	N/m <sup>2</sup>	- - /
	1,88E+06	N/m	J –
Yield strength	33-38	MPa	
Tensile strain to failure	0.05	%	
Compressive modulus	63.8-70.4	GPa	]
Compressive strength	390	MPa	-
	3,90E+08	N/m <sup>2</sup>	
Bending strength	45	Mpa	
Flexural modulus	70-74	GPa	
Flexural strength	40-45	MPa	
Shear modulus	27-29	GPa	
Bulk modulus	37-40	GPa	
Poisson ratio	0.22-0.24		
Fracture toughness	1-1.3	MPa/m <sup>2</sup>	
Thermal shock resistance	up to 130	С	
Permittable deflection =\$/60	0,3333	m	

Material safety factor : 1.2

•Permittable stress levels / deflection

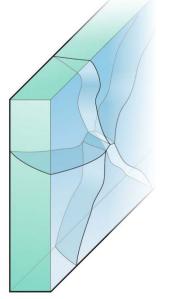
• Redundancy : extra layer of glass not included in calculations

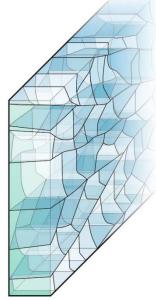
• Use of laminated heatstrengthened glass • Damage sensitivity,  $\Sigma$ : vulnerability based on probabilistic failure causes

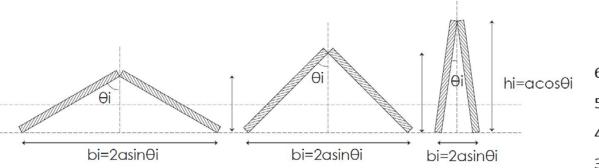
• Relative resistance r: ratio between actions on and resistance of elements

• Redundancy : margin between damage and failure and failure and collapse

• Fracture mode: breakage and injury potential







# FOLDING ANGLE Əi

Moment of inertia ly ly =  $2t\cos\theta$ i hi3 /12

First moment of area Sy = ta2 sin0i

Deflection w = 5ql4/384Ely

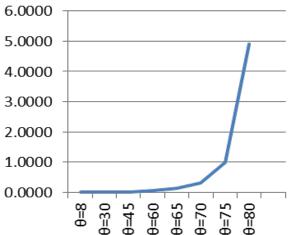
Bending moment middle M = ql2 / 8

Maximum flexural stress  $\sigma = Mz / Iy$ 

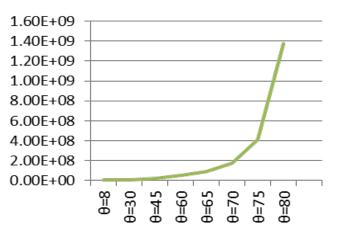
Principle Force max.  $F = \sigma t$ 

Maximum shear stress  $\tau = VQ / Ib$ 

# DEFLECTION



MAX STRESS



# t=4\*0.15 a=3m Span 20m

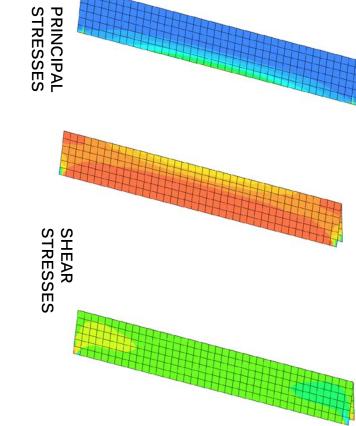
**Reaction force** Deflection Bending moment middle 6,78E+04 0,00242

