

SEAKEEPING STANDARD SERIES FOR OBLIQUE SEAS

(A SYNOPSIS)

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Abstract - The seakeeping performance in oblique seas for a series of 72 cruiser-stern hull forms has been evaluated analytically and is presented in a systematic way. The hull form series have been created by Loukakis and Chrysosostomidis (1975) by extending the principal characteristics of the Series 60 to cover usual shipbuilding practice. In that work, however, only the seakeeping performance in head seas was presented. Recently, the seakeeping performance of the Extended Series 60 was re-evaluated for both head seas and oblique seas. The complete results are presented in tabular and graphical form as a function of the principal characteristics of the ship, the Froude number (including $F_n=0$, missing in the original series), the non-dimensional modal wave period and the heading angle in a separate NTUA report (Grigoropoulos et al, 1994). In the present paper, the results for one case are given in tabular form accompanied by graphical representation. They include: heave, pitch, bending moment amidships, added resistance, absolute vertical acceleration and relative vertical motion at the bow and the stern regions and relative vertical velocity at stations 2 and 4 where slamming is likely to occur.

Keywords : seakeeping responses, Seakeeping Standard Series, Series 60, oblique seas, strip theory, vertical ship motions, added resistance in waves

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NOMENCLATURE

- A** = wave amplitude
- B** = beam
- C_B** = block coefficient
- Fn** = Froude number, $Fn = V_s / \sqrt{gL_{WL}}$
- g** = acceleration of gravity, 9.81 m/sec²
- H_{1/3}** = significant wave height, in [m]
- K_{yy}** = longitudinal radius of gyration about LCG
- K_{yy'}** = longitudinal radius of gyration of the forward part of the ship about LCG_F
- L, L_{BP}** = length between perpendiculars
- L_{WL}** = length on design waterline, for the Extended Series 60 $L_{BP} = 0.983 L_{WL}$
- LCB** = longitudinal position of centre of buoyancy
- LCF** = longitudinal position of centre of flotation
- LCG** = longitudinal position of centre of gravity
- LCG_F** = distance of the centre of gravity of the forebody from amidships
- RAO** = Response Amplitude Operator
- RMS** = Root-Mean-Square value
- RM** = amplitude of relative bow motion
- SSS** = Seakeeping Standard Series
- t** = thrust deduction factor
- T** = draft
- T_p** = modal period
- T_{p'}** = non-dimensional modal period, $T_p' = T_p / \sqrt{L/g}$
- V_s** = ship speed

W_F = weight of forward part of ship

W_T = total weight of ship

β = heading angle, $\beta=180^\circ$ corresponds to head seas

θ = pitch amplitude

κ = wave number

λ = wave length

μ_v = RAO of bending moment, $\mu_v = \frac{\text{Bending Moment Amplitude}}{\rho g A B L^2}$

ρ = specific density of sea water

σ_{aw} = RAO of added resistance, $\sigma_{aw} = \frac{\text{Mean Added Resistance}}{\rho g A^2 (B^2 / L)}$

ω = circular frequency

1. INTRODUCTION

The seakeeping performance of a ship can either be predicted using computer codes or measured in a seakeeping basin. However, during a feasibility study or in the preliminary ship design phase, the hull lines of the vessel are not yet available and hence, neither of the aforementioned methods is applicable. In an attempt to assist the naval architect in predicting the seakeeping behaviour in such cases, Loukakis and Chryssostomidis (1975) presented the Seakeeping Standard Series (SSS) for cruiser-stern ships. In that work the authors extended the principal characteristics of the Series 60 to cover the usual shipbuilding practice and they computed the seakeeping performance of the resulting series analytically. Thus, they generated a set of tables containing the motion characteristics in head seas of 72 Extended Series 60 hull forms. The information was given for a systematic variation of the principal ship geometric parameters i.e. block coefficient C_B , length-to-beam ratio L/B and beam-to-draft ratio B/T . The results were

presented in tabular form for various fully developed seas, expressed in terms of significant wave height to length ratio $H_{1/3}/L_{BP}$ and ship speed V_s , expressed as non-dimensional Froude number $F_n = V_s / \sqrt{gL_{WL}}$, where L_{WL} is the length of the vessel at the design waterline.

Since their presentation, the series have been extensively used in naval architecture practice as well as a teaching tool. The usefulness of the series has been appreciated, especially in studies on the effect of hull form parameters on the seakeeping behaviour of ships. In this respect, the papers of Beukelman and Huijser (1977), Schmitke and Murdey (1980), Lee (1983), Pawlowski (1983), Loukakis et al (1983), Grigoropoulos and Loukakis (1988, 1990) and Wilson (1985) should be mentioned. Furthermore, Bhattacharyya (1978) included the series in his book on the dynamics of marine vehicles.

Recently, Townsin et al (1994) recognized the significance of the series and underlined their two strong points, the wide range of hull forms and the number of the seakeeping responses calculated. However, it was pointed out that, the seakeeping performance of the 72 hull forms from the Extended Series 60 has been evaluated analytically only for head seas, while the $H_{1/3}/L_{BP}$ ratio range used, starting from $H_{1/3}/L_{BP} = 0.015$, corresponds to only relatively high sea states for the longer ships of today.

The aforementioned shortcomings of the series have also been noticed by the authors of the original paper. The inconvenient selection of the $H_{1/3}/L_{BP}$ ratios is closely connected to the use of single-parameter modelling of the sea state (fully developed seas), while the two-parameter spectral models are better representations of the actual sea conditions. Since the series refer to vertical motions only, which are linear with respect to the wave height, or to added resistance, which is proportional to the square of the wave height, these shortcomings could be remedied by appropriate scaling of the $H_{1/3}$ for the same modal

period T_p . However, it would be more convenient if the results were presented for a range of modal periods and for unity significant wave height.

