

Jib Rest Design for Crane Barge Type CB6324

A New Design Approach

Master Thesis
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Abstract

The Damen Crane Barge type CB6324 is a transshipment barge, designed to operate in harbours, inshore and in near coastal waters worldwide, mainly for the on- and offloading of bulk carriers. In the second generation of this crane barge, optimization of the equipment and overall design is sought, increasing usability of the barge and reducing overall costs. The same is sought in the design of a jib rest, as it is found to be heavy and cost reduction can be realized by reducing weight. Additionally, it is suspected that the jib rest can be hit by the crane grab during operation.

The new optimized design is restrained by the need to be able to withstand the load of the crane jib in travel conditions overseas and during its complete operational lifetime. As the barge needs to be able to operate worldwide, identification of Ultimate Limit Loads is key. Lloyds Register provides a conservative method to determine the loads acting on the floating crane barge and its equipment, based on extreme weather conditions. As specific data on the CB6324 is available, a different calculation method is proposed, using motion responses, sea state data and design criteria as an input. The method is based on probability of encountering a sea state, the probability of non-exceedance of a wave in such a sea state, and the operability of the barge, to predict the maximum probable accelerations encountered.

Finally, analysis of two first generation crane barges built, show some fatigue crack forming on the jib rests during transport to the operational locations. This signifies an interest of a fatigue analysis and estimation, adding up to three design objectives:

- A) Finding load cases using modeled motion responses and sea state probability
- B) Finding a new jib rest design that complies to the criteria set by Damen and with the load cases found with design objective A
- C) Finding an estimation on the fatigue lifetime on both the first and new generation jib rest

For the determination of the accelerations, a tool is developed, which automates the calculations for the maximum expected loads in worldwide near shore operation and transport. The barge motion responses in regular waves are determined and used as the primary input. Secondary input is composed of wave data, direction and spreading. Wind speed, roll, pitch, flooding angles and (bending) stresses in the crane pedestal are used as additional criteria. As an output, it gives an overview of operability over the world, along with a selection of the maximum expected accelerations in the critical directions.

Parallel to the load calculations and development of the tool, the design of the new jib rest is made. Starting with a concept and building up detail as more information is brought to light in every calculation step. The resulting design is composed of a 15 ton, hinged, steel A-frame with a weight reduction of approx. 5 ton compared to the first generation. Fatigue prediction shows a significant improvement on the fatigue lifetime, as a transport load case estimates the new design spending approx. 10% of its fatigue lifetime in comparison to approx. 200% in the old design.

More weight reduction could be achieved by further research on whether the f_r is a reasonable restriction for this barge. The stiffness of the jib rest is increased significantly to comply to this criterium in dropped down position. This weight reduction can only be realised if a less conservative load case calculation method is used, like the one proposed in this thesis.

Afstuderen is net watertrappen

Preface

Workspace Coupling and Python

If there is one thing I have adapted from programmers, is the drive to automate everything that is automatable, especially calculations. Some have asked me if it is really beneficial to spend time on automating a calculation, as it might be used just a couple of times and doing it by hand would suffice. A good argument. But. Engineering relies on software. From communication to numerical modelling. It uses software tools to facilitate in the design, often acquired in large and expensive packages. In my opinion, (structural) engineering is still done in a somewhat oldfashioned way and is falling behind on the rapid development strategies of software design. There is a lot that can be learned from this other discipline. If we are relying so much on this software, we should at least have an idea how the software is structured and see how calculations are automated. This way, we might find a faster way of improving the engineering design cycle.

Making an attempt to revolutionize the engineering design strategies might be a little far fetched to do next to a design thesis. During this thesis, a reasonably smaller step towards automation is done. A lot of data about this thesis' subject - the crane barge - is scrambled over different drawings and manuals. And there are several software tools used in the process, all using the same parameters and information. Therefore, next to a tool described in this thesis, a small linking script is built, to keep all parameters at one location. It handles all parameters, variables, switches, inputs and outputs, coupling the entire workspace. Programs and programming languages like Excel, Latex, Eclipse, Tikz, PRECAL_R, Multiframe and Autocad are all linked together, minimizing the cases of redoing. For me, this has fulfilled the need for automation.

Instead of using a large and paid package to do make the code in, I have chosen to use and learn an upcoming and open source programming language; Python. In hindsight, learning a new programming language at the beginning of a thesis while programming a model, designing a jib rest and mainly gaining more knowledge about structural engineering and sea keeping, might not have come at a perfect time. Nevertheless, I am glad I have taken this opportunity to do so.

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There are quite some people who helped me complete this design thesis, by giving guidance, support and ideas. I would like to thank them, and some in particular:

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Charlotte de Jong, for her support in these busy times. Now we can close a big chapter and start with new adventures!

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Glossary

- A_{grab} 18
- A_{total} total added mass matrix 32
- A_{tot} crosssectional area of the barge 13
- $A_{x,z}^{jib}$ Horizontal area jib side 46
- A_z response amplitude 32
- B_{total} total added damping matrix 32
- CI CI 10
- CR CR 10
- C_{total} total added restoring matrix 32
- F_{pre} pretension force according to Liebherr (10% SWL) 46
- F_w^{YH} force due to wind pressure on jib 46
- F_x^{YH} force due to gravity in x direction for load case YH 46
- F_y^{YH} force due to gravity in y direction for load case YH 46
- F_z^{YH} force due to gravity in z direction for load case YH 46
- $H_s[m]$ significant wave height 44
- H_s significant wave height 35, 37
- H_{ba} height of the barge 13
- I_{xx} moment of inertia over x-axis 12, 13
- J_{xx} torsional moment of inertia over x-axis 13
- L_{ba} length of the barge 14
- OP operability 39, 41
- PA parkability 39, 41
- $R(H_s, T_z, \omega)$ response spectrum 37
- $S_\zeta(H_s, T_z, \omega, \mu)$ wave spectrum 37
- T_z zero up crossing period 35, 37
- $V(p, w)$ weighed criteria score 23, 27, 29, 31
- W_{ba} width of the barge 13, 14
- W_{tot} W_{tot} 10
- X surge 32
- Y sway 32
- Z heave 32
- Φ roll 32, 44
- Ψ yaw 32
- Θ pitch 32, 44
- \bar{F} total excitation vector 32

\bar{x} motion response vector 32

ϵ response phase 33

γ_σ acceleration - stress ratio 53

$\gamma_{f,E,a}$ environmental load factor in load combination a 46

$\gamma_{f,E,b}$ environmental load factor in load combination b 46

$\gamma_{f,G,a}$ permanent load factor in load combination a 46

$\gamma_{f,G,b}$ permanent load factor in load combination b 46

γ_m material load factor 46

γ non-dimensional peak shape parameter 36

κ_x^{jib} roll radius of gyration 33

κ_y^{jib} pitch radius of gyration 33

κ_z^{jib} yaw radius of gyration 33

κ radii of gyration 32

λ_b buckling ratio ix, 51

λ_b lower boundary of allowable buckling ratio 11

$\mu_{p,j}$ ratio between loads transferred to jib rest and crane pedestal 46

μ wave direction 33, 37

ω_{slew} Slewing speed 18

ω_{slew} 18

ω wave frequency 32, 33, 36, 37

ϕ^{grab} maximum admissible heel for operation (Grab) 11

ϕ_{flood} maximum heel allowable heel 10

ϕ 14

ρ_{air} density of air at 0 deg C 18, 46

σ_s spectral width parameter 36

σ_{fat} rough allowable fatigue stress estimation 10

θ^{grab} maximum admissible pitch for operation (Grab) 11

θ_w wave phase 32

ζ^* maximum consistency error 10

$a_{RP0,y,beam}$ Y-direction set as primary direction in RP:0, while under load of beam waves, defined as load case YB 43

$a_{RP0,y,head}$ Y-direction set as primary direction in RP:0, while under load of head waves, defined as load case YH 43

$a_{RP0,z,beam}$ Z-direction set as primary direction in RP:0, while under load of beam waves, defined as load case ZB 43

$a_{RP0,z,head}$ Z-direction set as primary direction in RP:0, while under load of head waves, defined as load case ZH 43

a_x^{YH} acceleration in x direction for load case YH 46
 a_y^{YH} acceleration in y direction for load case YH 46
 a_z^{YH} acceleration in z direction for load case YH 46
 b_a section width 13
 c_{beam} within the beam of the barge 9, 10, 23, 27, 29, 31
 c_{compl} complexity of the structure ix, 9, 10, 23, 27, 29, 31
 c_{deck} fatigue or structural impact on deck 10, 23, 27, 29, 31
 c_{fabr} fabrication time and costs 9, 10, 23, 27, 29, 31
 c_{fat} fatigue resistance 9, 10, 23, 27, 29, 31
 c_{flex} time and effort needed to move from resting- to working position 9, 10, 23, 27, 29, 31
 c_{grab} 18
 c_{insusc} insusceptible to cargo/bulk spillage 9, 10, 23, 27, 29, 31
 c_j drag coefficient of the jib 46
 c_{maint} maintenace needed including inspection intervals 9, 10, 23, 27, 29, 31
 c_{mass} weight of the structure 9, 10, 23, 27, 29, 31
 c_{mat} availability of the material 9, 10, 23, 27, 29, 31
 c_{mot} effect on motions of the barge in different sea states 9, 10, 23, 27, 29, 31
 c_{platf} provide inspection platform 9, 10, 23, 27, 29, 31
 c_{size} overall size of the structure 10, 23, 27, 29, 31
 c_{sturd} sturdiness of the structure 9, 10, 23, 27, 29, 31
 c_{trav} travel conditions 9, 10, 23, 27, 29, 31
 c_{winch} obstruction of the winches 10, 23, 27, 29, 31
 c_{zone} out of working zone ix, 9, 10, 23, 27, 29, 31
 d_{grab} width grab 18
 f_r lower boundary of allowable modal frequency i, 51, 52, 60
 $m_0(H_s, T_z)$ spectral moment 37
 m_{BW} ballasted mass to get
 glscog in the barge centre 14
 m_{CT} mass crane top 46
 m_{JR} approx. mass jib rest (6324JRv9) 85
 m_{grab} mass of grabber 18
 m_{jib} mass jib 33, 46
 m_{ped} mass pedestal 46
 $p_{N.E.}$ probability of non exceedance vii, 37, 42, 43, 56
 p_w^{YH} wind pressure for load case YH 46
 p unweighed criteria score rated from 0 to 1 23, 27, 29, 31

$r_{1/3}$ significant response 37, 39
 r_m maximum response 37, 39
 t_a section thickness 13
 t_{eff} effective thickness 12, 13
 u_w wind speed 44, 46
 v_{wind}^{max} maximum wind speed at parking condition 11
 v_{work}^{wind} maximum wind speed at working condition 11, 18
 w criteria weight 23, 27, 29, 31
 $x_{C.O.G.}^{jib}$ coordinates of jibs C.O.G. 33
 x_{mass} 14
 $y_{C.O.G.}^{jib}$ 33
 $z_{C.O.G.}^{jib}$ 33
ACK aft-centerline-keel 32
BWM best-worst method ix, 9, 10, 23, 27, 29, 31
CB6324 Crane Barge type 6324 vii, 2, 4, 5, 7, 9, 12, 20, 61
CFD cumulative fatigue damage theory 53
Class rules and regulations set by classification bureaus, e.g. Lloyd's Register (LR) or Det Norske Veritas and Germanischer Lloyd (DNV GL) 4–6, 35, 39, 45, 59
COG centre of gravity vii, 8, 32, 33, 47
DNV GL Det Norske Veritas and Germanischer Lloyd 4, 5, 10, 35, 46
FAT fatigue class 53
FCP fatigue crack propagation theory 53
FEM finite element modelling 7, 47, 57
FoS factor of safety 7, 45, 46
IIW International Institute of Welding 53
joint distribution 3-parameter Weibull probability density function and lognormal distribution vii, 35, 36
JR jib rest vii, 2–7, 9, 12, 18, 20, 21, 23, 24, 45, 47, 49, 51–53, 55, 59–61, 74, 85, 86
LH Liebherr 2, 10, 12
LR Lloyd's Register 4, 5, 10, 39, 43, 45, 47, 59, 61
MCDM multi-criteria decision-making 9
MDEM Marine Design Engineering Mykolayiv 57
MF Multiframe 49, 57
NX Siemens NX 57

P&B Damen Pontoons and Barges 2–5, 9, 10, 12, 18, 51, 59

PRECAL_R Marins PRECAL software to determine ship behaviour in waves 7, 32, 33

Python Python 3.0, a numerical programming language 35

RAO reponse amplitude operator vii, 6, 7, 32–35, 37, 39, 65

RHS rectangular hollow sections 12, 20

RP reference point vii, 32–34, 37, 38, 43, 47, 85

S-N curve stress cycle curve 53

SCF stress concentration factor 53

ULS ultimate limit state ix, 5, 43–47, 53, 57, 59

YH load case with y as primary direction and barge under beam waves 46

1 Introduction



Figure 1.1: Current crane barge design

The Damen CB6324 is a transshipment barge, designed to operate in harbours, inshore and near coastal waters worldwide, mainly for the on- and offloading of bulk carriers. It provides a flexible solution for the transfer of goods in areas which may prove difficult to reach by roads, or where waterdepth in harbours cannot dock larger carriers.

Damen is an international shipyard group, focusing on the market of ship building, repair and conversion. Their design philosophy is standardization; reducing development costs and delivery time, while maintaining quality. This results in a wide range of standardized products, from pontoons to tugs, platform supply vessels and high speed crafts.

Transshipment is considered a new niche market and the CB6324 is, with its second generation, at an early stage of standardization. It evolved from a standard pontoon with a Liebherr (LH) CBG 350 floating cargo crane, along with some accomodation and a set of (optional) equipment, under the care of the product group Damen Pontoons and Barges (P&B). The crane is relatively high with a total height of 31 meters and its jib at 20m, which needs to be supported by a JR in stowed condition during its down time and transport.

1.1 Problem Description

The first generation of the JR design is a stiff K-frame shaped support, positioned on the fore part of the barge. The jib itself is lengthened by an extension where it rests on the JR, moving the JR a couple of meters away from the working range of the crane. In the second generation, optimization of the equipment arrangement and overall design is sought, increasing the usability of the barge and reducing the overall costs. The same is sought in the design of the JR, as - despite of the jib rest extension - it is suspected that the JR can be hit during operation.

Additionally, the complete construction of the JR is found to be heavy, and cost reduction can be realised by reducing weight. An optimised design might allow this reduction, under the restraint that it will be able to withstand the load of the jib in travel conditions and during its lifetime. As the barge needs to be able to operate worldwide, identification of these loads is key.

Finally, two of the first generation crane barges built, show fatigue crack growth in the JR before starting its operation.

1.2 Design Objectives

Applying to most equipment on top of floating structures, the loads acting on the jib rest are governed by the motions of the barge at sea. And with additional criteria described by P&B as shown in section 3 and with the design methodology description in section 2, the following three design objectives are found:

- A) Finding the load cases using modelled motion responses and sea state probability
- B) Finding a new jib rest design complying with criteria listed in section 3 and based on the load cases found in
- C) Finding an estimation on the fatigue lifetime of both the old and new JR design

1.3 Scope

Although analysing the complete barge will be beneficial to any insights about the its purpose and its functionality, the focus of this thesis will lay on the design of solely the JR. Any results from calculations done might bring up recommendations on enhancements for a third generation crane barge, but for this generation, no alterations will be done to the barge hull, structure or crane.

2 Methodology and Design Objectives

This section describes a summary of steps that will be made to get to a JR design. The criteria provided by P&B give a starting point (section 2.1), and through a review of rules and regulations set by classification bureaus, e.g. LR or DNV GL (DNV GL), an alternative, more specific calculation method for load cases is proposed (section 2.2). This method, the design and an additional fatigue analyses results into the design objectives and a complete overview of the design process (section 2.3).

2.1 Criteria

The need of a new JR design is initiated by P&B, providing the following main criteria:

1. **Lightweight**; total construction should be lighter than the current K-frame design
2. **Out of working zone**; retractable, moveable or in any way out of the working zone of the crane
3. **Load Capacity**; capable of withstanding loads and damage because of the heavy work environment
4. **Flexible**; Quickly be placed in operating position or travelling position, if the design allows transition between these positions. On/offloading operation time is in the range of 12h to 24h, therefore the position transition time should not take over 1h
5. **Worldwide Operable**; in a wide range of environmental and sea conditions at harbours, shipyards and coastal areas
6. **Insusceptible to cargo on deck**; any bulk spilled on deck should not effect use of the jip rest
7. **Provide a inspection platform**; a platform should be located in the top of the jip rest, for inspection and maintenance of the jip rest and crane boom connection, and should be accessible from main deck
8. **Within beam**; placement of the jib rest in any condition should not be extended over the sides of the CB6324

These criteria are very understandable and do not need a lot of explanation as to why they are considered to be the boundary conditions for a JR, or even any offshore structure. However, this leaves a lot of room for interpretation and a need of definition. As satisfying the criteria *lightweight* or *sturdy* could have opposing effects on eachother for example. Their importance is quantified along with the other main and secondary criteria in section 3 to have a measureable choice of concepts, but their conformity in definition can best be approached by determining loads on the structure and using safety factors on the resulting stresses. Looking for an optimum between the criteria requires a clear view on the expected loads and well defined geometry of the structure.

2.2 Load Identification

These loads are directly related to the most unfavourable motions and environmental conditions the barge would encounter during its operation or transport to its final location, or any foreseeable hazardous events. The motions are dependent on many factors and varies significantly between ship types, geometry and location [4, 5, 27]. Class have made an effort to set up guidelines for these motions, with DNV GL emphasizing on finding the unique motion behaviour per individual case and LR stating representative motions that envelope the motions of a large range of ship types. The guidelines or rules set up by LR stated in a couple of paragraphs state [29]:

” 2.11.3: In the stowed condition, the crane, its stowage arrangements and the structure in way are to be designed to withstand forces resulting from the following two design combinations:

- a Acceleration normal to deck of $\pm 1,0g$
Acceleration parallel to deck in fore and aft direction of $\pm 0,5g$
Static heel of 30 deg
Wind of $63m/s$ acting in fore and aft direction.

- b Acceleration normal to deck of $\pm 1, 0g$
Acceleration parallel to deck in transverse direction of $\pm 0, 5g$
Static heel of 30 deg
Wind of $63m/s$ acting in a transverse direction.

2.11.4 Alternatively, where the crane is to be fitted to a conventional ship and the ships characteristics are known, the forces may be calculated using accelerations obtained from consideration of the ships motions given in Table 4.2.2, together with the force due to a wind speed of $63m/s$ acting in the most unfavourable direction.

2.11.7 Proposals to use other values are to be substantiated by calculations and will be subject to special consideration.”

And DNV GL [5]:

” 2.6.1 The wave load analysis may be carried out for operating conditions with specific wave environments at the considered site, and may also be carried out for transit conditions as alternative to the requirements given in the DNV Rules for Classification of Ships Pt.3 Ch.1 Sec.5.”

The envelope of application in LR’s first guideline uses very conservative values for wind and wave loadings. They consider a maximum roll angle of $= 30.00$ deg and a wind speed of $v_{wind}^{max} = 63.00$ m/s , plus accelerations which might be on the high side for a crane barge. A roll angle of this magnitude alone would cause flooding, without even considering the wind loads acting on the barge and the relatively high crane on deck. And it could be contemplated whether a near shore crane barge would or could need to survive a hurricane at open sea. When the barge would find itself in such a hazardous environmental condition, the crane and its pedestal would withstand these loads from a ‘allowable-stress’ point of view, but would also endure high load cycles which would result in shortening of lifetime due to fatigue. In which either the crane or the pedestal would fail. Designing a JR for these conditions will result in an overdimensioned construction, and consequently more use of material.

Keeping the weight in mind, a step towards a more detailed sea motion analysis is made and the resulting loads provide a ULS condition for the JR design. This ULS is expected to be less conservative then proposed by Class and is used for the dimensioning of the JR. When a final design is reached that complies to this ULS and criteria, load cases derived from Class will be applied and the JR dimensions are scaled up if necessary. This thesis will provide a different approach to the calculation of load cases, but when push comes to shove, the JR will have to comply with Class rules.

The sea motion analysis brings in the need of specific sea state data and from the design and sales point of view, one of the criteria set by P&B for the CB6324 is that the barge can operate near shore, all over the world, and is backed up by the demand expectations [14]. Stating a criterium such as this brings in a lot of different sea states to take in to account. Plus, as reviewed in section 5, the transport tow brings in another number of these states. The different sea states and probability of occurrence are highly dependant on geological location and time of year [10, 15]. Figure 2.1 shows a map of all nautical areas of which past sea states are recorded and compiled into scatter diagrams [4]. Additionally, this figure shows possible transport routes and the crane barge demand per area. One could discuss whether to use a global or averaged scatter instead of a large number of area data. This would simplify the calculations, but would not give any insights whether *all over the world* is a reasonable criteria or not, and since we are looking for a *maximum* response, averaging would seem unsuitable. Specific operating site scatter data would be the most preferable, but zooming in at these areas is considered to be a sufficient level of detail for now.

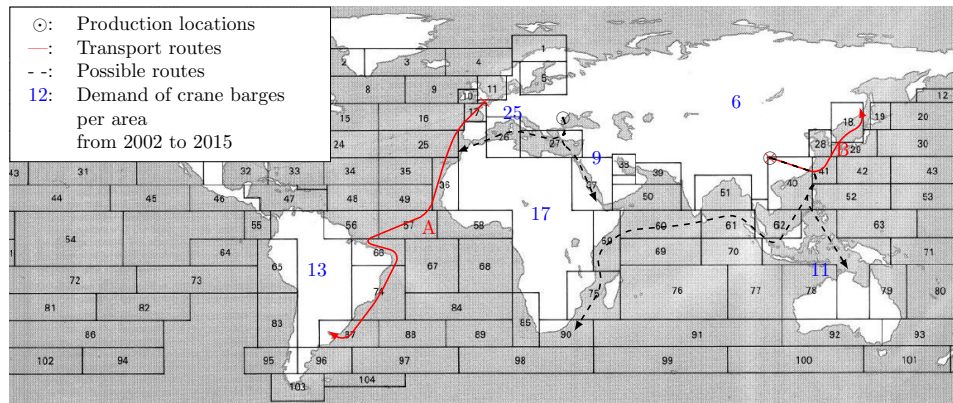


Figure 2.1: Nautical areas and possible transport routes

Worldwide operability is a bold statement for a near shore transshipment barge and introduces a call for quantification of this operability. Class provides rules and guidelines on when the crane is still allowed to operate and when it should be stowed. If the crane has to be stowed a large amount of the time, it might not even be worthwhile to position the barge at that location. And if a certain sea state would cause flooding or damage to the equipment on board, that sea state should be avoided at all times. Consequently, its probability can be disregarded from the motion analyses.

To get a more realistic estimation on the loads endured by the JR, a set of criteria, Class rules, sea states and several layers of probability have to be taken into account. This results in a large number of dependent variables and a calculation method that needs to be transparent, expandable and adaptable. As more insight about the expected loads will develop when all responses are found in a particular sea state and can be compared with the criteria.

Handling different calculation methods and dependent variables, while storing a wide range of intermittent data which can be altered and reused at a later stage, asks for a custom calculation tool. A tool which is object oriented and can be changed on the fly. RAOs, scatters, spectra and responses can be initialised as objects, allowing all calculations and manipulations to be done locally in the object itself. Because there are a large number of these scatters, spectra and responses to be analysed only one object per type needs to be checked and tested. Detailed information on the programming structure and code can be found in appendix E.

An additional outcome of the operability is the expected time in stowed position. Overlapping this with the probability a sea state occurs and its wave spectrum provides an estimation on the load cycles the structure would encounter. And thus a possibility to estimate the fatigue lifetime of the JR. A review of the old JR design shows some fatigue crack growth (section 5) and fatigue is considered as a field of interest for this design thesis.

Parallel to the load calculations and development of a tool, a design of a new JR is made, using the information and insights found in every calculation step. Starting with a concept and by adding detail in both the geometry and the evaluation of the design. Finishing with a simplification of the barge, crane and jib rest combination and the loads as load cases.

2.3 Design Objectives

The load and fatigue estimation using motions and probability, and the requirement of a new JR, brings this report to the following design objectives:

- A) Finding the load cases using modelled motion responses and sea state probability
- B) Finding a new jib rest design complying with criteria listed in section 3 and based on the load cases found in
- C) Finding an estimation on the fatigue lifetime of both the old and new JR design

To reach these objectives, the complete design cycle can be represented as a flow diagram as seen in fig. 2.2. Starting with preliminary calculations on any expected imposed forces because of deformations of the barge itself and defining the position of the JR where it could be hit (section 4). Modelling the CB6324 as a simple rectangular barge, a set of RAOs are found (section 7). These are used along with criteria (section 8.3), sea states per nautical area (section 8.1), and factor of safety (FoS) (section 9) as an input for the solver. This solver returns a set of load cases. The complete workflow, including the solver is described in section 8. With these load cases the dimensioning of the basic shape of JR can be made, and checked for nominal stress, buckling, natural frequencies and any supplementary criteria (section 10). As a final step in the design, a fatigue assessment is made along with structural detailing parallel to the engineering and a finite element modelling (FEM) analysis (sections 11 and 12).

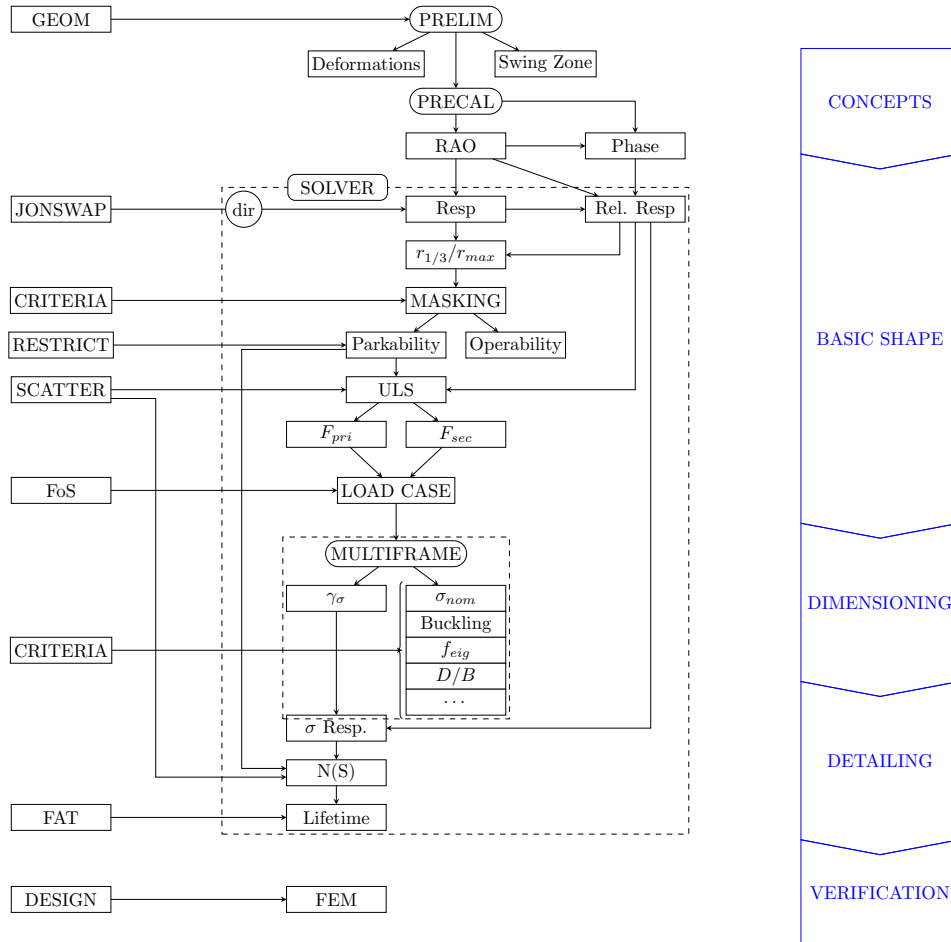


Figure 2.2: Model flowchart

2.4 Assumptions

The design and analysis of the JR rely on a set of simplifications and assumptions in the methodology and model. The load identification, predicting the motions of the barge and fatigue analysis are affected by a broad range of subjects. And before trying to open Pandora's box and investigating all its contents, this section will describe the assumptions made. However, the concept of the model and the programming of the tool is developed with the ability to add functionality where it is required, without having to redo a complete remodelling. The assumptions are as follows:

1. In the sea motion analysis, the barge is simplified as a rectangular barge, disregarding the sloped bow or stern. Mainly because the tool Marins PRECAL software to determine ship behaviour in waves (PRECAL_R) used for the motion behaviour is validated for this hull shape. Since the sloped parts of the hull is small in respect to the length and width of the barge, disregarding these parts is assumed to have little effect to the motion responses. Additionally, the overall motions are the focus in this analysis and not the drag or speed characteristics, what would bring in more interest in the hull shape.

2. For the determination of the motion responses, the barge is assumed levelled and without any load on deck, as this would be the case during any transport overseas. This is done by finding the COG of the barge, the pedestal, crane top and jib, and ballasting the forward ballast tanks until the total COG is located centrally on the x and y axes.
3. Worldwide operability as a design criteria suggests a usage of an average scatter diagram in response probability analysis. But since the scatter diagrams differ significantly per nautical area - or any location for that matter - and the barge will operate in near coastal areas all over the world, a list of these areas is taken into account. The specific shape of these scatters are adapted from [4].
4. In addition to the coastal areas as described above, areas that are located in probable transport routes are included in the list of scatters.
5. The long term wave statistics described in the scatters per area are assumed to be a sufficient level of detail. And any responses found from these scatters are regarded to have sufficient accuracy for load determination.
6. It is assumed that all sea states can be described by a JONSWAP wave spectrum [4].

CONCEPT PHASE

3 Quantification of Design Criteria

Quantification of the design criteria set by P&B gives the means for a more substantiated choice of concepts. This section describes a method to rate and weigh the criteria. And next to the design aspect, there are some more quantified criteria to which the new JR and crane will have to comply.

In correspondence with P&B, the different design aspects of the JR are categorized as the following design criteria:

1. **Lightweight**; total construction should be lighter than the current K-frame design
2. **Out of working zone**; retractable, moveable or in any way out of the working zone of the crane
3. **Load Capacity**; capable of withstanding loads and damage because of the heavy work environment
4. **Flexible**; Quickly be placed in operating position or travelling position, if the design allows transition between these positions. On/offloading operation time is in the range of 12h to 24h, therefore the position transition time should not take over 1h
5. **Worldwide Operable**; in a wide range of environmental and sea conditions at harbours, shipyards and coastal areas
6. **Insusceptible to cargo on deck**; any bulk spilled on deck should not effect use of the jib rest
7. **Provide a inspection platform**; a platform should be located in the top of the jip rest, for inspection and maintenance of the jip rest and crane boom connection, and should be accessible from main deck
8. **Within beam**; placement of the jib rest in any condition should not be extended over the sides of the CB6324

Design criteria are used as a quantification of all aspects involved, to compare different concepts and measure their design restrictions. To be able to make a trade-off between the concepts in a later stage, the design aspects are projected as criteria in a multi-criteria decision-making (MCDM) problem. To determine the weigh factors of these criteria, the BWM is used.

The following criteria are derived from the design specifications, with additional secondary design criteria:

- weight of the structure (c_{mass})
- out of working zone (c_{zone})
- sturdiness of the structure (c_{sturd})
- time and effort needed to move from resting- to working position (c_{flex})
- travel conditions (c_{trav})
- insusceptible to cargo/bulk spillage (c_{insusc})
- provide inspection platform (c_{platf})
- within the beam of the barge (c_{beam})
- maintenace needed including inspection intervals (c_{maint})
- fatigue resistance (c_{fat})
- effect on motions of the barge in different sea states (c_{mot})
- availability of the material (c_{mat})
- complexity of the structure (c_{compl})
- fabrication time and costs (c_{fabr})

- overall size of the structure (c_{size})
- obstruction of the winches (c_{winch})
- fatigue or structural impact on deck (c_{deck})

Initially, c_{zone} and c_{compl} are selected as the most and least important criteria - respectfully, which the importance of the remaining criteria are compared to, on a scale of one to five. In ?? the criteria are denoted with their scores: 'Most/c' describes how much more important the most important criteria is compared to criteria j . In the same way 'c/Least' compares the importance of criteria j to the least.

With these values, the weigh factor or weight of the criteria can be calculated. This weight is calculated by using the Excel Solver plugin, by keeping the maximum error as low as possible, and keeping the total of the weight one. The maximum consistency error (ζ^*), CI (CI), total of the weigh factors, and CR (CR) can be found in table 3.1, with c_{zone} as *best* and c_{compl} as *least preferable*.

Table 3.1: Best-worst method

Criteria	Most/c	c/Least	Weight	$\zeta(Most)$	$\zeta(Least)$	$\epsilon(Most)$	$\epsilon(Least)$	description
c_{mass}	1	5	0.11	0.04	0.36	0.36	0.36	weight of the structure
c_{zone}	1	5	0.11	0.00	0.58	0.58	0.14	out of working zone
c_{sturd}	2	4	0.09	0.70	0.30	0.70	0.01	sturdiness of the structure
c_{flex}	2	4	0.09	0.68	0.23	0.68	0.04	time and effort needed to move from resting-to working position
c_{trav}	2	4	0.09	0.72	0.35	0.72	0.00	travel conditions
c_{insusc}	3	3	0.05	0.69	0.59	0.69	0.03	insusceptible to cargo/bulk spillage
c_{platf}	2	4	0.09	0.68	0.22	0.68	0.04	provide inspection platform
c_{beam}	3	3	0.05	0.56	0.71	0.71	0.00	within the beam of the barge
c_{maint}	2	3	0.07	0.35	0.39	0.39	0.33	maintenace needed including inspection intervals
c_{fat}	3	3	0.05	0.58	0.70	0.70	0.02	fatigue resistance
c_{mot}	4	2	0.03	0.43	0.44	0.44	0.28	effect on motions of the barge in different sea states
c_{mat}	5	1	0.03	0.71	0.30	0.71	0.01	availability of the material
c_{compl}	5	1	0.02	0.58	0.00	0.58	0.14	complexity of the structure
c_{fabr}	4	2	0.03	0.58	0.37	0.58	0.14	fabrication time and costs
c_{size}	4	2	0.03	0.55	0.38	0.55	0.17	overall size of the structure
c_{winch}	3	3	0.05	0.56	0.72	0.72	0.00	obstruction of the winches
c_{deck}	4	2	0.03	0.22	0.53	0.53	0.19	fatigue or structural impact on deck

These weigh factors are found with a minimum CR as seen in table 3.2, and therefore are accepted as the weigh factors used.

Table 3.2: Consistency variables for BWM

Consistency	Value
ζ^*	0.72
CI	2.30
W_{tot} (W_{tot})	1.00
CR	0.31

Apart from the more weighed criteria factors a list of relevant restrictions and criteria are defined by P&B, LH, LR and DNV GL. These restrictions are used in the model as described in section 8.3:

- $\phi_{flood} = 14.99$ deg ; maximum heel allowable heel
- $\sigma_{fat} = 100.00$ MPa ; rough allowable fatigue stress estimation

- $\lambda_b = 3.00$; lower boundary of allowable buckling ratio
- $v_{work}^{wind} = 20.00$ m/s ; maximum wind speed at working condition
- $v_{wind}^{max} = 63.00$ m/s ; maximum wind speed at parking condition
- $\theta^{grab} = 3.00$ deg ; maximum admissable pitch for operation (Grab)
- $\phi^{grab} = 3.00$ deg ; maximum admissable heel for operation (Grab)

4 Preliminary Analysis

During the conceptualising of any JR designs, a set of analyses of the barge itself are required. The design philosophy of Damen is standardising; decreasing engineering and development costs and speeding up fabrication and delivery times. In the same way the CB6324 is developed, using a standard pontoon from the product range, and adding a LH crane to it, with any additional required equipment and structures like accommodation and winches. Only minor changes are made to the construction itself, keeping the main dimensions as set in the pontoon template. This template enables a wide range of applications and its hull and deck construction is designed to withstand any high loads. With a tough hull structure, the pontoon could be considered stiff, but will always have to allow some deformation, depending on the waves it encounters, especially during a tow or operation near shore. Making deformation one of the analyses (section 4.1). Another interesting analysis comes from one of the design criteria for the JR described in section 3: Out of the working zone. The criterium coming from P&B might be substantiated by Murphys law, where having a risk of hitting the JR is enough to develop a new design which minimises this risk. Therefore, an analysis is made to identify the zone on deck in which the crane operates in section 4.2.

4.1 Deformation

Added structures or modifications to the hull have a direct effect on the stiffness of the complete barge itself. Again, the pontoon template might be considered stiff, but a large construction like a stowed crane spanning over a wide distance on deck could function as a frame. This frame could add stiffness to the complete construction and result in large unwanted stresses in the crane, its pedestal or JR. The added stiffness can be subsided by choice of connections between the hull structure and its substructures as described in section 6, but any deformations of the barge during a tow or in operating conditions could lead to forced displacements of the crane and JR.

Barge as Beam Modelling Modelling the CB6324 as a beam provides an effective approach to determine the preliminary deformations of the barge in towing conditions. For this approach, the bending and torsion stiffness is taken into account, as a small deformation of the barge over x and y axis can propagate to a relative large deflection of the jib rest and crane pedestal at a height of 18.08 m .

To determine the stiffness of the barge, the barge sections are simplified as a rectangular hollow sections (RHS) with the dimensions seen in fig. 4.1. For the moment of inertia over x-axis (I_{xx}) an effective thickness (t_{eff}) is used as the hull thickness, since deck, bottom and walls of the barge are composed of stiffened plates.

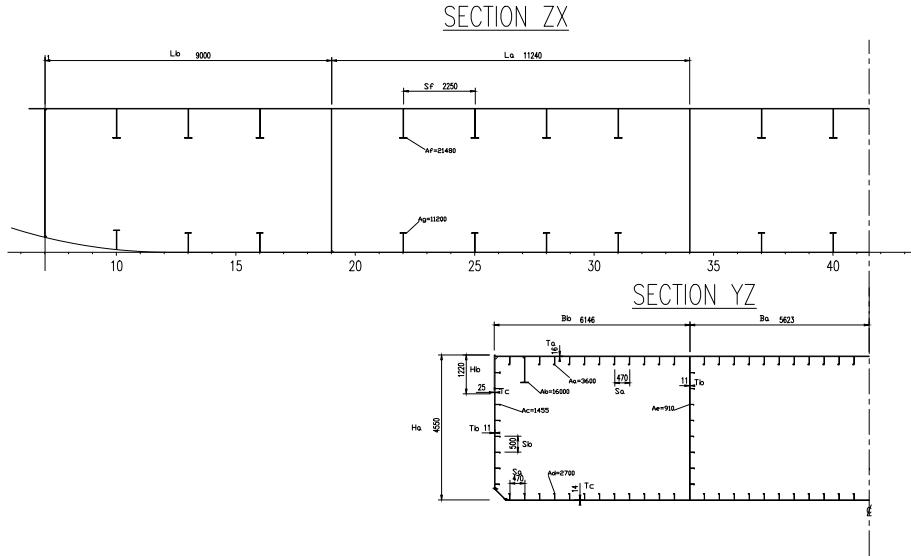


Figure 4.1: Simplified front and side view of the barge sections, with dimensions

With these main dimensions, the following torsional moment of inertia over x-axis (J_{xx}) is determined according [30]:

$$J_{xx} = \frac{4 \cdot A_{tot}}{\oint ds/t}$$

$$J_{xx} = \frac{4 \cdot W_{ba} \cdot H_{ba}}{\sum b_a/t_a} = 8.28 \text{ m}^4$$

And the I_{xx} :

$$I_{xx} = \sum \frac{1}{12} \cdot t_{eff}^3 \cdot W_{ba} + \frac{H_{ba} - t_a}{2} \cdot W_{ba} \cdot t_{eff} = 8.69 \text{ m}^4$$

Using these stiffnesses, the forces and moments acting on the barge can be modelled according:

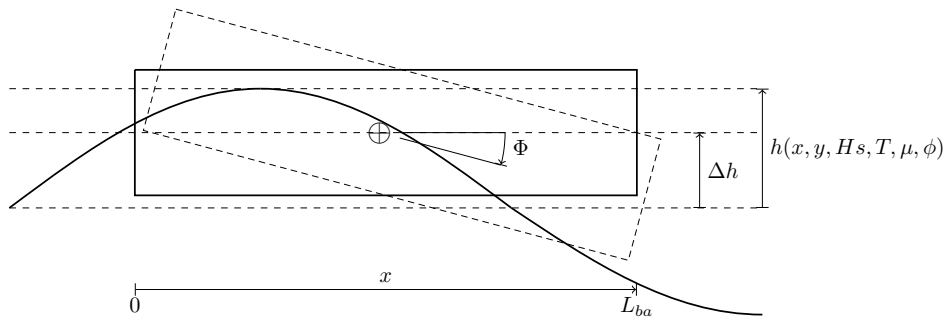


Figure 4.2: Sketch of translation and rotation due to wave height and barge mass

$$\begin{aligned}\sum F(x) &= \int_{-0.5 \cdot W_{ba}}^{0.5 \cdot W_{ba}} \int_0^{L_{ba}} \rho \cdot g \cdot h^*(x, y, Hs, T, \mu, \phi) \delta x \delta y - m_t \cdot g &= 0 \\ \sum M(x) &= \int_{-0.5 \cdot W_{ba}}^{0.5 \cdot W_{ba}} \int_0^{L_{ba}} \rho \cdot g \cdot h^*(x, y, Hs, T, \mu, \phi) \cdot x \cdot \delta x \delta y - m_t \cdot x_{mass} \cdot g &= 0\end{aligned}$$

with [15]:

$$\begin{aligned}h(x, y, Hs, T, \mu, \phi) &= a \cdot \cos \{x \cdot b + y \cdot c - \phi\} \\ a &= \frac{Hs}{2} \\ b &= k \cdot \cos(\mu) \\ c &= k \cdot \sin(\mu) \\ k &= (2\pi/Ts)^2 / g \\ h^*(x, y, Hs, T, \mu, \phi) &= h(x, y, Hs, T, \mu, \phi) - \sin(\theta \cdot x) - \Delta h\end{aligned}$$

Integrating over $x : (0, L_{ba})$ and $y : (-0.5 \cdot W_{ba}, 0.5 \cdot W_{ba})$, and writing this in matrix form gives:

$$\begin{bmatrix} L_{ba} \cdot W_{ba} & L_{ba} \cdot (1/2 L_{ba}^2 - x_{mass} \cdot x \cdot L_{ba}) \\ 1/2 L_{ba}^2 \cdot W_{ba} & W_{ba} \cdot 1/3 \cdot L_{ba}^3 - 1/2 \cdot L_{ba}^2 \cdot x_{mass} \end{bmatrix} \begin{bmatrix} -a \cdot (2 \cdot \sin(c \cdot 1/2 \cdot W_{ba})) \\ -a \cdot \left(\frac{2 \cdot \sin(c \cdot 1/2 \cdot W_{ba}) \cdot (b \cdot L_{ba} \cdot \sin(b \cdot L_{ba} + \phi)) + \cos(b \cdot L_{ba} + \phi) - \cos(\phi)}{c \cdot b^2} + m_{BW}/\rho \right) \end{bmatrix}$$

In this preliminary stage, $H_{tow}^{max} = 4.30$ m is assumed to be the largest wave height the barge would encounter and using the barge length to find a wave period of $T_{\lambda=l} = 6.35$ s, a regular wave is used to solve the matrix. Figures 4.3 to 4.8 show the deformations, rotations and stresses found in the hull structure for a head wave and different wave phases. In the 3D graphs, the top mesh represents the barge and bottom mesh the wave.