### Maximum comress. stress -5,44E+06 Maximum tensile. stress 5,04E+06 Shear stress 1,07E+06

PRINC.FORCE [N/m] -2,4E+05 2,27E+05

# SIMPLI FIED F FOLD

DEFORMATION



t=4\*0.15 a=3m Span 20m

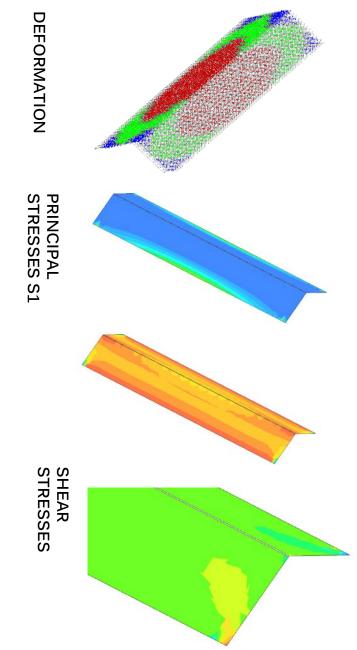
**Reaction force** 

0,0107
s -1,14E+07
9,05E+06
1,95E+06

1,26E+06 0,0107

> PRINC.FORCE [N/m] -5,13E+05 4,07E+05

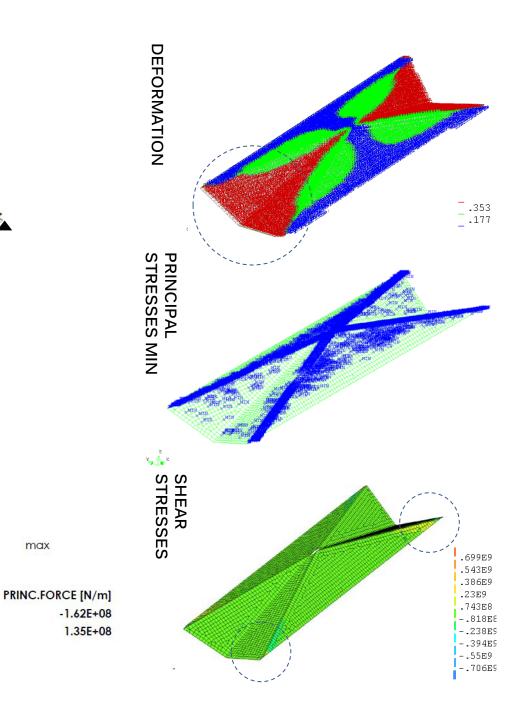
# SIMPLIFIED F FOLD

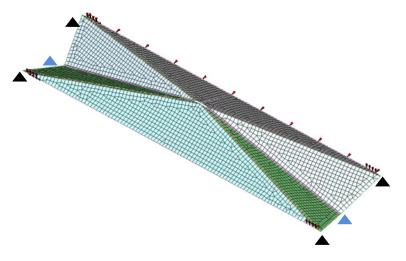


# PLANAR L-FOLD

deployment angle	θ=60	
Reaction force	N	
Deflection	m	0.500
Bending moment middle	Nm	
Maximum comress. stress	N/m2	-3.59E+09
Maximum tensile. stress	N/m2	2.99E+09

max



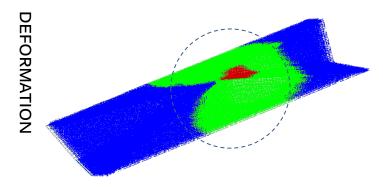


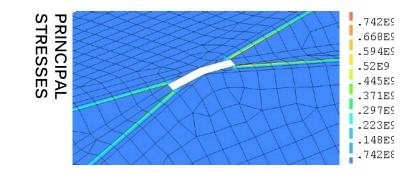
# ADDITIONAL SUPPORTS

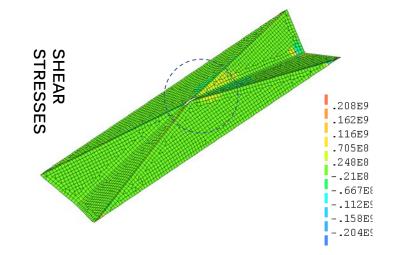
deployment angle	
Reaction force	
Deflection	0.1 at to
Bending moment middle	
	PRINC.
Maximum comress. stress	-5.99E+08
Maximum tensile. stress	3.43E+07
	1.10E+07
Shear stress	1.96E+08
	1.48E+07

op

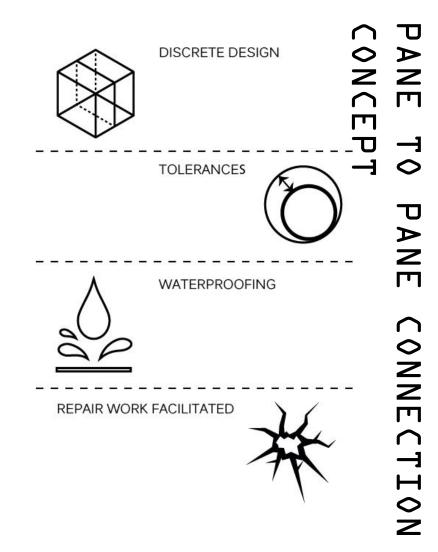
PRIN	C.FORCE [N/m]	
9E+08 -2.70E+		
3E+07	1.54E+06	
0E+07	4.95E+05	
6E+08		
8E+07		