In addition, the absence of the zero-speed responses from the seakeeping tables, prevented the use of the series in some applications e.g. the design of stationary ship-factories or storage ships. Finally, scant usable information exists in the literature for ship responses in oblique seas, although such knowledge can be useful for ship routing and seakeeping operability studies.

With the above in mind, it was decided to re-evaluate the seakeeping performance of the Extended Series 60 for all headings, using the same seakeeping responses as the initial paper and including a wider range of non-dimensional sea states as well as zero-speed responses. The strip theory of Salvesen, Tuck and Faltinsen (1970) has again been used for the estimation of the ship motions and bending moments. The energy method of Gerritsma and Beukelman (1972), as extended by Loukakis and Sclavounos (1978), has been used for the prediction of added resistance in head to beam seas.

The usage of the same theories for analytical predictions after some twenty three years, underlines the remarkable fact that the simple strip theory continues to give results useful for practical purposes in an efficient computationally manner and it has not been superseded by newer three-dimensional theories, except for the zero-speed case. In this respect, it is interesting to note that in two recent International Conferences on Ship and High Speed Craft Motions & Manoeuvrability, no less than 8 authors were using strip theory for predictions in the absence of other tools, even at very high Fns ! For a review of seakeeping theories and their applicability one can refer to Odabasi and Hearn (1977), Hearn and Donati (1980) and to the reports of the Seakeeping Committee of ITTC (1978,

1984, 1993), where the usefulness of strip theory is generally recognized, at least for cruiser stern ships sailing at zero to moderate speeds.

In order to demonstrate the applicability of strip theory to oblique ship responses, the analytical results in regular waves have been compared to the experimental results conducted a long time ago at Wageningen (Vossers et al, 1960 and 1961). Figures 1, 2, 3 and 4 show that for a $L_{BP} = 120$ m, $C_B = 0.7$, $L/B = 7$, $B/T = 3.0$ Series 60 hull form, the predictions for pitching motion, relative bow motion, bending moment and added resistance can be used in practice, with the bending moment and the added resistance results being the weaker predictions (in the case of added resistance, the measured added thrust is approximately converted to added resistance using the measured thrust deduction factor in calm water, $t = 0.184$). Furthermore, the analytically estimated bow acceleration and added resistance responses in regular waves, for the S-175 containership proposed by the I.T.T.C. for comparison studies, are shown in Figures 5 and 6 with the respective experimental results conducted in the towing tank of the Laboratory for Ship and Marine Hydrodynamics at NTUA (Grekoussis et al, 1986).

The SSS in oblique seas contain, in tabular form and as a function of the principal characteristics of the ship, the Froude number, the non-dimensional modal wave period and the heading angle, the results of the aforementioned computations, Table 1. Due to obvious space limitations, the seakeeping responses for only one case could be accommodated in the present paper in tabular form, accompanied by graphical representation. The complete results, for all 72 cases, are available in a separate technical report accompanied by a PC floppy disk (Grigoropoulos et al, 1994). The results of the Seakeeping Tables can be interpolated for the prediction of the seakeeping performance of cruiser-stern ships not necessarily with Series 60 hull forms.

Finally, it should be noted that only vertical responses have been considered in the seakeeping tables. The lateral ship responses, are highly dependent on the non-linear behaviour of rolling motion, where roll damping is mostly induced by bilge keels, active fins and other anti-rolling devices.

2. CONTENTS OF THE SEAKEEPING TABLES

The seakeeping performance of the 72 Extended Series 60 hull forms has been calculated for all headings and for the same seakeeping responses as the initial paper, that is : heave, pitch, wave bending moment amidships, added resistance, absolute vertical acceleration at stations 2, 4 and 20, relative vertical motion at stations 2, 4 and 20 and relative vertical velocity at stations 2 and 4.

The acceleration and the relative motion have been calculated at three points along the ship, the AP (station 20), 20% aft of FP (station 4) and 10% aft of FP (station 2), while the relative velocity has been computed at the latter two points in the bow region. The above points for the calculation of the relative motions and velocities have been selected so that the random events (propeller emergence, deck wetness and bottom slamming) could be estimated. The vertical acceleration, depending on the wave direction, has its maximum value in the FP and AP regions.

Since vertical ship responses and added resistance vary linearly with the significant wave height $H_{1/3}$ and its square respectively, they have been calculated for sea states following the Bretschneider two-parameter spectral model (Bretschneider, 1959) with $H_{1/3}$ equal to unity. The calculations have been performed for a range of eight modal periods, with non-dimensional values $T_p' = T_p / \sqrt{L_{BP} / g}$ ranging from 1.5 to 5.0 at 0.5 intervals. These values of T_p' correspond to $T_p = 3.4 \div 11.3$ sec for a 50 m vessel, to $T_p = 4.8 \div 16.0$ sec

for a 100 m vessel and to $T_p = 6.8+22.6$ sec for a 200 m vessel. Thus, they correspond to sea states appropriate for the determination of the seakeeping responses of different size ships, if the naturally observed relationship between wave height and wave period is taken into account.

The results are in the form of integer values in the range 0 - 9999. In order to restrict the results in this range, the following "non-dimensionalizations" have been used:

$$\text{Heaving motion} = (\text{RMS heave at amidships}) * 10^6 / (L_{BP} H_{1/3})$$

$$\text{Pitching motion} = (\text{RMS pitch in degrees}) * 10^4 / H_{1/3}$$

$$\text{Bending moment} = (\text{RMS bending moment at amidships}) * 10^9 / (\rho g L_{BP}^4 H_{1/3})$$

$$\text{Added resistance} = (\text{mean added resistance}) * 10^{10} / (\rho g L_{BP}^3 H_{1/3})$$

$$\text{Relative motion} = (\text{RMS relative motion}) * 10^6 / (L_{BP} H_{1/3})$$

$$\text{Relative velocity} = (\text{RMS relative velocity}) * 10^5 / (\sqrt{gL_{BL}} H_{1/3})$$

$$\text{Acceleration} = (\text{RMS acceleration}) * 10^5 / (g H_{1/3}),$$

where all results refer to unit significant wave height.