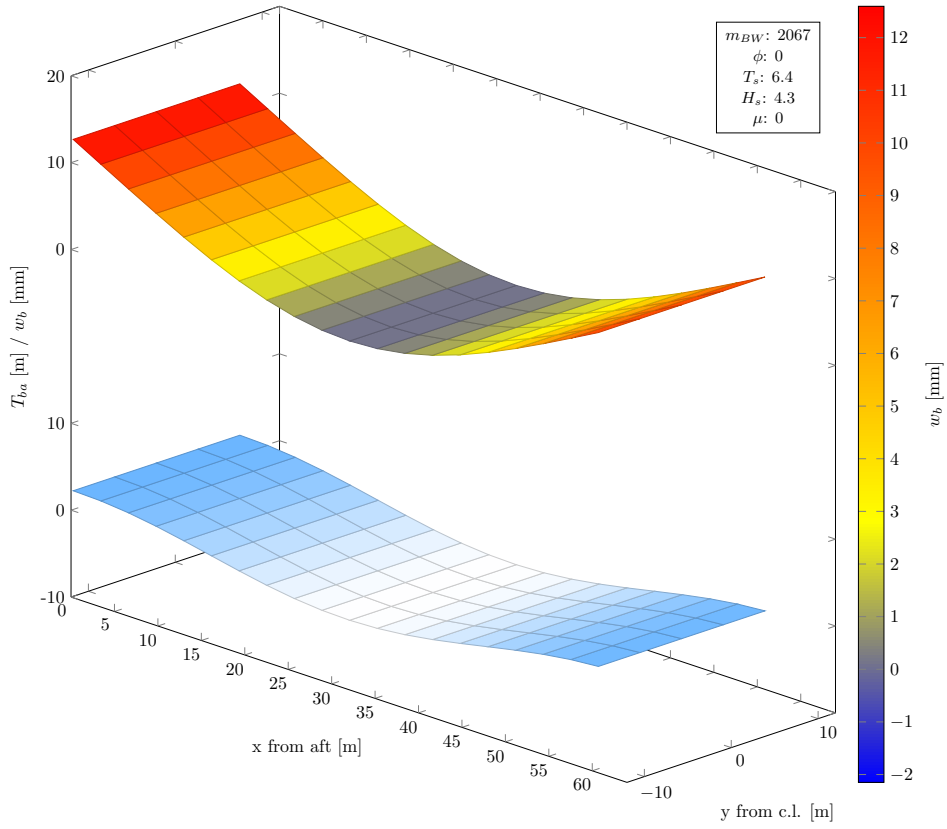


Figure 4.3: Deformations of the barge with respect to z-axis

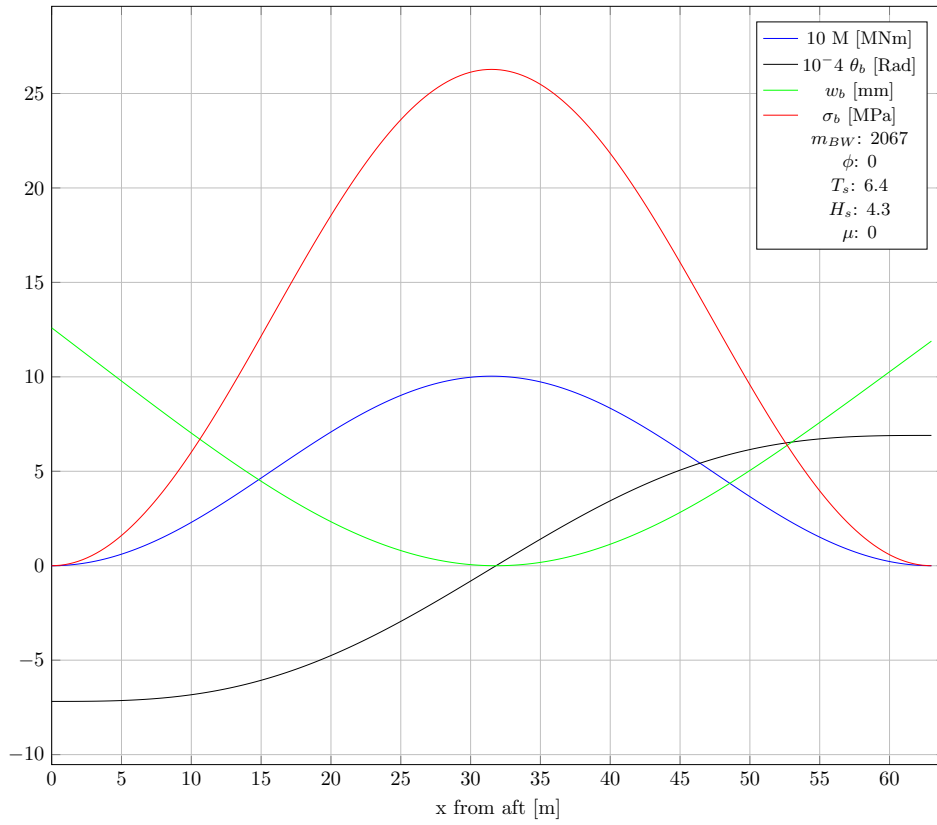


Figure 4.4: Deformations and bending stresses of the barge

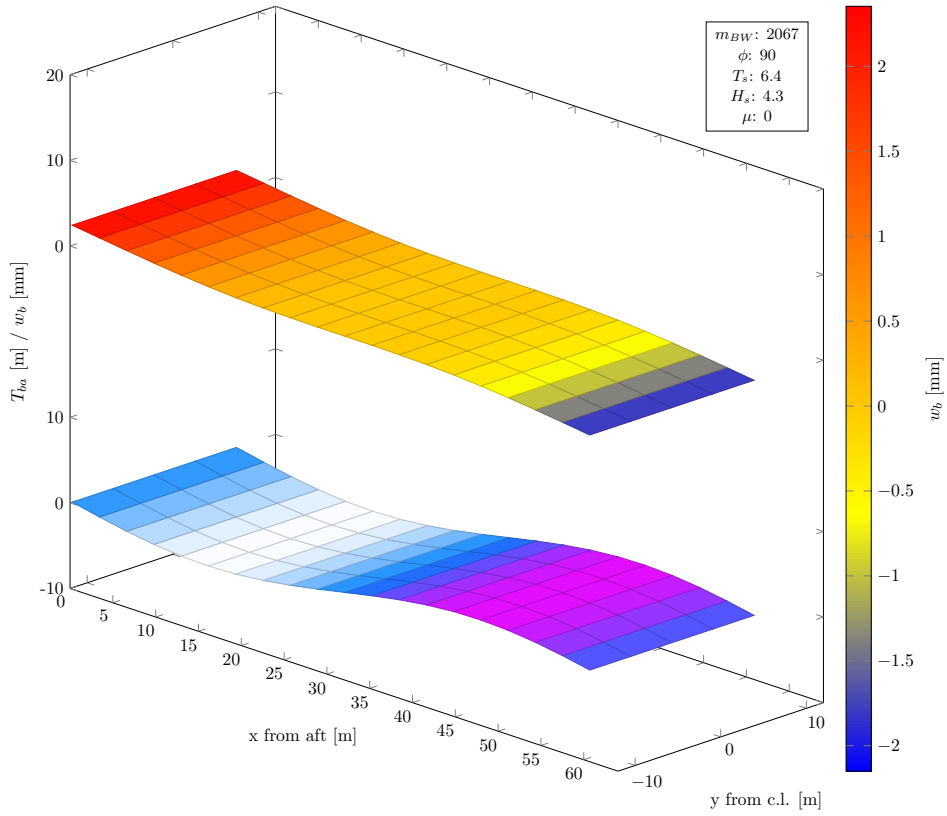


Figure 4.5: Deformations of the barge with respect to z-axis

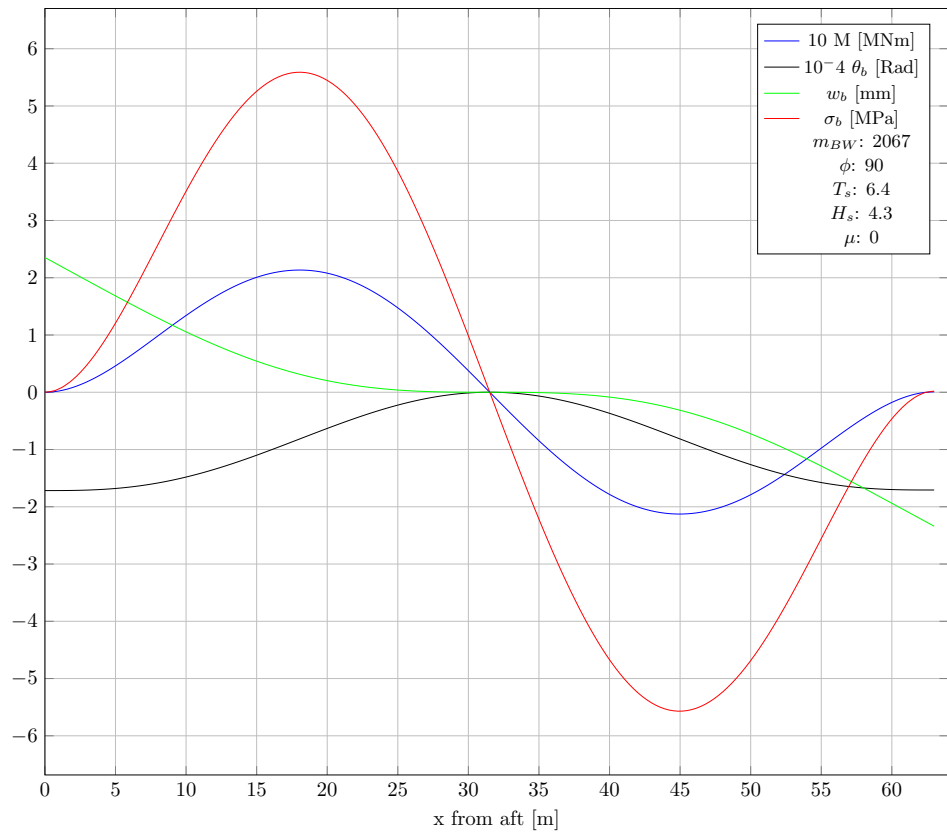


Figure 4.6: Deformations and bending stresses of the barge

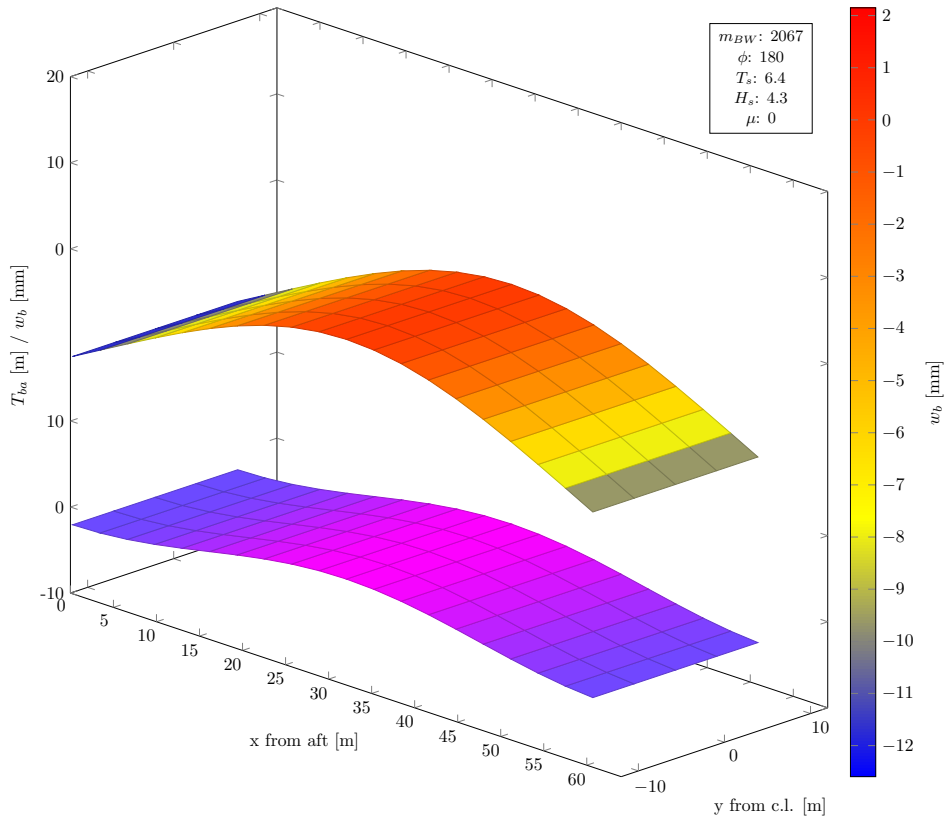


Figure 4.7: Deformations of the barge with respect to z-axis

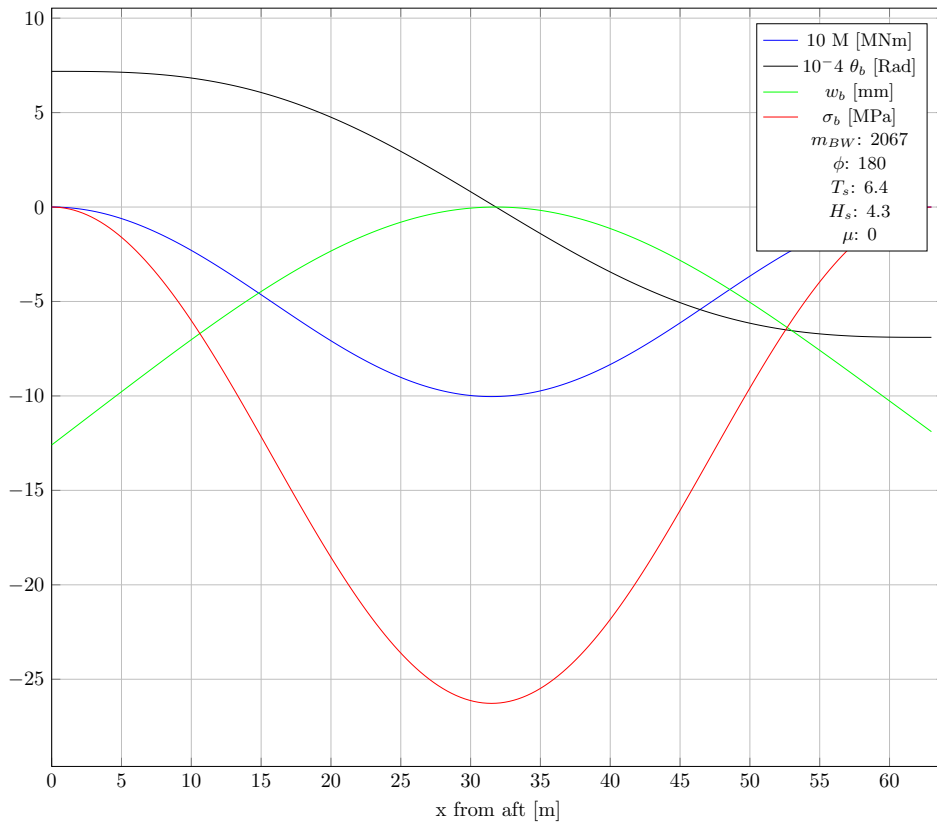


Figure 4.8: Deformations and bending stresses of the barge

This analysis shows relatively low deformations and stresses for a regular wave of $H_{low}^{max} = 4.30$ m , what indicates a stiff barge. With a side note that the total mass is assumed evenly distributed over the complete length and width of the barge, and without taking the deformation of the crane, the pedestal or JR into account. Whether further analysis of the hull structure should be done by including different wave directions and torsion, is up to P&B. But in the writers opinion, the barge itself is considered as a stiff plane in all further calculations.

4.2 Swing Area

The JR being out of the working zone on deck is mainly dominated by the outreach of the crane, equipment on deck and any space reserved for cargo. The outreach range is enlarged by the rotational speed of the crane and by wind acting on the grab or load, defined as the swing area. A part of this swing out is compensated by extending the crane jib and increasing the distance between the pedestal and JR, but to quantify this distance, the swing out is calculated using the crane specifications.

The Slewing speed (ω_{slew}) is used to determine the swing out of the grab, keeping the distance from deck constant at a varying boom angle. The total force directed outward is considered as the maximum wind acting on the grab plus the rotational force, resulting in the following equations to be solved (eq. (1)) and a schematic in fig. 4.9.

$$0.5 \cdot c_{grab} \cdot \rho_{air} \cdot A_{grab} \cdot v_{work}^{wind2} / m_{grab} + \omega_{slew}^2 \cdot (R_{tot} + r_{out}) - \tan(\arcsin(r_{out}/l)) \cdot 9.81 = 0 \quad (1)$$

$$R = R_{tot} + r_{out} + 0.5 \cdot d_{grab}$$

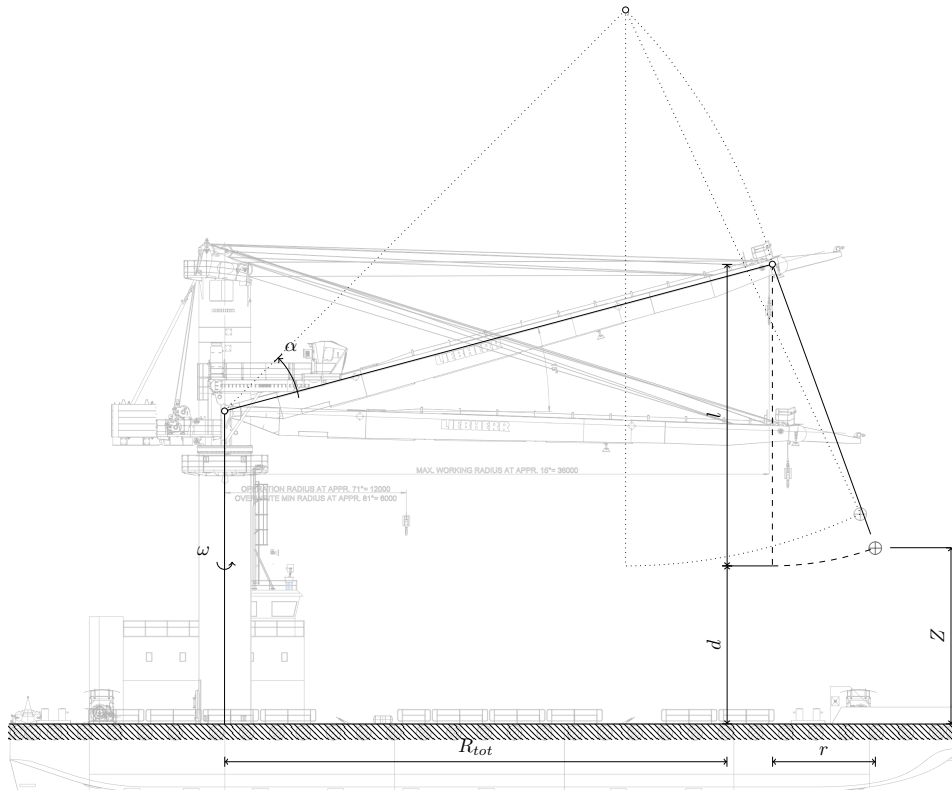


Figure 4.9: Schematic of the swingout of the grab, using the maximum slewing speed

Varying the boom angle and grab or hook distance from deck, the swing out is found for three different configurations: Grab operation without wind, grab operation with wind and (slow) container placement operation (figs. 4.10 and 4.11). This shows that the complete JR would either need to be removed completely, or dropped to an angle of approx. 30 deg relative to deck to ensure no collision can take place.

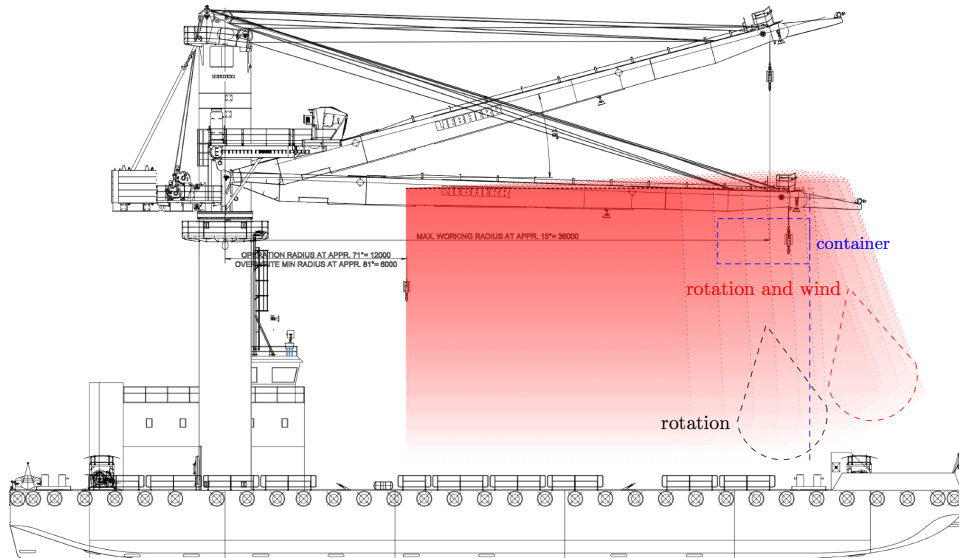


Figure 4.10: Calculated swing out of the grab, and the maximum outreach of the container

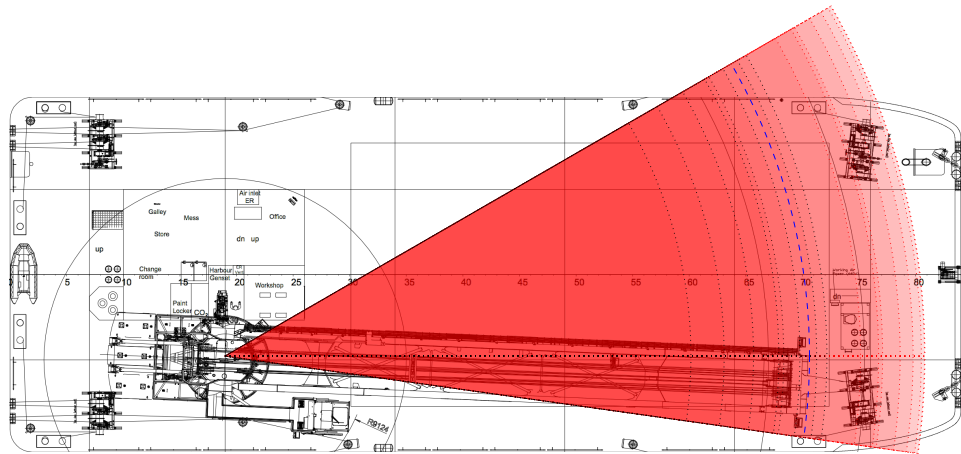


Figure 4.11: Calculated swing out of the grab, and the maximum outreach of the container, top view

5 Current Design Analysis

Two CB6324 are built and both showed issues even before operation. As any product in such an early stage of its life, any issues are to be resolved by the manufacturer and in this case, initiated a design overhaul of the JR. In this section, the old JR design is described, its issues are analysed and set as a benchmark for criteria score.

5.1 Description

The current JR is a K-framed structure of welded RHS members, providing the elevation needed for parking the jib. It is located in front of the crane in longitudinal direction, approx. a meter further than the maximum unloaded outreach of the crane, to reduce the chance of collision with the grab during transshipment operation. See fig. 5.1 for the location and fig. 5.2 for the construction details. This elevation of the JR is bound by the maximum angle in which the boom can be lowered. Lowering it more would either damage the hinges at the crane pedestal or would result in unwanted forces in the crane jib and crane itself. Optionally, the whole JR can be tilted, to provide even more distance between the grab and the rest itself, by use of a cylinder and a hinge on the bottom of the JR as seen in fig. 5.3.

5.2 Fatigue Failure

Two of the CB6324 are operating in Russia and Uruguay and were transported from China and Rotterdam respectively. During either of these transports the barge is towed and the jib is fixed onto the JR by straps with a pretension described by Liebherr [21]. Towing reports show that the tow is continued as long the maximum wave height predictions stay below 4.30 m [26, 3].

During the towages both jib rests were damaged and were showing fatigue cracks like seen in fig. 5.4.

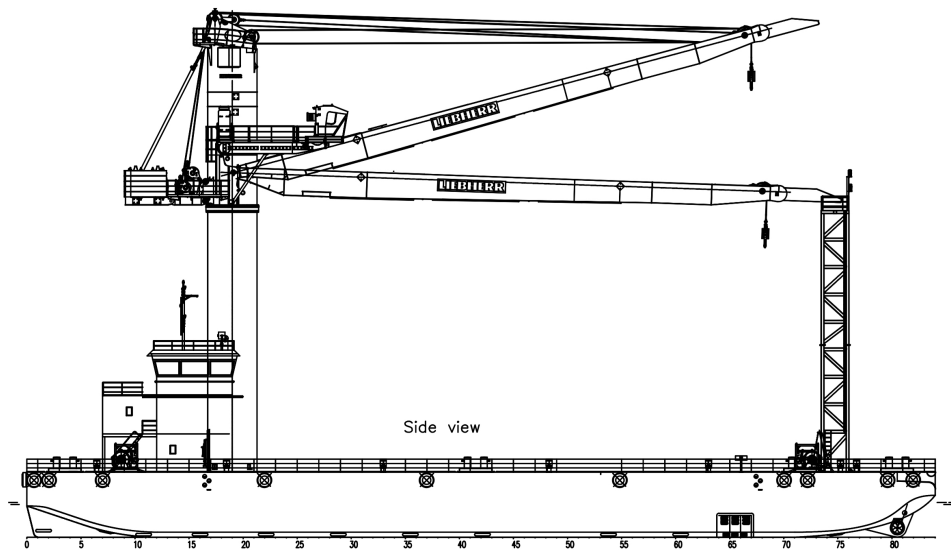


Figure 5.1: Side view of the current CB6324 design

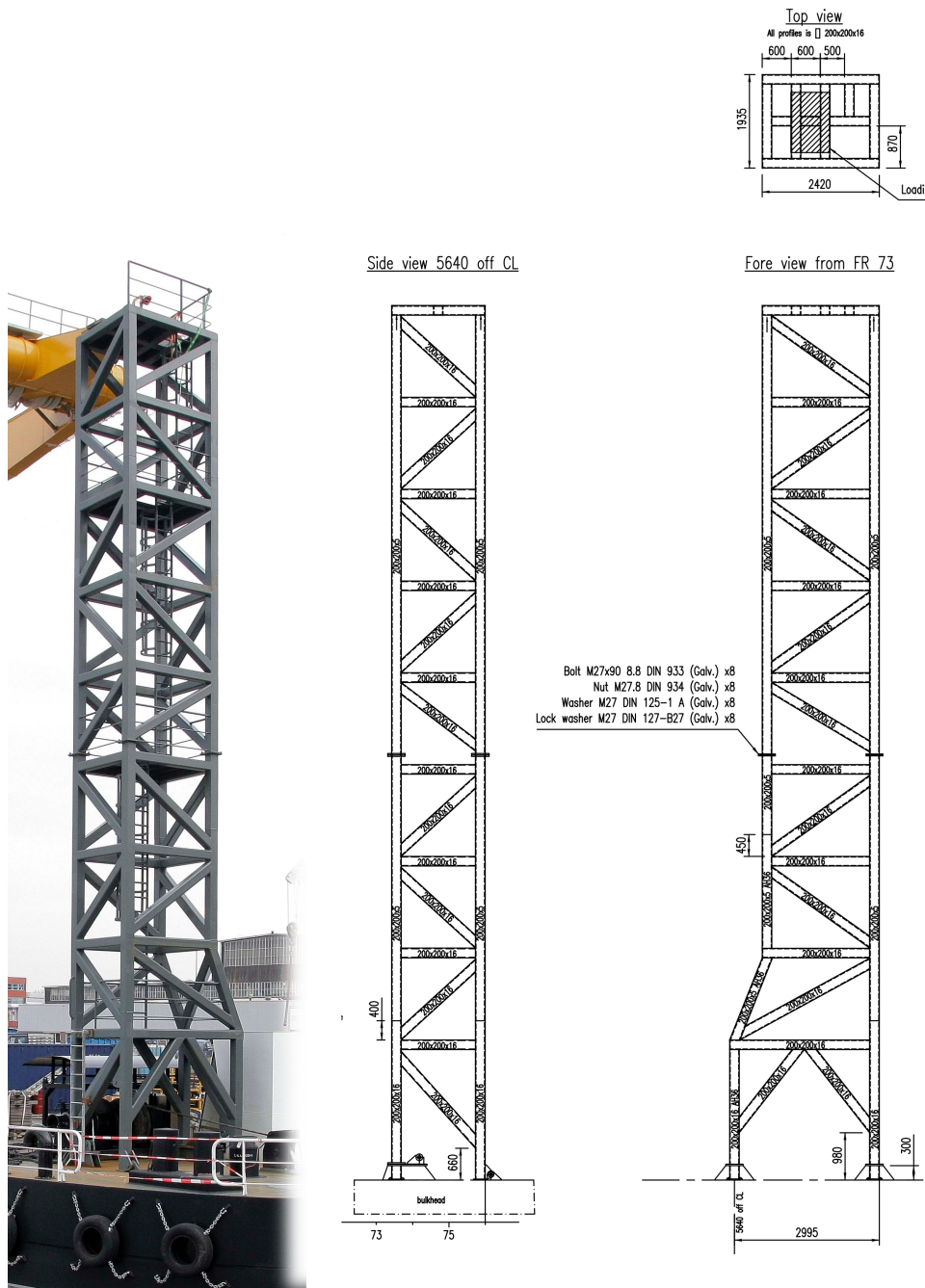


Figure 5.2: Construction details of the current JR design

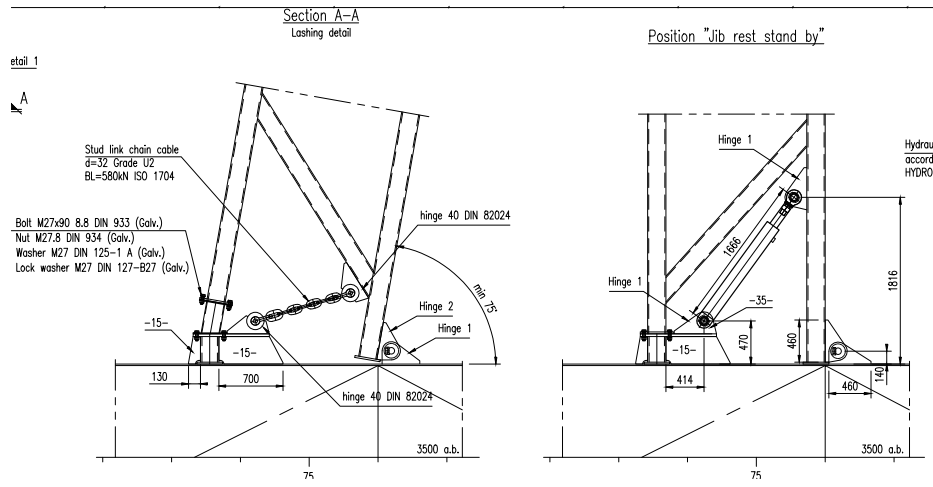


Figure 5.3: Hinge detail



Figure 5.4: Fatigue failure

5.3 Design Benchmark

Looking at the design parameters described in section 3, a benchmark can be set by grading the current JR design on the criteria, which results in the scorecard as seen in table 5.1. In this table the criteria are shown with their scores, weights, weighted scores and its motivation, and summing it up to an overall benchmarked total score of $S_{tot} = 0.53$.

Table 5.1: Current design BWM score

Criteria	$V(p, w)$			motivation
	p	w		
c_{mass}	0.00	0.11	0.00	Benchmarked as Heavy
c_{zone}	0.00	0.11	0.00	Benchmarked as 'in working zone'
c_{sturd}	1.00	0.09	0.09	A stiff K-frame with 200x200x16 profiles
c_{flex}	0.25	0.09	0.02	Hinged with cilinder; chain and bolted removable skids (dwg:523907-540-003); time consuming
c_{trav}	1.00	0.09	0.09	Stiff frame unaffected by weather conditions
c_{insusc}	1.00	0.05	0.05	Hinges welded on deck and easy acces
c_{platf}	1.00	0.09	0.09	Top frame used as platform
c_{beam}	1.00	0.05	0.05	Within beam
c_{maint}	0.25	0.07	0.02	A lot of welds to be checked; painting of nooks and crannies
c_{fat}	0.00	0.05	0.00	Stiff frame; large number of welds; weld failing during first transport
c_{mot}	0.50	0.03	0.02	centre of gravity at half of crane boom height
c_{mat}	1.00	0.03	0.03	Steel Grade A; standard material used in marine applications
c_{compl}	0.50	0.02	0.01	Simple welding/production procedures; but a lot of welds and different sizes
c_{fabr}	0.50	0.03	0.02	Large number of welds
c_{size}	0.50	0.03	0.02	Overall size slightly on the bigger size
c_{winch}	1.00	0.05	0.05	K-frame provides room for winches
c_{deck}	0.25	0.03	0.01	4-point welded connections on deck
			0.53	

6 Concepts

Before providing a solution for a new JR design, the complete barge with its drawings is analysed, looking for any information what could affect the design and the predefined criteria. Categorizing sub challenges and providing them with sub solutions gives a schematic overview of the more 'complex' overall system. Several viable combinations of the sub solutions are evaluated and scored according the weigh factors found in section 3, of which the three best concepts rise. Further analysis, during the development of the solver and definition of the basic shape, some of the sub solutions of two concepts are merged, as it provides a better design.

6.1 Morphologic Analysis

Figure 6.1 represents the categorisation of sub solutions, where *frame type* denotes the frame of the JR itself, *transition* the if and how it is removed or relocated. How this relocation is realised is described as *actuation*, connection between the jib and jib rest as the *jib connection*, *jib connection location* the actual location beneath or near the jib and the *unused location* as the place the JR is stored during operation.

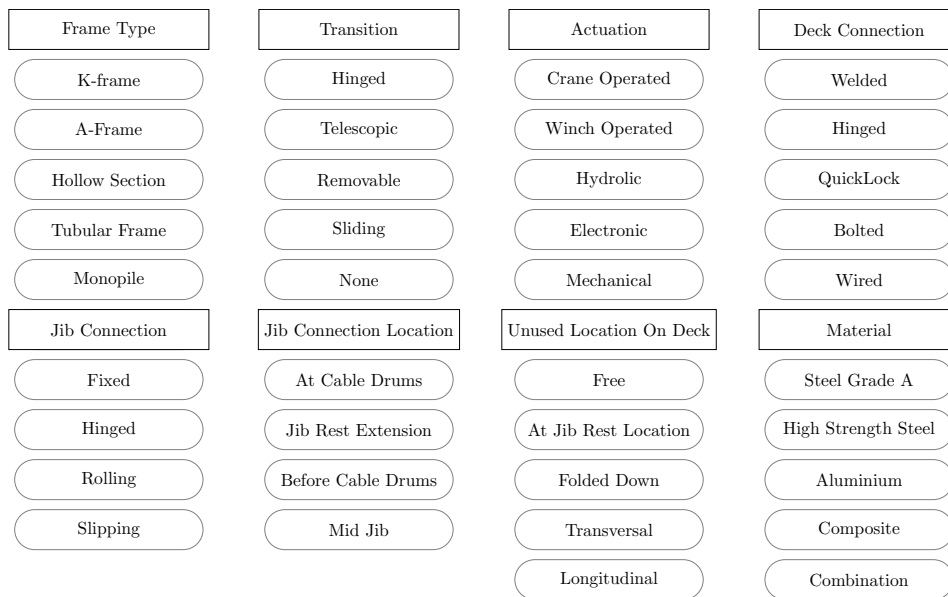


Figure 6.1: Morphologic Overview

6.2 Concept I

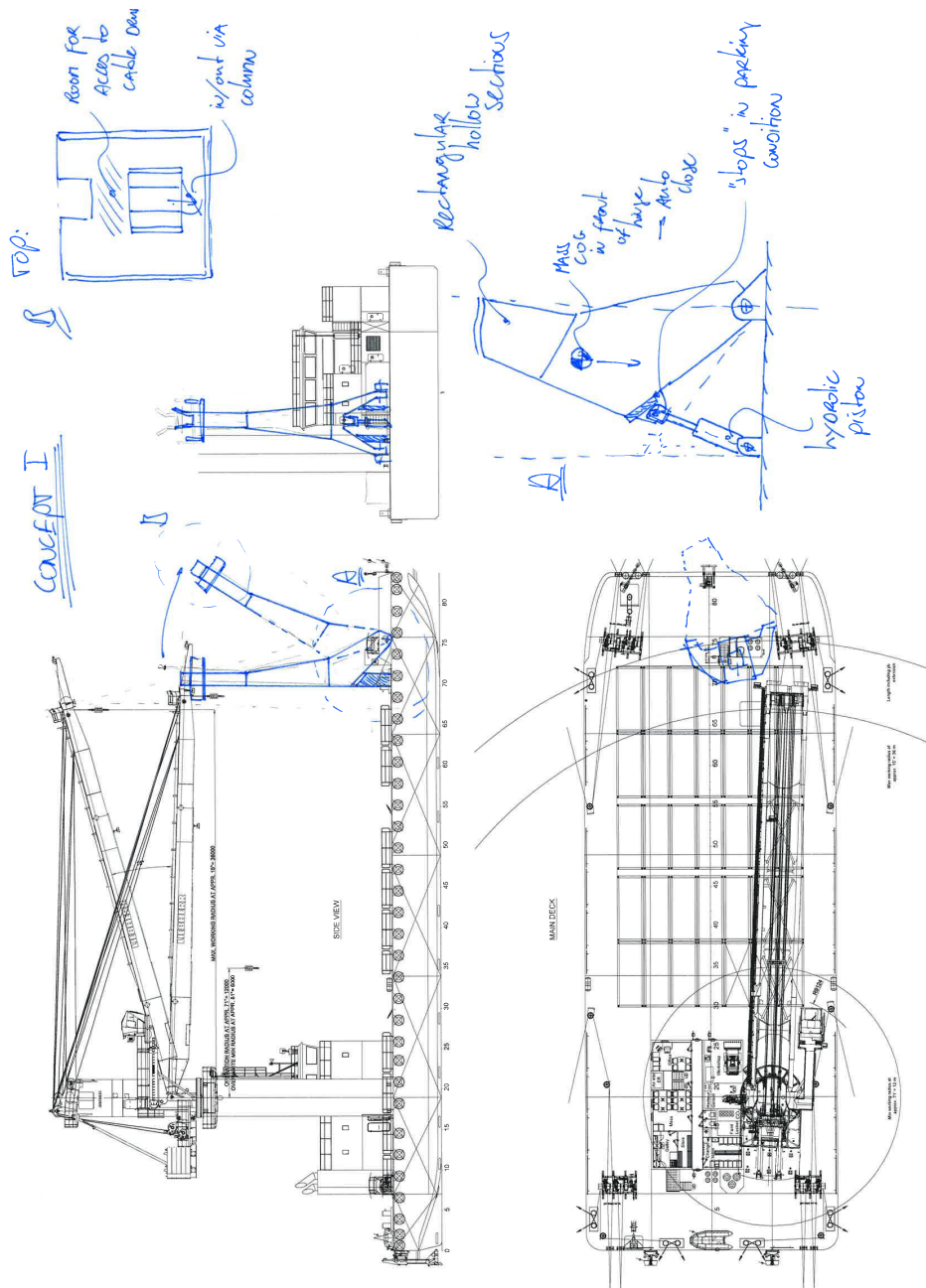


Figure 6.2: Sketch of Concept I

This first concept is based on a hydraulically operated, bottom hinged rectangular hollow section functioning as a boom rest. The hollow section mimics the shape of a crane boom, where steel plates are welded to each other to create the section, allowing automated welding techniques.