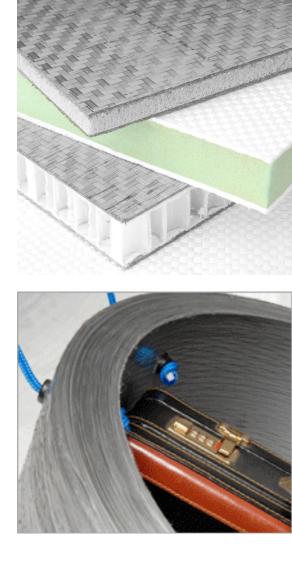




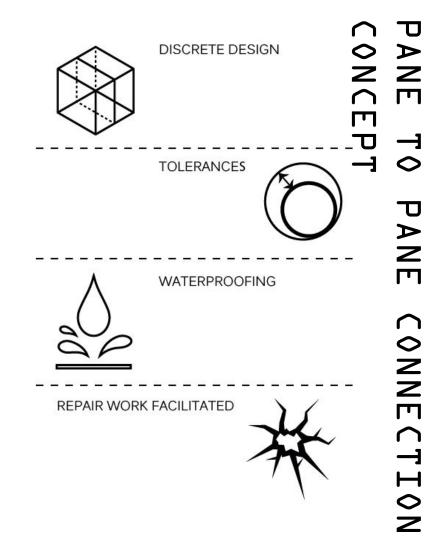


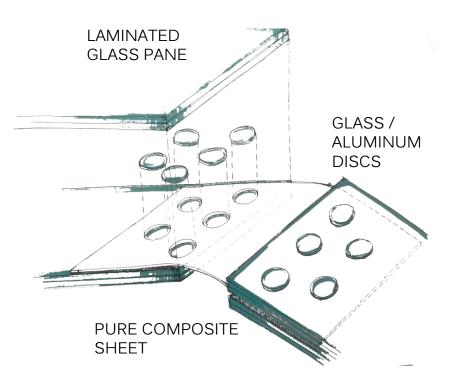
# PART 4: HINGED CONNECTION DEVELOPMENT

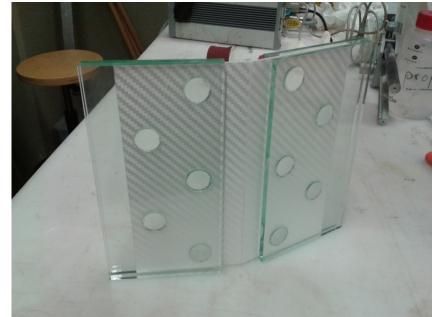


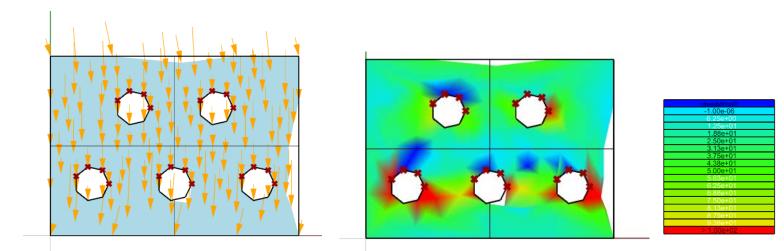


	Test method	Value	Unit
Bulk density	ASTM D792	0,78	g/cm <sup>3</sup>
Young's modulus	ISO 527-4	5,5	GPa
Tensile strength	ISO 527-4	200	Мра
Tensile strain to failure	ISO 527-4	9	%
Flexural modulus	ISO 178	4,5-5,5	GPa









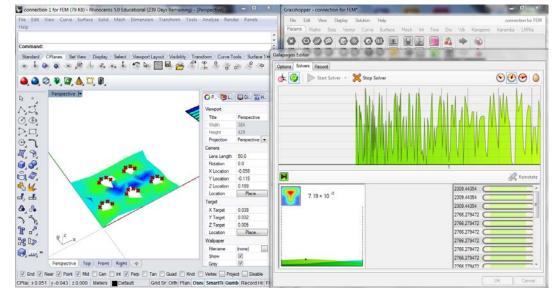
# CONNECTION OPTIMIZATION

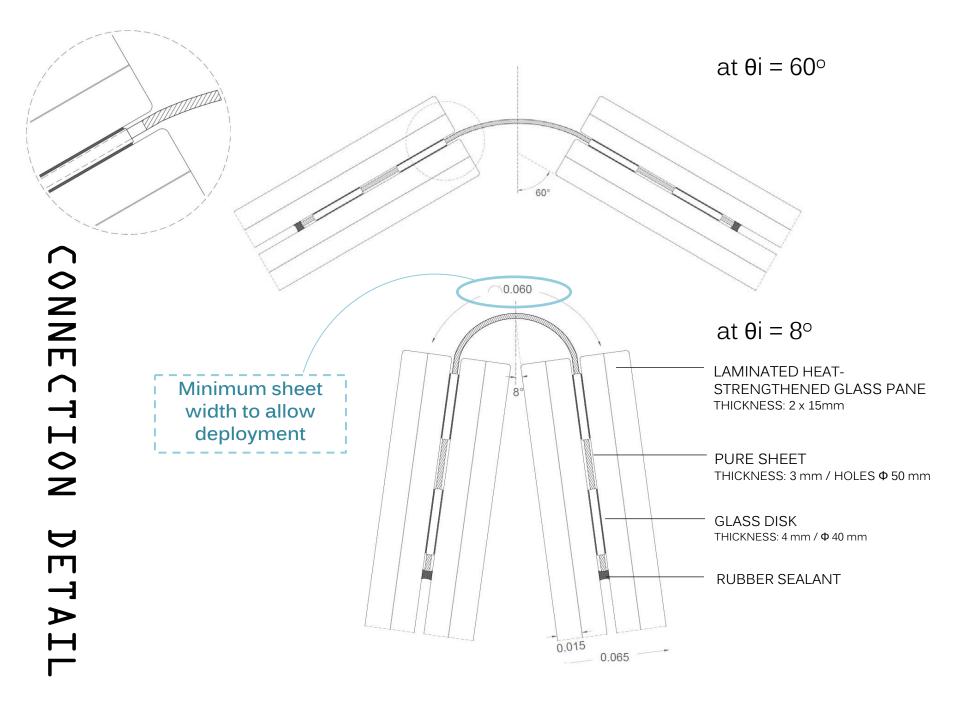
Based on steel construction regulations

Looking for adequate dimensioning of the discs

Minimalize induced Principal forces

♥ DISK DIAMETER : 20 mm





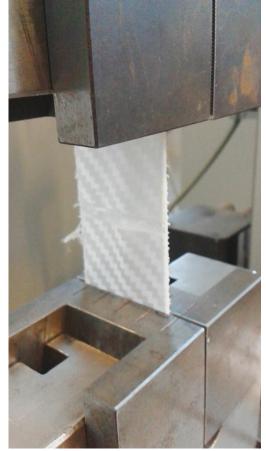
# LAB TESTS

MATERIAL PROPERTIES:

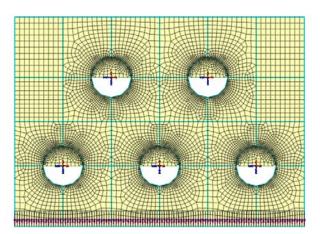
• Fatigue test on PURE sheet (notched and un-notched version)

CONNECTION PROPERTIES:

 Pull-out test on PURE sheet – glass disc connection principle (different sheet thicknesses)