In this fashion, three pages are necessary for the tabular presentation of the results for each hull form and a sample page is shown in Table 1.

The seakeeping responses have been calculated for each of the 72 hull forms of the Extended Series 60 with $C_B = 0.55$ (0.05) 0.90, $L/B = 5.5, 7.0$ and 8.5 and $B/T = 2.0, 3.0$ and 4.0, at four ship speeds corresponding to Froude numbers 0.0, 0.1, 0.2 and 0.3 and for heading angles ranging from head seas (180°) to following seas (0°) at 15° intervals. It should be noted that the radius of gyration K_{yy} has been assumed to be equal to $0.24 L_{BP}$, while the weight of the forebody W_F and the distance of the centre of gravity of the forebody LCG_F from amidships are connected to C_B by the following relations:

$$\frac{W_F}{W_T} = 0.20C_B + 0.36 \quad (1)$$

$$\frac{LCG_F}{L_{BP}} = 0.10C_B + 0.13 \quad (2)$$

where W_T is the total weight of the vessel.

Furthermore, the longitudinal radius of gyration for the forebody about the LCG_F of the vessel K_{yy} has been taken equal to $0.125 L_{BP}$.

The justification of these choices has been described in Loukakis and Chryssostomidis (1975).

The three-parameter Extended Lewis-form family, proposed by Athanassoulis and Loukakis (1985) has been used for the representation of the hull forms. Besides to the sectional breadth, draft and area, the sectional KB is used in the conformal mapping of the sections to the unit circle. Thus, the actual longitudinal KB(x) distribution was taken into account during the computations. On the contrary the two-parameter Lewis-form family (Lewis, 1929) has been used for the calculations in the initial paper.

3. DISCUSSION

Seakeeping predictions as a tool for designers of merchant ships is not of paramount importance to the ship design spiral. Merchant ships are primarily designed to carry a given amount of deadweight at a prescribed speed. They have, however, to sail through rough seas and their seakeeping qualities are therefore of some importance, especially in the form of the sustained sea speed. In this situation, the analytical contents of the seakeeping tables do support the practising naval architect in including seakeeping considerations in ship design and operations.

This is true in particular for the case of oblique seas, since most of the reference material pertains to head seas only. This is a void the present series can help to fill, as they pertain to ship responses in all headings in realistic seaways and as strip theory is well known to predict real life with adequate engineering approximation for the hull form and the speeds of the series.

Using the tables, sufficient information can be obtained for a qualitative and quantitative estimation of the seakeeping qualities of any hull form resembling the parent. Furthermore, taking advantage of the quite wide ranges of the C_B coefficient and the L/B and B/T ratios of the data base, conclusions can be drawn on the effect of any variation of these parameters on the seakeeping performance of the ship to be designed. Since, according to usual practice, these parameters are always determined at the preliminary ship design stage, when the hull form is only vaguely defined, the proposed series can support the designer for the creation of a hull form with good seakeeping qualities.

Moreover, the existence of information about seakeeping responses in oblique seas can help the naval architect broaden his understanding about what happens at sea. Using as example the central ship of the series, the following responses are plotted: heave (Fig. 7), pitch (Fig. 8), bending moment (Fig. 9), mean added resistance (Fig. 10), all at Froude number $F_n = 0.20$ and bow acceleration at $F_n = 0.00, 0.10, 0.20$ and 0.30 (Figs. 11, 12, 13 and 14).

Obviously, the non-dimensional results are ported to real life, i.e. the ship in real sea states, via the ship length and the wave height. But, nevertheless, the shape of the corresponding curves is interesting per se. Thus, heave resonance occurs around 100° heading angle (Fig. 7) and pitch resonance for the lower sea states (low values of T_p) is not for head seas but for headings close to the 90° minimum, from both directions (Fig. 8).

Similar behaviour pertains for the bending moment (Fig. 9) and even added resistance is not largest for head seas, when in the lower sea states (Fig. 10). For the case of bow acceleration, the results (Figs. 11, 12, 13 and 14) are even more interesting as the resonance is both pronounced and far away, (around beam seas), from head seas at zero speed. Gradually, as speed increases, the situation moves toward the conventional wisdom that head seas induce larger responses, although this is not true for the three lower sea states even at $F_n = 0.30$.

4. CONCLUSION

The widely recognized usefulness of the seakeeping standard series has been extended by including the zero speed and the oblique seas cases.

The zero-speed and the oblique seas results can be of further use to the designer in the case of special ships, which operate at rest or when oblique seas operation is of importance. Thus, for a given route of the vessel under investigation, the designer, using the tables and the related environmental data, can estimate the operational characteristics of the proposed hull form and decide upon necessary modifications.

However, in addition to the hull form parameters considered in the initial series, additional parameters referring to the waterplane area (C_w and LCF) and the LCB position affect the seakeeping behaviour of ships too. The same is true for the shape of the bow region sections (U or V) as well as for above water characteristics of the hull form (flare, stem angle and others). These parameters can not be examined within the scope of SSS, as they would increase dramatically the number of the hull variants, which is inconsistent with the stated intention of using the Tables during the feasibility study and the preliminary design stage only.

Thus, the selection of main hull form parameters should be accompanied by a subsequent selection of the waterline form parameters C_w and LCF and the longitudinal distribution of KB. The statistical method of Bales (1980) or the direct technique proposed by Grigoropoulos and Loukakis (1988, 1990) could assist the designer in this phase. Both techniques refer to head seas results.

