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Delft University of Technology

Deriving mathematical manoeuvring models for bare hull ships using viscous-flow calculations

by

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International Conference on Computational Methods in Marine Engineering MARINE 2007 P. Bergan, J. García, E. Oñate and T. Kvamsdal (Eds) ©CIMNE, Barcelona, 2007

DERIVING MATHEMATICAL MANOEUVRING MODELS FOR BARE HULL SHIPS USING VISCOUS-FLOW CALCULATIONS

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Key words: CFD, RANS, Viscous Flow, Ship Manoeuvring, Mathematical Model.

1 INTRODUCTION

In the present paper, the work conducted by the author regarding efficient determination of hydrodynamic coefficients for manoeuvring ships within the manoeuvring work package of the EU project VIRTUE¹ is presented. Based on various viscous-flow calculations for steady drift motion, steady yaw motion and combined drift/yaw motion, a mathematical model for the bare hull forces and moments is derived. The results show that using accurate viscous-flow calculations, a considerable improvement in the prediction of the forces and moments on ships compared to conventional empiric methods published in open literature is obtained.

Two hull forms are considered in this study. The first ship is the Hamburg Test Case (HTC), a single-screw ship. The second ship is the MARIN LNG carrier with twin gondolas, see also Jurgens et al. [1].

2 CALCULATIONS

Series of calculations have been conducted in order to be able to derive the required hydrodynamic coefficients, see Table 1. More information regarding the solver used for the calculation, the method of solution, nomenclature and a sensitivity study conducted for the HTC can be found in Toxopeus [2]. The experimental values were obtained by HSVA within VIRTUE.

| Series | HTC | MARIN LNG |
|-----------------|---|--|
| püre drift | $\beta = 0, 2.5, 5, 10, 15^{\circ}$ | $\theta = 0, 2.5, 5, 10, 20, 30^{\circ}$ |
| pure yaw | $\gamma = 0.1, 0.15, 0.2, 0.3, 0.4, 0.556$ | - |
| combined motion | $(\beta,\gamma) = (5^{\circ},0.2), (10^{\circ},0.2), (6^{\circ},0.4), (10^{\circ},0.4), (15^{\circ},0.4)$ | - |

Table 1: Overview of calculations

¹Part of the work conducted for this paper has been funded by the Commission of the European Communities for the Integrated Project VIRTUE under grand 516201 in the 6th Research and Technological Development Framework Programme (Surface Transport Call).

3 MATHEMATICAL MANOEUVRING MODEL

Mathematical manoeuvring models for the bare hull consist in general of two different components: (added) mass coefficients and damping coefficients. In earlier work (e.g. Vassalos et al. [3], Bulian et al. [4]), it was found that the sensitivity of the manoeuvrability to changes in the added mass coefficients is small. Therefore, no calculations are required to obtain the added mass coefficients and the added mass coefficients are approximated by using empiric formulae, see e.g. Clarke et al. [5].

The damping coefficients in the mathematical model are derived in several steps:

- 1. Linear coefficients for simple motions (slope of curves at $\beta = 0$ resp. $\gamma = 0$)
- 2. Non-linear coefficients for pure transverse motion and pure rotation using empirical relations (based e.g. on work of Hooft [6])
- 3. Other non-linear components for simple motions
- 4. Cross-terms, based on combined motions

This approach is chosen to enable accurate modelling of the linearised behaviour for coursekeeping (1.), realistic modelling of the low speed/harbour manoeuvring characteristics (2.) and accurate modelling of non-linear manoeuvres (3. and 4.). The following non-dimensionalised mathematical model for the transverse force Y and yawing moment N is obtained:

$$Y' = Y'_{\beta} \cdot |\cos\beta| \cdot \sin\beta + Y'_{\gamma} \cdot \cos\beta \cdot \gamma + Y'_{\beta|\beta|} \cdot \sin\beta \cdot |\sin\beta| + Y'_{\beta|\gamma|} \cdot \beta |\gamma| + Y'_{ab} \cdot |\cos^{a_{y}}\beta \cdot \sin^{b_{y}}\beta| \cdot sign \sin\beta$$
(1)

$$N' = N'_{\beta} \cdot \cos\beta \cdot \sin\beta + N'_{\gamma} \cdot |\cos\beta| \cdot \gamma + N'_{u\gamma c} \cdot |\cos\beta \cdot \gamma^{c_{n}}| \cdot sign\gamma + N'_{\gamma|\gamma|} \cdot \gamma \cdot |\gamma| + \left(N'_{\beta\beta\gamma} \cdot \beta + N'_{\beta\gamma\gamma} \cdot \gamma \cdot sign \cos\beta\right) \cdot \beta\gamma + N'_{ab} \cdot |\cos^{a_{n}}\beta \cdot \sin^{b_{n}}\beta| \cdot sign (\cos\beta \cdot \sin\beta)$$
(2)

With equations (1) and (2) as the mathematical formulation for the bare hull manoeuvring forces, the hydrodynamic derivatives are determined using the results of the available viscous-flow calculations presented in Table 1. Table 2 shows the obtained manoeuvring coefficients.