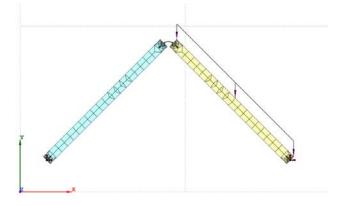




# FEM TESTS

# CONNECTION PROPERTIES:

- Pull-out test on PURE sheet glass disc connection principle (different sheet thicknesses)
- Deformation upon out-of-plane compression (different loadcase scenarios)



# FATIGUE TEST

NOTCHED

MAXIMUM STRAIN h0 [mm] : 92,5

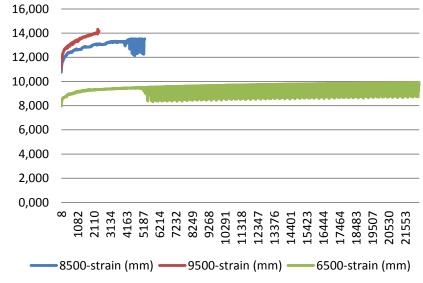
12,000 10,000 8,000 6,000 4,000 2,000 0.000 566,8 1862,8 3036,6 646,3 965,4 .261,3 2150,5 2443,8 4550,3 4855,4 5804,2 344,7 3335,9 3941,4 5164,7 5,7 2736,1 3634,2 4242,2 5475,2 6123,1 6435,2 4500-strain (mm) 5500-strain (mm) 3500-strain (mm)

# strain/time 1sec

# UNNOTCHED

MAXIMUM STRAIN h0 [mm] : 91,960

# strain/time 1sec



1.5 mm PURE sheet UN-NOTCHED : 90 MPa



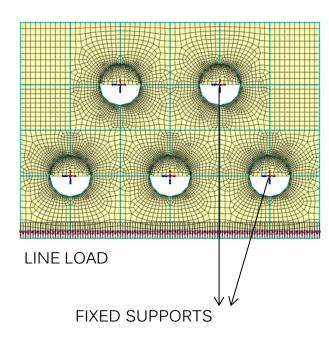




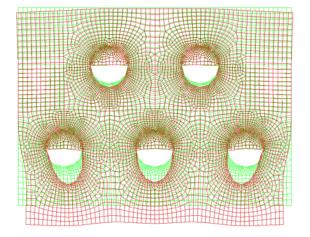
36.73 MPa
73.62 MPa 22 Mpa Shear strength Glass/glass adhesion

ľ	SINGLE 1.5mm /1 DISK Specimen Description 1 glass breakage around disks - PURE deformed 2 PURE deformed + failure 3 glass breakage- glue did not fail 4 glue failure	Standard travel mm 8.954 11.539 9.344 11.081	Standard force N 2354 2557 2129 2497	
-		Average force	2384	N
2	DOUBLE 1.5mm /2 DISK Specimen Description 5 glue failure+ glass disk breakage(between disks and with plate) 6 glue failure d sk/pane-small PURE deformation 7 glue failure d sk/pane-small PURE deformation-glass breakage (disk)	Standard travel mm 8.228 14.239 7.670	Standard force N 3955 4913 4132	
ion	(dif. pat)8 vlue failure disk/pane-small PURE deformation-glass breakage (disk)	10.744 Average force	<b>4207</b> 4417	N



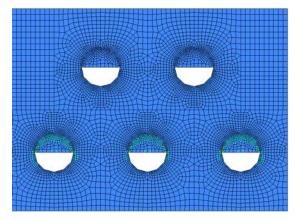


# DEFORMATION



PRINCIPAL STRESSES IN-PLANE

.639E8
.575E8
.511E8
.447E8
.383E8
.319E8
.256E8
.192E8
.128E8
.639E7



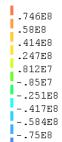
# DISK DIAMETER : 20 mm

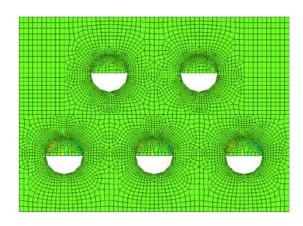
	1	.5mm PURE:	3mm PURE:	5mm PURE:
displacement	general	0.00008 [m]	0.00004 [m]	0.000025 [m]
a	round the holes	0.00007 [m]	0.000036 [m]	0.000022 [m]
prinicipal stress max		3.76E+07 [N/m2]	1.88E+07 [N/m2]	1.13E+07 [N/m2]
a	round the holes	37.60 [Mpa]	18.80 [Mpa]	11.30 [Mpa]
	on the edge	5.24E+06 [N/m2]	2.61E+06 [N/m2]	1.57E+06 [N/m2]
		5.24 [Mpa]	2.61 [Mpa]	1.57 [Mpa]
principal force max		5.64E+04 [N/m]	5.64E+04 [N/m]	5.65E+04 [N/m]