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LABELS OF FIGURES

Fig. 1. Analytically estimated (Salvesen-Tuck-Faltinsen strip theory, 1970) and experimentally measured (Vossers et al, 1960) pitch RAO for a 120-metre Series 60 ship with $C_B = 0.70$, $L/B = 7.0$ and $B/T = 3.0$ at $F_n = 0.20$.

Fig. 2. Analytically estimated (Salvesen-Tuck-Faltinsen strip theory, 1970) and experimentally measured (Vossers et al, 1960) relative bow motion RAO for a 120-metre Series 60 ship with $C_B = 0.70$, $L/B = 7.0$ and $B/T = 3.0$ at $F_n = 0.20$.

Fig. 3. Analytically estimated (Salvesen-Tuck-Faltinsen strip theory, 1970) and experimentally measured (Vossers et al (1960) bending moment RAO for a 120-metre Series 60 ship with $C_B = 0.70$, $L/B = 7.0$ and $B/T = 3.0$ at $F_n = 0.20$.

Fig. 4. Analytically estimated (Loukakis and Sclavounos, 1978) and experimentally derived (Vossers et al 1960) added resistance RAO for a 120-metre Series 60 ship with $C_B = 0.70$, $L/B = 7.0$ and $B/T = 3.0$ at $F_n = 0.20$.

Fig. 5. Comparison of strip theory prediction to experimental results for the vertical acceleration RAO at a position 15% L aft of the F.P of the S-175 standard ship adopted by ITTC. Head seas.

Fig. 6. Comparison of strip theory prediction to experimental results for the added resistance RAO of the S-175 standard ship adopted by ITTC. Head seas.

Fig. 7. Analytically estimated RMS heave for a set of heading angles for the Series 60 central ship at $F_n = 0.20$.

Fig. 8. Analytically estimated RMS pitch for a set of heading angles for the Series 60 central ship at $F_n = 0.20$.

Fig. 9. Analytically estimated RMS bending moment for a set of heading angles for the Series 60 central ship at $F_n = 0.20$.

Fig. 10. Analytically estimated RMS added resistance for a set of heading angles for the Series 60 central ship at $F_n = 0.20$.

Fig. 11. Analytically estimated RMS vertical acceleration at station 2 (10% aft of FP) for a set of heading angles for the Series 60 central ship at $F_n = 0.00$.

Fig. 12. Analytically estimated RMS vertical acceleration at station 2 (10% aft of FP) for a set of heading angles for the Series 60 central ship at $F_n = 0.10$.

Fig. 13. Analytically estimated RMS vertical acceleration at station 2 (10% aft of FP) for a set of heading angles for the Series 60 central ship at $F_n = 0.20$.

Fig. 14. Analytically estimated RMS vertical acceleration at station 2 (10% aft of FP) for a set of heading angles for the Series 60 central ship at $F_n = 0.30$.

Table 1. Seakeeping responses per meter of $H_{1/3}$ for Series 60 with $C_B=0.700$, $L/B=7.0$, $B/T=3.0$ and $LCB=1.0\% L$