The width of the structure would increase from top to deck to prevent having large bending stresses on deck or in the construction itself, without having a lot of impact on the deck area. In fig. 6.3 the used configuration elements are highlighted.

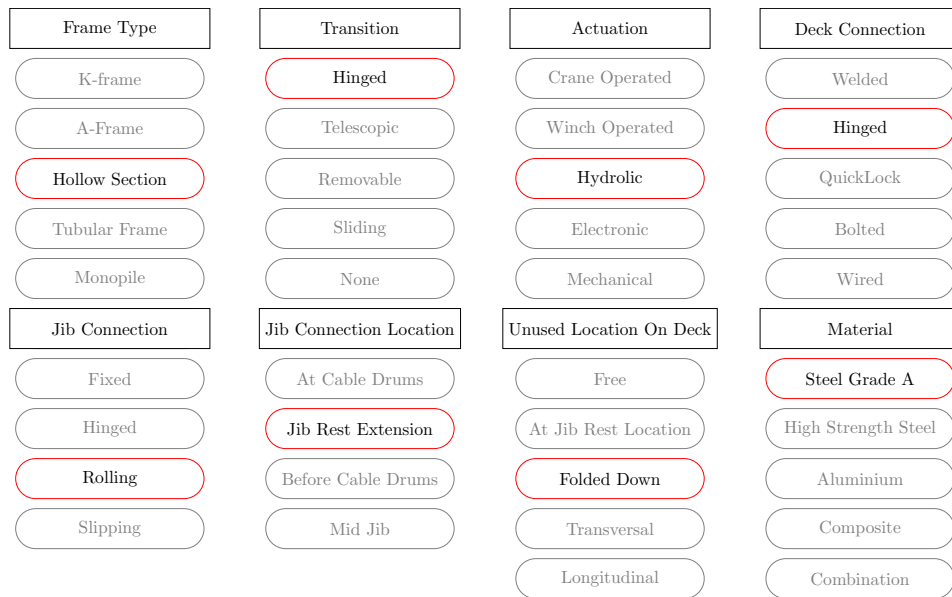


Figure 6.3: Morphologic overview concept I

Using the criteria set up in section 3, the concept scores can be found in table 6.1, with an overall score of 0.88 .

Table 6.1: BWM score of concept I

Criteria	$V(p, w)$			motivation
	p	w		
c_{mass}	1.00	0.11	0.11	From the first estimations of size; the structure can be slender and thin walled
c_{zone}	0.75	0.11	0.08	Hinge and cylinder will make the jib rest be able to move out of the way; but if the structure should be self returning; the rest could still be in the way
c_{sturd}	1.00	0.09	0.09	Even though it is thin walled; the hollow rectangular structure should be sturdy; and can be increased by increasing the thickness of the plating
c_{flex}	1.00	0.09	0.09	Using the hydraulics; it should be very simple to change from parking to working position
c_{trav}	0.75	0.09	0.07	Cylinder must be able to withstand forces due to wind and wave forces when it is returning to its parked condition
c_{insusc}	0.75	0.05	0.04	Hinges and cylinder are exposed to bulk spillage; if no protective measures are taken
c_{platf}	1.00	0.09	0.09	Structure is very capable to contain a service platform
c_{beam}	1.00	0.05	0.05	If placed in x direction; there is no overhang to either PS or SB
c_{maint}	0.75	0.07	0.05	Minimum number of welds and rectangular hollow sections with a large area would allow for minimum maintenance. However; bearings and the cylinder would require scheduled maintenance.
c_{fat}	1.00	0.05	0.05	The low number of welds and low number of tension concentrations minimises potential crack initiations. Only the base and top of the structure - two point hinges; one point cylinder; and the jib cradle are the significant hotspots
c_{mot}	0.75	0.03	0.02	Lowered mass and low centre of gravity would affect the barge motions as less
c_{mat}	1.00	0.03	0.03	Grade A steel will be available at all production locations
c_{compl}	0.75	0.02	0.02	Other than shape of the to be welded plating; no complex parts are in the structure
c_{fabr}	1.00	0.03	0.03	Simple material; welds and drawings
c_{size}	1.00	0.03	0.03	
c_{winch}	1.00	0.05	0.05	Placement out of the way of and next to the winches
c_{deck}	0.25	0.03	0.01	three point connection to the deck result in high concentrations of forces
			0.88	

6.3 Concept II

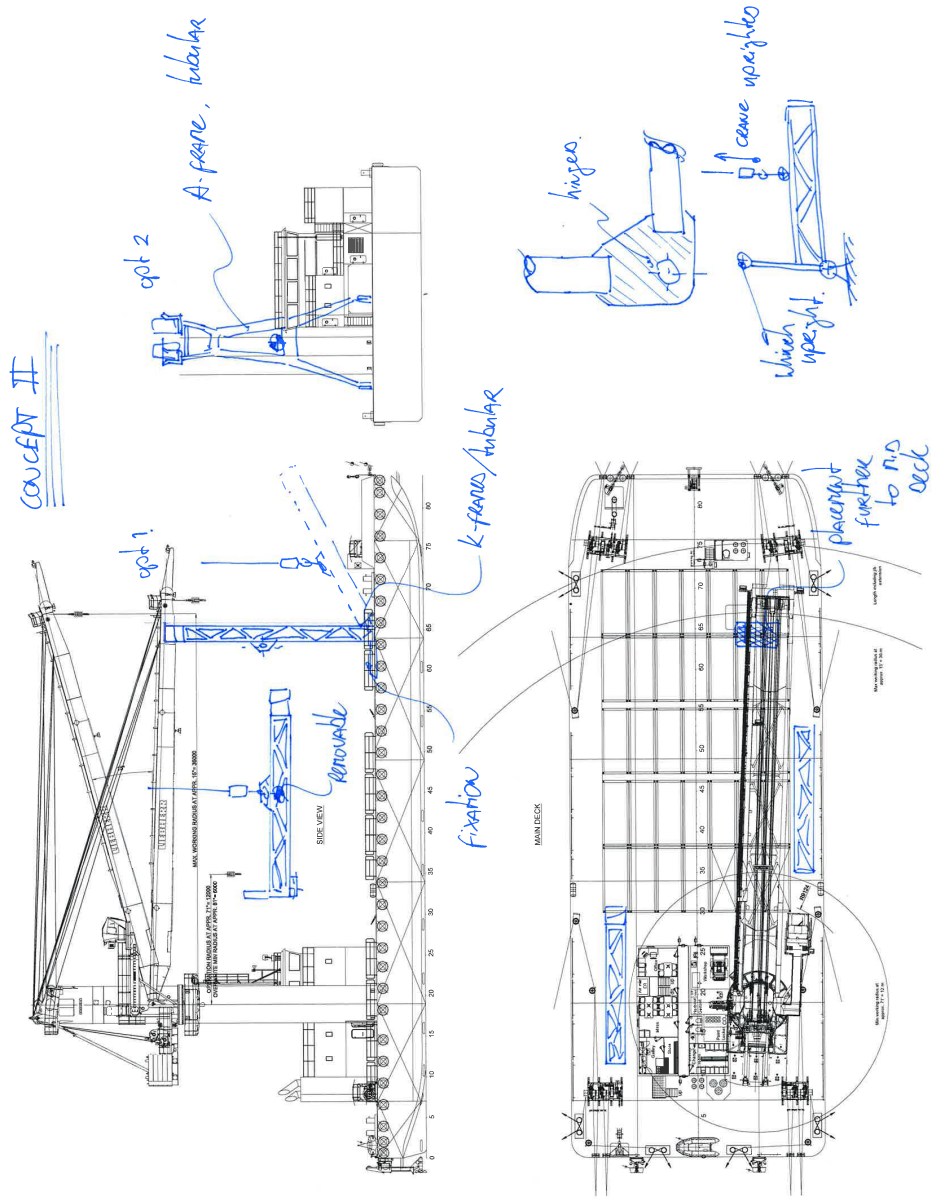


Figure 6.4: Sketch of Concept II

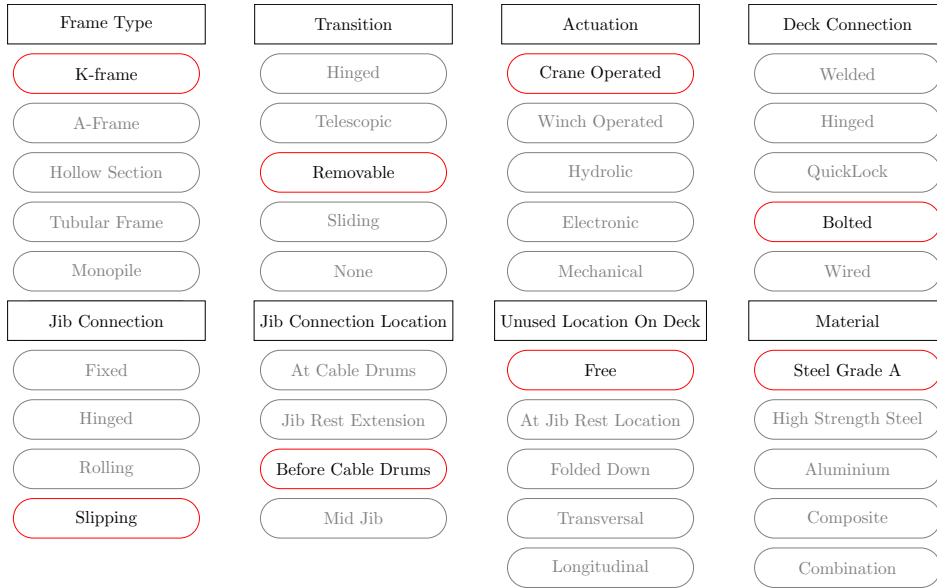


Figure 6.5: Morphologic overview concept II

Table 6.2: BWM score of concept II

Criteria	$V(p, w)$			motivation
	p	w		
c_{mass}	0.75	0.11	0.08	Designed shape results in smaller frame parts in the top
c_{zone}	1.00	0.11	0.11	complete removal of the jib rest
c_{sturd}	1.00	0.09	0.09	Tubular members
c_{flex}	0.50	0.09	0.04	Placement / removal is time consuming
c_{trav}	0.50	0.09	0.04	Placement in harsh weather conditions might prove difficult
c_{insusc}	1.00	0.05	0.05	Other than the deck connection; the structure is insusceptable to bulk damage (depending on storing position)
c_{platf}	1.00	0.09	0.09	Structure is very capable to contain a service platform
c_{beam}	1.00	0.05	0.05	
c_{maint}	0.50	0.07	0.03	A lot of welds to be checked; painting of nooks and crannies
c_{fat}	0.50	0.05	0.02	large number of welds
c_{mot}	0.75	0.03	0.02	Lowered mass and low centre of gravity would affect the barge motions as less
c_{mat}	1.00	0.03	0.03	Grade A steel will be available at all production locations
c_{compl}	0.50	0.02	0.01	large number of welds
c_{fabr}	0.50	0.03	0.02	large number of welds
c_{size}	1.00	0.03	0.03	
c_{winch}	1.00	0.05	0.05	Placement out of the way of and next to the winches
c_{deck}	1.00	0.03	0.03	three point connection to the deck result in high concentrations of forces
			0.79	

6.4 Concept III

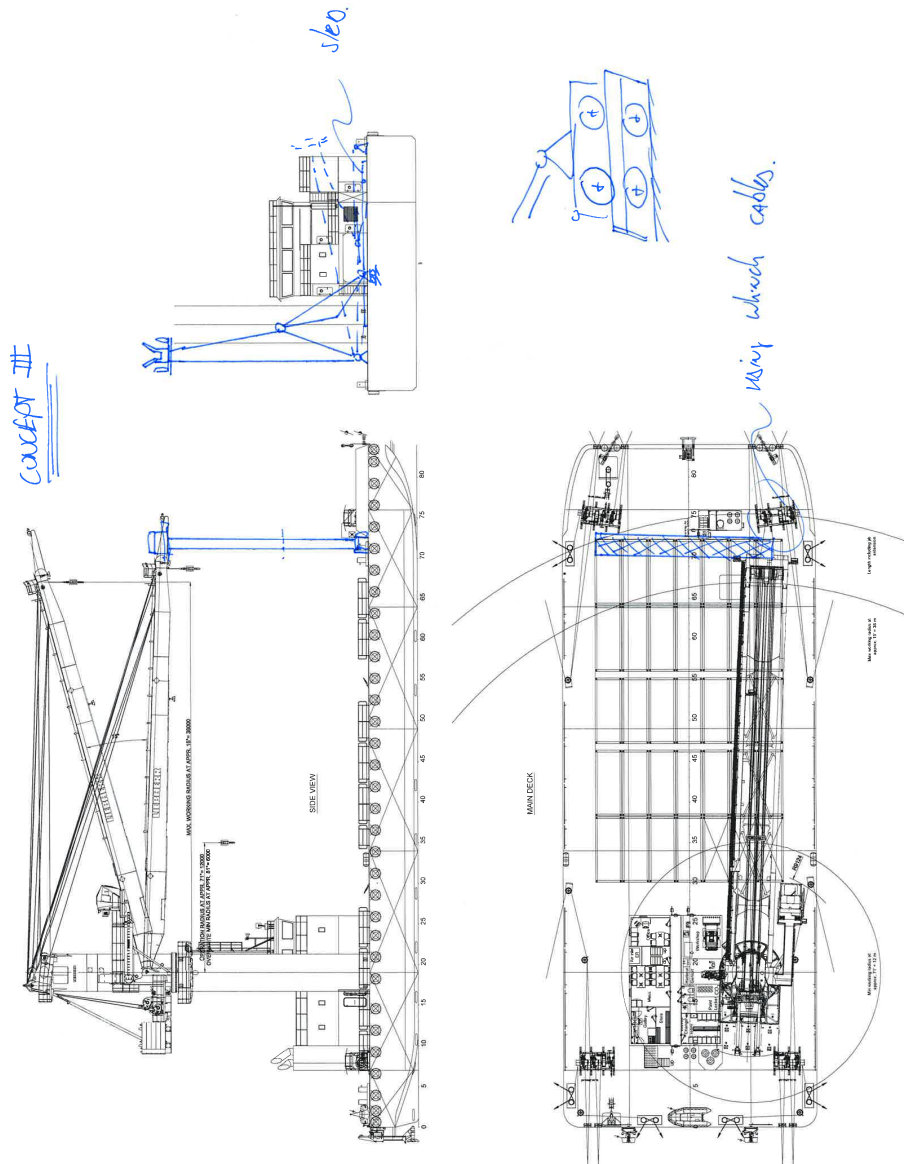


Figure 6.6: Sketch of Concept III



Figure 6.7: Morphologic overview concept III

Table 6.3: BWM score of concept III

Criteria	$V(p, w)$			motivation
	p	w	V	
c_{mass}	0.50	0.11	0.05	hinging construction would be large
c_{zone}	1.00	0.11	0.11	
c_{sturd}	0.50	0.09	0.04	dropped down would be out of the swing zone
c_{flex}	1.00	0.09	0.09	multiple hinges has impact on the overall sturdyness
c_{trav}	0.50	0.09	0.04	using either hydraulics or winch cables
c_{insusc}	0.50	0.05	0.02	large area which could be affected by cargo spill
c_{platf}	1.00	0.09	0.09	
c_{beam}	1.00	0.05	0.05	a lot of sliding or hinging parts
c_{maint}	0.25	0.07	0.02	
c_{fat}	0.75	0.05	0.04	
c_{mot}	1.00	0.03	0.03	
c_{mat}	1.00	0.03	0.03	
c_{compl}	0.25	0.02	0.01	
c_{fabr}	0.50	0.03	0.02	
c_{size}	0.25	0.03	0.01	
c_{winch}	0.50	0.05	0.02	
c_{deck}	0.25	0.03	0.01	
			0.67	

7 Response Amplitude Operators

After conceptualizing and having part of the preliminary results done, a step towards encountered motions is taken by calculating the barge responses. RAOs are the base of predicting any acceleration, and is used as the main input of the solver, which makes this one of the most critical subjects in the model. Deticated validated software is used to calculate the responses, as integration into the tool would be possible and interesting, but out of the scope of this thesis.

The responses of the barge are found by solving the equation of motion (eq. (2)) [19, 8] in any point, with respect to the and are represented by a RAO and a phase for every motion reponse.

$$\vec{F} = (-\omega^2(M + A_{total}) - i\omega B_{total} + C_{total})\vec{x} \quad (2)$$

Where:

- A_{total} = total added mass matrix
- B_{total} = total added damping matrix
- C_{total} = total added restoring matrix
- \vec{F} = total excitation vector
- \vec{x} = motion response vector
- ω = wave frequency

PRECAL_R is a tool which handles ship geometry and mass to calculate the motion responses. In a nutshell, PRECAL_R needs dampening coefficients, masses, draft and the barge hull geometry as a panel model to calculate the rigid body motions, while passing a regular 1m high wave through the centre of the barge at different frequencies. Resulting are the rigid body motions at the COG, namely; surge (X), sway (Y), heave (Z), yaw (Ψ), pitch (Θ) and roll (Φ) in the form of [19, 15, 8]:

$$X = A_x \cos(\omega t + \epsilon_x) \quad (3)$$

Where:

- A_z = response amplitude
- θ_w = wave phase

From which the RAOs follow. Combining the main motions with location information, results in relative motions in any point from the COG. And the needed accelerations can be found by integrating these relative motions twice, which results for RAOs for accelerations.

The barge itself is simplyfied as a rectangular hull, without any sloping or bow shape. The pedestal, crane boom and crane top are modelled as mass elements with their radii of gyration (κ) and the relative distance from aft-centerline-keel (ACK) to their local COGs introduced as RPs, as shown in fig. 7.1. Additionally some locations near the jib rest are also added as RPs. Presumably for this design stage, the heaviest environmental loads would occur during transport to its first operating location. During this transport, the barge is unmanned, presumed with an empty deck and levelled by ballasting. Arriving at the following values as input for the PRECAL_R calculations []:

Crane Jib

m_{jib}	39.20 t
$x_{C.O.G.}^{jib}$	32.40 m
$y_{C.O.G.}^{jib}$	-5.40 m
$z_{C.O.G.}^{jib}$	24.31 m
κ_x^{jib}	1.48 m
κ_y^{jib}	10.96 m
κ_z^{jib}	12.51 m
RP	0

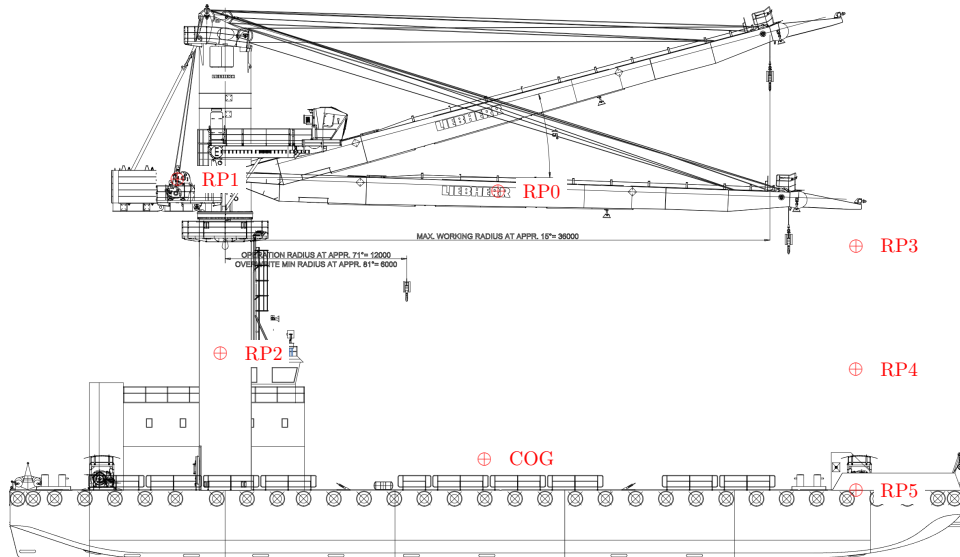


Figure 7.1: Local COGs as RP

Varying the wave frequency (ω) and wave direction (μ), PRECAL_R gives an output in the form of RAO and response phase (ϵ) per ω per motion or RP, as plotted in fig. 7.2, fig. 7.3 and ??.

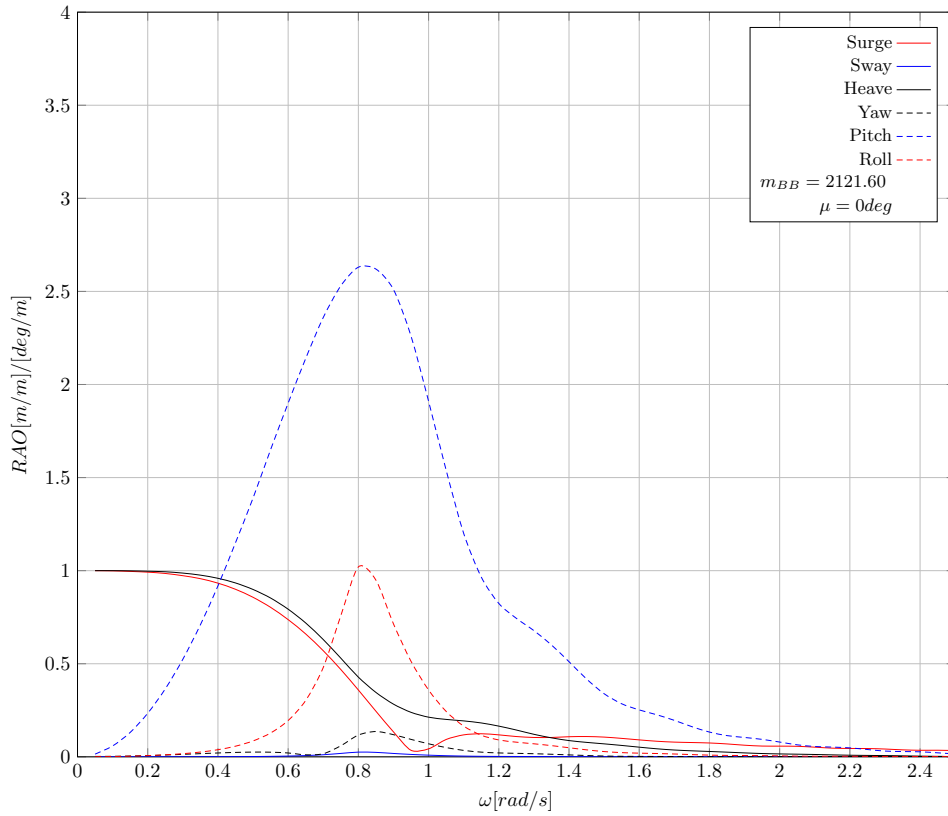


Figure 7.2: RAOs for head waves

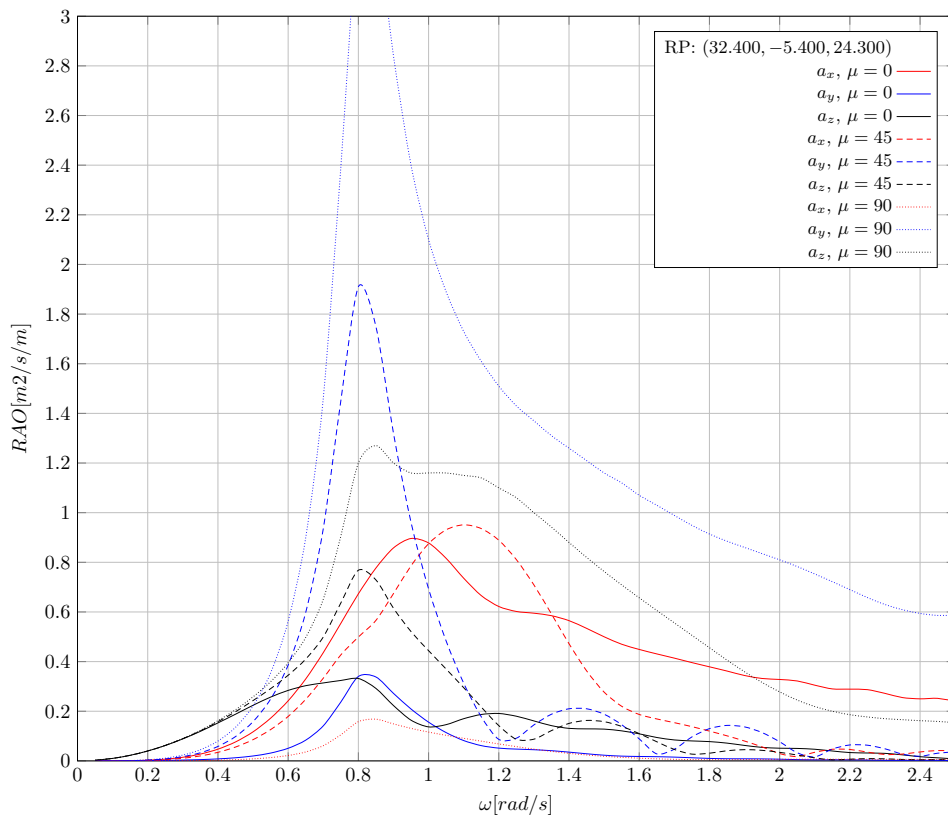


Figure 7.3: Acceleration RAOs for RP:0

BASIC SHAPE PHASE

8 Solver

The solver built in Python and its contents described in this chapter, combines the acceleration RAOs with a range of representations of sea states and digests it into a map of significant or maximum responses that the barge would encounter in those sea states. These responses are compared with all criteria and any structural or Class imposed restrictions to provide the operability of the barge. From this operability, or rather its parkability - explained in section 8.4 - and the probability of occurring seastates, the maximum accelerations and concequently the maximum loads are found. Keeping the structure of the model as seen in fig. 2.2 in mind.

8.1 Sea Modelling

The significant wave height (H_s) and zero up crossing period (T_z) that are measured during a sea state and its occurance are recorded in the scatters and could be seen as stochastic distributions [31, 17, 22]. DNV GL provides parameters for all these joint distribution scatters formulated as a joint distribution [23, 4] (eqs. (4) and (5)). fig. 8.1 shows a graph of the joint distribution.

$$p(H) = \frac{\beta_{H_s}}{\alpha_{H_s}} \left\{ \frac{H - \gamma_{H_s}}{\alpha_{H_s}} \right\}^{\beta_{H_s} - 1} \cdot \exp - \left(\frac{H - \gamma_{H_s}}{\alpha_{H_s}} \right)^{\beta_{H_s}} \quad (4)$$

$$p(T_z|H) = \frac{1}{\sigma t \sqrt{2\pi}} \cdot \exp \left\{ -\frac{(\ln t - \mu)^2}{2\sigma^2} \right\} \quad (5)$$

$$= p(H)p(T_z|H) \quad (6)$$

With:

$$\mu = 0.70 + a_1 H^{a_2}$$

$$\sigma = 0.07 + b_1 \exp(b_2 H)$$

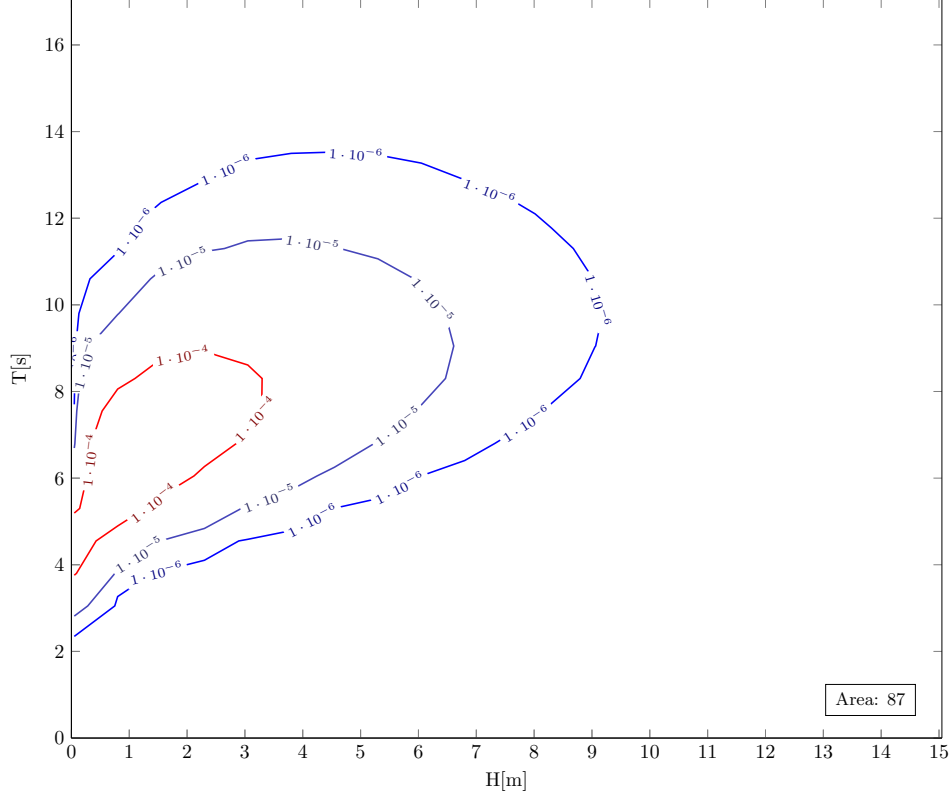


Figure 8.1: Joint distribution of area 87

This concludes the modelling of when and if the sea states will occur, but these joint distributions do not contain all information about the distribution of wave heights and periods during this sea state. This is where an irregular wave description in the form of a wave spectrum is introduced [16, 19]. The wave surface elevation can be described by a superpositioning of an infinite number of waves with varying amplitudes and wave frequencies. Its information of occurring or its (physical) amplitude per ω are stored in variance or energy density spectra, respectively. Common models are the Bretschneider-, Pierson-Moskowitz or the JONSWAP-spectra [16, 4]. The latter is considered to be the most representative in this design and is used for every sea state analysed. The equation describing the JONSWAP can be found in eq. (8) [4, 19].

$$S_J(\omega) = A_\gamma \cdot \frac{5}{16} \cdot H_s^2 \cdot \omega_p^4 \cdot \omega^{-5} \cdot \exp\left(-\frac{5}{4} \left(\frac{\omega}{\omega_p}\right)^{-4}\right) \cdot \gamma \cdot \exp\left(-0.5 \left(\frac{\omega - \omega_p}{\sigma \omega_p}\right)^2\right) \quad (7)$$

Where:

$$\omega_p = 2\pi / (T_z \cdot (1.30301 - 0.01698 \cdot \gamma + 0.12102/\gamma)) \quad (8)$$

$$\gamma = \text{non-dimensional peak shape parameter} = 3.3$$

$$\sigma_s = \text{spectral width parameter}$$

$$\sigma_s = \sigma_a \text{ for } \omega \leq \omega_p = 0.07$$

$$\sigma_s = \sigma_b \text{ for } \omega > \omega_p = 0.09$$

$$A_\gamma = 1 - 0.287 \ln(\gamma) \text{ normalizing factor} \quad (9)$$

Directional Spreading Since the barge responses and therefore the accelerations encountered are highly dependent on the wave direction, a distinction is made between head and beam waves. During transport and non-operating conditions while moored by one anchor it would encounter mainly beam waves, while as it is positioned next to a bulk carrier or using multiple point mooring, waves can hit the barge from all directions. As the barge has a simple rectangular shape, no distinction is made between head and

stern. And while the barge would seemingly encounter waves from one direction, the total wave energy described in the wave spectrum will have a directional spreading [4, 9], with a new spectrum modelled as:

$$S_{zeta}(\omega) = \cos^2(\mu) \cdot S_J(\omega) \quad (10)$$

Wind Coupling Apart from the accelerations of the barge, wind has a significant role in the total load endured by any structure on deck. Realistically, in any storm or weather condition, wind will pick up and will generate more waves adding on to the waves already developed at a (nearby) location. When the wind dies out, waves still contain a lot of energy and will die out long after the wind has passed. But for simplifying the interlinked environmental conditions and its probability of occurrence, the wind speed is set to the maximum speed encountered during such a sea state, according to the Beaufort scale [28, 32]. With a speed considered maximum when the $p_{N.E.}$ is 99%.

8.2 Responses

With a decent description of all sea states and calculated RAOs, the response spectrum ($R(H_s, T_z, \omega)$) per motion or acceleration in any RP can be found according to eq. (11), finding a response spectrum per sea state, directly from the sea state spectrum.

$$R(H_s, T_z, \omega) = \int |RAO(\mu)|^2 \cdot S_\zeta(H_s, T_z, \omega, \mu) d\mu \quad (11)$$

From all the spectra calculated with varying H_s and T_z , the significant response ($r_{1/3}$) can be calculated by using the spectral moment ($m_0(H_s, T_z)$) as seen in eqs. (12) and (13). And accordingly, with a $p_{N.E.}$ of 0.95, the maximum response (r_m) is found by use of eq. (14). [19, 4].

$$\begin{aligned} m_0(H_s, T_z) &= \int R(H_s, T_z, \omega) d\omega \\ &= \int \int |RAO(\mu)|^2 \cdot S_\zeta(H_s, T_z, \omega, \mu) d\mu d\omega \end{aligned} \quad (12)$$

$$r_{1/3} = 2\sqrt{m_0(H_s, T_z)} \quad (13)$$

$$\begin{aligned} r_m &= \sqrt{-\ln p_{N.E.} \cdot 2 \cdot m_0(T_z) \cdot H_s^2} \\ &= H_s \cdot \sqrt{-\ln(p_{N.E.}) \cdot 2 \cdot m_0(T_z)} \end{aligned} \quad (14)$$

As a result from these calculations, both the $r_{1/3}$ and r_m for any response can be plotted as done with the roll motion and acceleration in RP:0 in figs. 8.2 and 8.3, where the solid line represents a contour line of the significant response and the dashed line the maximum response, both with waves in head direction. Due to the usage of one shape parameter for the all JONSWAP spectra for every seastate, this response scales linearly with H_s , and would not need to be plotted in a contour map. However, the use of such a map simplifies the use of scatter diagrams, as these can be merged numerically and makes future change in wave spectra modelling possible. All remaining response plots per motion of RP can be found in appendix A

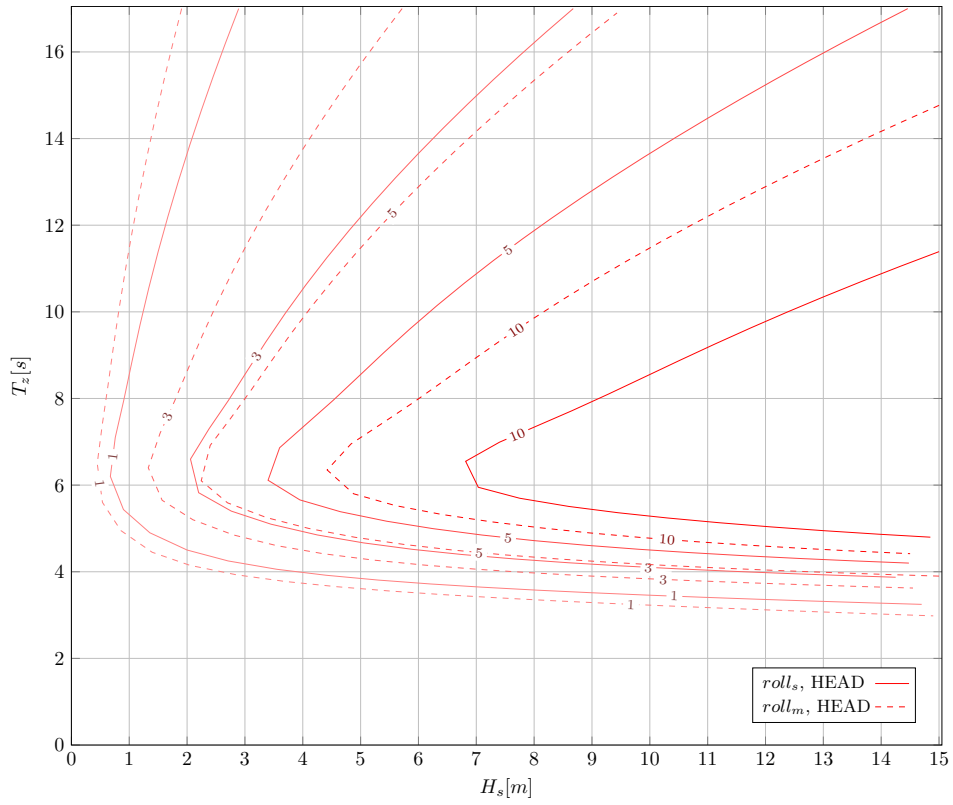


Figure 8.2: Roll responses for head waves

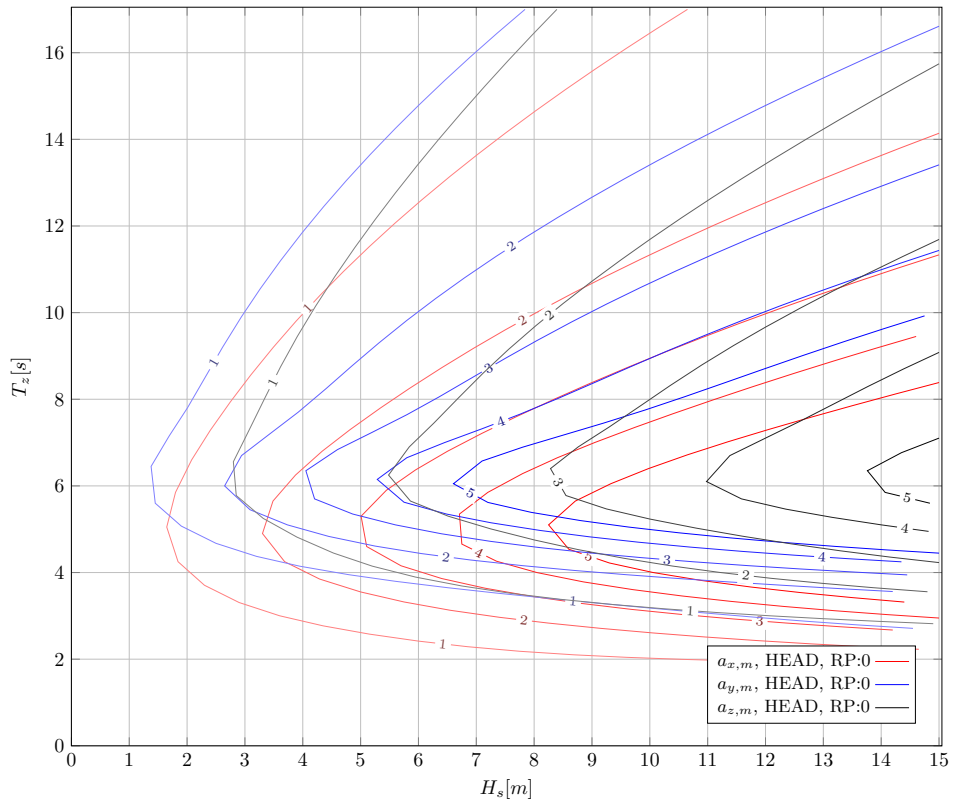


Figure 8.3: Maximum acceleration responses for head waves in RP:0

8.3 Criteria and Restrictions

With all responses calculated and mapped, design criteria, Class rules and structural restrictions regarding the crane and barge can be considered. Most of these restrictions or criteria are directly adapted from LR [21, 29]:

- $\phi^{grab} = 3.00$ deg
- $\theta^{grab} = 3.00$ deg
- $v_{work}^{wind} = 20.00$ m/s
- $v_{wind}^{max} = 63.00$ m/s
- $g_{max} = 4.91$ m/s²

Missing a restriction of an acceleration, when the crane driver would endure unpleasant accelerations in the crane driving position [2]:

- $\sqrt{m_0} = 288.00$ mm/s²

And for a stability criterium, the following roll angle is introduced:

- 14.99 deg

The RAOs are calculated with a regular wave with a height of 1 m. When the deck edge submerges, the responses of the barge would change significantly as green water flows on deck.

Lastly, a restriction of maximum stress in the pedestal is added. If the pedestal would fail, the jib rest is allowed to fail as well. The pedestal itself does comply to class regulations, but failure is expected to occur when the barge is in a rough seastate, resulting in high stress cycles in the base of the pedestal. This stress in the pedestal base is calculated as seen in appendix C and a rough estimation of allowable significant stress cycle is set to:

- $\sigma_{fat,ped} = 70.00$ Mpa

Comparing these criteria to either the $r_{1/3}$ or r_m maps, a new map can be created, showing a contour line of when this criteria would be exceeded and in what sea state. Resulting in the criteria map for beam waves in fig. 8.4. The most interesting lines here are *motion* - representing the unpleasant accelerations - and *pedestal stress*, because they define the lines between operability (*OP*), parkability (*PA*) and removal of the barge.