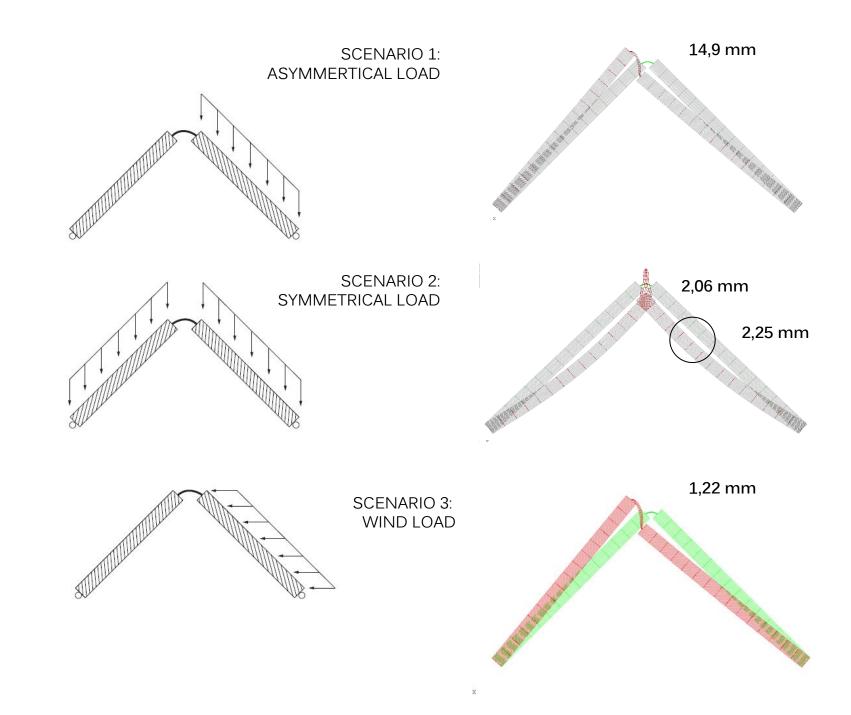
# DISK DIAMETER : 40 mm

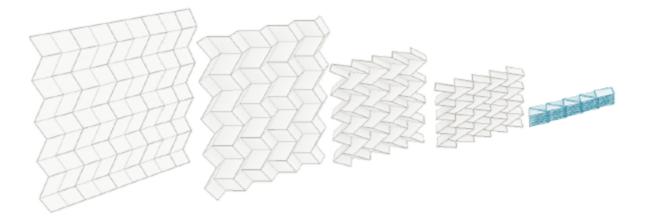
		0mm diameter
		3mm PURE:
displacement	general	0.00007 [m]
	around the holes	0.000057 [m]
prinicipal stress max		7.03E+07 [N/m2]
	around the holes	70.30 [Mpa]
	on the edge	2.59E+06 [N/m2]
		2.59 [Mpa]
principal force max		6.12E+06 [N/m]