		SHIP HEADING ANGLE IN DEGREES														SHIP HEADING ANGLE IN DEGREE																											
		HEAD							BEHM							FOLLOW							HEAD							BEHM							FOLLOW						
Fn	Tp	180	165	150	135	120	105	90	75	60	45	30	15	0	Fn	Tp	180	165	150	135	120	105	90	75	60	45	30	15	0														
HEAVE	0.0	1.5	238	253	286	286	428	1085	1348	754	320	222	201	186	187	0.1	1.5	123	124	136	224	486	992	1335	877	343	172	131	107	100													
	2.0	411	427	471	571	989	1798	2118	1517	845	506	382	335	330	2.0	403	427	534	801	1279	1877	2104	1508	788	437	294	239	223															
	2.5	631	662	761	958	1370	2002	2263	1835	1283	915	712	612	584	2.5	770	812	956	1241	1673	2137	2254	1774	1194	842	645	546	516															
	3.0	946	980	1087	1279	1605	2046	2235	1954	1555	1253	1060	954	922	3.0	1102	1142	1273	1512	1845	2168	2231	1888	1469	1179	995	893	861															
	3.5	1222	1253	1346	1506	1749	2056	2189	2001	1720	1490	1331	1237	1207	3.5	1360	1394	1500	1681	1922	2149	2186	1945	1647	1422	1268	1179	1149															
	4.0	1432	1458	1534	1660	1840	2055	2149	2021	1822	1650	1523	1447	1422	4.0	1545	1572	1655	1792	1967	2125	2147	1978	1782	1591	1467	1393	1368															
	4.5	1585	1606	1667	1765	1899	2053	2119	2032	1888	1758	1660	1599	1578	4.5	1677	1698	1762	1865	1994	2107	2119	1997	1839	1708	1610	1550	1530															
	5.0	1696	1712	1760	1837	1939	2049	2096	2037	1931	1832	1756	1707	1691	5.0	1770	1787	1837	1915	2011	2091	2096	2009	1892	1791	1713	1665	1648															
	0.2	1.5	64	68	99	182	389	876	1332	1045	274	121	86	74	70	0.3	1.5	38	41	47	110	275	768	1353	966	245	100	80	77	76													
	2.0	555	597	733	1001	1429	1910	2077	1490	722	399	261	205	191	2.0	547	589	725	1002	1434	1941	2046	1387	683	377	249	200	187															
2.5	1177	1226	1373	1631	1958	2242	2230	1711	1122	795	611	518	489	2.5	1437	1490	1647	1877	2156	2341	2199	1625	1076	764	588	500	473																
3.0	1521	1561	1683	1877	2103	2280	2215	1827	1403	1128	954	857	827	3.0	1957	1990	2078	2206	2336	2387	2193	1763	1359	1093	925	832	802																
3.5	1703	1736	1829	1974	2131	2238	2177	1896	1590	1373	1225	1139	1111	3.5	2128	2148	2201	2275	2343	2322	2162	1847	1551	1338	1194	1111	1083																
4.0	1809	1833	1904	2010	2124	2194	2142	1938	1716	1547	1426	1354	1330	4.0	2178	2191	2219	2257	2285	2263	2132	1902	1682	1515	1396	1325	1302																
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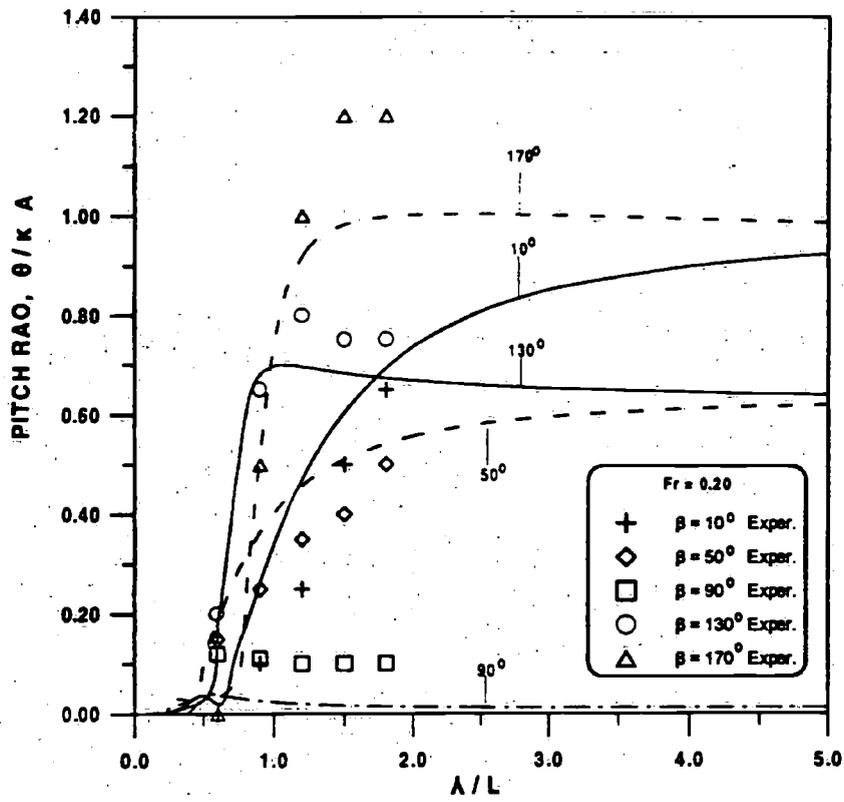