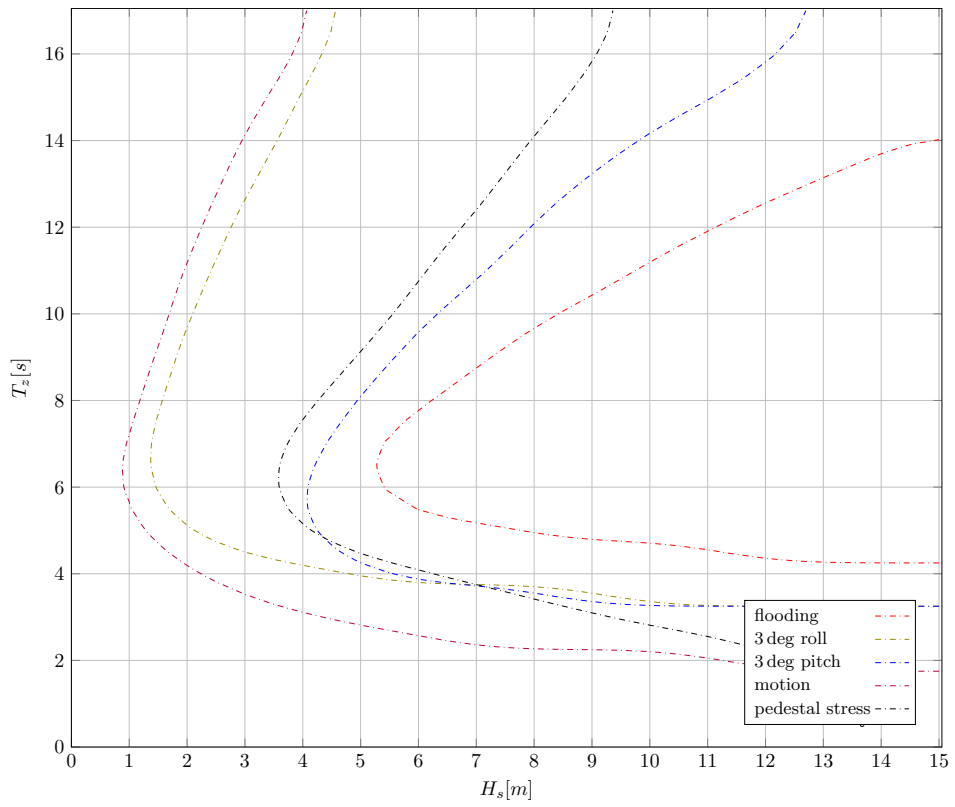


Figure 8.4: Criteria map

8.4 Masking and Operability

The criteria maps, defining the border between OP and PA , can be overlapped by the scatter diagrams of any coastal area the barge is supposed to operate. This creates an operability plot, based on long term wave statistics as seen in fig. 8.5. Here, the hashed area stands for the OP , which is bound by *motion* and the dotted area for PA and bound by *pedestal stress*. The area below these hashes can be summed up to get a total percentage of OP and PA . The legend shows this OP and the total parkability minus the operability as PA .

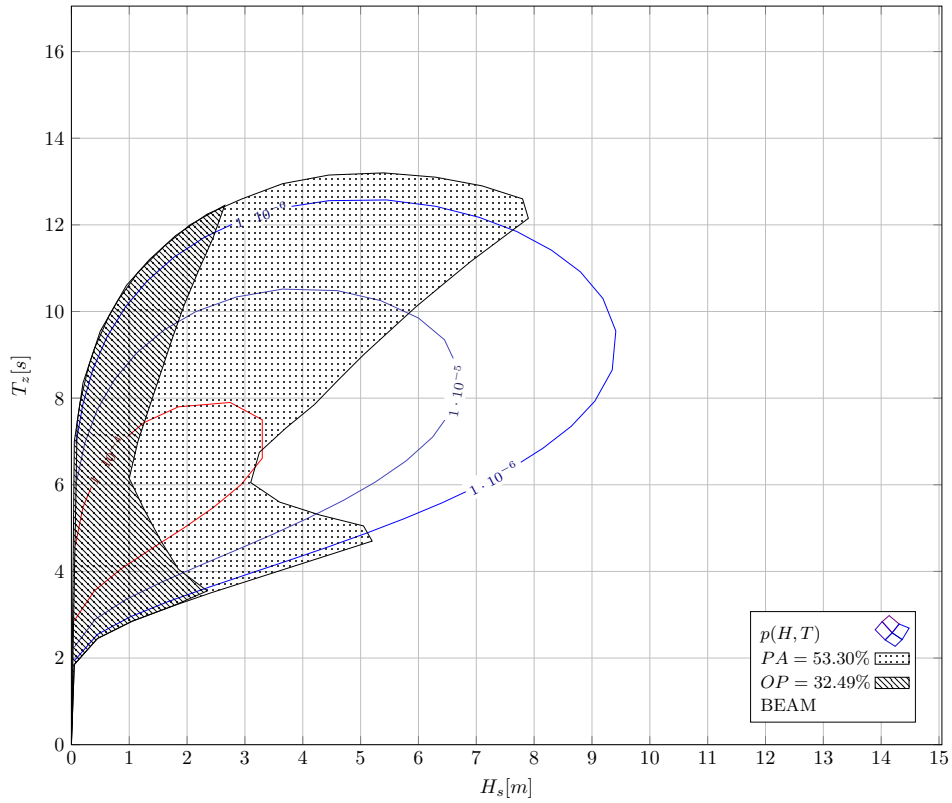


Figure 8.5: Operability map with beam waves

This overlapping of criteria, or masking of boundaries onto the scatter, is done for all nautical areas. The resulting OP (green) and PA (yellow) is shown in pie charts per area in figs. 8.6 and 8.7. Making a difference between head and beam main wave directions. These figures are visually very strong, requiring a recollection of the operability: The operability is calculated using the responses of the barge itself at open sea, without interaction with external equipment, shore or ships. And with the assumption that no shoaling or sheltering takes place and the crane will not be moving or actually operating, what would affect the heel and trim.

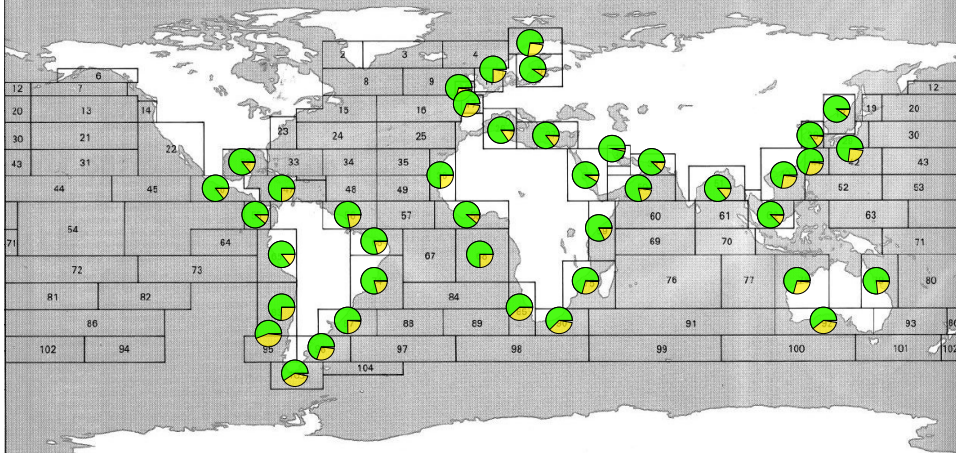


Figure 8.6: Operability map in head waves

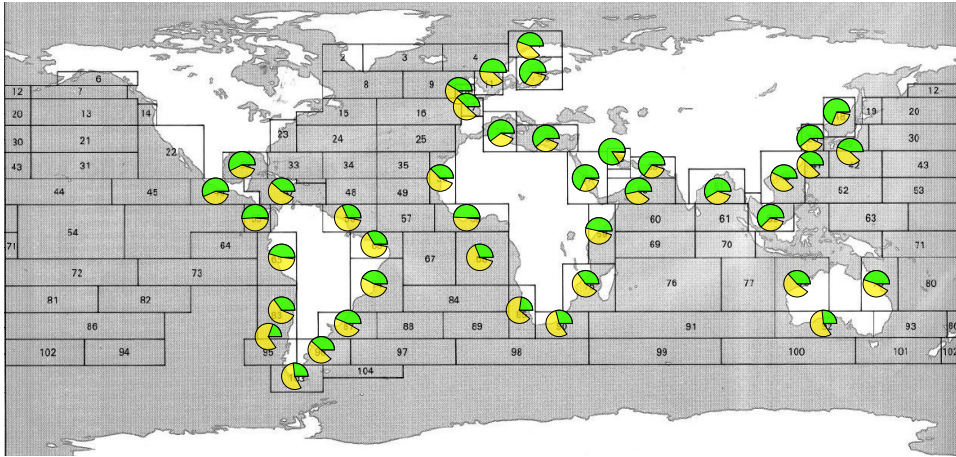


Figure 8.7: Operability map in beam waves

8.5 Probability Encountering Accelerations

Masking can also be done with the scatters and the acceleration response maps. This reveals the part of the scatter that is to be used in finding the maximum responses. The probability of encountering a maximum response is a combination of the $p_{N.E.}$ of the response in a seastate and the probability of occurrence of that seastate []. Simply put, the same accelerations can occur in different seastates, so the product of the probability of a seastate and the response in it can be summed to get a total $p_{N.E.}$. This relation is formulated in eq. (15).

$$p(a_r) = \int \int p(H_s, T_z) \cdot \exp\left\{\frac{-r^2}{2 \cdot m_0(H_s, T_z)}\right\} dH_s dT_z \quad (15)$$

This equation is solved by increasing the a_r and checking the sum until the probability drops below the assumed threshold, visualised in fig. 8.8 as different hashed areas.

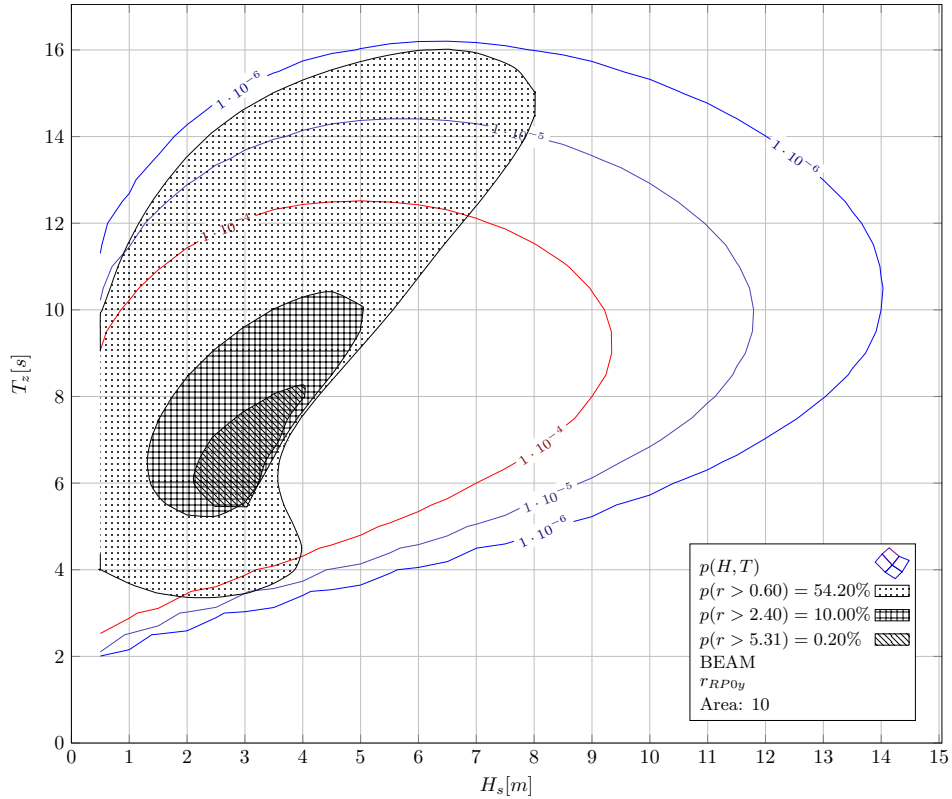


Figure 8.8: Combined $p_{N.E.}$ for area 10, RP:0 and acceleration in y direction

Mainly the maximum accelerations in y and z directions are of interest, and will not occur simultaneously since their phases are different in regard to the phase of the wave. Therefore, a primary direction is selected and calculated according eq. (15) at RP:0, the other directions and other RPs are set as secondary directions calculated with eq. (14), by using the mask from fig. 8.8 and taking the maximum out of that mask. The primary direction probability takes the chance of occurring seastate into account, while for the secondary directions it is presumed that the seastate is already reached. In this case the secondary accelerations are on the conservative side.

To get to a ULS, four states are calculated for every nautical area, from which the highest acceleration is selected as the ULS:

- Y-direction set as primary direction in RP:0, while under load of head waves, defined as load case YH ($a_{RP0,y,head}$)
- Z-direction set as primary direction in RP:0, while under load of head waves, defined as load case ZH ($a_{RP0,z,head}$)
- Y-direction set as primary direction in RP:0, while under load of beam waves, defined as load case YB ($a_{RP0,y,beam}$)
- Z-direction set as primary direction in RP:0, while under load of beam waves, defined as load case ZB ($a_{RP0,z,beam}$)

The solver itself, its programming structure, how it actually handles inputs and provides the output can be found in ???. Resulting in a set of the four cases, with highest probable acceleration in every RP, the roll and pitch motions, wave heights and wind speeds as seen in tables 8.1 to 8.3. And showing accelerations in x- and y-direction well below 0.50 g (4.91 m/s²) and in z-direction of well below 1 g (9.81 m/s²), as well as roll and pitch motion below 30 deg as LR states as motions to be used to calculate the ULS.

Table 8.1: ULS acceleration results from solver in m/s^2 (part 1)

	$a_{x,RP:0}$	$a_{y,RP:0}$	$a_{z,RP:0}$	$a_{x,RP:1}$	$a_{y,RP:1}$	$a_{z,RP:1}$	$a_{x,RP:2}$	$a_{y,RP:2}$	$a_{z,RP:2}$
YH	1.78	2.46	1.30	1.84	2.78	2.19	0.96	1.74	2.01
ZH	1.54	2.23	1.43	1.60	2.30	2.15	0.82	1.45	2.01
YB	0.95	3.62	1.53	0.98	3.35	1.78	0.46	2.05	1.71
ZB	1.20	3.54	1.66	1.25	3.35	2.02	0.58	2.05	1.93

Table 8.2: ULS acceleration results from solver in m/s^2 (part 2)

	$a_{x,RP:3}$	$a_{y,RP:3}$	$a_{z,RP:3}$	$a_{x,RP:4}$	$a_{y,RP:4}$	$a_{z,RP:4}$	$a_{x,RP:5}$	$a_{y,RP:5}$	$a_{z,RP:5}$
YH	1.50	2.40	2.12	0.88	1.68	2.12	0.32	0.99	2.12
ZH	1.30	1.99	2.02	0.75	1.39	2.02	0.26	0.81	2.02
YB	0.78	3.47	1.93	0.42	2.52	1.93	0.17	1.60	1.93
ZB	0.99	3.47	2.29	0.52	2.52	2.29	0.21	1.60	2.29

Table 8.3: ULS motion and sea state results from solver (part 3)

	Φ [deg]	Θ [deg]	H_s [m]	u_w [m/s]
YH	8.11	6.22	5	19.90
ZH	7.88	7.12	7	24.50
YB	10.50	2.97	3.50	15.96
ZB	10.50	3.29	3.50	15.96

DIMENSIONING PHASE

9 ULS Load Cases

One way to solely examine the loads and stresses on JR itself, the barge and crane combination can be simplified significantly. And by making use of the accelerations, motions and wind speeds found in section 8, all forces acting on the pedestal, crane and jib can also be simplified and superpositioned upon the JR. Making a distinction between the type of loads and by applying FoSs accordingly, a set of final load cases are defined and a complete ULS analysis can be done. But since the barge will have to comply with Class regulations, the first method defined by LR in section 2 is used to find a different set of loadcases. These load cases are used partially as a reference, but mainly as an ensurance of the useability of the design.

9.1 Model Load Cases

The whole barge, crane and JR configuration can be simplified as a rectangular frame as seen in fig. 9.1 and with only the design of the JR considered as the scope, the system can be simplified even further as in fig. 9.2. In this simplification, the crane masses and construction are distilled to a spring, and the forces acting on it in x direction are transferred to the JR top. To prevent the stowed crane acting as a stiff frame on deck, the JR top and bottom are presumed hinged.

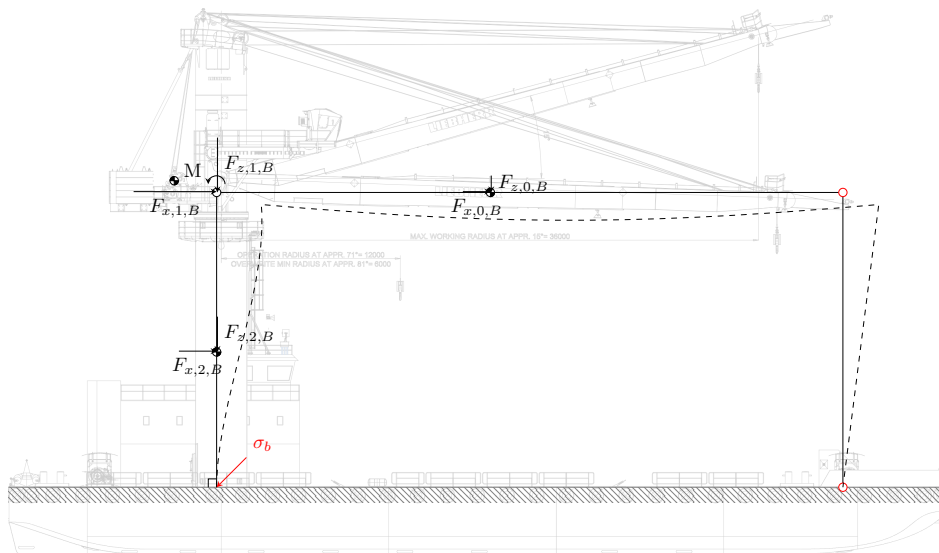


Figure 9.1: Crane barge simplified as a rectangular frame

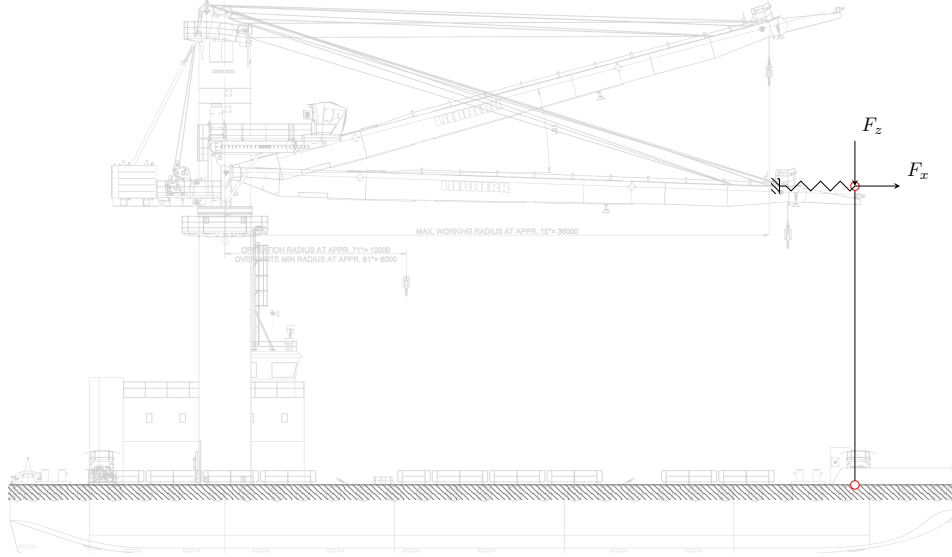


Figure 9.2: Further simplification of the system

With the crane modelled as a spring, the total force acting on the top of the jib with the accelerations found in section 8.5, the forces in x, y and z direction and wind pressures are calculated according:

$$F_x^{YH} = m_{jib} \cdot a_{x,RP0}^{YH} + m_{CT} \cdot a_{x,RP2}^{YH} + 1/2 \cdot m_{ped} \cdot a_{x,RP1}^{YH} = 490.69 \text{ kN} \quad (16)$$

$$F_y^{YH} = m_{jib} \cdot a_{y,RP0}^{YH} \cdot \mu_{p,j} = 42.15 \text{ kN} \quad (17)$$

$$F_z^{YH} = m_{jib} \cdot (a_{z,RP0}^{YH} + g) \cdot \mu_{p,j} + F_{pre} = 235.37 \text{ kN} \quad (18)$$

$$a_x^{YH} = a_{x,RP4}^{YH} = 0.88 \text{ m/s}^2 \quad (19)$$

$$a_y^{YH} = a_{y,RP4}^{YH} = 1.68 \text{ m/s}^2 \quad (20)$$

$$a_z^{YH} = a_{z,RP4}^{YH} = 11.93 \text{ m/s}^2 \quad (21)$$

$$F_w^{YH} = 1/2 \cdot c_j \cdot \rho_{air} \cdot u_w^2 \cdot A_{x,z}^{jib} \cdot \mu_{p,j} = 10.21 \text{ kN} \quad (22)$$

$$p_w^{YH} = 1/2 \cdot c_j \cdot \rho_{air} \cdot u_w^2 = 0.41 \text{ kN/m}^2 \quad (23)$$

Where:

YH : load case with y as primary direction and barge under beam waves

$\mu_{p,j}$: ratio between loads transferred to jib rest and crane pedestal

F_{pre} : pretension force according to Liebherr (10% SWL) [24]

DNV GL states a ULS analysis should take two sets of design load combinations into account, with different load factors. This FoS is split in a combination where functional and permanent loads are the main focus (a), and one where environmental have the focus (b), resulting in the following factors [6]:

- $\gamma_{f,G,a} = 1.20$: permanent load factor in load combination a
- $\gamma_{f,E,a} = 0.70$: environmental load factor in load combination a
- $\gamma_{f,G,b} = 1.00$: permanent load factor in load combination b
- $\gamma_{f,E,b} = 1.15$: environmental load factor in load combination b
- $\gamma_m = 1.15$: material load factor

And by combining these, the following overall factors are found:

- $FoS_{E,a} = 0.80$
- $FoS_{G,a} = 1.38$
- $FoS_{E,b} = 1.32$
- $FoS_{G,b} = 1.15$

Assuming the forces due to motion and gravity are functional and wind is environmental, the combination of different wave directions, primary acceleration directions and load factors lead to two ULS load cases (table 9.1 and ??).

Table 9.1: Load Case: 6 (YHa)

Fx @JRT	677.15	kN
Fy @JRT	66.38	kN
Fz @JRT	324.81	kN
py	0.41	kN/m ²
ax @JRM	1.21	m/s ²
ay @JRM	2.32	m/s ²
az @JRM	-16.46	m/s ²
FoS	1.00	-
Ref. Stress	235.00	Mpa

Table 9.2: Load Case: 7 (YBa)

Fx @JRT	358.62	kN
Fy @JRT	90.88	kN
Fz @JRT	330.25	kN
py	0.26	kN/m ²
ax @JRM	0.58	m/s ²
ay @JRM	3.48	m/s ²
az @JRM	-16.20	m/s ²
FoS	1.00	-
Ref. Stress	235.00	Mpa

An additional load case in operating or dropped down condition is composed, using only the response maps of RP:5 and the workability criteria and finding the highest response in table 9.3. For an impression on the positioning and the JR, see appendix D.

Table 9.3: Load Case: 8 (DD)

Fx @JRT	358.62	kN
Fy @JRT	90.88	kN
Fz @JRT	330.25	kN
py	0.26	kN/m ²
ax @JRM	0.58	m/s ²
ay @JRM	3.48	m/s ²
az @JRM	-16.20	m/s ²
FoS	1.00	-
Ref. Stress	235.00	Mpa

Where RP:3 is denoted as @JRT or jib rest top and RP:4 as @JRM or jib rest mid.

9.2 Load Cases Accepted By Class

Again, the conservative method of LR is used to provide a set of load cases to which the JR design has to comply, in order to assure the useability and applicability of the structure and can be found in tables 9.4 to 9.8. Any wind or loads due to acceleration are not simplified as one force at the jib rest top, but at their respective COGs, because the pedestal itself is also evaluated in the FEM calculations found in section 12. All details on their locations and denotations can be found in appendix D.

Table 9.4: Load Case: 1 (LRLC1)

m @JIB	39.20	t
m @CT	213.00	t
Fx @JIB	0.00	kN
Px @PED	0.00	kN/m ²
Fx @PT	0.00	kN
Fy @JIB	260.16	kN
Py @PED	1.58	kN/m ²
Fy @PT	125.50	kN
Mxx @PT	582.18	kNm
Myy @PT	0.00	kNm
Mzz @PT	-266.69	kNm
Fz @JRT	-45.00	kN
Px @JRCOG	0.00	kN/m ²
Py @JRCOG	3.92	kN/m ²
ax	0.00	m/s ²
ay	9.81	m/s ²
az	-18.31	m/s ²
FoS	1.56	-
Ref. Stress	235.00	Mpa

Table 9.5: Load Case: 2 (LRLC2)

m @JIB	39.20	t
m @CT	213.00	t
Fx @JIB	0.00	kN
Px @PED	1.58	kN/m ²
Fx @PT	109.09	kN
Fy @JIB	0.00	kN
Py @PED	0.00	kN/m ²
Fy @PT	0.00	kN
Mxx @PT	0.00	kNm
Myy @PT	566.32	kNm
Mzz @PT	0.00	kNm
Fz @JRT	-45.00	kN
Px @JRCOG	3.92	kN/m ²
Py @JRCOG	0.00	kN/m ²
ax	9.81	m/s ²
ay	0.00	m/s ²
az	-18.31	m/s ²
FoS	1.56	-
Ref. Stress	235.00	Mpa

Table 9.6: Load Case: 3 (LRDD1)

Px	0.20	kN/m ²
Py	0.00	kN/m ²
ax	9.81	m/s ²
ay	0.00	m/s ²
az	-18.31	m/s ²
FoS	1.56	-
Ref. Stress	235.00	Mpa

Table 9.7: Load Case: 4 (LRDD2)

Px	0.00	kN/m ²
Py	3.92	kN/m ²
ax	0.00	m/s ²
ay	9.81	m/s ²
az	-18.31	m/s ²
FoS	1.56	-
Ref. Stress	235.00	Mpa

Table 9.8: Load Case: 5 (LRUP)

Px	0.39	kN/m ²
Py	0.00	kN/m ²
ax	9.81	m/s ²
ay	0.00	m/s ²
az	-18.31	m/s ²
FoS	1.56	-
Ref. Stress	235.00	Mpa

10 Dimensioning

This section describes the usage and output of Multiframe (MF), a tool used to define the basic geometry of the concept, built up from a set of nodes, which are connected by members. The programme allows these members to have a section shape and material properties, and by applying loadcases, it calculates forces, moments and stresses in these members or nodes. In every design iteration, section geometry is adjusted to comply with any criteria or assumption, and nodes or members are added, altered or removed accordingly. Its output consists of forces, bending moments, bending and axial stresses, deflection and mass per node or member. Additionally to the static analysis, the software is capable of buckling and nodal analyses. Only the final and old design are relevant and described here.

10.1 Nominal Stress

A set of static analyses are made by applying the load cases to determine the nominal stresses per member in the new JR design. The same is done with the old design as a reference for the fatigue analysis done in section 11. The member numbering can be found in figs. 10.1 and 10.2.

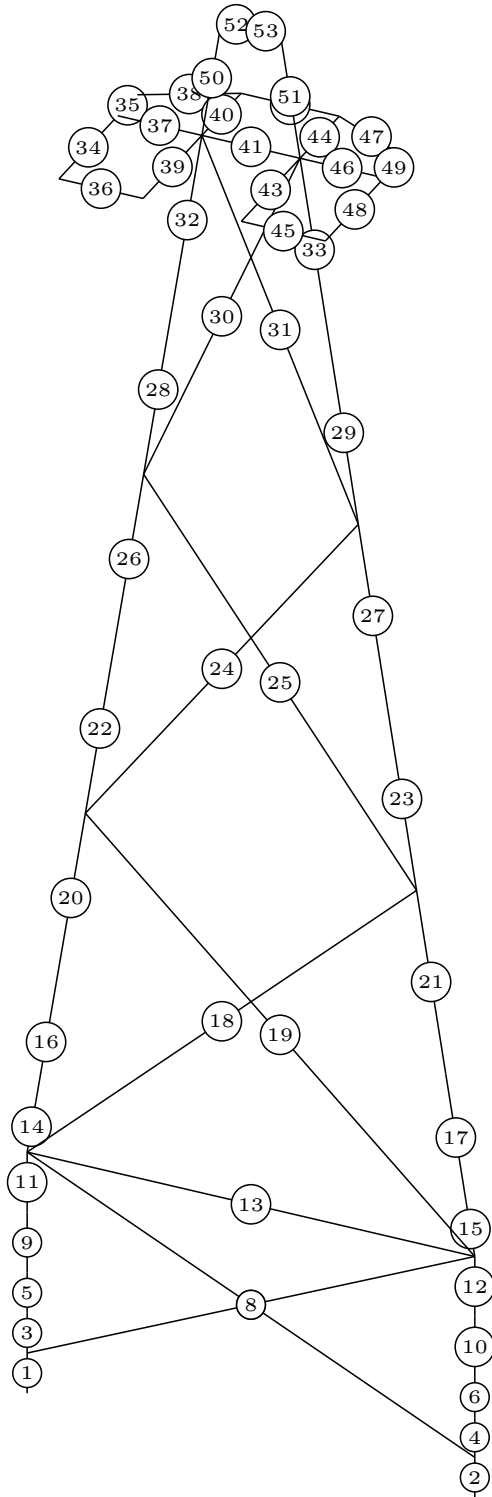


Figure 10.1: Member numbering of the new design

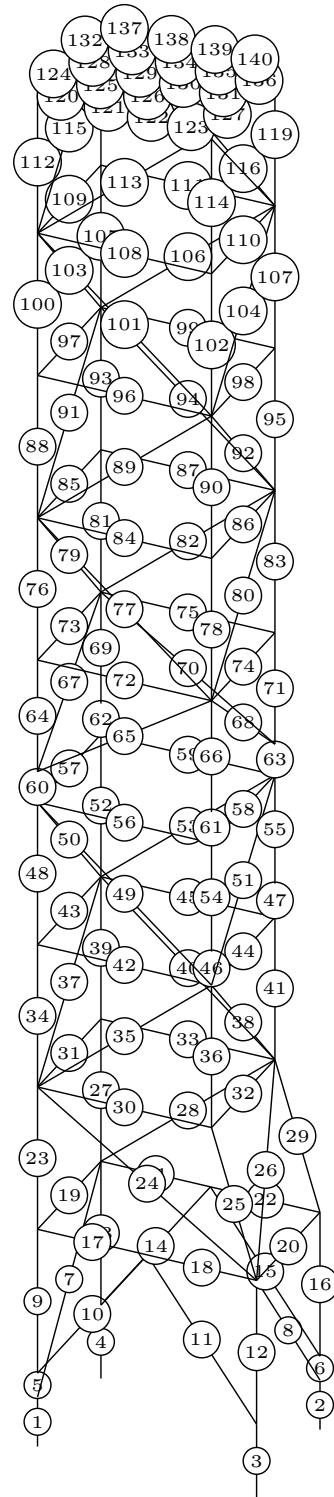


Figure 10.2: Member numbering of the old design

Per iteration, the outputs are parsed in the tool, to determine the maximum stresses in four outer locations on the perimeter of each member. The section and shape is then changed until the stresses are below a range of approx. 100 MPa, set as a first rough estimation of the allowable stress to minimize fatigue damage. The member numbers and their maximum stress levels are described as in fig. 10.3. Showing that all members satisfy the rough stress criterium.

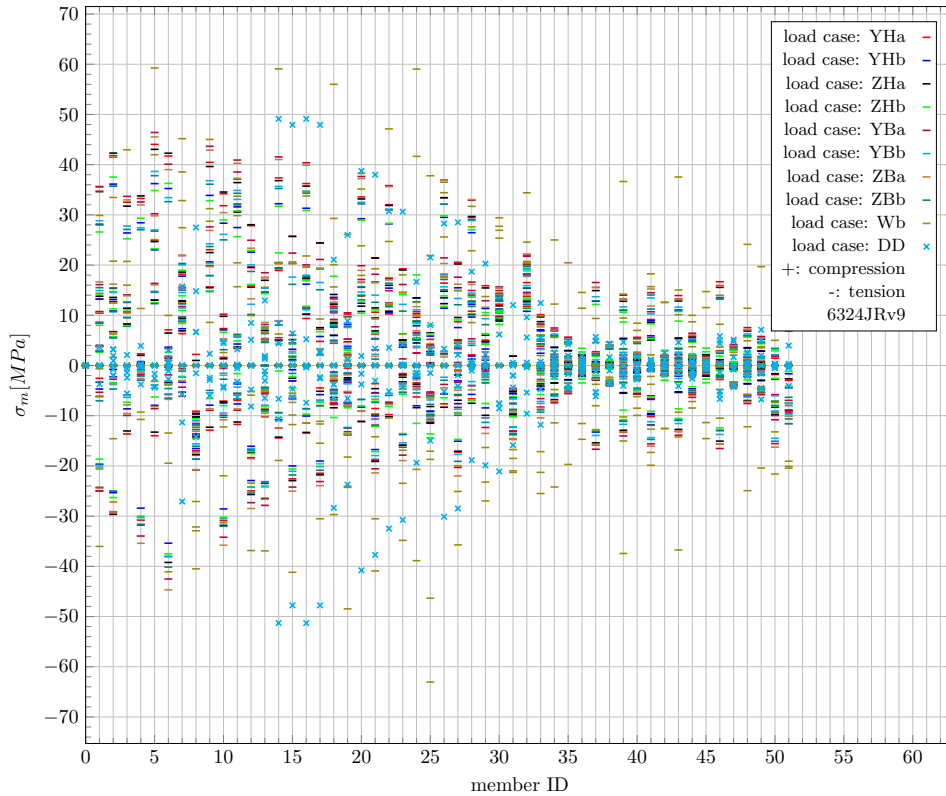


Figure 10.3: Stresses per member

10.2 Buckling

A second step in the iteration is a buckling analysis, shown as a buckling ratio (λ_b) relative to the load applied on top of the JR. In all iteration steps this factor stays well above 3.

Table 10.1: Buckling factor (λ_b) per load case

Load Case	λ_b
YHa	10.70
YHb	11.94
ZHa	11.18
ZHb	12.54
YBa	8.33
YBb	9.42
ZBa	8.59
ZBb	9.69
Wb	6.86
DD	

10.3 Natural Frequency

The last step in the design iteration is the modal analysis of the complete structure. As an additional criterium, P&B has defined a lower boundary of allowable modal frequency (f_r). Frequencies above f_r , will not be significantly generated by waves, engines or any other equipment on board and assures no resonance will occur. Tables 10.2 to 10.4 show the frequencies found for the JR in stowed, upright and free, and dropped down position, respectfully.

Table 10.2: Natural frequencies in stowed condition

Mode	Freq. [Hz]	Freq. [rad/s]	mod. mass [kg]	mod. mass [%]
1.0	5.14	0.31	1547.06	0.10
2.0	7.31	0.21	832.07	0.05
3.0	7.42	0.21	1468.74	0.10
4.0	9.46	0.17	3893.15	0.25
5.0	12.67	0.12	689.31	0.04
6.0	12.73	0.12	662.09	0.04
7.0	16.61	0.09	2978.04	0.19
8.0	17.42	0.09	109.47	0.01
9.0	17.72	0.09	1017.57	0.07
10.0	18.16	0.09	33.26	0.00

Table 10.3: Natural frequencies in operating, upright position

Mode	Freq. [Hz]	Freq. [rad/s]	mod. mass [kg]	mod. mass [%]
1.0	2.44	0.64	2353.93	0.15
2.0	4.79	0.33	1712.88	0.11
3.0	7.31	0.21	826.75	0.05
4.0	7.42	0.21	1556.06	0.10
5.0	12.19	0.13	1237.98	0.08
6.0	12.54	0.13	826.03	0.05
7.0	13.33	0.12	1886.90	0.12
8.0	14.60	0.11	609.82	0.04
9.0	17.01	0.09	141.00	0.01
10.0	17.73	0.09	1018.22	0.07

Table 10.4: Natural frequencies in operating, dropped down position

Mode	Freq. [Hz]	Freq. [rad/s]	mod. mass [kg]	mod. mass [%]
1.0	4.37	0.36	1709.18	0.11
2.0	5.36	0.29	1264.17	0.08
3.0	7.32	0.21	821.01	0.05
4.0	9.54	0.16	921.76	0.06
5.0	12.61	0.12	431.82	0.03
6.0	12.62	0.12	703.44	0.05
7.0	14.69	0.11	1331.40	0.09
8.0	17.69	0.09	1065.47	0.07
9.0	17.89	0.09	36.92	0.00
10.0	18.26	0.09	32.57	0.00

During the design iterations, the JR is stiffened up significantly in x-direction, to get the natural frequency above f_r , mainly in the dropped down condition. The free upright position is disregarded in this case, since bringing the frequency up would mean too much addition of stiffness by changing the complete design. Additionally, this position would only occur when transitioning from stowed to operating condition.

10.4 Weight

In each cycle, the mass of the total construction is checked to be below the mass of the old JR. And in the last cycle, the total mass of the JR is approximated on $m_{JR} = 14.46$ ton , includeing 2 ton additional weight due to stairs and other structural details. This is a couple of tonnes below the benchmarked mass of = 19.40 ton .

DETAILING PHASE

11 Fatigue

The fatigue cracks found in the first generation of the JR is found to be originated in the stiff and slender design. This causes high stress concentrations or hotspots in and near joints of the K-frame. During the transport, these hotspots are loaded cyclically allowing existing cracks or imperfections to grow until failure occurs. Making an accurate estimation of this crack growth and consequentially the fatigue life estimation is dependent on many factors like the number and amplitude of load cycles, compressive or tensional stress, mean stress, initial crack size, shape, material properties and temperature [1, 7, 25, 11, 13].

Different approaches are used to determine fatigue life, either based on cumulative fatigue damage theory (CFD) - like rainflow counting in combination with the Palmgren-Miner rule - or in combination with fatigue crack propagation theory (FCP) [20]. Commonly, stress concentrations are represented as a stress concentration factor (SCF) and a local nominal stress [13] and its cycles are determined by the load cycle history or time-domain simulation, which would need detailed history data or extensive simulations. Dirlik proposed an empirical model which provides the rainflow ranges using the spectral moments of an expected stress spectrum [18, 12] and its formula can be found in section 11.3.

11.1 FAT Codes and SN-Curves

At this design stage, only the basic shape and dimensioning is set up for the JR. The detailed construction is as recursively dependant on any stress concentration, as it is on the ULS. A prediction on the nominal stresses in the members can be made, which are dominated by the loads and dimensioning of the individual members and any connection between the members is dominated by SCFs and the nominal stresses. For a large number of different (welded) connections, the allowable nominal stress is standardized into stress cycle curves (S-N curves) and fatigue class (FAT) proposed by International Institute of Welding (IIW) [13], where any SCF is taken into account. For every connection or beam element, a type of standard connection is selected, and its FAT or S-N curve used as reference for further fatigue analysis.