# m2] m2] SHEAR m2] STRESSES m2]

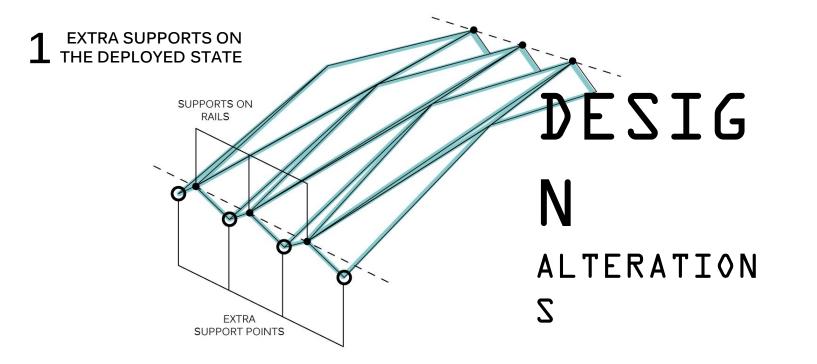


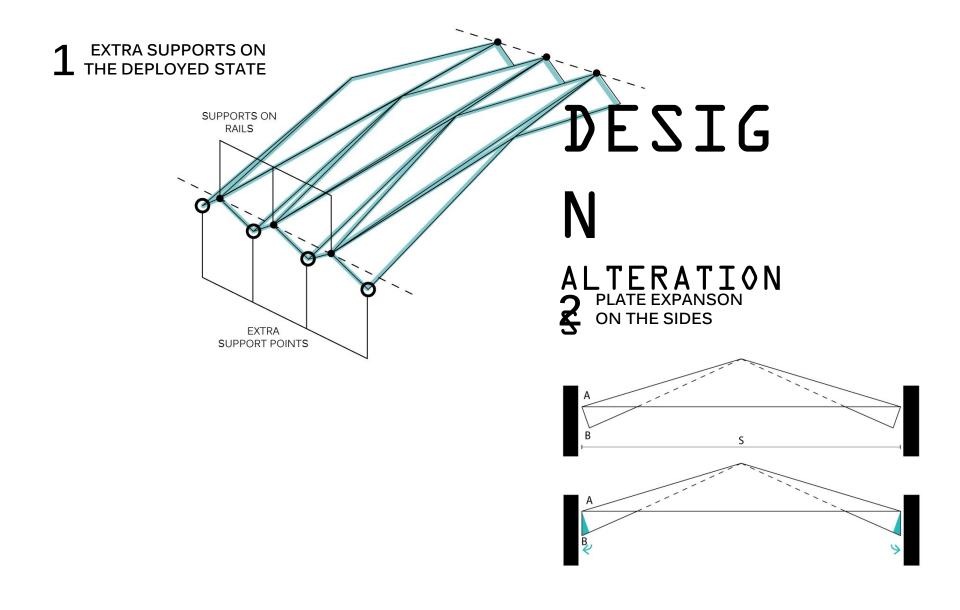


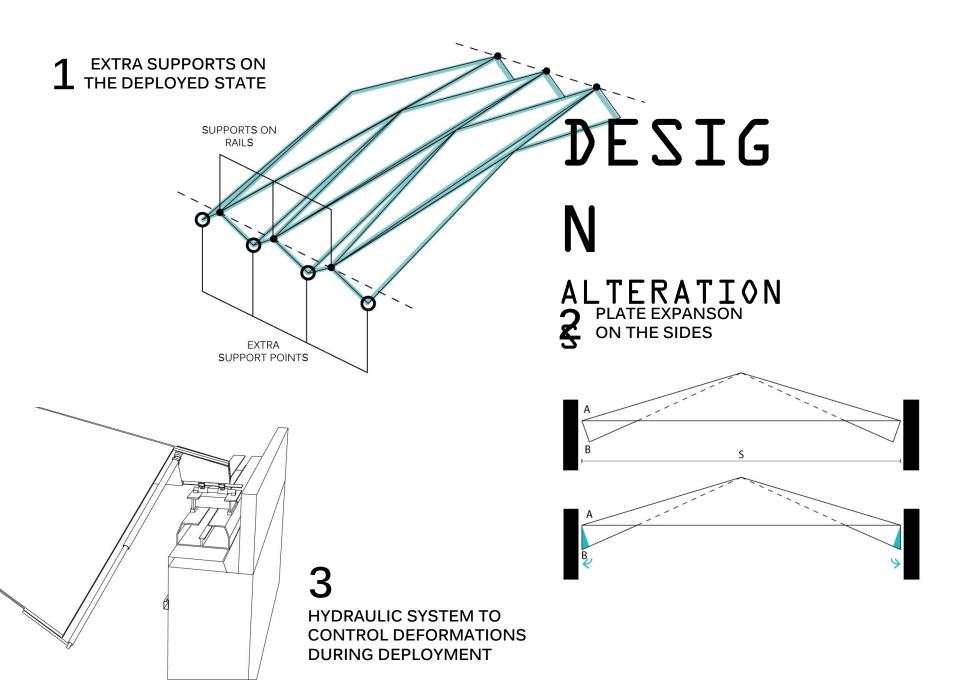


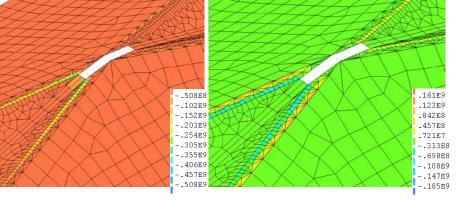


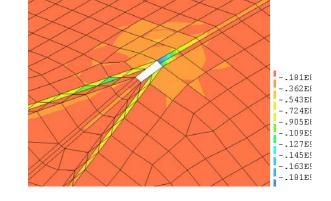
### PART 5: FINALISATION



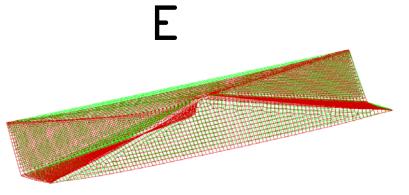




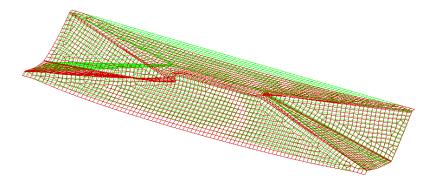




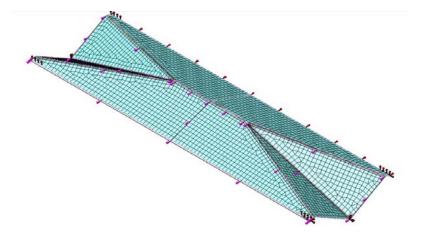
## BEFOR F



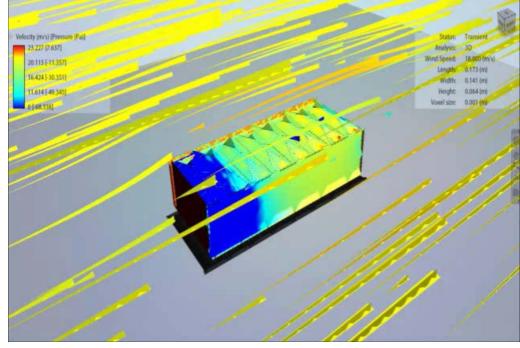
## AFTER



deployment angle Reaction force Deflection Bending moment middle	<b>θ=60</b> Ν m Nm	0.1	at top	2.83E+05 2.83E+03 0.04	at supports at the edges at top	NOT EVENLY DISTRIBUTED ALONG EDGE
		PRINC.FORCE [N/m]			[Mpa]	PRINC.FORCE [N/m]
Maximum comress. stress	N/m2	-5.99E+08	-2.70E+07	-1.38E+07	-13.8	-6.21E+05
Maximum tensile. stress	N/m2	3.43E+07	1.54E+06 close to the edges	1.28E+07	12.8	5.76E+05 at the mid bottom
		1.10E+07	4.95E+05 mid edge on themain panels	1.79E+08	179	8.06E+06 at the connection on top
Shear stress	N/m2	1.96E+08	peaks at the top	3.82E+07	38.2	at the connection at the top
		1.48E+07		-4.54E+07	-45.4	around suports

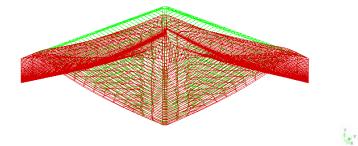


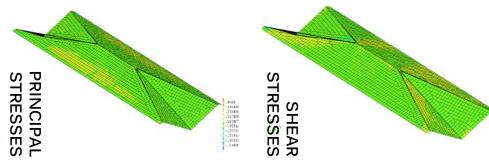
CHAMFERED 6FOLD VERSION deployment angle	WIND LOAD (V	(WIND LOAD+SELFWEIGHT)		
Reaction force	N			
Deflection	m	0.030	AT TOP	
Bending moment middle	Nm			
			PRINC.FORCE [N/m]	
Maximum comress. stress	N/m2	-6.80E+07	-3.06E+06	
Maximum tensile. stress	N/m2	1.30E+08	5.85E+06	
Shear stress	N/m2	2.97E+07		