Fig. 1

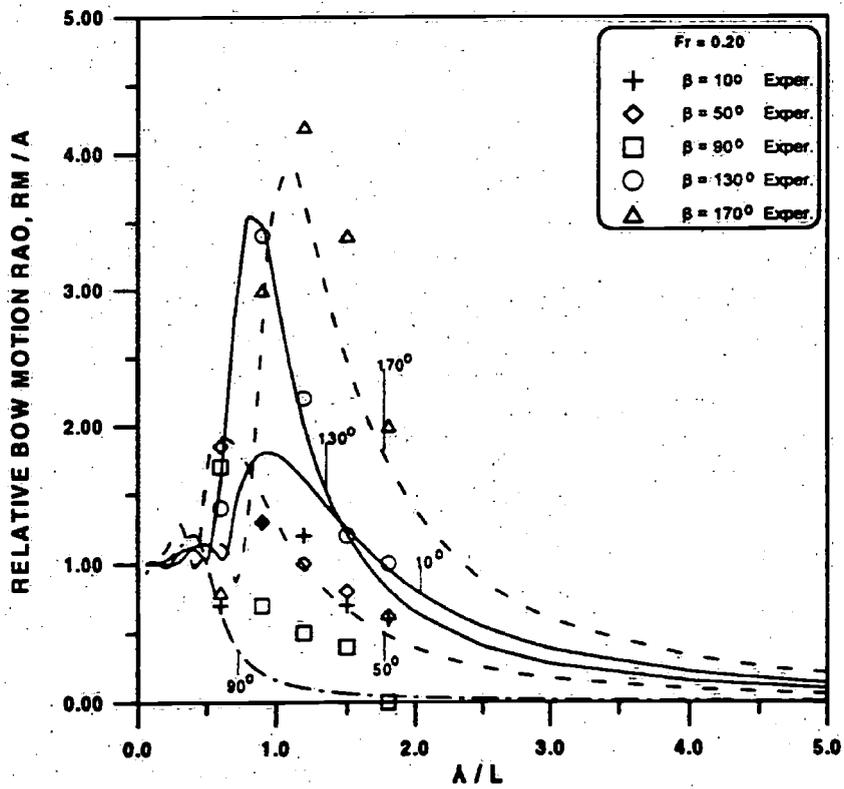


Fig. 2

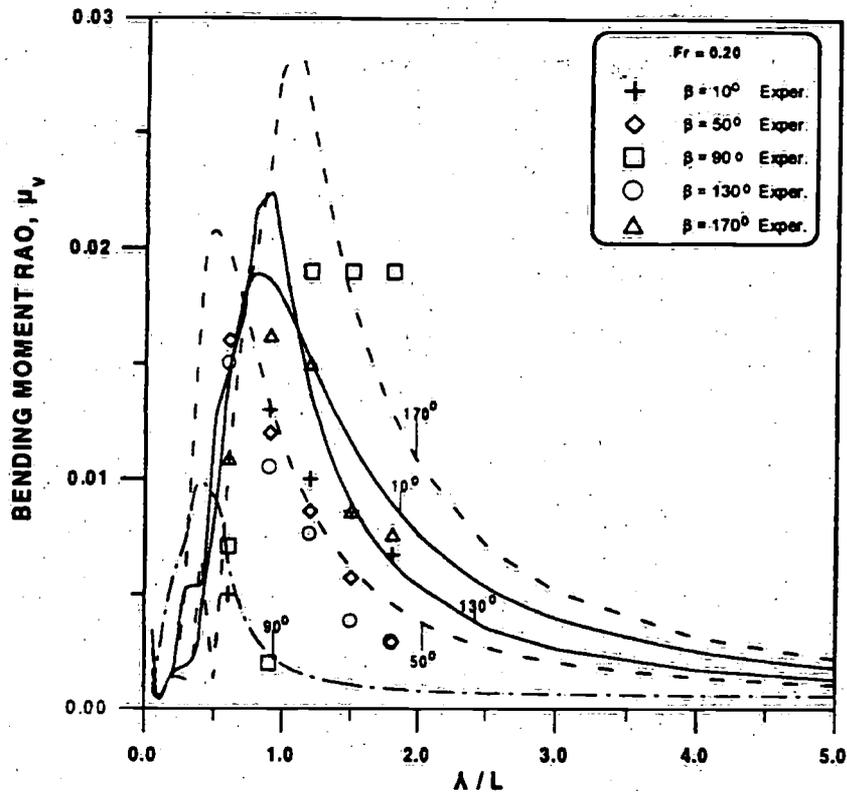


Fig. 3

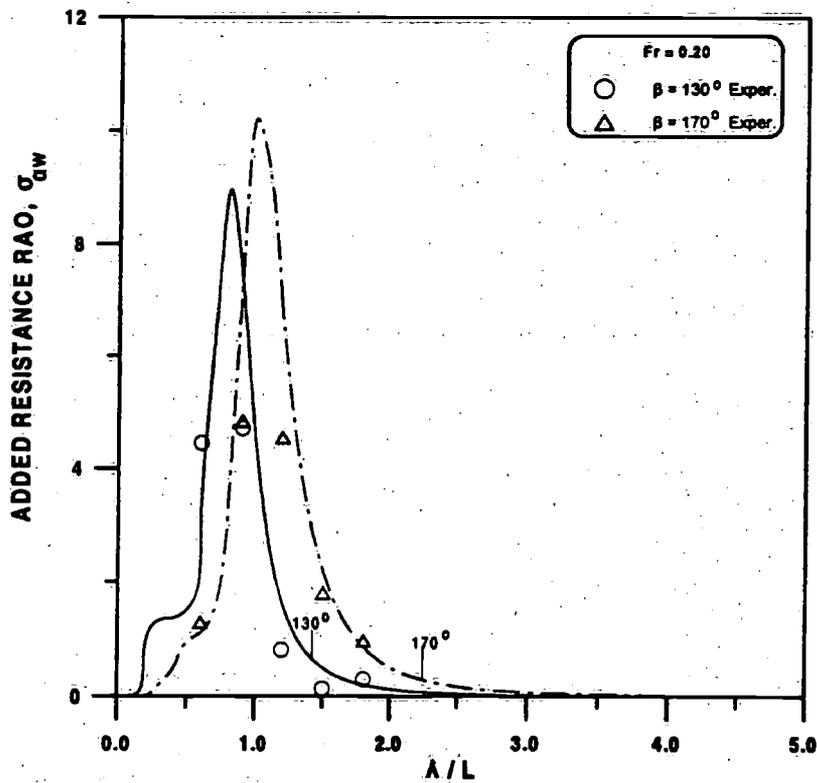