11.2 Stress Spectra

Nominal stresses found in the JR depend on individual member geometry and loads. Assuming the JR will be double hinged on both top and bottom, it will only be able to withstand loads in lateral and vertical directions. This implies that the structure will only need to withstand load cycles in those directions, presuming the stiffness of the hinge is negligible in comparison to the stiffness of the jib rest itself. By applying an acceleration of one in one of the directions, ratio between acceleration and stress (γ_σ) can be found per member per direction. This ratio is used as a transfer function, to get a probable stress spectrum per area according eq. (24) and plotted in fig. 11.1.

$$S_\sigma = |\gamma_\sigma|^2 S_a \tag{24}$$

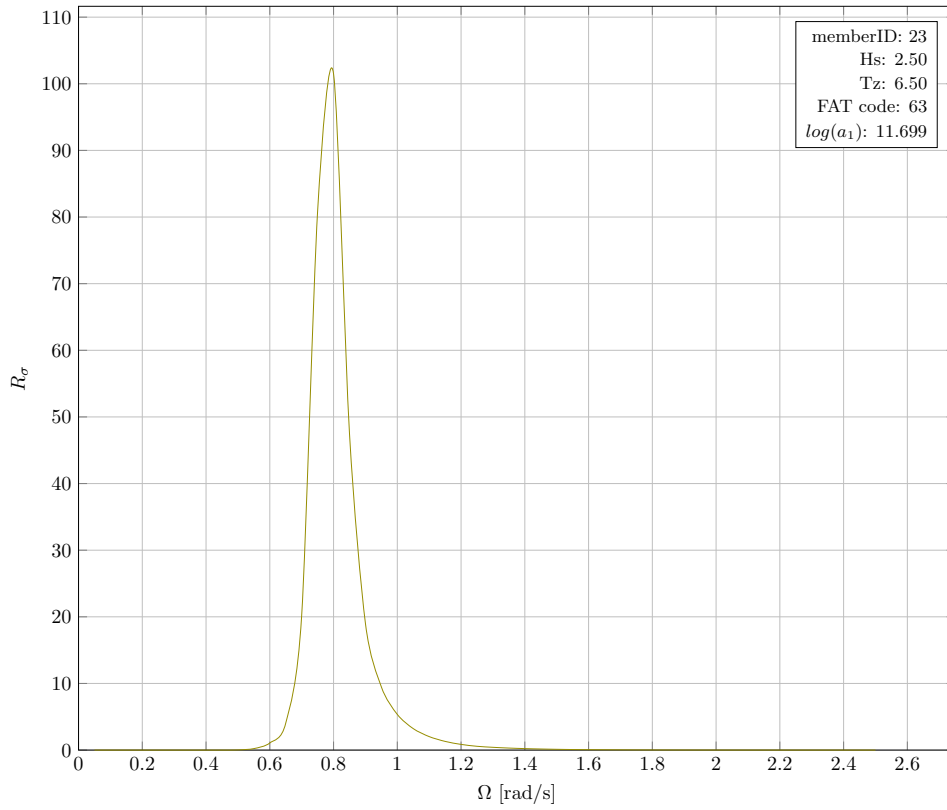


Figure 11.1: Stress spectrum in member no. 23 due to lateral acceleration

11.3 Fatigue Lifetime Estimation

Dirlik's has developed an empirical model to estimate the stress cycles encountered during a structure's lifetime, using a broad band load spectrum [12]. Its formula, using the spectral moments, representing the number of cycles per stress range is given in eq. (25) [12].

$$N(S) = E[P] \cdot T \cdot p(S) \quad (25)$$

Where:

$$p(S) = \frac{\frac{D_1}{Q} \cdot e^{-\frac{z}{Q}} + \frac{D_2 \cdot Z}{R^2} \cdot e^{-\frac{z^2}{2R^2}} + D_3 \cdot Z \cdot e^{-\frac{z^2}{2}}}{2\sqrt{m_0}}$$

$$D_1 = \frac{2(x_m - \gamma_d^2)}{1 + \gamma_d^2}$$

$$D_2 = \frac{1 - \gamma_d - D_1 + D_1^2}{1 - R}$$

$$D_3 = 1 - D_1 - D_2$$

$$Z = \frac{S}{2\sqrt{m_0}}$$

$$Q = \frac{1.25 \cdot (\gamma_d - D_3 - D_2 \cdot R)}{D_1}$$

$$R = \frac{\gamma_d - x_m - D_1^2}{1 - \gamma_d - D_1 + D_1^2}$$

$$\gamma_d = \frac{m_2}{\sqrt{m_0 \cdot m_4}}$$

$$x_m = \frac{m_1}{m_0} \cdot \sqrt{\frac{m_2}{m_4}}$$

T : time in seconds

At this detailing phase of the design, this formula is used for every member in every seastate, resulting in a histogram showing the expected cycles per seastate. Figure 11.2 shows the expected cycles for one of the members in the new JR design.

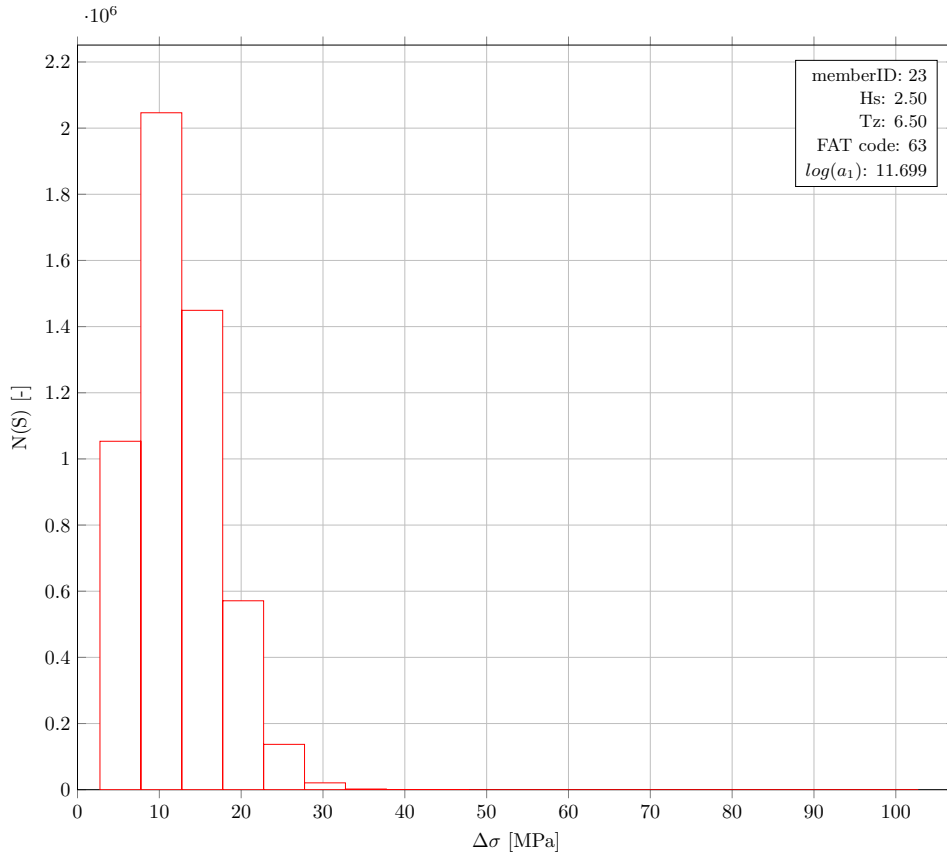


Figure 11.2: Expected cycles per stress range per year

The cycle estimation can directly be multiplied by the $p_{N.E.}$ per sea state and the probability of encountering such a sea state from the scatters. Taking the duration of a towing trip and its passed nautical areas from the voyage reports, and cumulating the damage using the Palmgren-Miner rule, an estimation of surpassed lifetime can be made. At this point, the accuracy of the total probability is highly debatable since this model uses the long term probability of a seastate and would not be very representable for a short term tow, especially since weather and tow conditions are monitored day by day.

As one case study, a rough towing trip is considered as a voyage of two weeks, in which $H_{s_{tow}}^{max} = 2.30$ m as from the reports and $T_{\lambda=l} = 6.35$ s as the response in head waves would be the highest in waves with this period. In this case, the two designs are subjected to such a tow and the expected damage per member is calculated. The old design shows significant damage in almost all of the vertical members, in both wave directions. Several would even exceed their allowable expected cycles by two or three times. The expected damage of the new design is a lot lower, with some members reaching approx. 7% of their total cycles. More detailed data on the cycles and estimations can be found in appendix B, along with two other fatigue cases, simulating a lifetime in or near one nautical area.

VERIFICATION PHASE

12 Final Design and Design Verification

As a last step, the complete detailed design is drawn with precise dimensioning of the main elements, such as the members, flanges and stiffeners, and rough dimensioning of the hinges, connecting eyes and secondary additions to the structure. The MF basic dimensioning, like node coördinates and section data is exported to Autocad and accompanied by any further data, making sure the materials with those dimensions are standardized and available worldwide. This results in the final design shown in fig. 12.1. Any further data, drawings or calculations made to get to this design can be found in appendix D.



Figure 12.1: Drawings of the final design (6324JRv9) *Blurred because it is considered classified*

Using these final dimensions, a FEM model is made in Siemens NX (NX) by Marine Design Engineering Mykolayiv (MDEM) and with the added loadcases defined in section 9 used as a verification of the ULS. Any minor adjustments to the structural detail of the design are done, mainly by looking at the peak stresses as shown in an example in fig. 12.2 and the natural frequencies as in an example shown fig. 12.3.

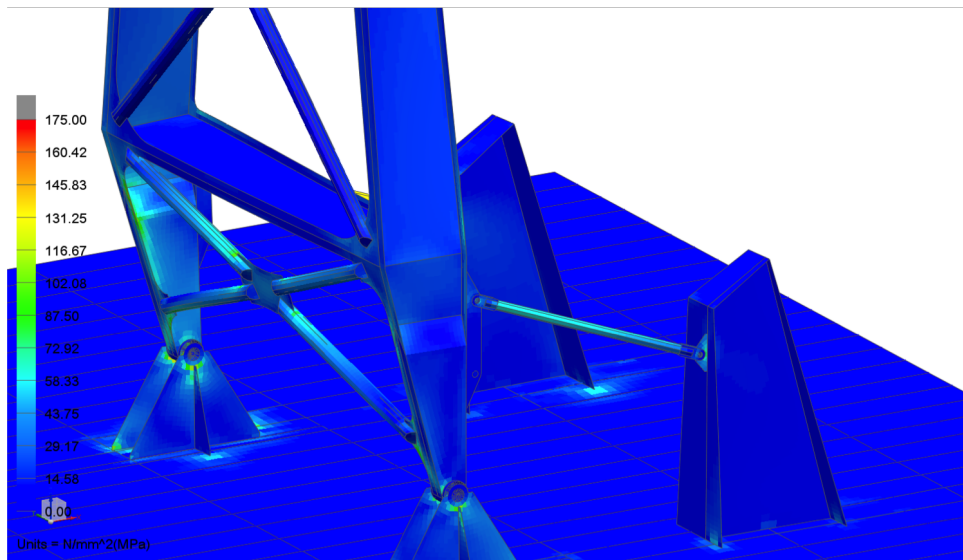


Figure 12.2: Stresses in bottom part

nm
:

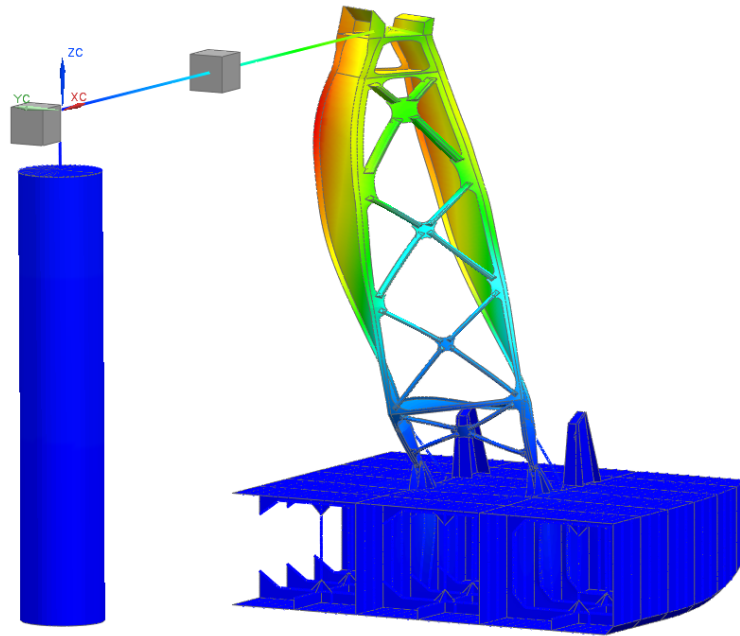


Figure 12.3: Natural frequency analysis

13 Conclusions

13.1 Discussion

13.1.1 Model Transparency

The barge is developed by combining a standard pontoon, standard crane and additional equipment. Using the standardized components enables an efficient and fast development, but also introduces a need for transparency, as standardization of the separate products could bring in different Class regulations and restrictions. Both the crane and pontoon are subjected to extreme design conditions, but their interaction may bring up a different insight on the overall design or even the applicability of the Class regulations. Identifying a bottle-neck in the design and knowing where its dimensioning or choice is originated, speeds up the development and optimisation. For example: If the average operability in an area is well below optimum, should the barge still be designed for this area? And one used as a criteria in this thesis: In what extreme sea state is the crane pedestal expected to fail, and should the JR be designed to withstand even higher accelerations? As standardization is one of the key design philosophies, it is beneficial to see what effect addition or adaptation of components on the barge. Depicting this in a graphically per sea state and allowing quick adjustments provides a fast way of gaining insight on the overall design.

13.1.2 Sea States

To get a more detailed idea about the forces the JR has to withstand, more detailed information about the location of operation is needed. Detailed information on local weather and sea states to get to a more accurate estimation on the loads. Knowing the loads are highly dependant on the location where the barge would need to operate, and a design criterium of worldwide operability, suggests that detailed information of all around the world would need to be analysed. As analysing all areas would be a difficult task, a representation of all sea states is chosen to be a JONSWAP-spectrum. These spectra are the base of finding the maximum responses and are considered to be sufficient for finding a reasonable worldwide ULS. In some areas and in some seastates, this JONSWAP-spectrum would not be the best fit, and it would be interesting to see if and how a different spectrum would result in different accelerations, and consequently a different ULS.

13.1.3 Worldwide Operability

Using the long term wave statistics and responses as a determination on operability, the barge shows promise in near coastal areas just above the equator and more sheltered seas, as long as the main wave direction are head waves. In this operability determination, only the responses of the barge itself are used, without interaction with other vessels, mooring or sheltering. If the barge would be moored in between a bulk carrier and feeder vessel, or would be transshipping in a harbour, this operability would increase. P&B states an ideal operation time would be around 90% and could be met in the more sheltered conditions. The operability seems to decrease below the equator and the barge would be less usable in near coastal areas around Argentina, South Africa and Australia, looking at a yearly average.

The first restricting condition in this operability analysis are the accelerations in the crane top, at the drivers position, ensuring comfortable operation. The following restriction is the allowable roll dictated by Class. According to this regulation, the crane will enter a safety mode when this angle is met. In this analysis, only responses of the barge are used to calculate the accelerations and roll or heel angles, and rotating and operation of the crane would add to these responses, reducing the operability.

13.1.4 Load Cases

As seen in section 9, there is a significant difference between the load cases found following Class and the load cases found using the method described in section 8. Lower loads are expected in the latter, because the more conservative method of LR is based on extreme weather conditions and the one proposed in this report takes more detailed information on the barge and its operational locations into account. Accepting the assumptions and simplifications of the more detailed method, would mean a large decrease in JR size. Maybe other structures on deck could be downsized as well. If this method is not accepted, it still shows the benefit of an alternative

calculation method, especially when weight reduction is one of the design criteria.

13.1.5 Jib Rest Design

A JR design is found, which folds down, and is lighter than the design of the first generation. The frequency analysis showed a need to increase the stiffness of the frame. The criteria of f_r for an eigenfrequency is originated in standards within Damen, to stay well away from frequencies induced by waves, propellor or engine. The applicability of this standard should be investigated more thoroughly, because the barge will not be engine propelled, and allowing a lower frequency would result in a less stiff frame and more weight reduction.

13.1.6 Pontoon Dimensions

The preliminary calculations show relatively low deformations (section 4.1), even in waves of approx. 4 m. In further calculations the deck is considered to have an infinite stiffness. This allows simplification of the modeling of the barge, crane and JR, but also shows a hull construction that might be too stiff for its application. If the deck load prerequisites and further analysis would allow this, there is a large potential for weight reduction.

13.1.7 Fatigue Lifetime Estimation

The estimation on fatigue lifetime is based on the stress spectra in every member by using an acceleration in either the y or z direction. The cumulative damage is then calculated by adding the stress cycles of both directions. Since the accelerations of in both directions are not coupled - neither phases are taken into consideration - this gives a rough estimation. A better estimation can be found by relating the two directions and by summing the stress levels instead of their cycles. The relating of the accelerations would need an introduction of a relation of the two directions by using the reponse phases. An other way to add detail to this estimation is by taking the effects of mean, tensional and compressive stresses into account.

13.2 Conclusions

LR provides a conservative method to determine loads acting on a structure on a floating crane, based on extreme values. As specific data on the CB6324 is available, a different method is proposed, using motion responses, sea state data and criteria as an input. This method is based on probability of encountering a sea state, the probability of non-exceedance of a wave in such a state and the operability, to predict accelerations. For this, a tool is developed, which automates the calculations for the maximum expected load in worldwide near shore operation and transport. It takes wave data, direction and spreading, wind speed, roll, pitch, flooding angle and pedestal (bending) stresses into account to find these loads. Therefore, objective item A is considered reached. The loads are as expected significantly lower than found by applying the method proposed by LR.

A new JR design follows from the load cases found with the proposed method. Peak stress levels in this design are well below the yield stress. The modal frequency in two of the three positionings comply with the criteria, being above $f_r = 4.00$ Hz . The buckling factor per member are all well above $\lambda_b = 3.00$. The JR is placed or can move out of the working zone and the total mass of the construction is reduced by $\Delta m = 4.94$ ton , making the JR comply to the main design criteria.

The tool is extended with a fatigue prediction component, which again uses the sea state data plus the geometry of the JR. The old design showed fatigue crack growth within two weeks in rough weather, as well as in the prediction as in the actual situation twice. The new is predicted to have spent a maximum of 7% of its fatigue lifetime.

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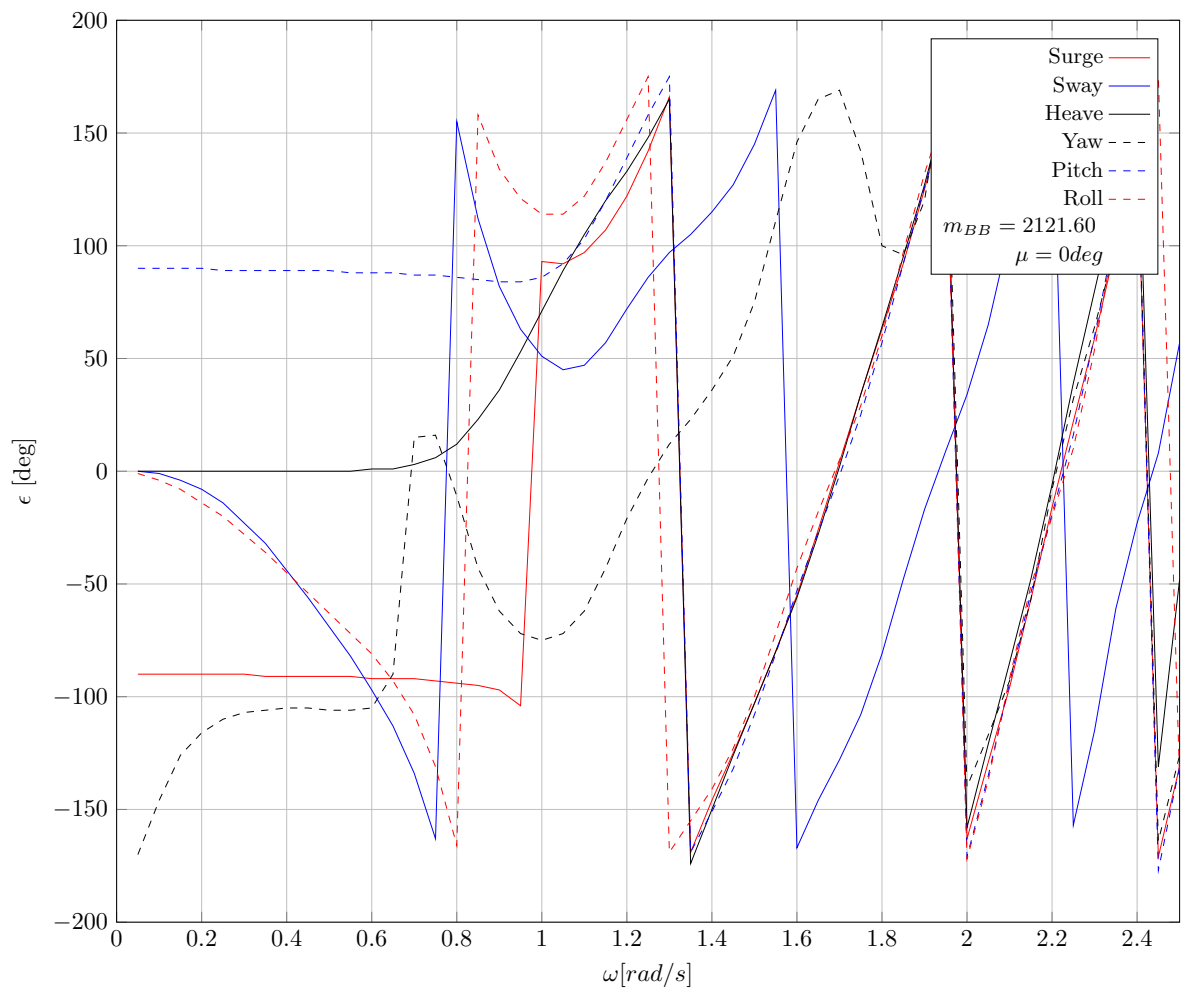
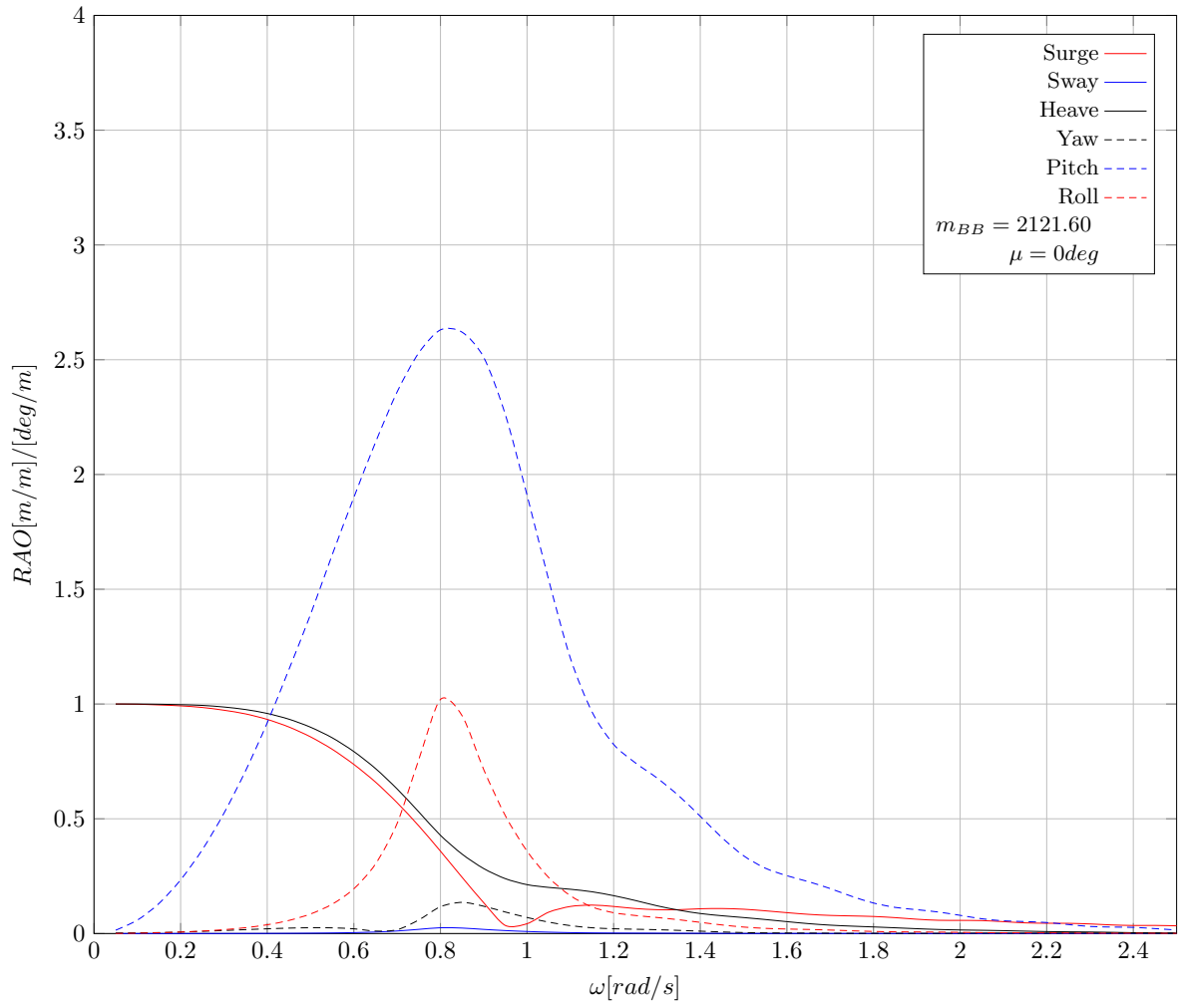
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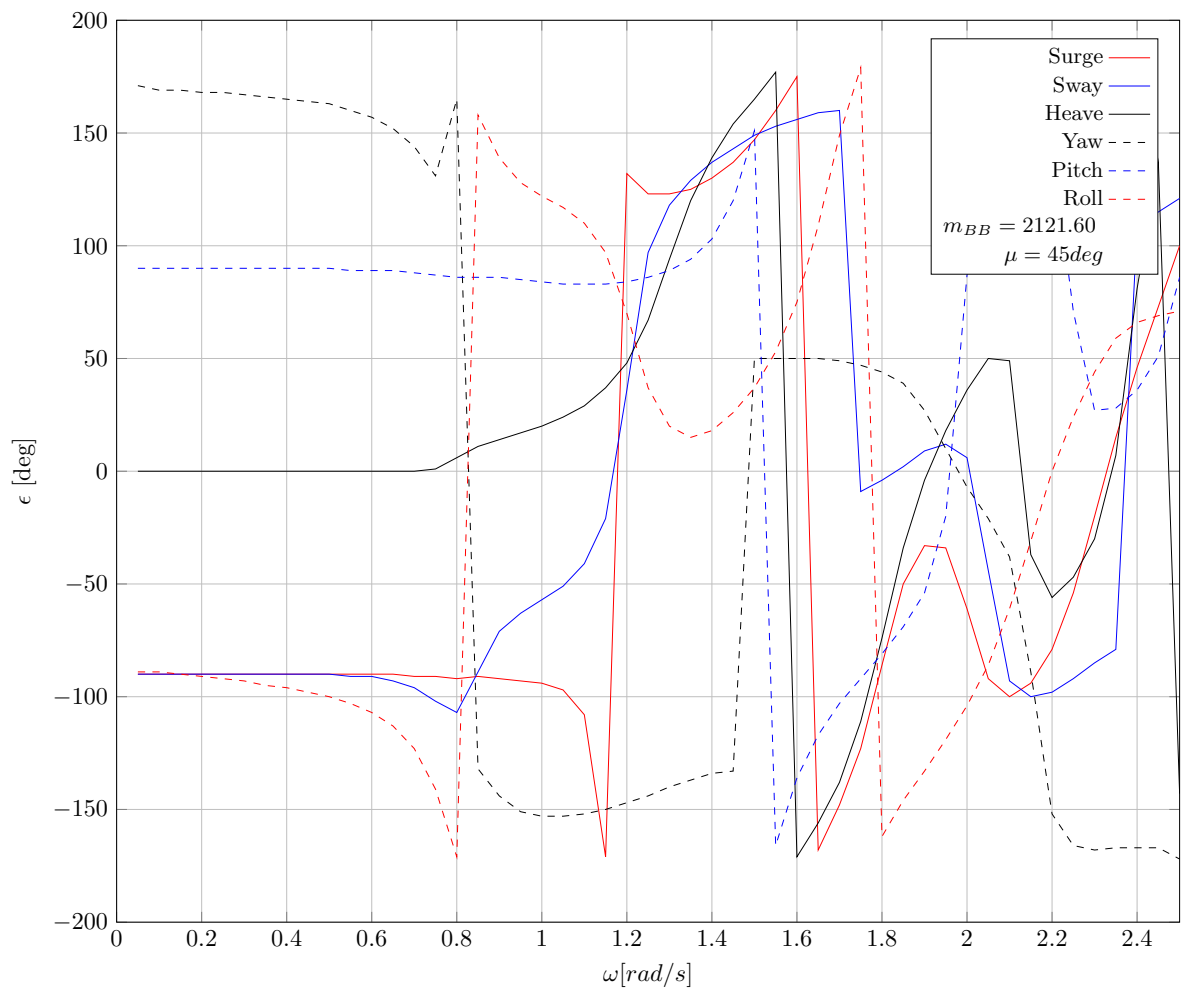
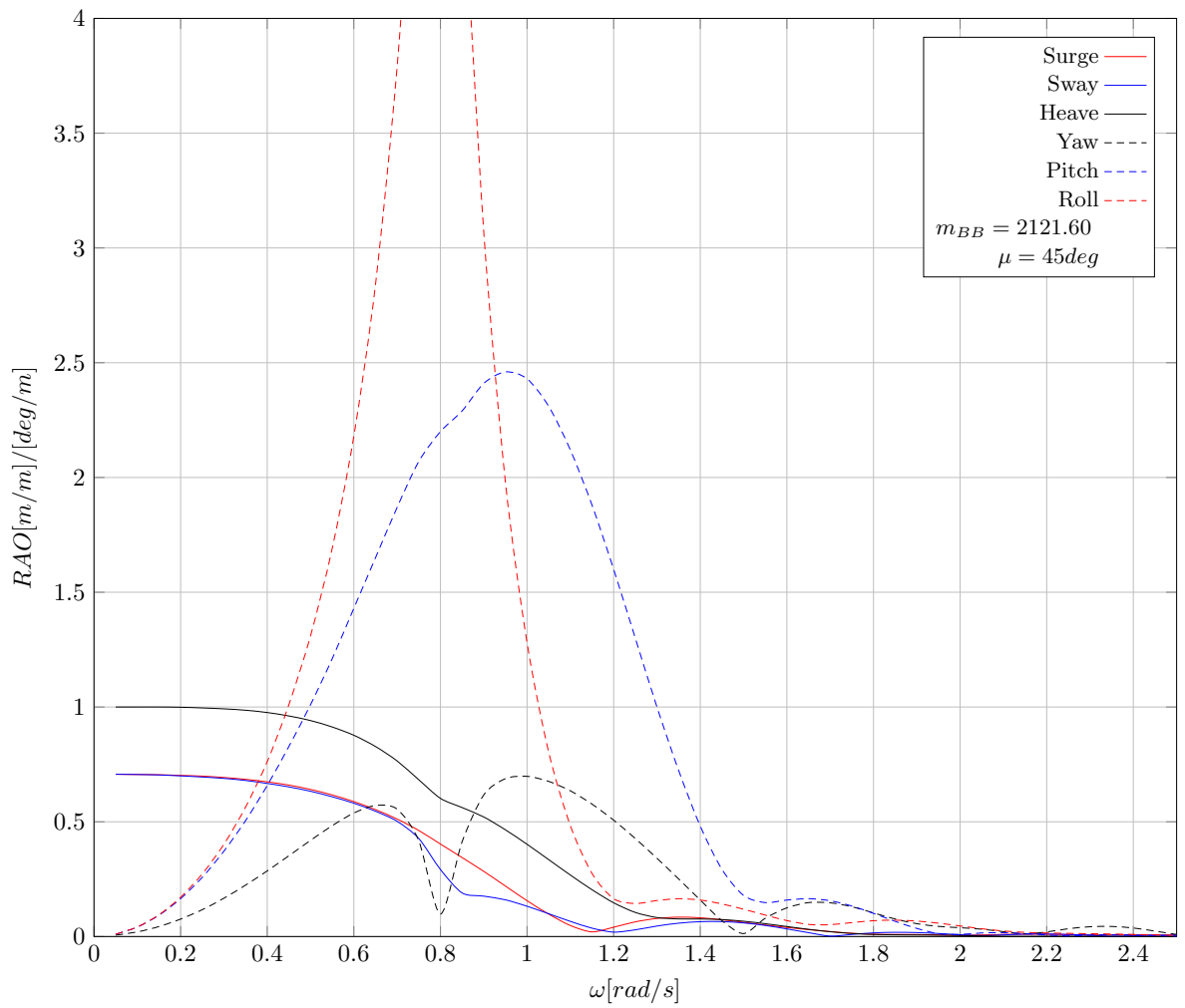
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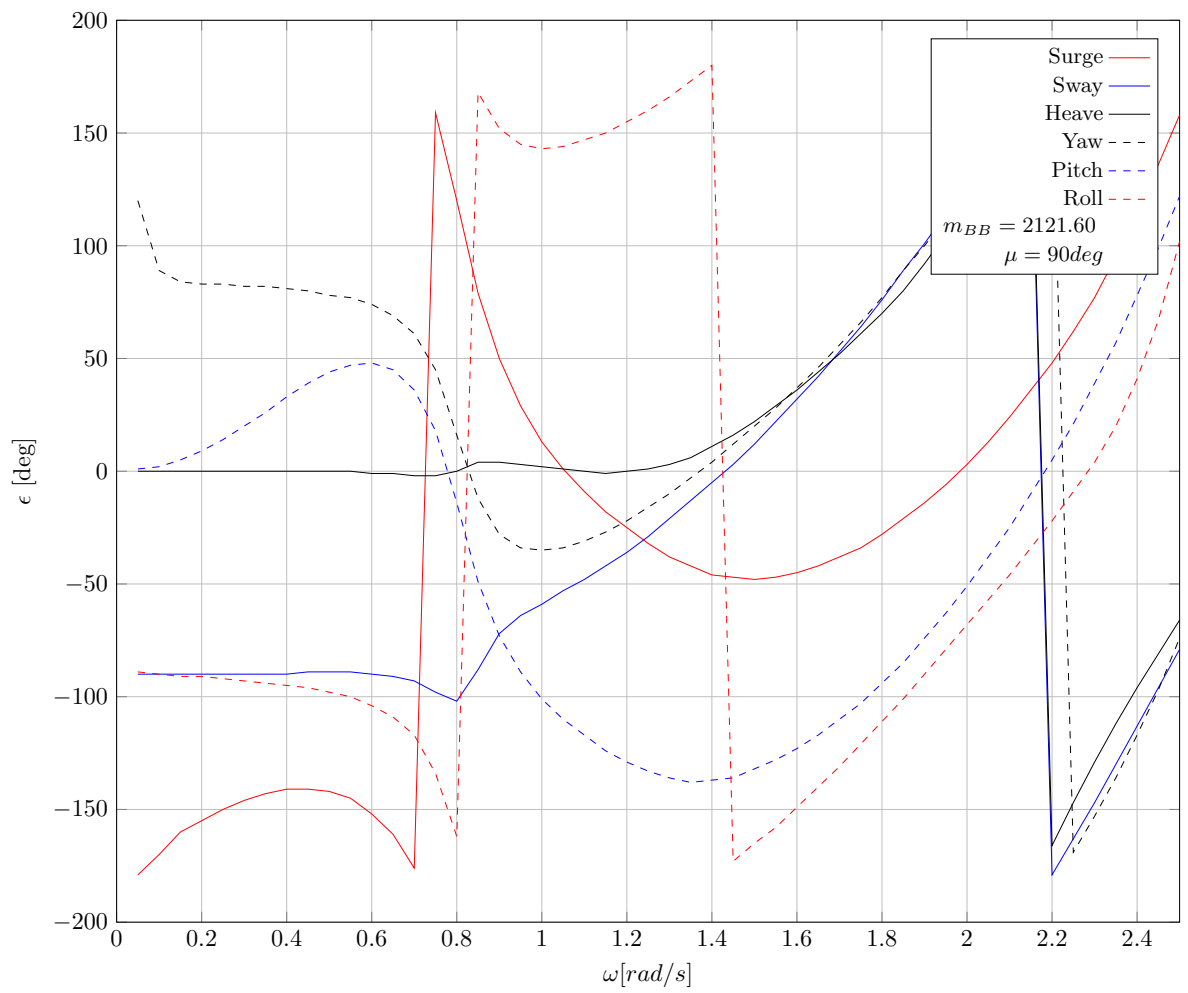
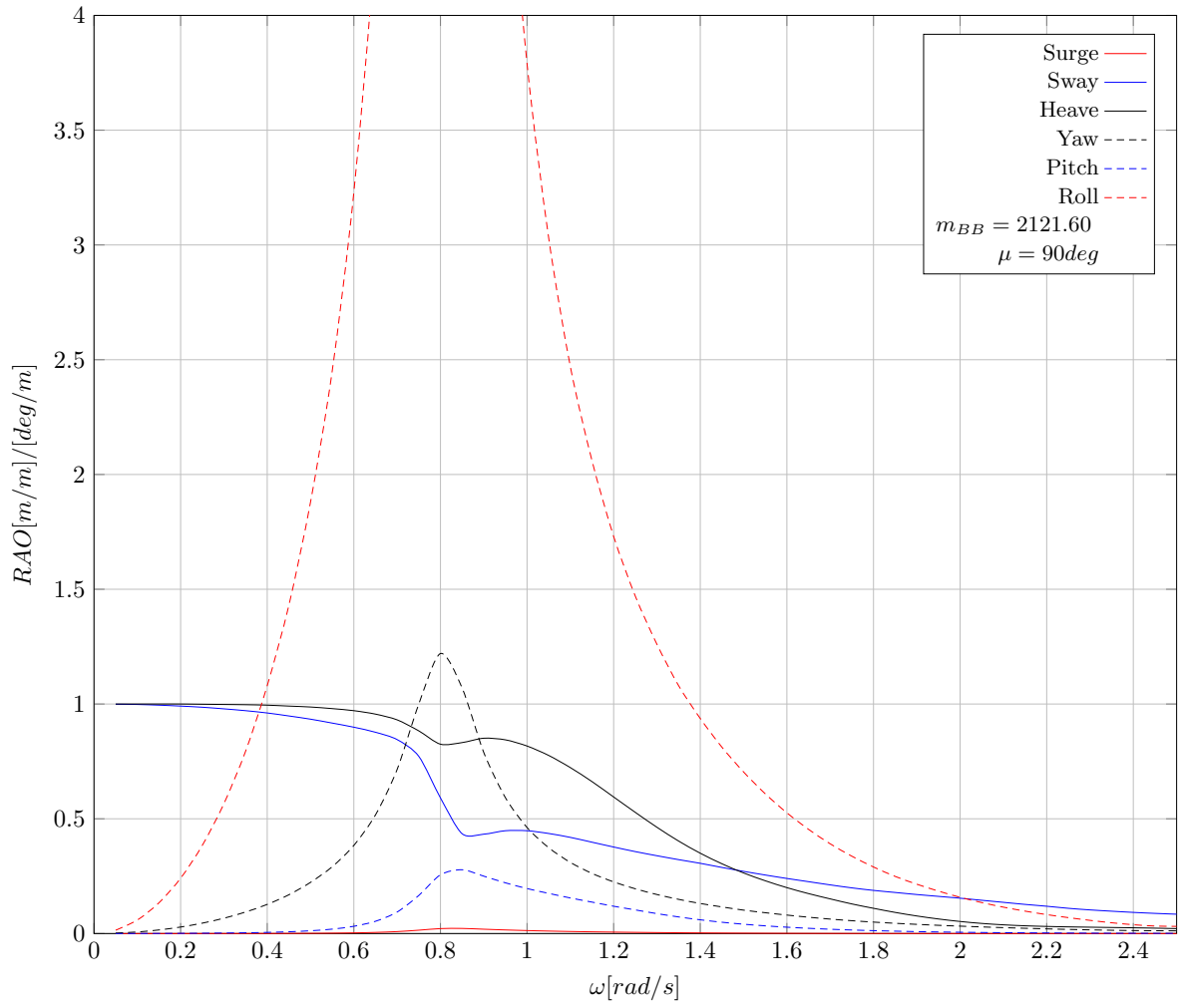
The following pages show the graphs relating to the barge motion responses found using the methodology described in section 2, in the following order:

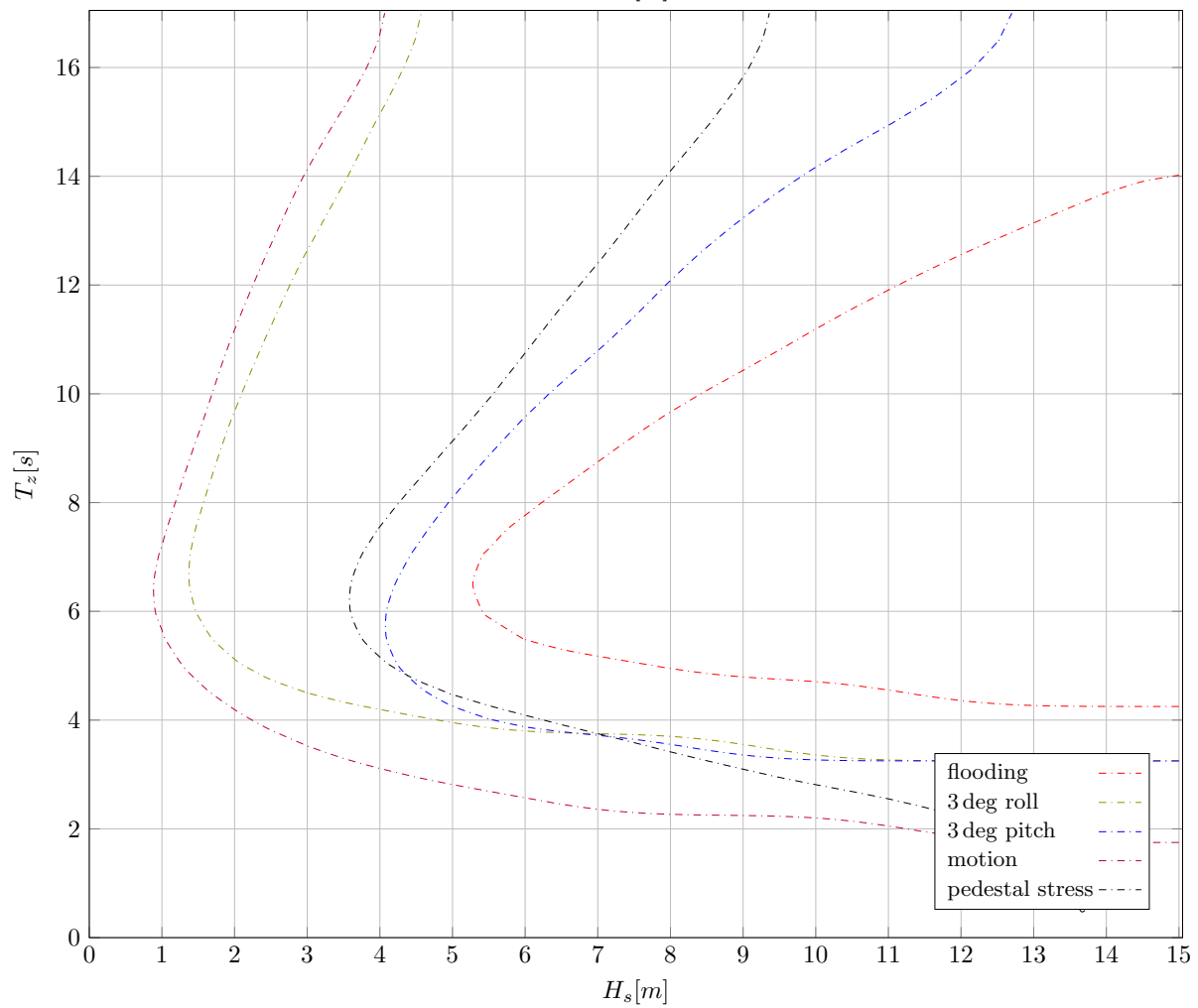
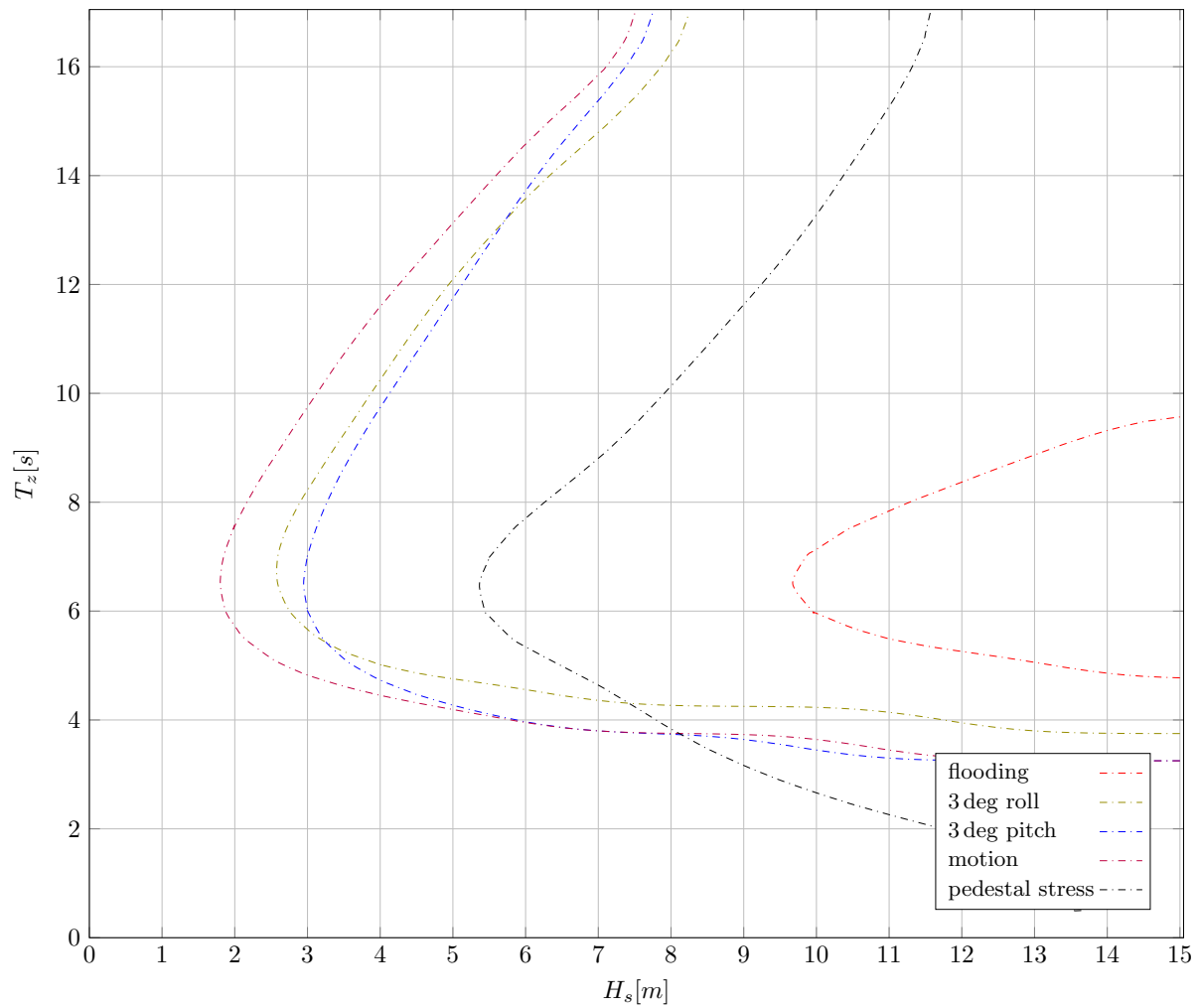
- RAOs
- Phases
- Criteria
- Motion responses calculated with head waves as main direction
- Motion responses calculated with beam waves as main direction

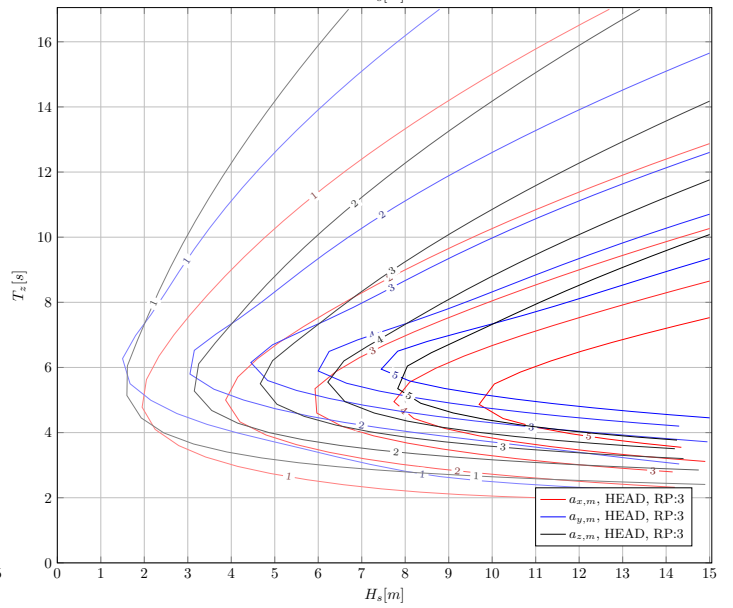
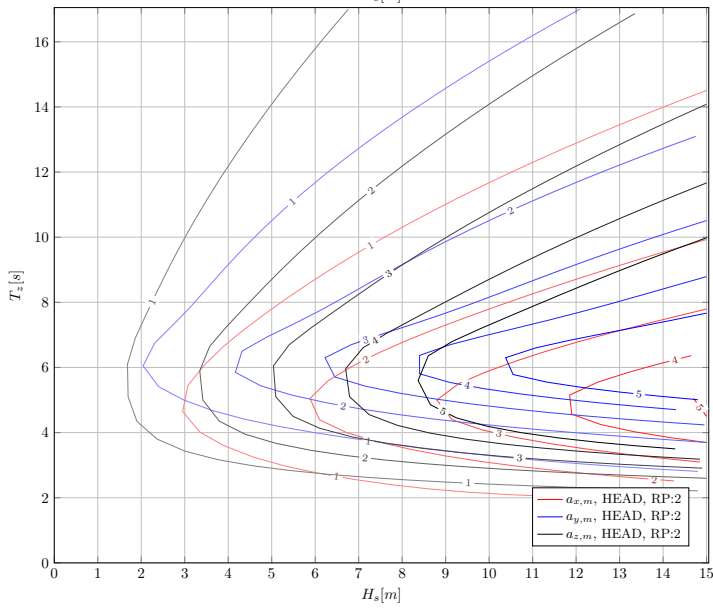
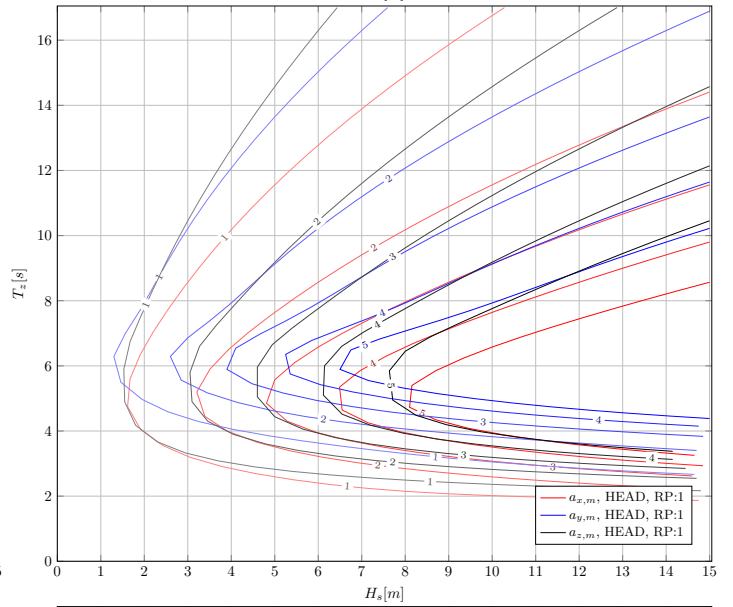
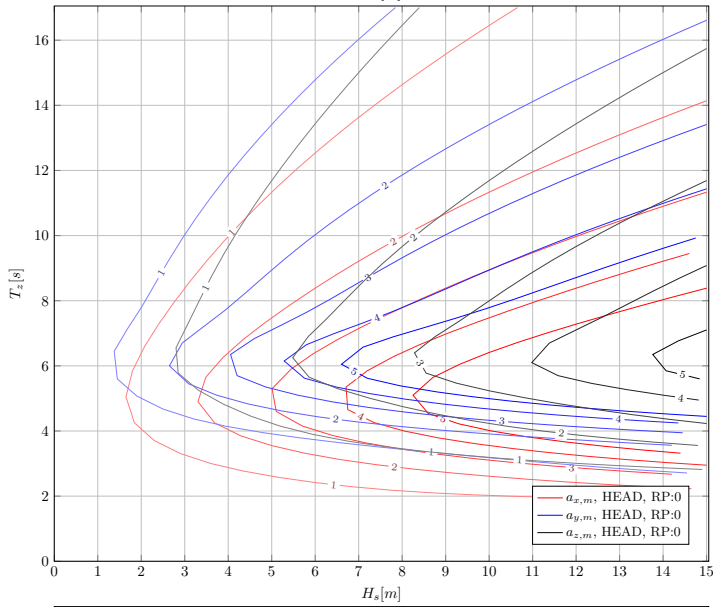
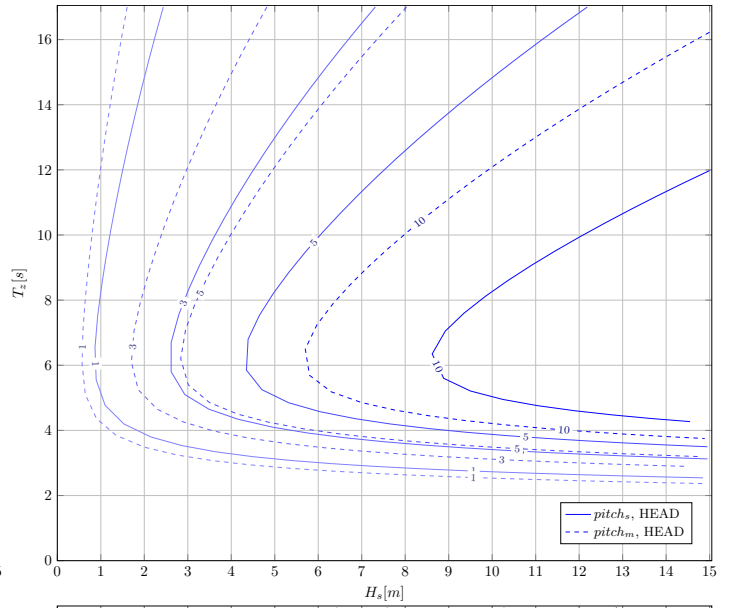
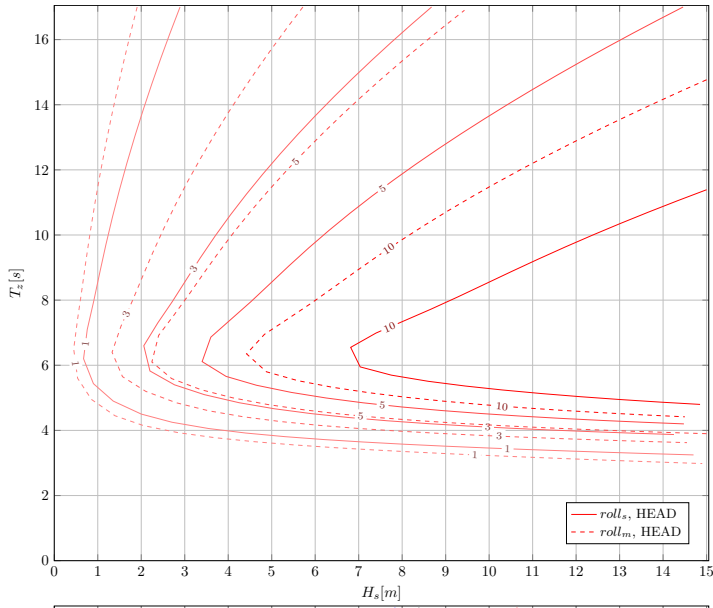
In these graphs, roll, pitch and yaw are in [deg], accelerations in [m/s²] and stresses in [MPa]

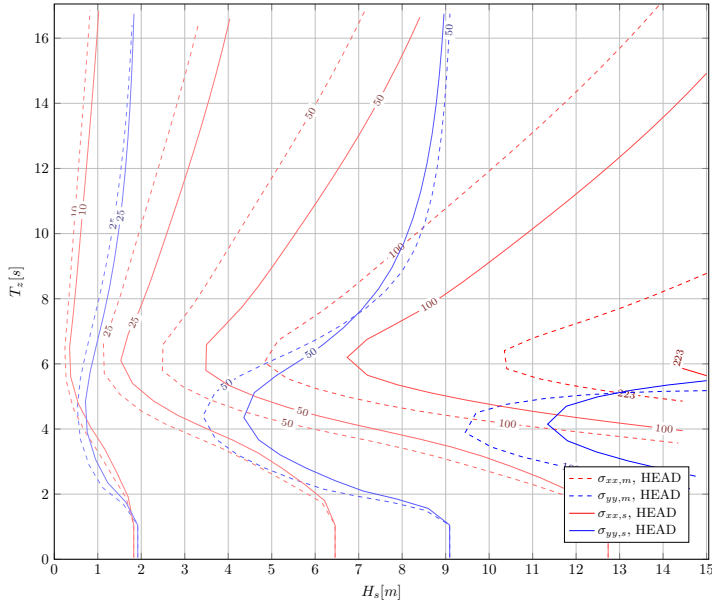
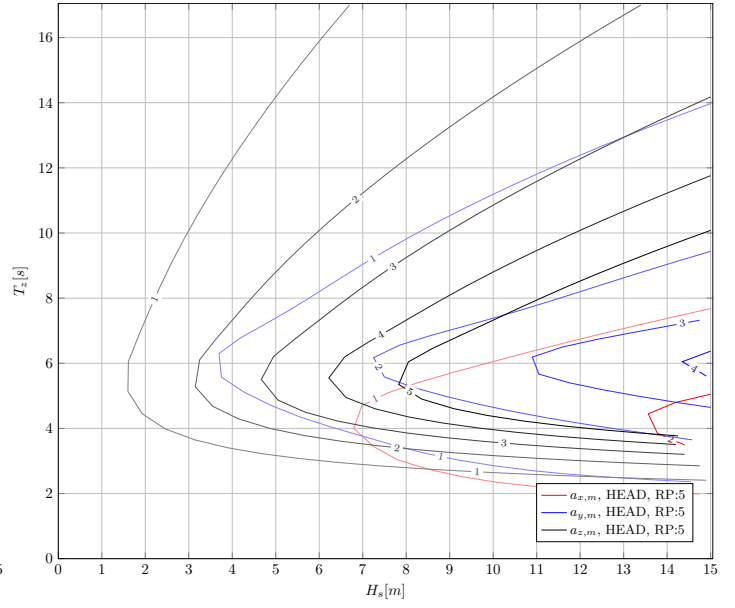
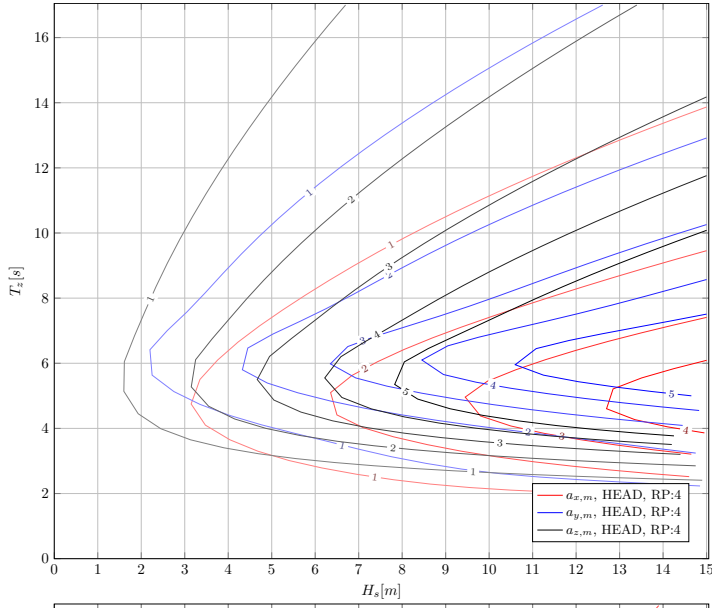


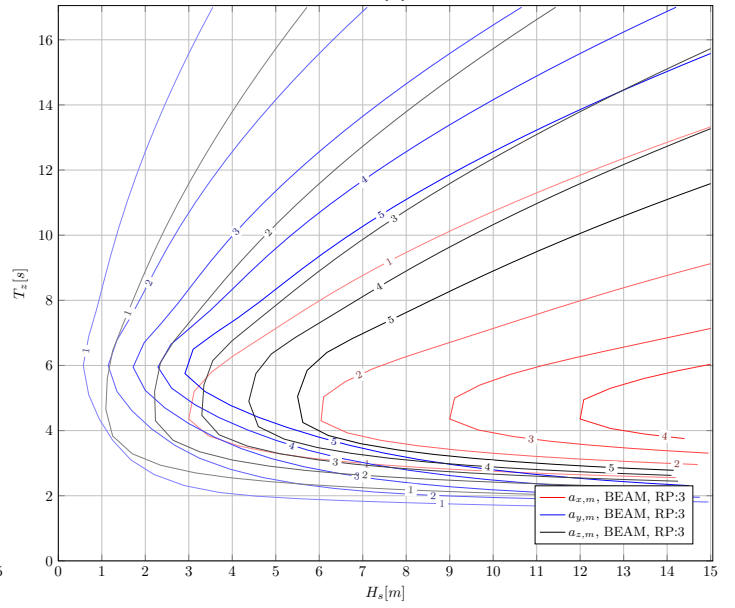
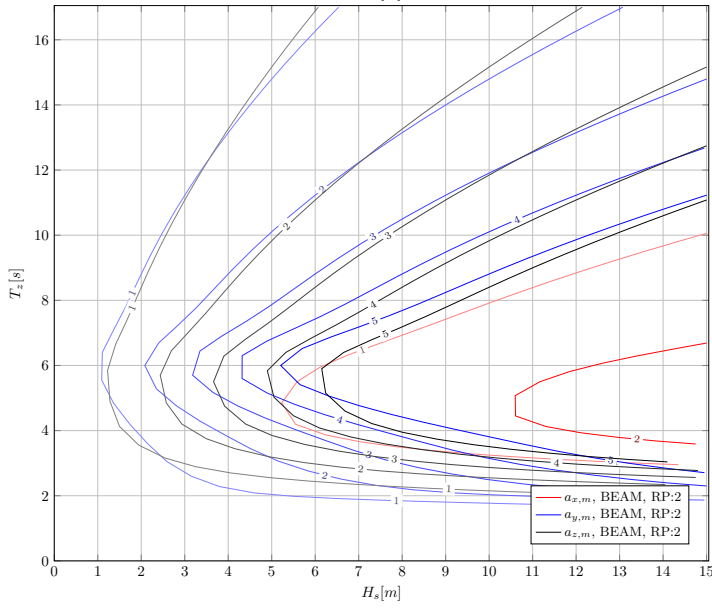
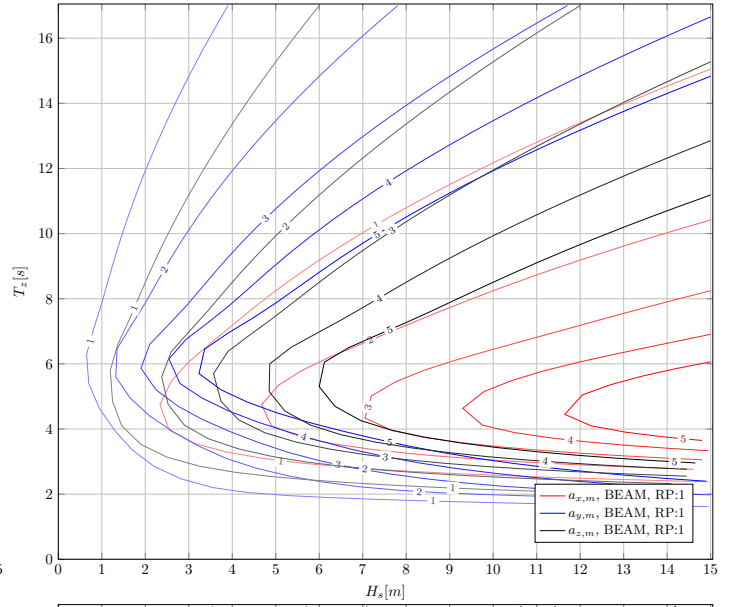
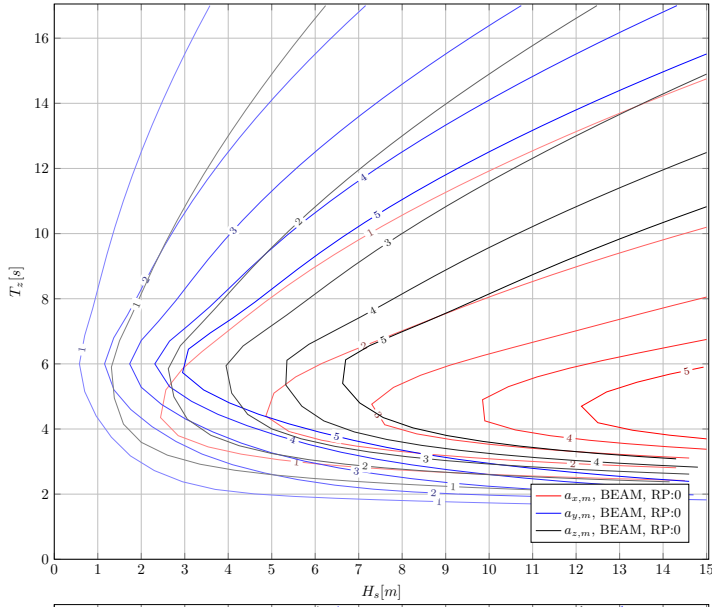
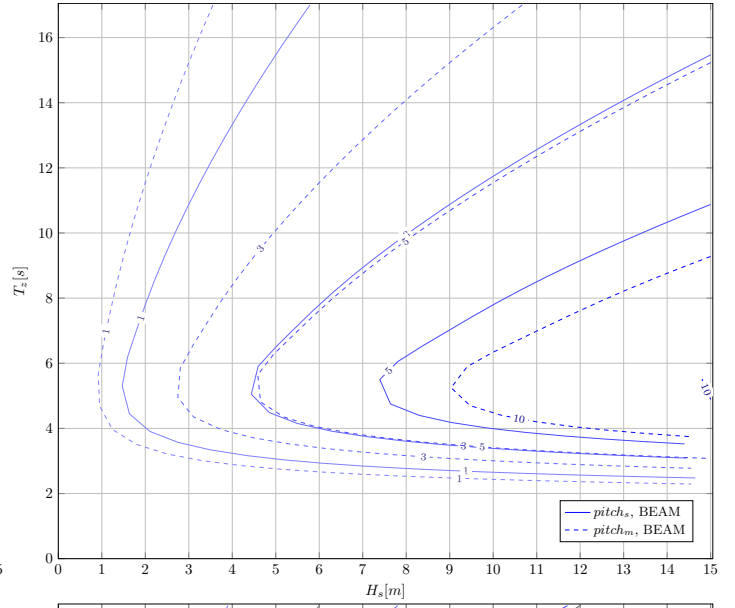
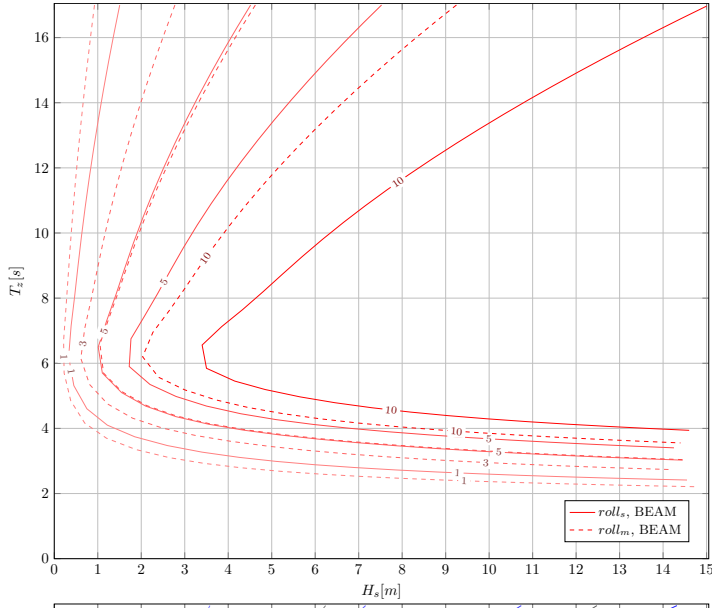


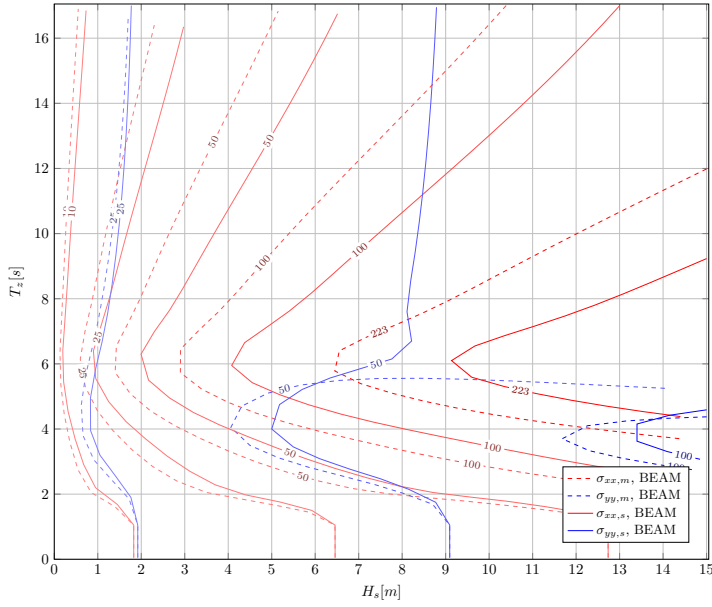
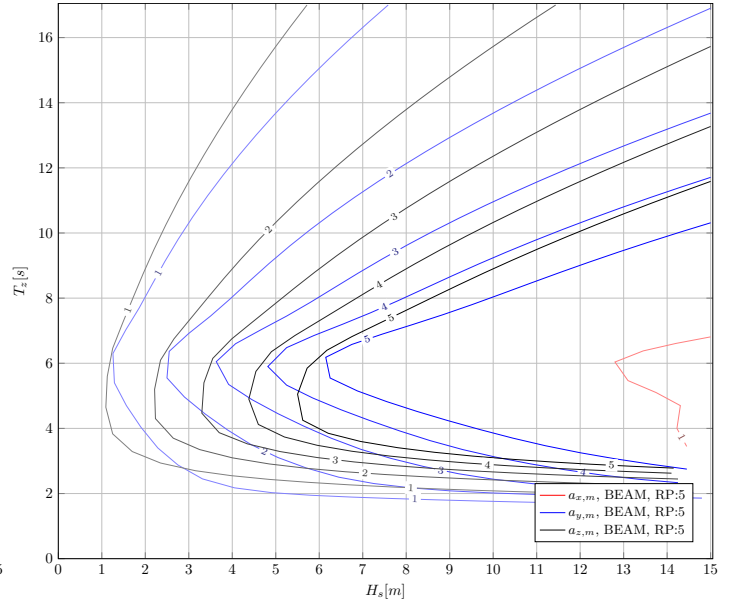
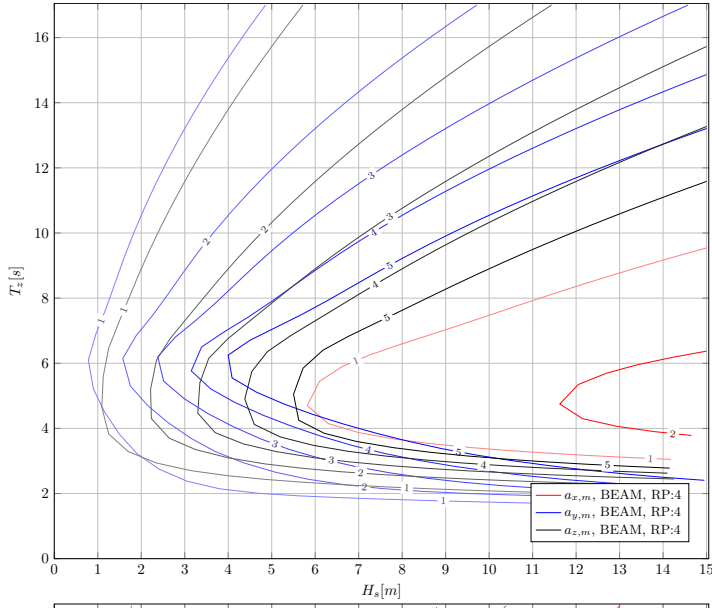












B Fatigue

In this section, more detailed information on the fatigue estimation is described. An analysis of the stress spectrum per member is done, using the final JR design, with the member numbering as shown in fig. B.1:

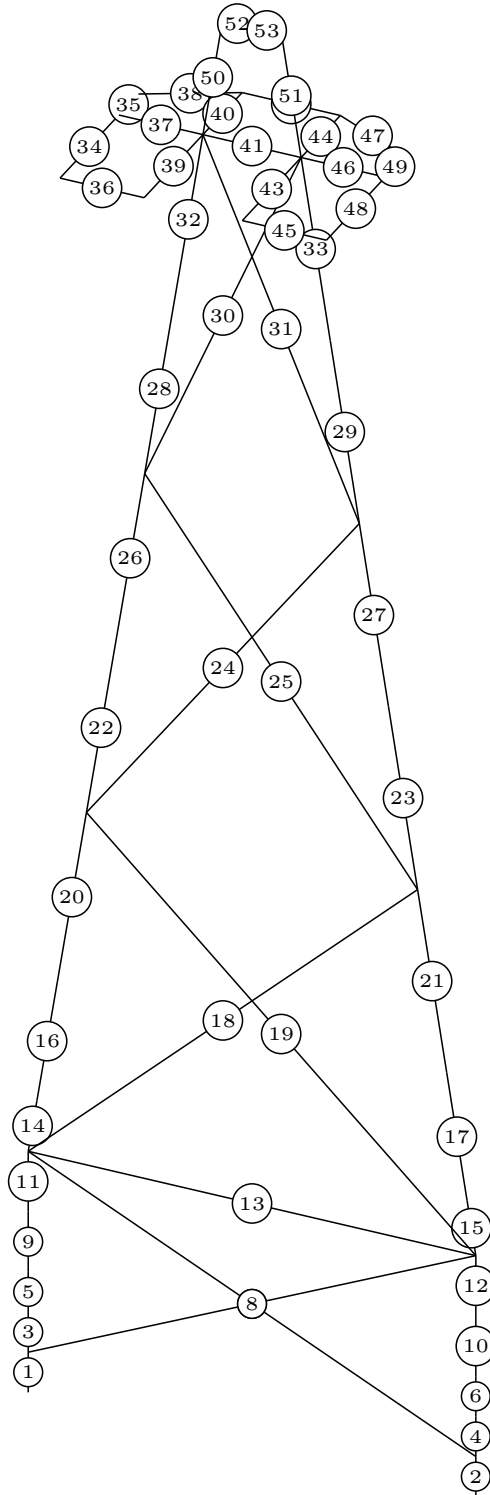


Figure B.1: Member numbering of the new design

Making a distinction between y and z direction, an acceleration of one is applied to simplified system as shown in section 9.1. Because the jib is governing in the loads, the mass of the JR itself is neglected and only the acceleration on the jib (RP:0) is considered relevant. In every member, the ratio between maximum stress and

the acceleration can be found and can be multiplied by the acceleration spectrum to get the stress spectrum. Finally Dirlik's method is used to get to a range of cycles.

At this stage, the detailed design provides information on member types and location of welds in the connections. With the weld connection types known, the FAT codes can be selected per member as seen in fig. B.2.

ID	loga	gammaB	gammaN	FAT	loga2	two sides! transverse attache	penetratio n weld, r<1/3	plate r< 1/6 or overlap	pinched pipe / hinge
1	11.855	3.028	0.116		71	15.091 E	F		
2	11.855	3.028	0.116		71	15.091 E	F		
3	11.699	3.229	0.478		63	14.832 E	F	F1	
4	11.699	3.229	0.478		63	14.832 E	F	F1	
5	12.01	4.468	0.93		80	15.35 E			
6	12.01	4.468	0.93		80	15.35 E			
7	11.398	3.481	0.449		50	14.33 E		F1	G
8	11.398	3.481	0.449		50	14.33 E		F1	G
9	12.01	3.37	1.015		80	15.35 E			
10	12.01	3.37	1.015		80	15.35 E			
11	11.699	2.498	1.584		63	14.832 E	F	F1	
12	11.699	2.498	1.584		63	14.832 E	F	F1	
13	12.01	0	0.2658		80	15.35 E			
14	11.855	2.997	0.755		71	15.091 E	F		
15	11.855	2.997	0.755		71	15.091 E	F		
16	12.01	2.345	0.754		80	15.35 E			
17	12.01	2.345	0.754		80	15.35 E			
18	11.398	2.643	0.43		50	14.33 E		F1	G
19	11.398	2.643	0.43		50	14.33 E		F1	G
20	11.699	2.711	0.85		63	14.832 E	F1		
21	11.699	2.711	0.85		63	14.832 E	F1		
22	11.699	2.875	0.525		63	14.832 E	F1		
23	11.699	2.875	0.525		63	14.832 E	F1		
24	11.398	3.093	0.1686		50	14.33 E	F1		G
25	11.398	3.093	0.1686		50	14.33 E	F1		G
26	11.699	2.644	0.831		63	14.832 E	F1		
27	11.699	2.644	0.831		63	14.832 E	F1		
28	11.699	2.402	0.52		63	14.832 E	F1		
29	11.699	2.402	0.52		63	14.832 E	F1		
30	11.398	2.581	0.567		50	14.33 E	F1		G
31	11.398	2.581	0.568		50	14.33 E	F1		G
32	11.699	2.659	0.767		63	14.832 E	F	F1	
33	11.699	2.659	0.767		63	14.832 E	F	F1	
34	12.01	0.25	0.458		80	15.35 E			
35	12.01	0.375	0.356		80	15.35 E			
36	12.01	0.042	0.081		80	15.35 E			
37	12.01	0.12	0.857		80	15.35 E			
38	12.01	0.066	0.256		80	15.35 E			
39	12.01	0.367	0.867		80	15.35 E			
40	12.01	0.068	0.233		80	15.35 E			
41	12.01	0	0.093		80	15.35 E			
42	12.01	0	0.119		80	15.35 E			
43	12.01	0.367	0.867		80	15.35 E			
44	12.01	0.068	0.233		80	15.35 E			
45	12.01	0.042	0.081		80	15.35 E			
46	12.01	0.12	0.857		80	15.35 E			
47	12.01	0.066	0.256		80	15.35 E			
48	12.01	0.25	0.458		80	15.35 E			
49	12.01	0.375	0.356		80	15.35 E			
50	11.855	1.771	1.183		71	15.091 E	F		
51	11.855	1.771	1.182		71	15.091 E	F		
52	11.855	3.53	1.195		71	15.091 E	F		
53	11.855	3.53	1.195		71	15.091 E	F		

Figure B.2: Member FAT code selection

The spectra and FAT codes provide the amount of cycles per time unit and the maximum number of cycles, respectively. By selecting a timerange - and acceleration spectrum - the fatigue lifetime can be estimated. For this, three different cases are examined; two cases with 20yr lifespan in different locations and one case with a two week transport in rough weather. Figure B.3 shows a sheet of the results of the analysis of the cases for the new design, where the first column represents the member number, the next three the percentage lifetime expected to be spent for the three different load cases, under head waves. The three following columns represent

the cases under beam waves and the final three show a cumulative damage expected under 70% head waves.

member ID	head			beam			70/30		
	case 1	case 2	rough trip	case 1	case 2	rough trip			
0	0.01	0.01	0	0.06	0.07	0.02	0.025	0.028	0.006
1	0.01	0.01	0	0.06	0.07	0.02	0.025	0.028	0.006
2	0.02	0.02	0	0.14	0.17	0.03	0.056	0.065	0.009
3	0.02	0.02	0	0.14	0.17	0.03	0.056	0.065	0.009
4	0.02	0.03	0.01	0.21	0.26	0.02	0.077	0.099	0.013
5	0.02	0.03	0.01	0.21	0.26	0.02	0.077	0.099	0.013
6	0.07	0.08	0.01	0.61	0.74	0.07	0.232	0.278	0.028
7	0.07	0.08	0.01	0.61	0.74	0.07	0.232	0.278	0.028
8	0.01	0.01	0	0.05	0.06	0.02	0.022	0.025	0.006
9	0.01	0.01	0	0.05	0.06	0.02	0.022	0.025	0.006
10	0	0	0	0.04	0.05	0.01	0.012	0.015	0.003
11	0	0	0	0.04	0.05	0.01	0.012	0.015	0.003
12	0	0	0	0	0	0	0	0	0
13	0	0	0	0	0	0	0	0	0

Figure B.3: Expected cumulative damage per member

C Pedestal Stress

Two outer stresses in the pedestal are analysed. One maximum over the local x-axis of the pedestal in the outer edge of the circular section, where bending and normal stresses are combined. And the same over the y-axis. The calculations are constructed into a function which returns both stresses. Wind forces are taken into account by using wind speeds found from the wind coupling and assumed to be working perpendicular on the selected axis. This is combined with the forces coming from the accelerations in all respective reference points and the crane or jib rest elements modeled as point masses. The function and its calculations can be found in the following code snippet:

```
1     if 3.6 > Tp / math.sqrt(Hs) or 5 < Tp / math.sqrt(Hs):
2         rep = 0
3
4     SPM = np.zeros(len(omegas))
5     SJ = SPM.copy()
6     for iW, w in enumerate(omegas):
7         if w == 0:w=0.05;
8         SPM[iW] = 5/16 * Hs ** 2 * math.pow(wp, 4) * math.pow(w, -5) * np.exp(-5 / 4 * math.pow(w / wp, -4))
9         if w <= wp:
10            s = sa
11        else:
12            s = sb
13        SJ[iW] = Ag * SPM[iW] * math.pow(gamma, np.exp(-0.5 * math.pow((w - wp) / (s * wp), 2)))
14
15    if representable:
16        return (SJ, rep)
17    else:
18        return SJ
19
20 def windSpeed(Hs:float):
21     return 7.3515 * math.pow(Hs, 0.6187) #made with a fit from excel
22
23 def windLoad(U, Cd, rho, A):
24     return 0.5 * rho * math.pow(U, 2) * Cd * A * 9.81
25
26 def pedestalStress(aRP0, aRP1, aRP2, roll, pitch, U, par:dict):
27
28     APed = math.pi * (pow(par['crRadiusPedistal']['value'], 2) -
29                     pow(par['crRadiusPedistalInner']['value'], 2))
30     forcesPed = np.zeros((3, 3)) # iRP, xyz, mu, Omega :: RAOa= # xyz, refpoint, U, mu, Omega, RAO
31
32     armPed = np.zeros((3, 3)) #iRP, xyz
33     armPed[0, 0] = 0 #par['pcalJibX']['value'] - par['pcalPedX']['value']
34     armPed[0, 1] = 0 #par['pcalJibY']['value'] - par['pcalPedY']['value']
35     armPed[0, 2] = abs(par['pcalJibZ']['value'] - par['baHeight']['value'])
36     armPed[1, 0] = abs(par['pcalCraneTopX']['value'] - par['pcalPedX']['value'])
37     armPed[1, 1] = 0 #par['pcalCraneTopY']['value'] - par['pcalPedY']['value'] # No moment due to swivel
38     armPed[1, 2] = abs(par['pcalCraneTopZ']['value'] - par['baHeight']['value'])
39     armPed[2, 0] = 0 #abs(par['pcalPedX']['value'] - par['pcalPedX']['value'])
40     armPed[2, 1] = 0 #abs(par['pcalPedY']['value'] - par['pcalPedY']['value'])
41     armPed[2, 2] = abs(par['pcalPedZ']['value'] - par['baHeight']['value'])
42     #iRP, xyz
43     # also presuming wind gusts from x and y direction!!!!
44     forcesPed[0, 0] = par['pcalMassJib']['value'] * aRP0[0] * 1000
45     forcesPed[0, 1] = par['pcalMassJib']['value'] * aRP0[1] * \
46         (1 - par['pllForceRatio']['value']) * 1000
47     forcesPed[0, 2] = - 1 * par['pcalMassJib']['value'] * (aRP0[2] + 9.81) * \
48         (1 - par['pllForceRatio']['value']) * 1000
49     forcesPed[1, 0] = par['pcalMassCraneTop']['value'] * aRP1[0] * 1000
50     forcesPed[1, 1] = par['pcalMassCraneTop']['value'] * aRP1[1] * 1000
51     forcesPed[1, 2] = - 1 * par['pcalMassCraneTop']['value'] * (aRP1[2] + 9.81) * 1000
52     forcesPed[2, 0] = par['pcalMassPed']['value'] * aRP2[0] * 1000
53     forcesPed[2, 1] = par['pcalMassPed']['value'] * aRP2[1] * 1000
54     forcesPed[2, 2] = - 1 * par['pcalMassPed']['value'] * (aRP2[2] + 9.81) * 1000
55
56     #adding wind
57     forcesPed[0, 1] += (1 - par['pllForceRatio']['value']) * windLoad(U,
58                                                                    par['pllDragCoeff']['value'],
59                                                                    par['pllRhoAir']['value'],
```

```

60                                                                 par['crAreaJibSide']['value'])
61 forcesPed[2, 0] += windLoad(U,
62     par['pllDragCoeffPed']['value'],
63     par['pllRhoAir']['value'],
64     par['crRadiusPedistal']['value'] * 2 * par['crHr']['value'] )
65
66 forcesPed[2, 1] += windLoad(U,
67     par['pllDragCoeffPed']['value'],
68     par['pllRhoAir']['value'],
69     par['crRadiusPedistal']['value'] * 2 * par['crHr']['value'] )
70
71 forcesPed[1, 0] += windLoad(U,
72     par['pllDragCoeffPed']['value'],
73     par['pllRhoAir']['value'],
74     par['crRadiusPedistal']['value'] * 2 * par['crHr']['value'] )
75
76 forcesPed[1, 0] += windLoad(U,
77     par['pllDragCoeffPed']['value'],
78     par['pllRhoAir']['value'],
79     par['crRadiusPedistal']['value'] * 2 * par['crHtot']['value'] )
80
81 forcesPed[1, 1] += windLoad(U,
82     par['pllDragCoeffPed']['value'],
83     par['pllRhoAir']['value'],
84     par['crRadiusPedistal']['value'] * 2 * par['crHtot']['value'] )
85
86 transPitch = np.zeros((3, 3))
87 transRoll = transPitch.copy()
88 relativeForces = forcesPed.copy()
89
90
91 for iRP in (0, 1, 2):
92     transPitch = rotation_matrix([0, 1, 0], math.radians(pitch))
93     transRoll = rotation_matrix([1, 0, 0], math.radians(roll))
94     relativeForces[iRP, :] = np.dot(transPitch, relativeForces[iRP, :])
95     relativeForces[iRP, :] = np.dot(transRoll, relativeForces[iRP, :])
96
97 forcesPedZ = np.sum(relativeForces[:, 2]) #iRP, xyz, mu, Omega
98 momentsPed = np.empty((3,)) # XYZ, Mu, Omegas
99 momentsPed[0] = relativeForces[0, 1] * armPed[0, 2] +\
100     relativeForces[1, 1] * armPed[1, 2] +\
101     relativeForces[2, 1] * armPed[2, 2] +\
102     relativeForces[0, 2] * armPed[0, 1] +\
103     relativeForces[1, 2] * armPed[1, 1] +\
104     relativeForces[2, 2] * armPed[2, 1] # OK! #iRP, XYZ
105
106 momentsPed[1] = relativeForces[0, 0] * armPed[0, 2] +\
107     relativeForces[1, 0] * armPed[1, 2] +\
108     relativeForces[2, 0] * armPed[2, 2] +\
109     relativeForces[0, 2] * armPed[0, 0] -\
110     relativeForces[1, 2] * armPed[1, 0] +\
111     relativeForces[2, 2] * armPed[2, 0]
112
113 momentsPed[2] = relativeForces[0, 0] * armPed[0, 1] +\
114     relativeForces[1, 0] * armPed[1, 1] +\
115     relativeForces[2, 0] * armPed[2, 1] +\
116     relativeForces[0, 1] * armPed[0, 0] +\
117     relativeForces[1, 1] * armPed[1, 0] +\
118     relativeForces[2, 1] * armPed[2, 0]
119
120
121 stressXX = abs(momentsPed[0] * par['crRadiusPedistal']['value'] /\
122     par['pllIrrPed']['value']) +\
123     abs(forcesPedZ / APed)
124
125 stressYY = abs(momentsPed[1] * par['crRadiusPedistal']['value'] /\
126     par['pllIrrPed']['value']) +\

```

D Engineering

This appendix holds detailed information on the structural details of the jib rest with its calculations. This section is omitted intentionally, as the information it contains is considered classified.