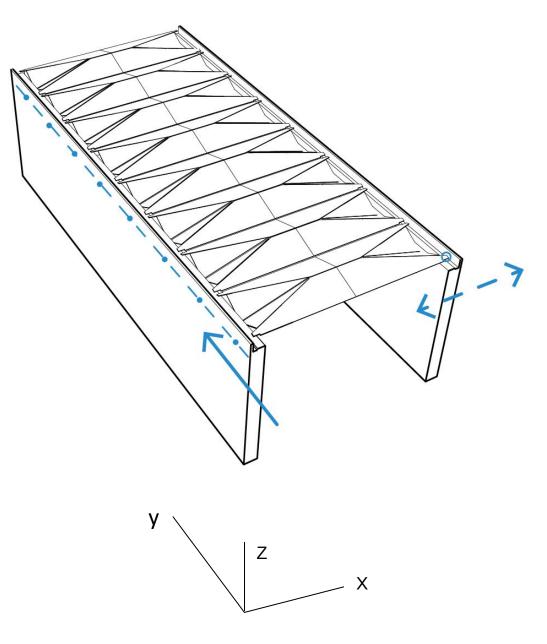


## WIND LOAD EFFECT







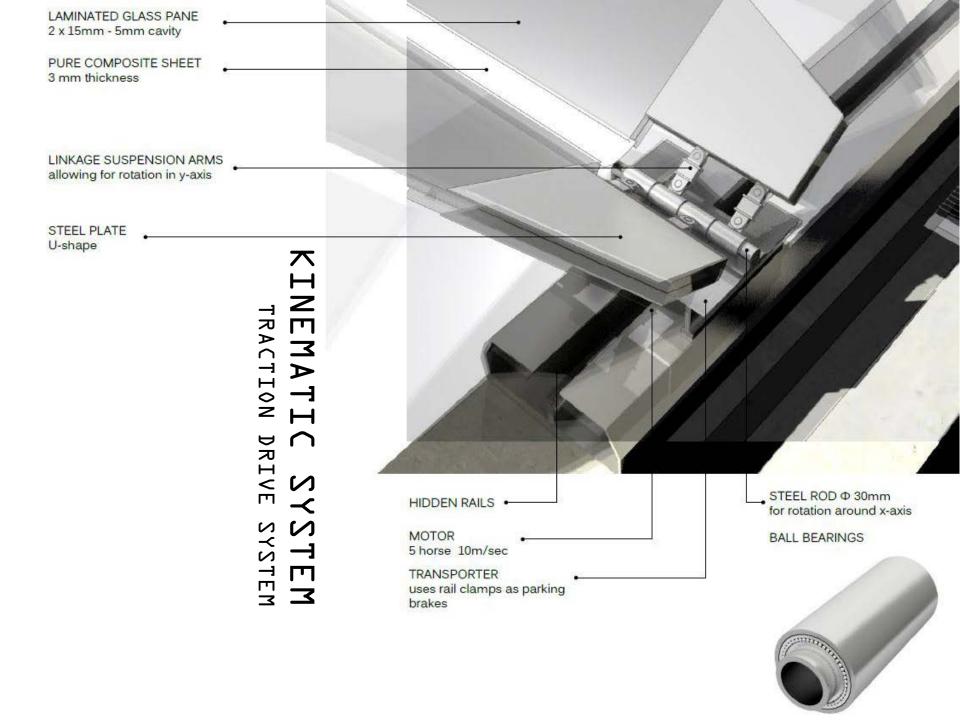


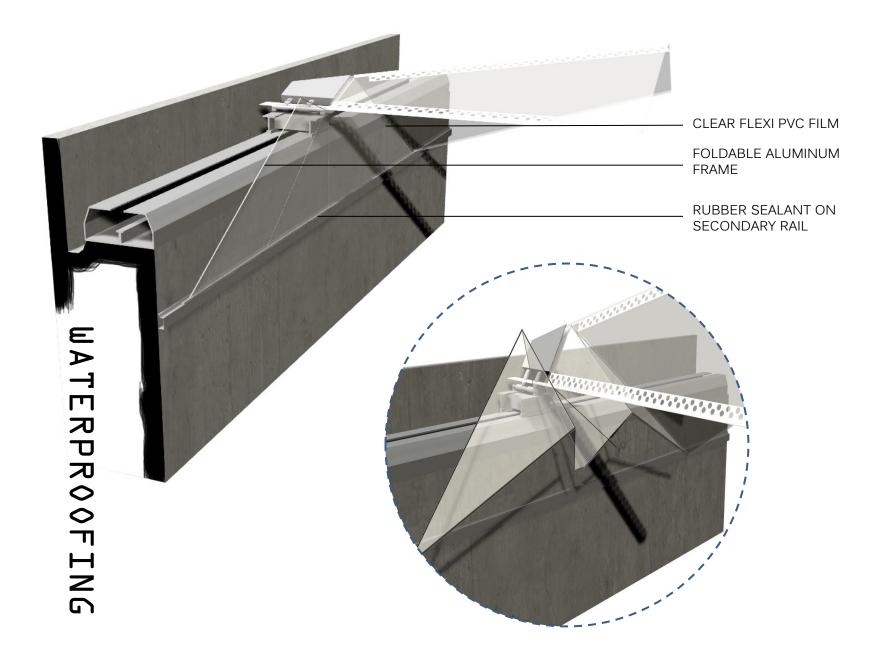
1. Translation on y-axis allowed during folding process and hindranced at the enclosed state. KINEMATIC