Fig. 4

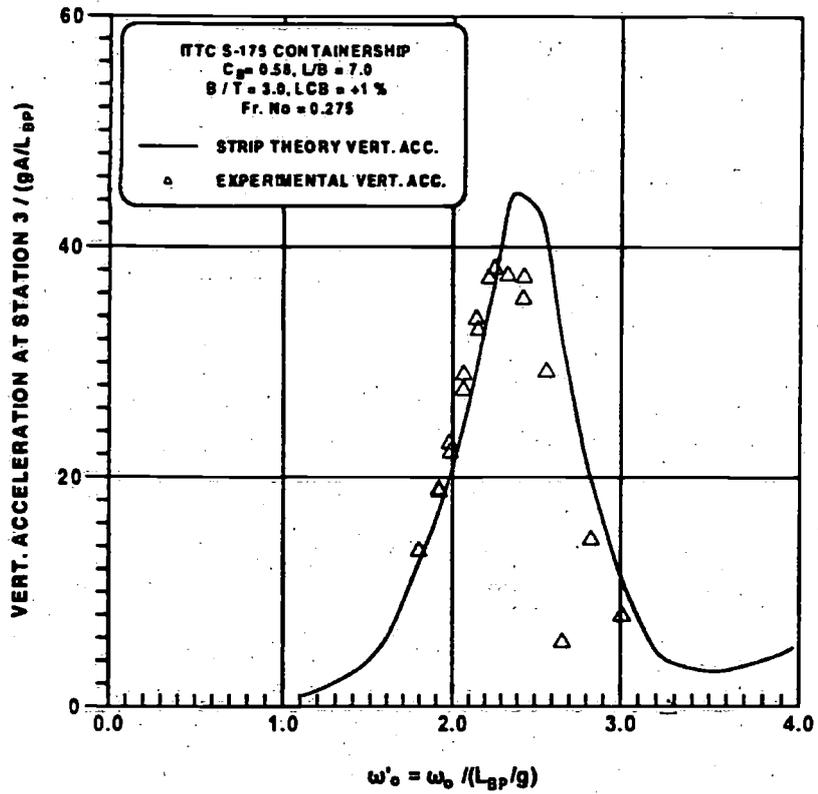


Fig. 5

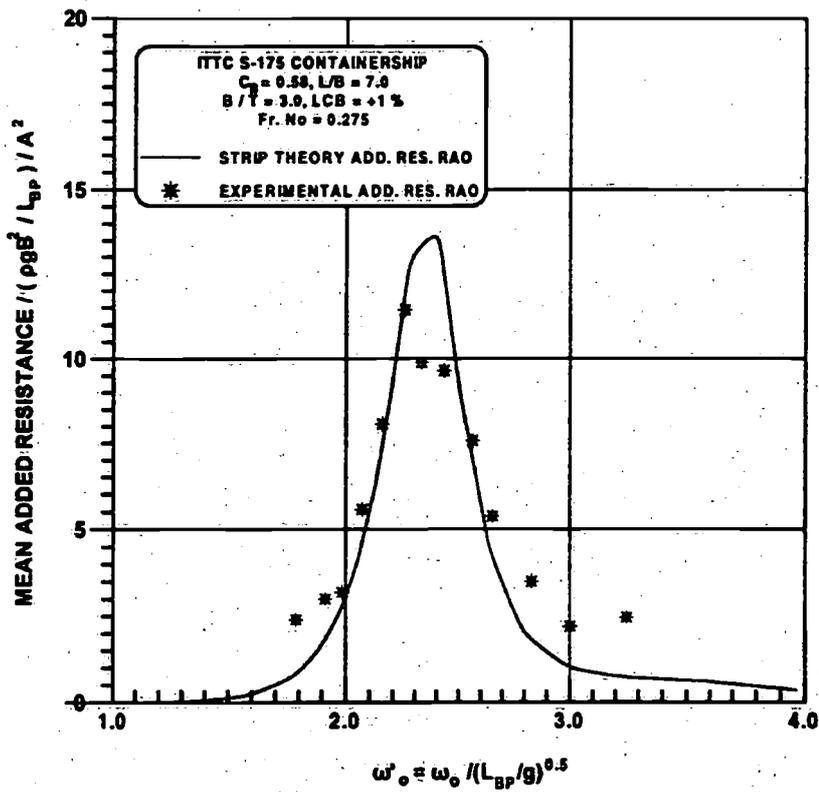


Fig. 6

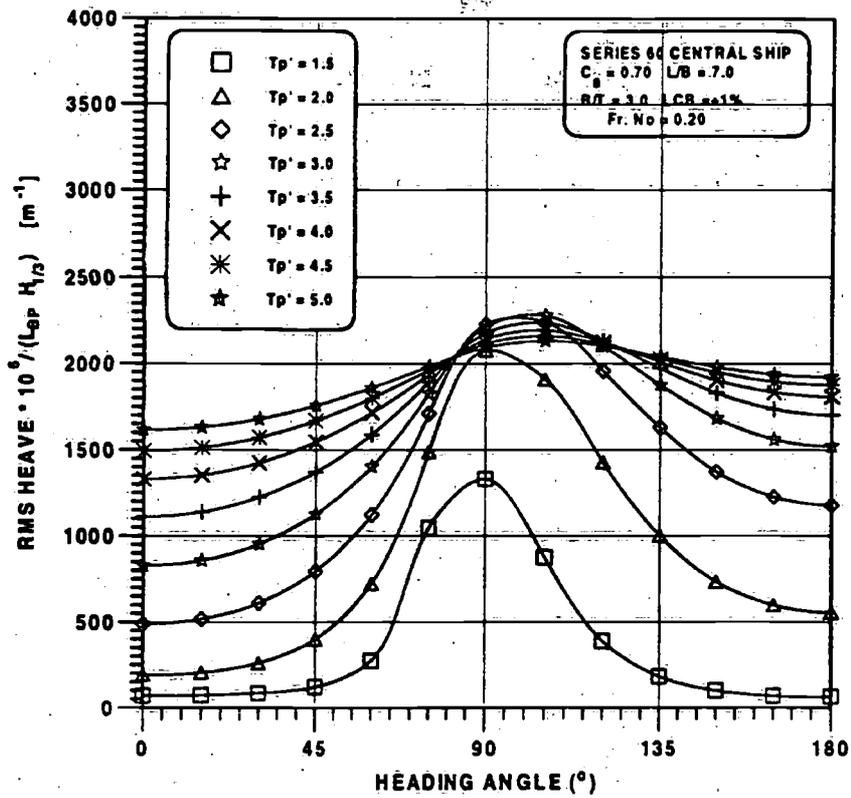


Fig. 7