| Step | Coefficient | HTC | MARIN LNG | Coefficient | HTC | MARIN LNG |
|------|----------------------|---------|-----------|--------------------------|---------|-----------|
| 1 | Y'_{β} | 0.1830 | 0.0416 | N'_{β} | 0.1403 | 0:0894 |
| | Y'_{γ} | 0.0250 | - | N'_{γ} | -0.0270 | - |
| 2 | $Y'_{\beta \beta }$ | 1.1100 | 0.9662 | | -0.0375 | -0.1755 |
| 3 | Y'_{ab} | -0.6552 | -0.9802 | N'ab | 0.1314 | -0.0373 |
| | a_y | 3 | 2 | an | 1 | 3 |
| | b_y | 2 | 3 | bn | 3 | 2 |
| | | | | $N'_{n\gamma c}$ | -0.0073 | - |
| | | | | Cn | 2 | - |
| 4 | $Y'_{\beta \gamma }$ | 0.1635 | - | $N'_{\beta\beta\gamma}$ | -0.8682 | - |
| | | | | $N'_{\beta\gamma\gamma}$ | 0.2753 | - |

Table 2: Estimated bare hull manoeuvring coefficients

In Figure 1, good agreement with the experiments (designated exp) is seen for the results based on the viscous flow calculations (designated cfd) and based on the mathematical model



(designated cfd-fit). Only the HTC results for the transverse force Y for pure yaw (γ) deviate from the measurements. The magnitude of the Y force during pure rotation is however very small and of less significance than the other force or moment components.

Figure 1: Comparison between experiments and predicted forces and moments

In Table 3, the linear derivatives for the HTC are given according to empirical formulae in open literature. Furthermore, the derivatives based on the measurements (exp), on the viscous-flow calculations (cfd) and on the slender-body theory (sb, see e.g. Toxopeus [7]) are presented. The errors in the predictions compared to the experimental values are shown in Figure 2. The error is defined as error = (prediction/experiment - 1). It is seen that in general, both the viscous-flow calculations and the slender-body coefficients approximate the experimental values better than the other empiric formulae. Especially the de-stabilising (N'_{β}/Y'_{β}) and stabilising $N'_{\gamma}/(Y'_{\gamma} - m')$ arms are closer to the experimental ones.

4 CONCLUSIONS

The study presented in this paper demonstrates that the forces and moments acting on a ship hull in manoeuvring conditions can be accurately predicted using viscous-flow calculations. Comparisons with empiric formulae proposed in the past show that better linear hydrodynamic derivatives can be obtained when using CFD.

REFERENCES

- [1] A.J. Jurgens, R. Hallmann, and J. Tukker. Experimental investigation into the flow around a manoeuvring LNG carrier on shallow water. NAV International Conference on Ship and Shipping Research, June 2006.
- [2] S.L. Toxopeus. Calculation of hydrodynamic manoeuvring coefficients using viscous-flow calculations. 7th ICHD International Conference on Hydrodynamics, pages 493-502, October 2006.
- [3] D. Vassalos, S.H. Park, and B.S. Lee. Developing a manoeuvring design capability for naval vessels. PRADS Practical Design of Ships and Other Floating Bodies, pages 1.616-1.617, September 1995.
- [4] G. Bulian, R.G. Nicolosi, and Francescutto A. On the effect of uncertainty modeling in the hydrodynamic derivatives of a ship manoeuvring mathematical model. 7th ICHD International Conference on Hydrodynamics, pages 359-368, October 2006.
- [5] D. Clarke, P. Gedling, and G. Hine. The application of manocuvring criteria in hull design using linear theory. *Transactions RINA*, pages 45-68, 1983.
- [6] J.P. Hooft. The cross flow drag on a manoeuvring ship. Ocean Engineering, Vol. 21(3):329-342, 1994.
- [7] S.L. Toxopeus. Validation of slender-body method for prediction of linear manoeuvring coefficients using experiments and viscous-flow calculations. 7^{th} ICHD International Conference on Hydrodynamics, pages 589–598, October 2006.
- [8] K. Kijima, Y. Nakiri, and Y. Furukawa. On a prediction method for ship manoeuvrability. International Workshop on Ship Manoeuvrability - 25 Years CPMC at HSVA, October 2000.

| Method | Ref | Y'_{β} | N'_{β} | N'_{β}/Y'_{β} | Y'_{γ} | N'_{γ} | $N_{\gamma}'/(Y_{\gamma}'-m')$ |
|----------|-----|--------------|--------------|-------------------------|---------------|---------------|--------------------------------|
| Kijima | [8] | 0.373 | 0.134 | 0.359 | 0.158 | -0.054 | 0.730 |
| Vassalos | [3] | 0.373 | 0.110 | 0.294 | 0.067 | -0.053 | 0.323 |
| Clarke | [5] | 0.357 | 0.139 | 0.390 | 0.067 | -0.053 | 0.323 |
| Norrbin | [5] | 0.365 | 0.130 | 0.356 | 0.092 | -0.078 | 0.553 |
| exp | | 0.175 | 0.137 | 0.782 | 0.032 | -0.039 | 0.192 |
| cfd | | 0.183 | 0.140 | 0.767 | 0.025 | -0.027 | 0.130 |
| sb | [7] | 0.253 | 0.151 | 0.598 | 0.058 | -0.038 | 0.215 |

Table 3: Comparison of linear coefficients, HTC



Figure 2: Comparison between errors in prediction of linear coefficients