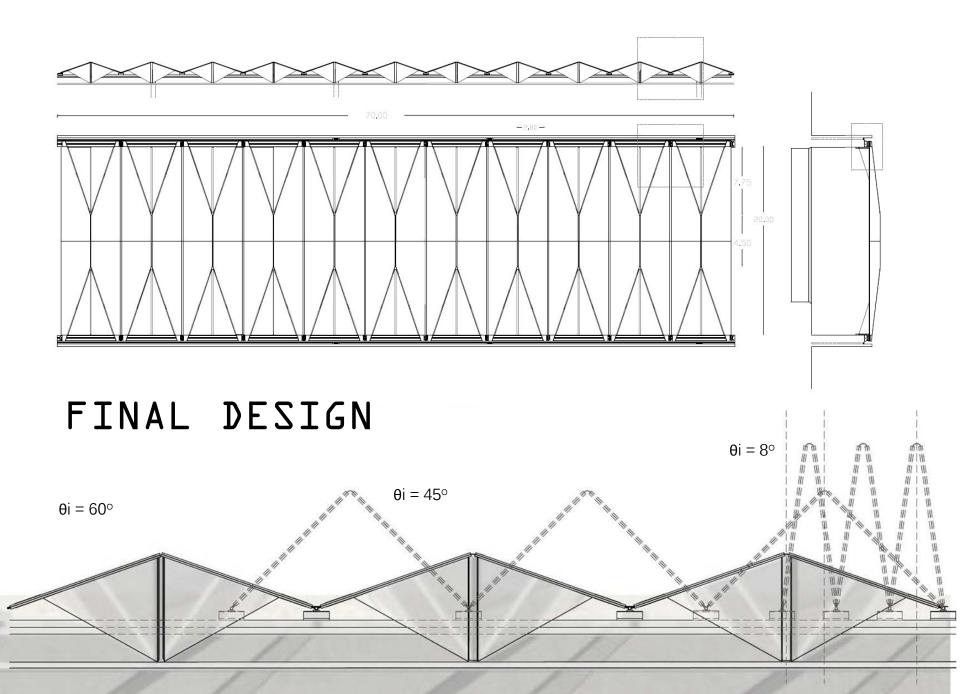
SYSTEM

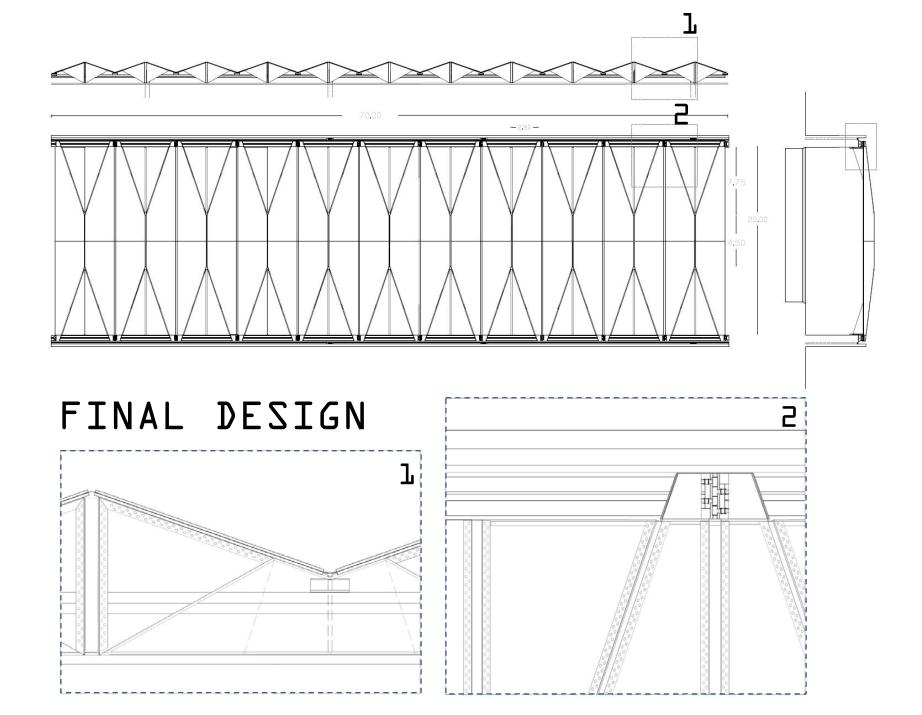
2. Tolerances on x-axis to avoid moments caused by the induced deflection.

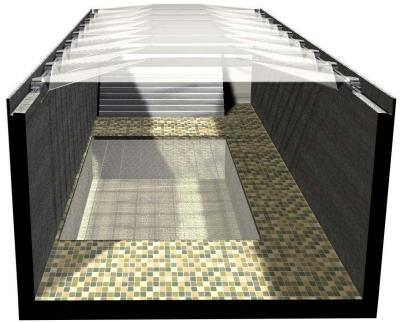
3. Increasing the area of support as much as possible to avoid stress concentration











# CRITERIA FVALIATION

### DESIGN CRITERIA

Provide natural lighting and maximum transparency Self-supported glass plate structure (no frame) Deployable on one side (fully adaptable) Feasibility

### STRUCTURAL CRITERIA

Controlled element deformation + stress levels (all

phases of deployment)

General shape stability (all phases of deployment)

Glass element redundancy

Damage sensitivity - Fracture mode – Safety factors

#### DETAILING CRITERIA

Discrete design- Invisible connection

Tolerances

Restriction of gaps- waterproofing

Repair work facilitated

BC maintained by connection detailing