E Code

This section shows the most relevant code that is used to program the solver and any related sub functions. Its basic object structure is graphed in an UML-diagram in fig. E.1. Followed by the actual Python code of the solver.

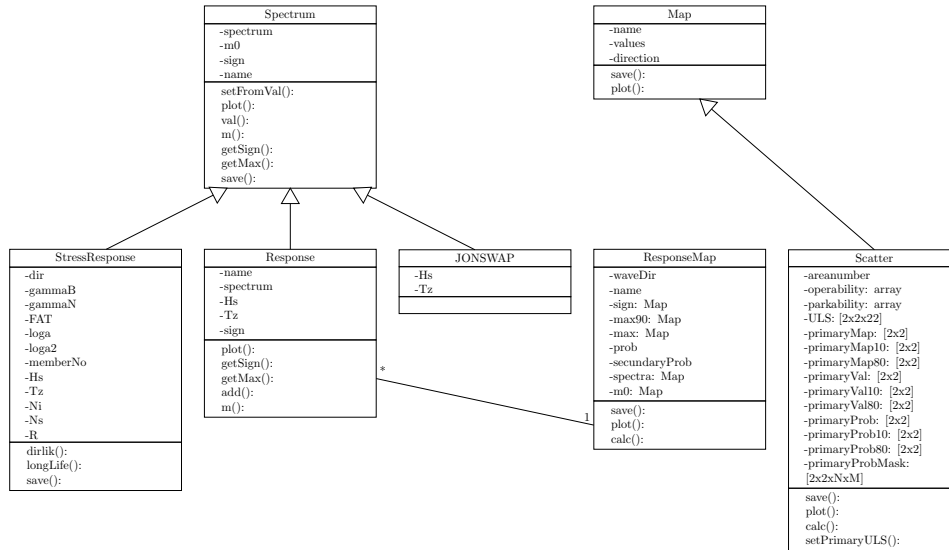


Figure E.1: UML diagram of the solver

```

1 from CalcFunctions import jonswap, pedestalStress, windSpeed, windLoad, \
2 buildScatter, combinedProb, savePickle, loadPickle, rotation_matrix, multiframe
3 import pickle as pk
4 import matplotlib.pyplot as plt
5 import numpy as np
6 import math as math
7 import TikzWriter as tk
8 import CalcFunctions as cf
9 import warnings
10
11
12
13 class RAO:
14
15     def __init__(self, location):
16         self.valueArray = np.loadtxt(location, delimiter = ",", skiprows=1)
17         self.valueArray[:, 7] = self.valueArray[:, 7] / 180 * math.pi
18
19     def val(self, mu):
20         #RAO, phase
21         return self.valueArray[self.valueArray[:, 3] == mu, 6], \
22             self.valueArray[self.valueArray[:, 3] == mu, 7]
23
24     def plot(self, mu):
25         global omegas
26         plt.plot(omegas, self.val(mu)[0])
27
28
29 class Spectrum:
30
31     def __init__(self, spectrum, name=""):
32         self.spectrum = spectrum
33         self.m0 = self.m(0)
34         self.sign = self.getSign()
35         self.name = name
36
37     def spectrum(self):
38         return self.spectrum
39
40     def set(self, spectrum):

```

```

41         self.spectrum = spectrum
42
43     def setFromVal(self, val):
44         self.spectrum = 0.5 * np.power(np.abs(val), 2) / deltaOmega
45
46     def plot(self):
47         global omegas
48         plt.plot(omegas, self.spectrum, label="S(" + self.name + ") " + \
49                 "m0:%.2f"%self.m0)
50
51     def val(self):
52         global deltaOmega
53         return np.sqrt(2 * self.spectrum * deltaOmega)
54
55     def m(self, n):
56         global omegas, deltaOmega
57         return np.sum(np.multiply(np.power(omegas, n), self.spectrum) * deltaOmega)
58
59     def getSign(self):
60         global deltaOmega
61         # AMPLITUDE!!!
62         #return 2 * math.sqrt(self.m(0))
63         return 2 * math.sqrt(np.sum(self.spectrum * deltaOmega))
64
65     def getMax(self, prob):
66         return math.sqrt(-math.log(1 - prob) * 2 * self.m0)
67
68     def save(self):
69         global savePath
70         np.savetxt(savePath + self.name + "Spectrum.csv", self.sign,
71                   delimiter=",")
72
73 class JONSWAP(Spectrum):
74
75     def __init__(self, Hs, Tz):
76         global omegas
77         global gamma
78         self.Hs = Hs
79         self.Tz = Tz
80         Spectrum.__init__(self, cf.jonswap(omegas, Hs, Tz, gamma),
81                           name="JONSWAP" + str(int(np.argmax(heights >= Hs))) + \
82                           "_" + str(int(np.argmax(periods >= Tz))))
83
84 class Response:
85
86     def __init__(self, response, Hs, Tz, dirW, name):
87
88         self.responseDir = response
89         if dirW == 0:
90             self.waveDir = "HEAD"
91         elif dirW == 90:
92             self.waveDir = "BEAM"
93         else:
94             self.waveDir = str(int(dirW))
95
96         if len(np.shape(response)) > 1:
97             self.spectrum = Spectrum(np.sum(response, 0), name + self.waveDir)
98         else:
99             self.spectrum = Spectrum(response, name + self.waveDir)
100         self.sign = self.spectrum.getSign()
101         self.Tz = Tz
102         self.Hs = Hs
103         self.name = name
104
105     def plot(self):
106         plt.title("Hs:{:.2f} Tz:{:.2f} dir:{}".format(self.Hs, self.Tz,
107                                                       self.waveDir))
108         self.spectrum.plot()
109
110     def getSign(self):
111         return self.spectrum.getSign()
112
113     def getMax(self, prob):

```

```

114         return self.spectrum.getMax(prob)
115
116     def add(self, value):
117         self.spectrum.spectrum += value
118
119     def m(self, n):
120         return self.spectrum.m(n)
121
122     class Map:
123
124         global heights
125         global periods
126
127         def __init__(self, values=None, name="Nameless", dirW=""):
128
129             if values == None:
130                 values = np.zeros((len(heights), len(periods)))
131
132             self.values = values
133             self.name = name
134             self.dirW = dirW
135
136         def save(self, subfolder="", header=""):
137             global savePath
138             np.savetxt(savePath + subfolder + "" + self.name + "Map" + \
139                 str(self.dirW) + ".csv",
140                 self.values, delimiter=",",
141                 header=header)
142
143         def plot(self, levels=[0.5]):
144             plt.contour(heights, periods, self.values.transpose(),
145                 label=self.name, levels=levels)
146
147         def __str__(self):
148             return str(self.values)
149
150     class ResponseMap:
151
152         def __init__(self, name="Response", dirW="", prob=0.95, secondaryProb=0.90):
153             self.waveDir = dirW
154             self.name = name
155             self.sign = Map(name=self.name + "Sign")
156             self.max90 = Map(name=self.name + "Max90")
157             self.max = Map(name=self.name + "Max")
158             self.prob = prob
159             self.secondaryProb=secondaryProb
160             self.spectra = Map(values=np.zeros((len(heights), len(periods)),
161                 dtype=object), name=self.name)
162             self.m0 = np.zeros((len(heights), len(periods)))
163
164         def save(self):
165             global savePath
166             np.savetxt(savePath + 'Responses/' + self.name + "SignMap" + \
167                 str(self.waveDir) + ".csv", self.sign.values, delimiter=",")
168             np.savetxt(savePath + 'Responses/' + self.name + "MaxMap" + \
169                 str(self.waveDir) + ".csv", self.max.values, delimiter=",")
170             # selfFile = bz2.BZ2File(savePath + 'Responses/' + self.name + str(self.waveDir) + ".json.bz2", 'wb')
171             # json.dump(self, selfFile)
172             # selfFile.close()
173             cf.savePickle(self, self.name + str(self.waveDir),
174                 savePath + 'Responses/')
175
176         def plot(self):
177             self.sign.plot()
178             self.max.plot()
179
180         def calc(self):
181             print("calculating response map for: " + self.name + " " + self.waveDir)
182             for iM, spec in np.ndenumerate(self.spectra.values):
183                 if not spec == 0:
184                     self.sign.values[iM] = spec.getSign()
185                     self.max.values[iM] = spec.getMax(self.prob)
186                     self.max90.values[iM] = spec.getMax(self.secondaryProb)

```

```

187         self.m0[iM] = spec.m(0)
188
189     def load(self):
190         # selfFile = bz2.BZ2File(savePath + 'Responses/' + self.name + str(self.waveDir) + ".json.bz2", 'rb')
191         # s = json.load(selfFile)
192         # selfFile.close()
193         # return cf.loadPickle(self.name + str(self.waveDir), savePath + 'Responses/')
194         s = cf.loadPickle(self.name + str(self.waveDir), savePath + 'Responses/')
195         self.waveDir = s.waveDir
196         self.name = s.name
197         self.sign = s.sign
198         self.max90 = s.max90
199         self.max = s.max
200         self.prob = s.prob
201         self.secondaryProb = s.secondaryProb
202         self.spectra = s.spectra
203         # print(self.spectra)
204         self.m0 = s.m0
205
206
207     class Scatter(Map):
208
209         global heights
210         global periods
211
212         def __init__(self, values, areaNumber):
213             Map.__init__(self, values, name="Scatter" + str(int(areaNumber)))
214             self.areaNumber = areaNumber
215             self.operability = [0, 0]
216             self.parkability = [0, 0]
217             self.ULS = np.zeros((2, 2, 22)) #[direction; Y/Z; xRP0, yRP0, ..., zRP5, roll, pitch, Hs, U]
218             self.primaryMap80 = np.empty((2, 2), dtype=object)
219             self.primaryMap10 = np.empty((2, 2), dtype=object)
220             self.primaryMap = np.empty((2, 2), dtype=object)
221             self.primaryVal80 = np.empty((2, 2), dtype=object)
222             self.primaryVal10 = np.empty((2, 2), dtype=object)
223             self.primaryVal = np.empty((2, 2), dtype=object)
224             self.primaryProb80 = np.empty((2, 2), dtype=object)
225             self.primaryProb10 = np.empty((2, 2), dtype=object)
226             self.primaryProb = np.empty((2, 2), dtype=object)
227             self.primaryProbMask = np.empty((2, 2, len(heights), len(periods)))
228
229         def plot(self, maskMap = None, levels=[1e-6, 1e-5, 1e-4]):
230             global heights
231             global periods
232
233             if maskMap == None:
234                 maskMap = np.ones((len(heights), len(periods)))
235             elif isinstance(maskMap, Map):
236                 maskMap = maskMap.values
237
238             output = np.multiply(self.values, maskMap)
239             plt.contour(heights, periods, output.transpose(), label="Area " + \
240                 str(self.areaNumber), levels=levels)
241         def setPrimaryULS(self):
242             self.ULS[0, 0, 1] = self.primaryVal[0, 0]
243             self.ULS[1, 0, 1] = self.primaryVal[1, 0]
244             self.ULS[0, 1, 2] = self.primaryVal[0, 1]
245             self.ULS[1, 1, 2] = self.primaryVal[1, 1]
246
247         def save(self):
248             global savePath
249             for iD, dirW in enumerate(("HEAD", "BEAM")):
250                 for iYZ, YZ in enumerate(("Y", "Z")):
251                     np.savetxt(savePath + 'Scatter/Area' +
252                         str(int(self.areaNumber)) + ".csv",
253                         self.values, delimiter=",")
254                     self.primaryMap[iD, iYZ].save(subfolder="Areas/" + dirW + "/" +
255                         YZ + "/" + self.name, header="v," +
256                         str(round(self.primaryVal[iD, iYZ], 2)) +
257                         ",p," + str(round(self.primaryProb[iD, iYZ], 3)))
258
259                     self.primaryMap10[iD, iYZ].save(subfolder="Areas/" + dirW + "/" +

```

```

260         YZ + "/" + self.name, header="v," +
261         str(round(self.primaryVal10[iD, iYZ], 2)) +
262         ",P," + str(round(self.primaryProb10[iD, iYZ], 3)))
263     self.primaryMap80[iD, iYZ].save(subfolder="Areas/" + dirW + "/" +
264     YZ + "/" + self.name, header="v," +
265     str(round(self.primaryVal80[iD, iYZ], 2)) +
266     ",P," + str(round(self.primaryProb80[iD, iYZ], 3)))
267
268     ULSfile = open(savePath + "ULS/ULS.csv", 'a')
269     valStr = ""
270     for val in self.ULS[iD, iYZ, :]: valStr += str(round(val, 2)) + ",";
271     ULSfile.write(dirW + "," + YZ + "," + str(int(self.areaNumber)) +
272     ", " + valStr + "\n")
273     ULSfile.close()
274
275     probVal = [[round(self.primaryVal[iD, iYZ], 2),
276     round(self.primaryVal10[iD, iYZ], 2),
277     round(self.primaryVal80[iD, iYZ], 2)],
278     [round(self.primaryProb[iD, iYZ], 3),
279     round(self.primaryProb10[iD, iYZ], 3),
280     round(self.primaryProb80[iD, iYZ], 3)]]
281     np.savetxt(savePath + "Areas/" + dirW + "/" + YZ + "/" +
282     self.name + "Values.csv", probVal, delimiter=',')
283     np.savetxt(savePath + "Areas/" + dirW + "/" + YZ + "/" +
284     self.name + "ProbMask.csv", self.primaryProbMask[iD, iYZ], delimiter=',')
285
286     def saveOperability(self):
287         global savePath
288         np.savetxt(savePath + "Operability/Area" + self.name + ".csv", [self.operability, self.parkability],
289         delimiter=",",
290         header="OPHead, OPBeam\n PAHead, PABeam")
291
292
293     class StressResponse(Spectrum):
294         global Srange
295         global deltaStress
296         global fatSlope
297
298         def __init__(self, spectrum=None, factor = 1, memberNo=0, Hs=10, Tz=10,
299         gammaB=0.1, gammaN=0.1, loga=10, loga2=14, FAT=32,
300         dirW="DIRECTION", comment="BN"):
301             self.accelResponse = spectrum
302             Spectrum.__init__(self, spectrum=spectrum * factor,
303             name="StressResponse" + comment + str(int(memberNo)) +
304             "-" + str("%.1f"%Hs) + "-" + str("%.1f"%Tz))
305             self.dirW = dirW
306             self.gammaB = gammaB
307             self.gammaN = gammaN
308             self.FAT = FAT
309             self.loga = loga
310             self.loga2 = loga2
311             self.memberNo = memberNo
312             self.Hs = Hs
313             self.Tz = Tz
314             if np.max(spectrum) == 0.0:
315                 self.Ni = np.zeros(np.shape(Srange))
316                 self.NS = np.zeros(np.shape(Srange))
317             else:
318                 self.Ni = self.longLife() # longlife cycles
319                 self.NS = self.dirlik(Srange) # dirlik cycles
320             self.R = np.sum(np.divide(self.NS, self.Ni)) * deltaStress #Damage per second per sea state
321             # print(self.Ni)
322             # print(self.NS)
323
324         def dirlik(self, Srange):
325
326             N = np.zeros(len(Srange))
327             pSLower = 0
328             if self.m0 > 0:
329                 for iS, S in enumerate(Srange):
330                     m0 = self.m0
331                     m1 = self.m(1)
332                     m2 = self.m(2)

```

```

333         m4 = self.m(4)
334         xm = m1 / m0 * math.sqrt(m2 / m4)
335         gam = m2 / math.sqrt(m0 * m4)
336         D1 = 2 * (xm - gam ** 2) / (1 + gam ** 2)
337         R.A = (gam - xm - D1 ** 2) / (1 - gam - D1 + D1 ** 2)
338         D2 = (1 - gam - D1 + D1 ** 2) / (1 - R.A)
339         D3 = 1 - D1 - D2
340         Z = S / (2 * math.sqrt(m0))
341         Q = 1.25 * (gam - D3 - D2 * R.A) / D1
342         try:
343             P1 = D1 / Q * math.exp(-Z / Q)
344         except OverflowError:
345             P1 = 0
346         P2 = D2 * Z / R.A ** 2 * math.exp(-Z ** 2 / (2 * R.A ** 2))
347         P3 = D3 * Z * math.exp(-Z ** 2 / 2)
348
349
350         pS = (P1 + P2 + P3) / (2 * math.sqrt(m0))
351
352         EP = math.sqrt(m4 / m2)
353
354         N[iS] = EP * pS * 3.15e7
355     return N
356
357     def longLife(self):
358
359         #         mMatrix = fatSlope * np.ones(np.shape(Srange))
360         #         mMatrix[Srange <= self.FAT] = 5
361         #         logaMatrix = self.loga * np.ones(np.shape(Srange))
362         #         logaMatrix[Srange <= self.FAT] = self.loga2
363         #
364         #         Ni = np.power(10, (logaMatrix - np.multiply(mMatrix, np.log10(Srange))))
365         #         Ni[Srange <= self.FAT] = 1e100
366         Ni = np.power(10, (self.loga - np.multiply(3, np.log10(Srange))))
367         Ni[Ni > 1e7] = np.power(10, (self.loga2 -
368                                     np.multiply(5, np.log10(Srange[Ni > 1e7]))))
369
370         return Ni
371
372
373     def save(self):
374         global savePath
375         global omegas
376
377         with open(savePath + "Members/Fatigue/" + self.name + self.dirW +
378                 ".csv", 'w') as fileStress:
379             stressString = "memberID,{0}".format(str(int(self.memberNo)))
380             stressString += ",Hs,{:.2f}".format(self.Hs)
381             stressString += ",Tz,{:.2f}".format(self.Tz)
382             stressString += "\nS [MPa],"
383             for val in Srange: stressString += str(val) + ","
384             stressString += "\nNS (dirlik) [1/s],"
385             for val in self.NS: stressString += str(val) + ","
386             stressString += "\nNi (long life),"
387             for val in self.Ni: stressString += str(val) + ","
388             stressString += "\nA Spectrum,"
389             for val in self.accelResponse: stressString += str(val) + ","
390             stressString += "\nS Spectrum:,"
391             for val in self.spectrum: stressString += str(val) + ","
392             fileStress.write(stressString)
393
394     class Member:
395
396         def __init__(self, number, FAT=32, gamma=[0, 0, 0, 0]):
397             self.number = number
398             self.FAT = FAT
399             self.gamma = gamma
400
401         # jons can either be a spectrum (JONSWAP object) or just an array
402
403     def directionalResponse(rao, jons, dirW, wavePhase=-1, directionality=True):
404         global mus
405         global deltaMu

```

```

406     global deltaOmega
407     global omegas
408     indexMu = np.argmax(mus == dirW)
409
410     #dirRad = dirW / 180 * math.pi
411     cosFactor = 0
412     cosCheck = 0
413
414     directions = (np.arange(-0.5 * math.pi, 0.5 * math.pi, deltaMu / 180 * math.pi))
415     respSpec = np.zeros((len(directions), len(omegas))) * 1j
416
417     for iMu, mu in enumerate(directions):
418         if directionality:
419             cosFactor = math.pow(math.cos(mu), 2) * 2 / math.pi * \
420                 (mus[1] - mus[0]) / 180 * math.pi
421             cosCheck += cosFactor
422             #print(cosCheck)
423         else:
424             if round(mu,1) == 0.0:
425                 cosFactor = 1
426             else:
427                 cosFactor = 0
428
429         if wavePhase == -1:
430             #plt.plot(rao.val(mus[abs(indexMu + iMu - 6)])[0], label=str(int(mus[abs(indexMu + iMu - 6)])) + " ")
431             respSpec[iMu] = np.multiply(jons.spectrum * cosFactor,
432                 np.power(rao.val(mus[abs(indexMu + iMu - 6)])[0], 2)) + \
433                 rao.val(mus[abs(indexMu + iMu - 6)])[1] * 1j
434         else:
435             respSpec[iMu] = np.multiply(np.multiply(jons.spectrum * cosFactor,
436                 np.power(rao.val(mus[abs(indexMu + iMu - 6)])[0], 2)),
437                 np.cos(rao.val(mus[abs(indexMu + iMu - 6)])[1] + wavePhase))
438
439     #exit(0)
440     #plt.plot(omegas, np.sum(respSpec, 0))
441     return np.abs(respSpec.real)
442
443 def solveProb(m0, pAccept, scatterValues, step=0.1):
444     pa = 1
445     val = 0.5
446     stepVal = step
447     p = np.zeros(scatterValues.shape)
448     pMap = np.zeros(scatterValues.shape)
449     pMap80 = pMap.copy()
450     pMap10 = pMap.copy()
451     sw80 = True
452     sw10 = True
453     # print("Solving")
454     m0[m0 == 0] = np.max(m0) / 10000000
455     while pa >= (1 - pAccept):
456         #t = tim.time()
457         p_r = np.exp(np.divide(-val ** 2, 2 * m0))
458
459         p_r[p_r > 1] = 0
460         p = np.multiply(scatterValues, p_r)
461         p [0,:] = 0
462         pa = np.sum(p)
463         val += stepVal
464
465
466         if pa < 0.80 and sw80:
467             sw80=False
468             pMap80 = p.copy()
469             val80 = val
470             p80 = pa
471
472         if pa < 0.10 and sw10:
473             sw10=False
474             pMap10 = p.copy()
475             val10 = val
476             p10 = pa
477
478     if pa <= 0.08:

```

```

479         stepVal = 0.005
480
481     #         print(pa)
482     pMap = p
483     mask = pMap > 1e-4
484     mask = mask.reshape(np.shape(scatterValues))
485     return ([val, val10, val80], [pa, p10, p80], [Map(values=pMap, name="Prob"),
486                                                  Map(values=pMap10, name="Prob10"),
487                                                  Map(values=pMap80, name="Prob80")], mask)
488
489
490 def main(rootPath, code, par, scatterPar, scatterCoastPar):
491
492     global omegas
493     global gamma; gamma = 3.3
494     global g; g = 9.81
495     global deltaOmega, deltaMu
496     global mus, heights, periods
497     global HEAD; HEAD = 0
498     global BEAM; BEAM = 90
499     designVersion = "6324Jrv8"
500     checkMultiframeVersion = "6324Jrv9" # designVersion
501     global savePath; savePath = rootPath + "/Data/Final/" + designVersion + "/"
502     global Srange
503     global fatSlope
504     global deltaStress
505
506     exceedProb = par['pllRespMaxPOper']['value']
507     secondaryExceedProb = par['pllProbAccelSecondary']['value']
508     compFatRatio = par['pllCompressionFatigueRatio']['value']
509     fatSlope = par['pllFatSlope']['value']
510
511
512     omegas = loadPickle("omegas2121ton", rootPath + code['PATH_PRECAL'] + \
513                       code['PRECAL_NAME'])
514     deltaHeights = 0.5
515     heights = np.arange(0, code['WEIBULL_H_END'], deltaHeights, dtype=np.ndarray)
516     heights[0] = 0.5
517     deltaPeriods = 0.5
518     periods = np.arange(0, code['WEIBULL_T_END'], deltaPeriods, dtype=np.ndarray)
519     periods[0] = 0.5
520     heightsMesh = heights.reshape((len(heights), 1)) * np.ones((1, len(periods)))
521     mus = loadPickle("mus2121ton", rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'])
522     wavePhases = np.arange(0, 2*math.pi, 0.25 * math.pi)
523     Srange = np.arange(2.5, 100, 5)
524     deltaStress = Srange[1] - Srange[0]
525
526     deltaMu = mus[2] - mus[1]
527     deltaOmega = omegas[1] - omegas[0]
528
529     #     cf.buildScatter2(["prob", "combProb"], fileName="CombinedProbabilityExampleNew", direction="BEAM", savePath=
530     #     cf.buildScatter2(["accel"], RP = [0], fileName="AccelResponseRP0New", direction="BEAM", savePath=savePath)
531     #     exit(0)
532
533     commentString = "2121ton"
534     roll = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
535              'Roll' + commentString + '.csv')
536     pitch = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
537              'Pitch' + commentString + '.csv')
538     surge = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
539              'Surge' + commentString + '.csv')
540     sway = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
541              'Sway' + commentString + '.csv')
542     heave = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
543              'Heave' + commentString + '.csv')
544     yaw = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] + 'Yaw'
545              + commentString + '.csv')
546
547     ay = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
548              'aYRP0' + commentString + '.csv')
549     az = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
550              'aZRP0' + commentString + '.csv')
551

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552 axRP0 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
553             'aXRP0' + commentString + '.csv')
554
555 axRP1 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
556             'aXRP1' + commentString + '.csv')
557 ayRP1 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
558             'aYRP1' + commentString + '.csv')
559 azRP1 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
560             'aZRP1' + commentString + '.csv')
561 axRP2 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
562             'aXRP2' + commentString + '.csv')
563 ayRP2 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
564             'aYRP2' + commentString + '.csv')
565 azRP2 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
566             'aZRP2' + commentString + '.csv')
567 axRP3 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
568             'aXRP3' + commentString + '.csv')
569 ayRP3 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
570             'aYRP3' + commentString + '.csv')
571 azRP3 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
572             'aZRP3' + commentString + '.csv')
573 axRP4 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
574             'aXRP4' + commentString + '.csv')
575 ayRP4 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
576             'aYRP4' + commentString + '.csv')
577 azRP4 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
578             'aZRP4' + commentString + '.csv')
579 axRP5 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
580             'aXRP5' + commentString + '.csv')
581 ayRP5 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
582             'aYRP5' + commentString + '.csv')
583 azRP5 = RAO(rootPath + code['PATH_PRECAL'] + code['PRECAL_NAME'] +
584             'aZRP5' + commentString + '.csv')
585
586 scatterItems = []
587 scatterSet = []
588
589 for scat in scatterPar.values():
590     scatterItems.append(int(float(scatterPar[scat['value']])))
591     scatterSet.append(scat)
592 for scat in scatterCoastPar.values():
593     scatterItems.append(int(float(scatterCoastPar[scat['value']])))
594     scatterSet.append(scat)
595 scatterItems = set(scatterItems)
596 # scatterItems = [10, 47]
597
598 #SWITCHES
599 loopULS = False
600 loopFatigue = False
601 calculateResponses = False
602 calculateULS = False
603 calculateOperability = False
604 readMultiframe = False
605 postProc = True
606
607 emptyArray = np.empty((len(heights), len(periods)), dtype=object)
608
609 waveSpectra = np.empty((len(heights), len(periods)), dtype=object)
610
611 rollResponseMap = [ResponseMap(name="Roll", dirW="HEAD", prob=exceedProb,
612                               secondaryProb=secondaryExceedProb),
613                  ResponseMap(name="Roll", dirW="BEAM", prob=exceedProb,
614                               secondaryProb=secondaryExceedProb)]
615 pitchResponseMap = [ResponseMap(name="Pitch", dirW="HEAD", prob=exceedProb,
616                               secondaryProb=secondaryExceedProb),
617                   ResponseMap(name="Pitch", dirW="BEAM", prob=exceedProb,
618                               secondaryProb=secondaryExceedProb)]
619 surgeResponseMap = [ResponseMap(name="Surge", dirW="HEAD", prob=exceedProb,
620                               secondaryProb=secondaryExceedProb),
621                   ResponseMap(name="Surge", dirW="BEAM", prob=exceedProb,
622                               secondaryProb=secondaryExceedProb)]
623 swayResponseMap = [ResponseMap(name="Sway", dirW="HEAD", prob=exceedProb,
624                               secondaryProb=secondaryExceedProb),

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625         ResponseMap(name="Sway", dirW="BEAM", prob=exceedProb,
626             secondaryProb=secondaryExceedProb)]
627 heaveResponseMap = [ResponseMap(name="Heave", dirW="HEAD", prob=exceedProb,
628             secondaryProb=secondaryExceedProb),
629         ResponseMap(name="Heave", dirW="BEAM", prob=exceedProb,
630             secondaryProb=secondaryExceedProb)]
631 yawResponseMap = [ResponseMap(name="Yaw", dirW="HEAD", prob=exceedProb,
632             secondaryProb=secondaryExceedProb),
633         ResponseMap(name="Yaw", dirW="BEAM", prob=exceedProb, secondaryProb=secondaryExceedProb)]
634
635 axRP0ResponseMap = [ResponseMap(name="AXRP0", dirW="HEAD", prob=exceedProb,
636             secondaryProb=secondaryExceedProb),
637         ResponseMap(name="AXRP0", dirW="BEAM", prob=exceedProb,
638             secondaryProb=secondaryExceedProb)]
639 ayRP0ResponseMap = [ResponseMap(name="AYRP0", dirW="HEAD", prob=exceedProb,
640             secondaryProb=secondaryExceedProb),
641         ResponseMap(name="AYRP0", dirW="BEAM", prob=exceedProb,
642             secondaryProb=secondaryExceedProb)]
643 azRP0ResponseMap = [ResponseMap(name="AZRP0", dirW="HEAD", prob=exceedProb,
644             secondaryProb=secondaryExceedProb),
645         ResponseMap(name="AZRP0", dirW="BEAM", prob=exceedProb,
646             secondaryProb=secondaryExceedProb)]
647
648 axRP1ResponseMap = [ResponseMap(name="AXRP1", dirW="HEAD", prob=exceedProb,
649             secondaryProb=secondaryExceedProb),
650         ResponseMap(name="AXRP1", dirW="BEAM", prob=exceedProb,
651             secondaryProb=secondaryExceedProb)]
652 ayRP1ResponseMap = [ResponseMap(name="AYRP1", dirW="HEAD", prob=exceedProb,
653             secondaryProb=secondaryExceedProb),
654         ResponseMap(name="AYRP1", dirW="BEAM", prob=exceedProb,
655             secondaryProb=secondaryExceedProb)]
656 azRP1ResponseMap = [ResponseMap(name="AZRP1", dirW="HEAD", prob=exceedProb,
657             secondaryProb=secondaryExceedProb),
658         ResponseMap(name="AZRP1", dirW="BEAM", prob=exceedProb,
659             secondaryProb=secondaryExceedProb)]
660
661 axRP2ResponseMap = [ResponseMap(name="AXRP2", dirW="HEAD", prob=exceedProb,
662             secondaryProb=secondaryExceedProb),
663         ResponseMap(name="AXRP2", dirW="BEAM", prob=exceedProb,
664             secondaryProb=secondaryExceedProb)]
665 ayRP2ResponseMap = [ResponseMap(name="AYRP2", dirW="HEAD", prob=exceedProb,
666             secondaryProb=secondaryExceedProb),
667         ResponseMap(name="AYRP2", dirW="BEAM", prob=exceedProb,
668             secondaryProb=secondaryExceedProb)]
669 azRP2ResponseMap = [ResponseMap(name="AZRP2", dirW="HEAD", prob=exceedProb,
670             secondaryProb=secondaryExceedProb),
671         ResponseMap(name="AZRP2", dirW="BEAM", prob=exceedProb,
672             secondaryProb=secondaryExceedProb)]
673
674 axRP3ResponseMap = [ResponseMap(name="AXRP3", dirW="HEAD", prob=exceedProb,
675             secondaryProb=secondaryExceedProb),
676         ResponseMap(name="AXRP3", dirW="BEAM", prob=exceedProb,
677             secondaryProb=secondaryExceedProb)]
678 ayRP3ResponseMap = [ResponseMap(name="AYRP3", dirW="HEAD", prob=exceedProb,
679             secondaryProb=secondaryExceedProb),
680         ResponseMap(name="AYRP3", dirW="BEAM", prob=exceedProb,
681             secondaryProb=secondaryExceedProb)]
682 azRP3ResponseMap = [ResponseMap(name="AZRP3", dirW="HEAD", prob=exceedProb,
683             secondaryProb=secondaryExceedProb),
684         ResponseMap(name="AZRP3", dirW="BEAM", prob=exceedProb,
685             secondaryProb=secondaryExceedProb)]
686
687 axRP4ResponseMap = [ResponseMap(name="AXRP4", dirW="HEAD", prob=exceedProb,
688             secondaryProb=secondaryExceedProb),
689         ResponseMap(name="AXRP4", dirW="BEAM", prob=exceedProb,
690             secondaryProb=secondaryExceedProb)]
691 ayRP4ResponseMap = [ResponseMap(name="AYRP4", dirW="HEAD", prob=exceedProb,
692             secondaryProb=secondaryExceedProb),
693         ResponseMap(name="AYRP4", dirW="BEAM", prob=exceedProb,
694             secondaryProb=secondaryExceedProb)]
695 azRP4ResponseMap = [ResponseMap(name="AZRP4", dirW="HEAD", prob=exceedProb,
696             secondaryProb=secondaryExceedProb),
697         ResponseMap(name="AZRP4", dirW="BEAM", prob=exceedProb,

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698         secondaryProb=secondaryExceedProb) ]
699
700 axRP5ResponseMap = [ResponseMap(name="AXRP5", dirW="HEAD", prob=exceedProb,
701     secondaryProb=secondaryExceedProb),
702     ResponseMap(name="AXRP5", dirW="BEAM", prob=exceedProb,
703     secondaryProb=secondaryExceedProb) ]
704 ayRP5ResponseMap = [ResponseMap(name="AYRP5", dirW="HEAD", prob=exceedProb,
705     secondaryProb=secondaryExceedProb),
706     ResponseMap(name="AYRP5", dirW="BEAM", prob=exceedProb,
707     secondaryProb=secondaryExceedProb) ]
708 azRP5ResponseMap = [ResponseMap(name="AZRP5", dirW="HEAD", prob=exceedProb,
709     secondaryProb=secondaryExceedProb),
710     ResponseMap(name="AZRP5", dirW="BEAM", prob=exceedProb,
711     secondaryProb=secondaryExceedProb) ]
712
713 ayRelResponseMap = [ResponseMap(name="AYRel", dirW="HEAD", prob=exceedProb,
714     secondaryProb=secondaryExceedProb),
715     ResponseMap(name="AYRel", dirW="BEAM", prob=exceedProb,
716     secondaryProb=secondaryExceedProb) ]
717 #azRelResponseMap = [ResponseMap(name="AZRel", dirW="HEAD", prob=exceedProb), ResponseMap(name="AZRel", dirW=
718
719 ULSfile = open(savePath + "ULS/ULS.csv", 'w')
720 ULSfile.write("dir,y/z,area,RP0aX,RP0aY,RP0aZ,RP1aX,RP1aY,RP1aZ,RP2aX," +
721     "RP2aY,RP2aZ,RP3aX,RP3aY,RP3aZ,RP4aX,RP4aY,RP4aZ,RP5aX," +
722     "RP5aY,RP5aZ,roll,pitch,Hs,U\n")
723 ULSfile.close()
724
725 #####
726 # SECOND ITERATION LOOP
727 #####
728 if loopULS:
729     if calculateResponses:
730         for iH, Hs in enumerate(heights):
731             for iT, Tz in enumerate(periods):
732                 waveSpectra[iH, iT] = JONSWAP(Hs, Tz)
733                 for iD, dirW in enumerate((HEAD, BEAM)):
734                     print("\rULS:Responses for Hs:{0} Tz:{1}".format(str(Hs),
735                         str(Tz)))
736
737 #####
738 # MOTIONS AND ACCELERATIONS
739 #####
740
741 #####
742 # calc resp
743 rollResponseMap[iD].spectra.values[iH, iT] =\
744     Response(directionalResponse(roll,
745         waveSpectra[iH, iT],
746         dirW,
747         directionality=True),
748         name="Roll", Hs=Hs, Tz=Tz, dirW=dirW)
749 pitchResponseMap[iD].spectra.values[iH, iT] =\
750     Response(directionalResponse(pitch,
751         waveSpectra[iH, iT],
752         dirW,
753         directionality=True),
754         name="Pitch", Hs=Hs, Tz=Tz, dirW=dirW)
755 surgeResponseMap[iD].spectra.values[iH, iT] =\
756     Response(directionalResponse(surge,
757         waveSpectra[iH, iT],
758         dirW,
759         directionality=True),
760         name="Surge", Hs=Hs, Tz=Tz, dirW=dirW)
761 swayResponseMap[iD].spectra.values[iH, iT] =\
762     Response(directionalResponse(sway,
763         waveSpectra[iH, iT],
764         dirW,
765         directionality=True),
766         name="Sway", Hs=Hs, Tz=Tz, dirW=dirW)
767 heaveResponseMap[iD].spectra.values[iH, iT] =\
768     Response(directionalResponse(heave,
769         waveSpectra[iH, iT],
770         dirW,

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771                                     directionality=True),
772         name="Heave", Hs=Hs, Tz=Tz, dirW=dirW)
773 yawResponseMap[iD].spectra.values[iH, iT] =\
774     Response(directionalResponse(yaw,
775                                     waveSpectra[iH, iT],
776                                     dirW,
777                                     directionality=True),
778         name="Yaw", Hs=Hs, Tz=Tz, dirW=dirW)
779
780 axRP0ResponseMap[iD].spectra.values[iH, iT] =\
781     Response(directionalResponse(axRP0,
782                                     waveSpectra[iH, iT],
783                                     dirW,
784                                     directionality=True),
785         name="AXRP0", Hs=Hs, Tz=Tz, dirW=dirW)
786 ayRP0ResponseMap[iD].spectra.values[iH, iT] =\
787     Response(directionalResponse(ay,
788                                     waveSpectra[iH, iT],
789                                     dirW,
790                                     directionality=True),
791         name="AYRP0", Hs=Hs, Tz=Tz, dirW=dirW)
792 azRP0ResponseMap[iD].spectra.values[iH, iT] =\
793     Response(directionalResponse(az,
794                                     waveSpectra[iH, iT],
795                                     dirW,
796                                     directionality=True),
797         name="AZRP0", Hs=Hs, Tz=Tz, dirW=dirW)
798
799 ayRelResponseMap[iD].spectra.values[iH, iT] =\
800     Response(directionalResponse(ay,
801                                     waveSpectra[iH, iT],
802                                     dirW,
803                                     directionality=True),
804         name="AYRP0", Hs=Hs, Tz=Tz, dirW=dirW)
805
806 axRP1ResponseMap[iD].spectra.values[iH, iT] =\
807     Response(directionalResponse(axRP1,
808                                     waveSpectra[iH, iT],
809                                     dirW,
810                                     directionality=True),
811         name="AXRP1", Hs=Hs, Tz=Tz, dirW=dirW)
812 ayRP1ResponseMap[iD].spectra.values[iH, iT] =\
813     Response(directionalResponse(ayRP1,
814                                     waveSpectra[iH, iT],
815                                     dirW,
816                                     directionality=True),
817         name="AYRP1", Hs=Hs, Tz=Tz, dirW=dirW)
818 azRP1ResponseMap[iD].spectra.values[iH, iT] =\
819     Response(directionalResponse(azRP1,
820                                     waveSpectra[iH, iT],
821                                     dirW,
822                                     directionality=True),
823         name="AZRP1", Hs=Hs, Tz=Tz, dirW=dirW)
824
825 axRP2ResponseMap[iD].spectra.values[iH, iT] =\
826     Response(directionalResponse(axRP2,
827                                     waveSpectra[iH, iT],
828                                     dirW,
829                                     directionality=True),
830         name="AXRP2", Hs=Hs, Tz=Tz, dirW=dirW)
831 ayRP2ResponseMap[iD].spectra.values[iH, iT] =\
832     Response(directionalResponse(ayRP2,
833                                     waveSpectra[iH, iT],
834                                     dirW,
835                                     directionality=True),
836         name="AYRP2", Hs=Hs, Tz=Tz, dirW=dirW)
837 azRP2ResponseMap[iD].spectra.values[iH, iT] =\
838     Response(directionalResponse(azRP2,
839                                     waveSpectra[iH, iT],
840                                     dirW,
841                                     directionality=True),
842         name="AZRP2", Hs=Hs, Tz=Tz, dirW=dirW)
843

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844     axRP3ResponseMap[iD].spectra.values[iH, iT] =\
845         Response(directionalResponse(axRP3,
846             waveSpectra[iH, iT],
847             dirW,
848             directionality=True),
849             name="AXRP3", Hs=Hs, Tz=Tz, dirW=dirW)
850     ayRP3ResponseMap[iD].spectra.values[iH, iT] =\
851         Response(directionalResponse(ayRP3,
852             waveSpectra[iH, iT],
853             dirW,
854             directionality=True),
855             name="AYRP3", Hs=Hs, Tz=Tz, dirW=dirW)
856     azRP3ResponseMap[iD].spectra.values[iH, iT] =\
857         Response(directionalResponse(azRP3,
858             waveSpectra[iH, iT],
859             dirW,
860             directionality=True),
861             name="AZRP3", Hs=Hs, Tz=Tz, dirW=dirW)
862
863     axRP4ResponseMap[iD].spectra.values[iH, iT] =\
864         Response(directionalResponse(axRP4,
865             waveSpectra[iH, iT],
866             dirW,
867             directionality=True),
868             name="AXRP4", Hs=Hs, Tz=Tz, dirW=dirW)
869     ayRP4ResponseMap[iD].spectra.values[iH, iT] =\
870         Response(directionalResponse(ayRP4,
871             waveSpectra[iH, iT],
872             dirW,
873             directionality=True),
874             name="AYRP4", Hs=Hs, Tz=Tz, dirW=dirW)
875     azRP4ResponseMap[iD].spectra.values[iH, iT] =\
876         Response(directionalResponse(azRP4,
877             waveSpectra[iH, iT],
878             dirW,
879             directionality=True),
880             name="AZRP4", Hs=Hs, Tz=Tz, dirW=dirW)
881
882     axRP5ResponseMap[iD].spectra.values[iH, iT] =\
883         Response(directionalResponse(axRP5,
884             waveSpectra[iH, iT],
885             dirW,
886             directionality=True),
887             name="AXRP5", Hs=Hs, Tz=Tz, dirW=dirW)
888     ayRP5ResponseMap[iD].spectra.values[iH, iT] =\
889         Response(directionalResponse(ayRP5,
890             waveSpectra[iH, iT],
891             dirW,
892             directionality=True),
893             name="AYRP5", Hs=Hs, Tz=Tz, dirW=dirW)
894     azRP5ResponseMap[iD].spectra.values[iH, iT] =\
895         Response(directionalResponse(azRP5,
896             waveSpectra[iH, iT],
897             dirW,
898             directionality=True),
899             name="AZRP5", Hs=Hs, Tz=Tz, dirW=dirW)
900
901     #####
902     # CALC AND SAVE MAPS
903     #####
904     for iD in (0, 1):
905         rollResponseMap[iD].calc()
906         pitchResponseMap[iD].calc()
907         surgeResponseMap[iD].calc()
908         swayResponseMap[iD].calc()
909         heaveResponseMap[iD].calc()
910         axRP0ResponseMap[iD].calc()
911         ayRP0ResponseMap[iD].calc()
912         azRP0ResponseMap[iD].calc()
913         axRP1ResponseMap[iD].calc()
914         ayRP1ResponseMap[iD].calc()
915         azRP1ResponseMap[iD].calc()
916         axRP2ResponseMap[iD].calc()

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917     ayRP2ResponseMap[iD].calc()
918     azRP2ResponseMap[iD].calc()
919     axRP3ResponseMap[iD].calc()
920     ayRP3ResponseMap[iD].calc()
921     azRP3ResponseMap[iD].calc()
922     axRP4ResponseMap[iD].calc()
923     ayRP4ResponseMap[iD].calc()
924     azRP4ResponseMap[iD].calc()
925     axRP5ResponseMap[iD].calc()
926     ayRP5ResponseMap[iD].calc()
927     azRP5ResponseMap[iD].calc()
928
929     ayRelResponseMap[iD].calc()
930
931     print("ULS:saving")
932     # save all maps
933     rollResponseMap[iD].save()
934     pitchResponseMap[iD].save()
935     surgeResponseMap[iD].save()
936     swayResponseMap[iD].save()
937     heaveResponseMap[iD].save()
938     axRP0ResponseMap[iD].save()
939     ayRP0ResponseMap[iD].save()
940     azRP0ResponseMap[iD].save()
941     axRP1ResponseMap[iD].save()
942     ayRP1ResponseMap[iD].save()
943     azRP1ResponseMap[iD].save()
944     axRP2ResponseMap[iD].save()
945     ayRP2ResponseMap[iD].save()
946     azRP2ResponseMap[iD].save()
947     axRP3ResponseMap[iD].save()
948     ayRP3ResponseMap[iD].save()
949     azRP3ResponseMap[iD].save()
950     axRP4ResponseMap[iD].save()
951     ayRP4ResponseMap[iD].save()
952     azRP4ResponseMap[iD].save()
953     axRP5ResponseMap[iD].save()
954     ayRP5ResponseMap[iD].save()
955     azRP5ResponseMap[iD].save()
956
957     ayRelResponseMap[iD].save()
958     #plt.show()
959
960 else:
961     print("ULS:loading")
962     for iD, dirW in enumerate((HEAD, BEAM)):
963         rollResponseMap[iD].load()
964         pitchResponseMap[iD].load()
965         surgeResponseMap[iD].load()
966         swayResponseMap[iD].load()
967         heaveResponseMap[iD].load()
968         axRP0ResponseMap[iD].load()
969         ayRP0ResponseMap[iD].load()
970         azRP0ResponseMap[iD].load()
971         ayRP1ResponseMap[iD].load()
972         axRP1ResponseMap[iD].load()
973         azRP1ResponseMap[iD].load()
974         axRP2ResponseMap[iD].load()
975         ayRP2ResponseMap[iD].load()
976         azRP2ResponseMap[iD].load()
977         axRP3ResponseMap[iD].load()
978         ayRP3ResponseMap[iD].load()
979         azRP3ResponseMap[iD].load()
980         axRP4ResponseMap[iD].load()
981         ayRP4ResponseMap[iD].load()
982         azRP4ResponseMap[iD].load()
983         axRP5ResponseMap[iD].load()
984         ayRP5ResponseMap[iD].load()
985         azRP5ResponseMap[iD].load()
986
987     ayRelResponseMap[iD].load()
988     #     ayRelResponseMap[iD].spectra.values[0, 10].plot()
989     #     plt.show()

```

```

990
991 #             print(s.spectra.values[10, 10].sign)
992 #             print(self.spectra.values[10, 10].sign)
993 #             print(self.sign)
994 #             print(s.sign)
995
996 #####
997 # Stresses in pedestal
998 #####
999
1000 stressResponseMapXX = [Map(values=None, dirW="HEAD", name="StressXX"),
1001                       Map(values=None, dirW="BEAM", name="StressXX")]
1002 stressResponseMapYY = [Map(values=None, dirW="HEAD", name="StressYY"),
1003                       Map(values=None, dirW="BEAM", name="StressYY")]
1004 for ID, dirW in enumerate((HEAD, BEAM)):
1005     stressArrayXX = np.zeros((len(heights), len(periods)))
1006     stressArrayYY = np.zeros((len(heights), len(periods)))
1007     for iH, Hs in enumerate(heights):
1008         for iT, Tz in enumerate(periods):
1009             stressArrayXX[iH, iT], stressArrayYY[iH, iT] = \
1010                 pedestalStress([axRP0ResponseMap[ID].sign.values[iH, iT],
1011                                ayRP0ResponseMap[ID].sign.values[iH, iT],
1012                                azRP0ResponseMap[ID].sign.values[iH, iT]],
1013                                [axRP1ResponseMap[ID].sign.values[iH, iT],
1014                                ayRP1ResponseMap[ID].sign.values[iH, iT],
1015                                azRP1ResponseMap[ID].sign.values[iH, iT]],
1016                                [axRP2ResponseMap[ID].sign.values[iH, iT],
1017                                ayRP2ResponseMap[ID].sign.values[iH, iT],
1018                                azRP2ResponseMap[ID].sign.values[iH, iT]],
1019                                rollResponseMap[ID].sign.values[iH, iT],
1020                                pitchResponseMap[ID].sign.values[iH, iT],
1021                                windSpeed(Hs),
1022                                par)
1023             stressResponseMapXX[ID].values = stressArrayXX / 1000000
1024             stressResponseMapYY[ID].values = stressArrayYY / 1000000
1025
1026             stressResponseMapXX[ID].save(subfolder = "Responses/")
1027             stressResponseMapYY[ID].save(subfolder = "Responses/")
1028             #stressResponseMapXX[ID].plot(levels=[30, 40, 50, 60, 70, 80])
1029
1030
1031 #####
1032 # MASKS
1033 #####
1034 motionMap = np.empty(2, dtype=object)
1035 rollMap = motionMap.copy()
1036 floodMap = motionMap.copy()
1037 pitchMap = motionMap.copy()
1038 stressMap = motionMap.copy()
1039
1040 for ID, dirW in enumerate((HEAD, BEAM)):
1041     motionMap[ID] = Map(values=(ayRelResponseMap[ID].m0 <=
1042                               par['pllRMSAccelWork']['value'] / 1000) *
1043                          (azRP0ResponseMap[ID].m0 <=
1044                            par['pllRMSAccelWork']['value'] / 1000),
1045                          dirW=dirW,
1046                          name="MaskMotion")
1047     rollMap[ID] = Map(values=rollResponseMap[ID].sign.values <=
1048                       par['crMaxHeelGrab']['value'],
1049                       dirW=dirW,
1050                       name="MaskRoll")
1051     pitchMap[ID] = Map(values=pitchResponseMap[ID].sign.values <=
1052                          par['crMaxTrimGrab']['value'],
1053                          dirW=dirW,
1054                          name="MaskPitch")
1055     floodMap[ID] = Map(values=((pitchResponseMap[ID].max.values <=
1056                                par['pllMaxHeel']['value']) *
1057                               (rollResponseMap[ID].max.values <=
1058                                par['pllMaxHeel']['value'])),
1059                          dirW=dirW,
1060                          name="MaskFlood")
1061     stressMap[ID] = Map(values=((stressResponseMapXX[ID].values <=
1062                                par['pllAllowableStressFAT']['value']) *

```

```

1063                                     (stressResponseMapYY[iD].values <=
1064                                     par['pllAllowableStressFAT']['value'])),
1065                                     dirW=dirW,
1066                                     name="MaskStress")
1067
1068     motionMap[iD].save(subfolder="Masks/")
1069     rollMap[iD].save(subfolder="Masks/")
1070     pitchMap[iD].save(subfolder="Masks/")
1071     floodMap[iD].save(subfolder="Masks/")
1072     stressMap[iD].save(subfolder="Masks/")
1073 #     #motionMap[iD].plot()
1074 #     #rollMap[iD].plot()
1075 #     #pitchMap[iD].plot()
1076 #     #floodMap[iD].plot()
1077 #     stressMap[iD].plot()
1078 #     #print(stressResponseMapXX[iD].values)
1079 #     #exit(0)
1080 #     plt.show()
1081 #     exit(0)
1082
1083 #####
1084 # SCATTER
1085 #####
1086
1087 areas = list()
1088 newAreas = areas.copy()
1089 ULSArea = np.zeros((2, 2))
1090 ULSValues = np.zeros((2, 2, 22))
1091 ULSMax = np.zeros((2, 2))
1092 for iA, area in enumerate(scatterSet):
1093     areaNumber = int(float(area['value']))
1094 #     scatterValues = np.genfromtxt(rootPath + code['PATH_SCATTER'] + '/Scatter' + str(areaNumber) + '.cs
1095 #     print(np.sum(scatterValues))
1096 #     #newScatterValues = cf.rebinOfficial(scatterValues, (len(heights), len(periods)))
1097 #     newScatterValues = cf.rebin(scatterValues, (len(heights), len(periods)))
1098 #     print(np.sum(newScatterValues))
1099 #     newScatterValues = newScatterValues / np.sum(newScatterValues)
1100 #     print(np.sum(newScatterValues))
1101 #     exit(0)
1102 #     if np.sum(newScatterValues) < 0.999:
1103 #         print("ERROR sum probability values of area " + str(areaNumber) + " is " + str(np.sum(scatterVa
1104 value = scatterSet[iA]
1105 scatterArray = cf.scatter2(heights, periods, value['extra'][1],
1106                             value['extra'][2],
1107                             value['extra'][3],
1108                             value['extra'][4],
1109                             value['extra'][5],
1110                             value['extra'][6],
1111                             deltaHeights,
1112                             deltaPeriods, returnfHs=False)
1113
1114 #     print(scatterArray)
1115 #     print(np.sum(scatterArray))
1116 #     plt.contour(heights, periods, scatterArray.transpose(), levels=[1e-7, 1e-6, 1e-5])
1117 #     plt.show()
1118 #     exit(0)
1119
1120 newScatterValues = scatterArray
1121 currentScatter = Scatter(newScatterValues, areaNumber)
1122 #     currentScatter.plot(maskMap=motionMap[1], levels = [1e-6, 1e-5, 1e-4])
1123 #     motionMap[1].plot()
1124 #     plt.show()
1125 #     exit(0)
1126 if calculateULS or calculateOperability:
1127     #####
1128     # PARKABILITY AND WORKABILITY
1129     #####
1130     for iD, dirW in enumerate((HEAD, BEAM)):
1131         workability = motionMap[iD].values * rollMap[iD].values * \
1132             pitchMap[iD].values * floodMap[iD].values * stressMap[iD].values
1133         parkability = floodMap[iD].values * stressMap[iD].values
1134         currentScatter.operability[iD] = np.sum(workability * newScatterValues)
1135         currentScatter.parkability[iD] = np.sum(parkability * newScatterValues)

```



```

1136         currentScatter.saveOperability()
1137
1138
1139     if calculateULS:
1140         #####
1141         # ULS
1142         #####
1143
1144     for iD, dirW in enumerate((HEAD, BEAM)):
1145         parkability = floodMap[iD].values * stressMap[iD].values
1146         for iYZ in (0, 1):
1147             if iYZ == 0: # =Y
1148                 m0array = ayRelResponseMap[iD].m0
1149             else:
1150                 m0array = azRP0ResponseMap[iD].m0
1151             [[currentScatter.primaryVal[iD, iYZ],
1152              currentScatter.primaryVal10[iD, iYZ],
1153              currentScatter.primaryVal80[iD, iYZ]],
1154              [currentScatter.primaryProb[iD, iYZ],
1155              currentScatter.primaryProb10[iD, iYZ],
1156              currentScatter.primaryProb80[iD, iYZ]],
1157              [currentScatter.primaryMap[iD, iYZ],
1158              currentScatter.primaryMap10[iD, iYZ],
1159              currentScatter.primaryMap80[iD, iYZ]],
1160              currentScatter.primaryProbMask[iD, iYZ]] = \
1161                 solveProb(m0array,
1162                            par['pllProbAccel']['value'],
1163                            np.multiply(newScatterValues, parkability))
1164             # plt.title(areaNumber)
1165             # currentScatter.plot(levels=[1e-5, 1e-6, 1e-7])
1166             # currentScatter.primaryMap80[iD, iYZ].plot(levels=[1e-6])
1167             # currentScatter.primaryMap10[iD, iYZ].plot(levels=[1e-6])
1168             # currentScatter.primaryMap[iD, iYZ].plot(levels=[1e-6])
1169             # plt.contour(heights, periods, currentScatter.primaryProbMask[iD, iYZ].transpose(), col=
1170             # plt.show()
1171             # exit(0)
1172             # plt.contour(heights, periods, currentScatter.primaryProbMask[iD, iYZ].transpose(),
1173             # plt.show()
1174             # exit(0)
1175         currentScatter.ULS[iD, iYZ] = \
1176             [np.max(np.multiply(axRP0ResponseMap[iD].max90.values,
1177                                currentScatter.primaryProbMask[iD, iYZ])),
1178              np.max(np.multiply(ayRP0ResponseMap[iD].max90.values,
1179                                currentScatter.primaryProbMask[iD, iYZ])),
1180              np.max(np.multiply(azRP0ResponseMap[iD].max90.values,
1181                                currentScatter.primaryProbMask[iD, iYZ])),
1182              np.max(np.multiply(axRP1ResponseMap[iD].max90.values,
1183                                currentScatter.primaryProbMask[iD, iYZ])),
1184              np.max(np.multiply(ayRP1ResponseMap[iD].max90.values,
1185                                currentScatter.primaryProbMask[iD, iYZ])),
1186              np.max(np.multiply(azRP1ResponseMap[iD].max90.values,
1187                                currentScatter.primaryProbMask[iD, iYZ])),
1188              np.max(np.multiply(axRP2ResponseMap[iD].max90.values,
1189                                currentScatter.primaryProbMask[iD, iYZ])),
1190              np.max(np.multiply(ayRP2ResponseMap[iD].max90.values,
1191                                currentScatter.primaryProbMask[iD, iYZ])),
1192              np.max(np.multiply(azRP2ResponseMap[iD].max90.values,
1193                                currentScatter.primaryProbMask[iD, iYZ])),
1194              np.max(np.multiply(axRP3ResponseMap[iD].max90.values,
1195                                currentScatter.primaryProbMask[iD, iYZ])),
1196              np.max(np.multiply(ayRP3ResponseMap[iD].max90.values,
1197                                currentScatter.primaryProbMask[iD, iYZ])),
1198              np.max(np.multiply(azRP3ResponseMap[iD].max90.values,
1199                                currentScatter.primaryProbMask[iD, iYZ])),
1200              np.max(np.multiply(axRP4ResponseMap[iD].max90.values,
1201                                currentScatter.primaryProbMask[iD, iYZ])),
1202              np.max(np.multiply(ayRP4ResponseMap[iD].max90.values,
1203                                currentScatter.primaryProbMask[iD, iYZ])),
1204              np.max(np.multiply(azRP4ResponseMap[iD].max90.values,
1205                                currentScatter.primaryProbMask[iD, iYZ])),
1206              np.max(np.multiply(axRP5ResponseMap[iD].max90.values,
1207                                currentScatter.primaryProbMask[iD, iYZ])),
1208              np.max(np.multiply(ayRP5ResponseMap[iD].max90.values,

```

```

1209         currentScatter.primaryProbMask[iD, iYZ]),
1210     np.max(np.multiply(azRP5ResponseMap[iD].max90.values,
1211         currentScatter.primaryProbMask[iD, iYZ])),
1212     np.max(np.multiply(rollResponseMap[iD].max90.values,
1213         currentScatter.primaryProbMask[iD, iYZ])),
1214     np.max(np.multiply(pitchResponseMap[iD].max90.values,
1215         currentScatter.primaryProbMask[iD, iYZ])),
1216     np.max(np.multiply(heightsMesh,
1217         currentScatter.primaryProbMask[iD, iYZ])),
1218     windSpeed(np.max(np.multiply(heightsMesh,
1219         currentScatter.primaryProbMask[iD, iYZ])))
1220     ] # roll, pitch, Hs, U
1221     currentScatter.setPrimaryULS()
1222     currentScatter.save()
1223
1224     for iD, dirW in enumerate((HEAD, BEAM)):
1225         for iYZ in (0, 1):
1226             print(":")
1227             print(ULSMax[iD, iYZ])
1228             print(currentScatter.primaryVal[iD, iYZ])
1229             if ULSMax[iD, iYZ] < currentScatter.primaryVal[iD, iYZ]:
1230                 ULSArea[iD, iYZ] = areaNumber
1231                 ULSValues[iD, iYZ] = currentScatter.ULS[iD, iYZ]
1232                 ULSMax[iD, iYZ] = currentScatter.primaryVal[iD, iYZ]
1233     newAreas.append(currentScatter)
1234
1235     #print(ULSArea)
1236
1237     if calculateULS:
1238         ULSfile = open(savePath + "ULS/LoadCases.csv", 'w')
1239         ULSfile.write("RP0aX,RP0aY/aYRel,RP0aZ,RP1aX,RP1aY,RP1aZ,RP2aX," +
1240             "RP2aY,RP2aZ,RP3aX,RP3aY,RP3aZ,RP4aX,RP4aY,RP4aZ," +
1241             "RP5aX,RP5aY,RP5aZ,roll,pitch,Hs,U\n")
1242         ULSCase = np.empty((2,2), dtype=str)
1243         ULSCase[0, 0] = "YH"
1244         ULSCase[0, 1] = "ZH"
1245         ULSCase[1, 0] = "YB"
1246         ULSCase[1, 1] = "ZB"
1247
1248         for iD, dirW in enumerate((HEAD, BEAM)):
1249             for iYZ in (0, 1):
1250                 ULSfile.write(ULSCase[iD, iYZ] + ",")
1251                 for val in ULSValues[iD, iYZ]: ULSfile.write(str(round(val, 2)) + ",")
1252                 ULSfile.write("\n")
1253         ULSfile.close()
1254     #####
1255     # FoS
1256     #####
1257     # TODO: include FoS here, then write to xlsx
1258
1259     #####
1260     # SECOND ITERATION LOOP
1261     #####
1262     if loopFatigue:
1263         savePath = rootPath + "/Data/Final/" + checkMultiframeVersion + "/"
1264         # TODO: write function that loads the response arrays again...
1265         #####
1266         # FATIGUE
1267         #####
1268         memberData = np.loadtxt(savePath + "Members/MemberData.csv",
1269             delimiter=",",
1270             skiprows=1,
1271             ndmin=2).transpose() #[ID, FAT, gammaB, gammaN]
1272     #     print("Fatigue member Data:")
1273     #     print(memberData)
1274     members = memberData[0, :]
1275     loga = memberData[1, :]
1276     #####
1277     # WARNING: in the old version, the mass is taken into account here, but should be done in the 'gamma' dir
1278     gammaB = memberData[2, :] #* par['pllForceRatio']['value'] * par['pcalMassJib']['value']
1279     gammaN = memberData[3, :] #* par['pllForceRatio']['value'] * par['pcalMassJib']['value'] * par['pllCompre
1280     FAT = memberData[4, :]
1281     loga2 = memberData[5, :]

```

```

1282 Sb = np.empty((len(heights), len(periods), len(members)), dtype=object)
1283 Sn = Sb.copy()
1284 iD = 1 # BEAM!!
1285 # ayRelResponseMap[0].spectra.values[0, 10].plot()
1286 # print(ayRelResponseMap[0].spectra.values[0, 5].spectrum.spectrum)
1287 # plt.show()
1288 # ayRelResponseMap[0].spectra.values[0, 1].plot()
1289 # ayRelResponseMap[0].spectra.values[0, 8].plot()
1290 # ayRelResponseMap[0].spectra.values[0, 10].plot()
1291 # print(ayRelResponseMap[1].spectra.values[0, 0].spectrum.spectrum)
1292 # plt.show()
1293 # exit(0)
1294
1295 for iD, dirW in enumerate(("HEAD", "BEAM")):
1296     for iH, Hs in enumerate(heights):
1297         for iT, Tz in enumerate(periods):
1298             if iT > 2:
1299                 try:
1300                     aySpec = ayRelResponseMap[iD].spectra.values[iH, iT].spectrum.spectrum
1301                     azSpec = azRP0ResponseMap[iD].spectra.values[iH, iT].spectrum.spectrum
1302                 except AttributeError as e:
1303                     print("empty spectrum (aRel) found in Hs:{0}({2}) Tz:{1}({3}) ".format(Hs, Tz, iH, iT))
1304                     ayRelResponseMap[iD].spectra.values[iH, iT].plot()
1305                     exit(0)
1306                 #print(ayRelResponseMap[iD].spectra.values[iH, iT].spectrum.spectrum)
1307                 #plt.show()
1308
1309                 print("empty spectrum (aRel) found in Hs:{0} Tz:{1} ".format(Hs, Tz, fmt="%0.2f"))
1310             else:
1311                 aySpec = np.zeros((np.shape(omegas)))
1312                 azSpec = np.zeros((np.shape(omegas)))
1313
1314                 for iMF, memberID in enumerate(members):
1315                     if isinstance(ayRelResponseMap[iD].spectra.values[iH, iT], Response) \
1316                         and isinstance(azRP0ResponseMap[iD].spectra.values[iH, iT], Response):
1317                         Sb[iH, iT, iMF] = StressResponse(spectrum=aySpec,
1318                                                         factor = ((1 + compFatRatio) / 2) **2 * gammaB[iMF],
1319                                                         memberNo=memberID,
1320                                                         Hs=Hs,
1321                                                         Tz=Tz,
1322                                                         gammaB=gammaB[iMF],
1323                                                         gammaN=0,
1324                                                         loga=loga[iMF],
1325                                                         loga2=loga2[iMF],
1326                                                         FAT=FAT[iMF],
1327                                                         dirW=dirW,
1328                                                         comment="Bend")
1329
1330                         Sn[iH, iT, iMF] = StressResponse(spectrum=azSpec,
1331                                                         factor = compFatRatio **2 * gammaN[iMF] **2,
1332                                                         memberNo=memberID,
1333                                                         Hs=Hs,
1334                                                         Tz=Tz,
1335                                                         gammaB=0,
1336                                                         gammaN=gammaN[iMF],
1337                                                         loga=loga[iMF],
1338                                                         loga2=loga2[iMF],
1339                                                         FAT=FAT[iMF],
1340                                                         dirW=dirW,
1341                                                         comment="Normal")
1342
1343                     if iH in (5, 10, 15, 20):
1344                         if iT in (12, 14):
1345                             if iMF in (0, 2, 7, 14, 20):
1346                                 Sb[iH, iT, iMF].save()
1347                                 Sn[iH, iT, iMF].save()
1348
1349
1350 fatigueTime = np.loadtxt(savePath + "Members/FatigueTime.csv",
1351                          delimiter=",",
1352                          skiprows=1,
1353                          ndmin=2).transpose() #[ID, FAT, gammaB, gammaN]
1354 maxFatCases = np.max(fatigueTime[0, :])

```

```

1355     fatCaseDamage = np.zeros((maxFatCases, len(members)))
1356     fatTowDamage = np.zeros(len(members))
1357
1358     parkabilityMap = floodMap[iD].values * stressMap[iD].values
1359     SbMap = np.zeros(np.shape(Sb[:, :, 0]))
1360     SnMap = SbMap.copy()
1361     #     print(Sb[10, 10, 0].R)
1362     for iM, memberID in enumerate(members):
1363         for iH, Hs in enumerate(heights):
1364             for iT, Tz in enumerate(periods):
1365                 try:
1366                     SbMap[iH, iT] = Sb[iH, iT, iM].R
1367                 except AttributeError:
1368                     print("no damage found for Hs:{0} Tz:{1} member:{2} sigmaB".format(Hs, Tz, memberID))
1369                 try:
1370                     SnMap[iH, iT] = Sn[iH, iT, iM].R
1371                 except AttributeError:
1372                     print("no damage found for Hs:{0} Tz:{1} member:{2} sigmaN".format(Hs, Tz, memberID))
1373
1374     SbMap[np.isnan(SbMap)] = 0
1375     SnMap[np.isnan(SnMap)] = 0
1376     #     print(SbMap[0::4])
1377     #     print(SnMap[0::4])
1378     #     print(SbMap[15, 12] * 3.16e7 * 20)
1379
1380
1381     #     SbMap = ReturnDamage(Sb[:, :, iM])
1382
1383     #     SnMap = ReturnDamage(Sn[:, :, iM])
1384
1385     for iA, areaNumber in enumerate(fatigueTime[1, :]):
1386     #         print("area: {0}".format(areaNumber))
1387         probMap = [area.values for area in newAreas if area.areaNumber == areaNumber][0]
1388         fatTime = fatigueTime[2, iA]
1389     #         print("time: {0}s".format(fatTime))
1390     #         damageB = np.sum(np.multiply(np.multiply(parkabilityMap, SbMap), probMap)) * fatTime
1391
1392         parkProbMap = np.multiply(parkabilityMap, probMap)
1393         normalizedParkProbMap = np.divide(parkProbMap,
1394                                           np.sum(parkProbMap)) # = will always moved into a sea state
1395         damageB = np.sum(np.multiply(normalizedParkProbMap, SbMap)) * fatTime
1396         damageN = np.sum(np.multiply(np.multiply(parkabilityMap, SnMap), probMap)) * fatTime
1397     #         print("damageB: {0}".format(damageB))
1398
1399         fatCaseDamage[fatigueTime[0, iA] - 1, iM] += damageB
1400         fatCaseDamage[fatigueTime[0, iA] - 1, iM] += damageN
1401
1402
1403     #####
1404     # TOUGH SEA TRIP
1405     #####
1406     tripLength = 1/2/12 #2 weeks! -> year
1407     HsTrip = par['pllSignWaveHeightWeather']['value']
1408     TzTrip = par['pllMaxDeformationPeriod']['value']
1409     SbTrip = Sb[np.argmax(heights >= HsTrip), np.argmax(periods >= TzTrip), iM]
1410     SnTrip = Sn[np.argmax(heights >= HsTrip), np.argmax(periods >= TzTrip), iM]
1411     #     plt.figure()
1412     #     SbTrip.plot()
1413     #     plt.legend()
1414     #     plt.show()
1415     SbTrip.name += "TowingTrip"
1416     SnTrip.name += "TowingTrip"
1417     SbTrip.save()
1418     SnTrip.save()
1419
1420     if SbTrip.R > 0:
1421         fatTowDamage[iM] += SbTrip.R * tripLength
1422     if SnTrip.R > 0:
1423         fatTowDamage[iM] += SnTrip.R * tripLength
1424     print(fatTowDamage)
1425
1426     outputFatDamage = np.vstack((members, fatCaseDamage))
1427     topRow = np.c_[0:maxFatCases+1]

```

```

1428         outputFatDamage = np.hstack((topRow, outputFatDamage))
1429         header = "membersID/fatigueCases and cumulative damage"
1430         np.savetxt(savePath + "Members/FatigueDamage" + dirW + ".csv",
1431                   np.round(outputFatDamage.transpose(), 2), delimiter=",", fmt="%.2f", header=header)
1432         header = "membersID/Towing and cumulative damage"
1433         np.savetxt(savePath + "Members/FatigueDamageTowingTrip" + dirW + ".csv",
1434                   np.round(np.vstack((members, fatTowDamage)).transpose(), 2),
1435                   delimiter=",",
1436                   fmt="%.2f",
1437                   header=header)
1438
1439     if readMultiframe:
1440         multiframe(rootPath=rootPath, code=code,
1441                   inputName=checkMultiframeVersion,
1442                   name=checkMultiframeVersion + "MultiframeOutput")
1443
1444     if postProc:
1445         ""
1446         mu = mus[6]
1447         rao = [np.loadtxt(rootPath + code['PATH_PRECAL'] + "6324Surge2121ton.csv", delimiter = ",", comments="#",
1448                          np.loadtxt(rootPath + code['PATH_PRECAL'] + "6324Sway2121ton.csv", delimiter = ",", comments="#",
1449                          np.loadtxt(rootPath + code['PATH_PRECAL'] + "6324Heave2121ton.csv", delimiter = ",", comments="#",
1450                          np.loadtxt(rootPath + code['PATH_PRECAL'] + "6324Yaw2121ton.csv", delimiter = ",", comments="#",
1451                          np.loadtxt(rootPath + code['PATH_PRECAL'] + "6324Pitch2121ton.csv", delimiter = ",", comments="#",
1452                          np.loadtxt(rootPath + code['PATH_PRECAL'] + "6324Roll2121ton.csv", delimiter = ",", comments="#",
1453
1454         #650,8
1455         mot = (0, 1, 2, 3, 4, 5) #surge, sway, ...
1456         motString = ("Surge", "Sway", "Heave", "Yaw", "Pitch", "Roll")
1457         lineStyle = ("red", "blue", "black", "black, dashed", "blue, dashed", "red, dashed")
1458         #     print(rao[mot][np.where(rao[mot][:, 3] == mu), -1])
1459         #     exit(0)
1460         #for iMu in (0, 3, 6):
1461         #     mu = mus[iMu]
1462
1463         tikzString = tk.beginFigure()
1464         tikzString += tk.begin2Daxis(options='xmin=0, xmax=2.5,ymin=-200,ymax=200, grid=major', xLabel = '$\\omega$')
1465
1466         for iMot in mot:
1467             phase = rao[iMot][np.where(rao[iMot][:, 3] == mu), -1][0]
1468             #     print(phase)
1469             #     print(omegas)
1470             tikzString += tk.addPlotCoordinates(omegas, phase, lineStyle[iMot])
1471             tikzString += tk.addLegend(motString[iMot])
1472
1473         tikzString += tk.addExtraLegend('$\\omega$pcalMassBallastedLong')
1474         tikzString += tk.addExtraLegend('$\\omega$ = ' + str(int(mu)) + 'deg$')
1475
1476         tikzString += tk.end2Daxis()
1477         tikzString += tk.endFigure()
1478
1479
1480         with open(rootPath + code['PATH_TIKZ'] + "/PHASE-Mu" + str(int(mu)) + "-2121ton.tikz", 'w') as tikzFile:
1481             tikzFile.write(tikzString)
1482
1483
1484
1485         #     tikzString = [r'''
1486         #         \begin{tikzpicture}
1487         #             \begin{axis}[axis lines= none, legend pos=south east, x=\textwidth/360, y=\textwidth/360,
1488         #             xmin=-180, xmax=180,
1489         #             ymin=-90, ymax=90]
1490         #                 \newcommand*\unit{\textwidth/360}
1491         #                 \newcommand*\workradius{5\unit}
1492         #                 \node[anchor = south west, inner sep = 0] at (-180,-90) {\includegraphics[trim = 2.25cm 2.8cm 1.
1493         #                 width=\textwidth]{../Pictures/worldmapDNVSeaStates.PNG}};
1494         #                 ''',
1495         #                 r'''
1496         #             \begin{tikzpicture}
1497         #                 \begin{axis}[axis lines= none, legend pos=south east, x=\textwidth/360, y=\textwidth/360,
1498         #                 xmin=-180, xmax=180,
1499         #                 ymin=-90, ymax=90]
1500         #                     \newcommand*\unit{\textwidth/360}

```

```

1501 #         \newcommand*{\workradius}{5\unit}
1502 #         \node[anchor = south west, inner sep =0] at (-180,-90) {\includegraphics[trim = 2.25cm 2.8cm 1.
1503 #         width=\textwidth]{../Pictures/worldmapDNVSeaStates.PNG}};
1504 #         ''
1505 #     ]
1506 #
1507 # for iA, area in enumerate(scatterCoastPar.values()):
1508 #     areaNumber = int(float(area['value']))
1509 #     areaOP = np.loadtxt(savePath + "/Operability/AreaScatter" +
1510 #         str(areaNumber) + ".csv", delimiter=",", comments="#" ) # [[OPH, OPB], [PAH, PAE
1511 #     areaB = float(area['extra'][7])
1512 #     areaH = float(area['extra'][8])
1513 #
1514 #     for iD, dir in enumerate(("HEAD", "BEAM")):
1515 #         OPdeg = 360 * areaOP[0, iD]
1516 #         PAdeg = 360 * areaOP[1, iD]
1517 #         areasOptions = ['fill opacity= 0.7, fill=green', 'fill opacity= 0.7, fill=yellow']
1518 #         tikzString[iD] += r"\draw [" + areasOptions[1] + "]" (" + str(areaB) + r", " + str(areaH) + r")
1519 #         tikzString[iD] += r"\draw [" + areasOptions[0] + "]" (" + str(areaB) + r", " + str(areaH) + r")
1520 #
1521 #
1522 #     tikzString[0] += tk.end2Daxis()
1523 #     tikzString[1] += tk.end2Daxis()
1524 #     tikzString[0] += tk.endFigure()
1525 #     tikzString[1] += tk.endFigure()
1526 #
1527 #     with open(savePath + "/Operability/OperabilityMapHead.tikz", 'w') as f:
1528 #         f.write(tikzString[0])
1529 #     with open(rootPath + code['PATH_TIKZ'] + "/OperabilityMapHead.tikz", 'w') as f:
1530 #         f.write(tikzString[0])
1531 #     with open(savePath + "/Operability/OperabilityMapBeam.tikz", 'w') as f:
1532 #         f.write(tikzString[1])
1533 #     with open(rootPath + code['PATH_TIKZ'] + "/OperabilityMapBeam.tikz", 'w') as f:
1534 #         f.write(tikzString[1])
1535 #
1536 #     multiframe(rootPath=rootPath, code=code, inputName=checkMultiframeVersion, name=checkMultiframeVersion
1537 #     cf.buildScatter2(["prob", "combProb"], fileName="CombinedProbabilityExampleNew", direction="HEAD", savePa
1538 #     cf.buildScatter2(["accel"], RP = [0], fileName="AccelResponseRP0New", direction="BEAM", savePath=save
1539 #     cf.buildScatter2(["accel"], RP = "Rel", fileName="AccelResponseRelativeNew", direction="BEAM", savePa
1540 #     cf.buildScatter2(["accel", "workcrit"], RP = [5], fileName="AccelResponseRP0andCrit", direction="BEAM"
1541 #     cf.buildScatter2(["accel", "workcrit"], RP = [5], fileName="AccelResponseRP0andCrit", direction="HEAD"
1542 #     cf.fatigueOutput("StressResponseBend23.2.5.6.5TowingTripBEAM", rootPath, code, par, fileName="FatigueStr
1543 #     cf.fatigueOutput("StressResponseBend1.2.5.6.5TowingTripBEAM", rootPath, code, par, fileName="FatigueStr
1544 #     cf.buildScatter2(["workcrit"], fileName="Criteria", direction="HEAD", savePath=savePath, heights=height
1545 #     cf.buildScatter2(["workcrit"], fileName="Criteria", direction="BEAM", savePath=savePath, heights=height
1546 #     cf.buildScatter2(["accel", "workcrit"], RP = [4], fileName="CylinderSelectionResponseHead", direction=

```