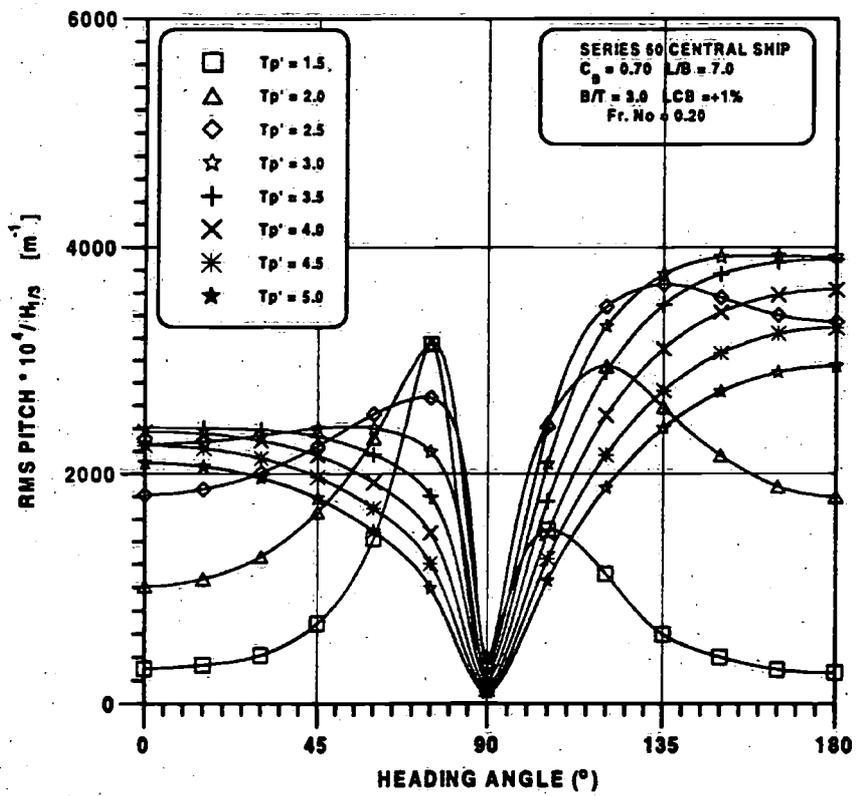


Fig. 8

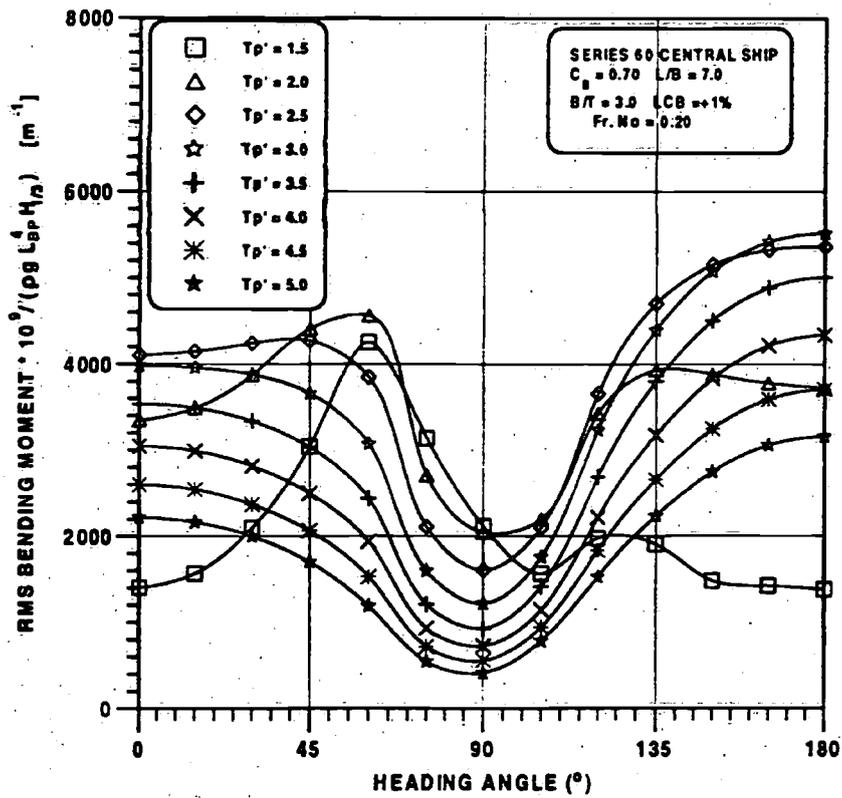


Fig. 9

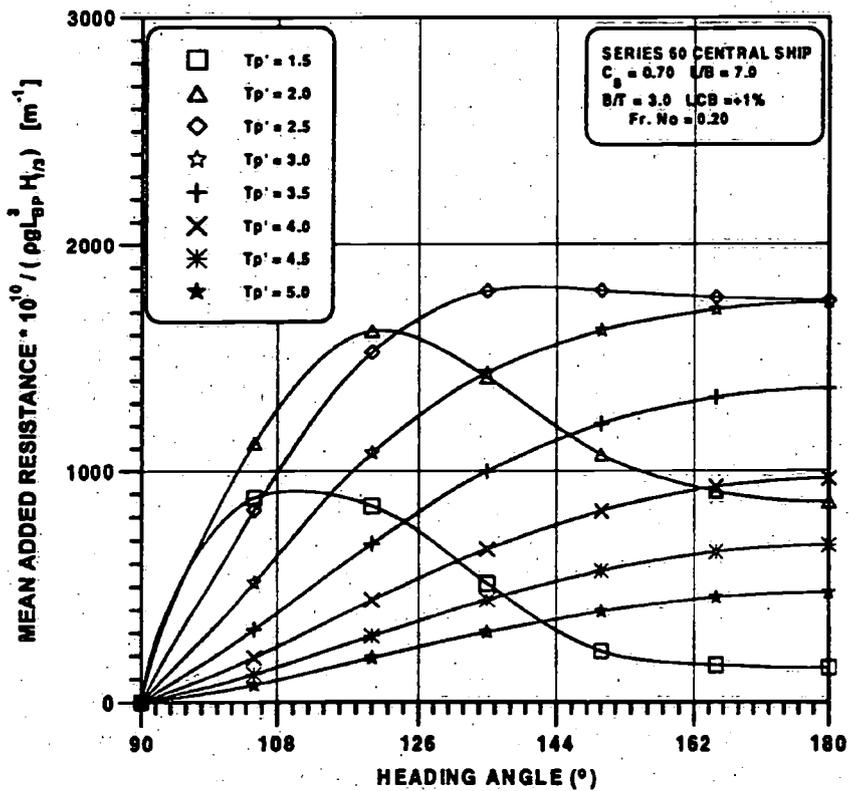


Fig. 10

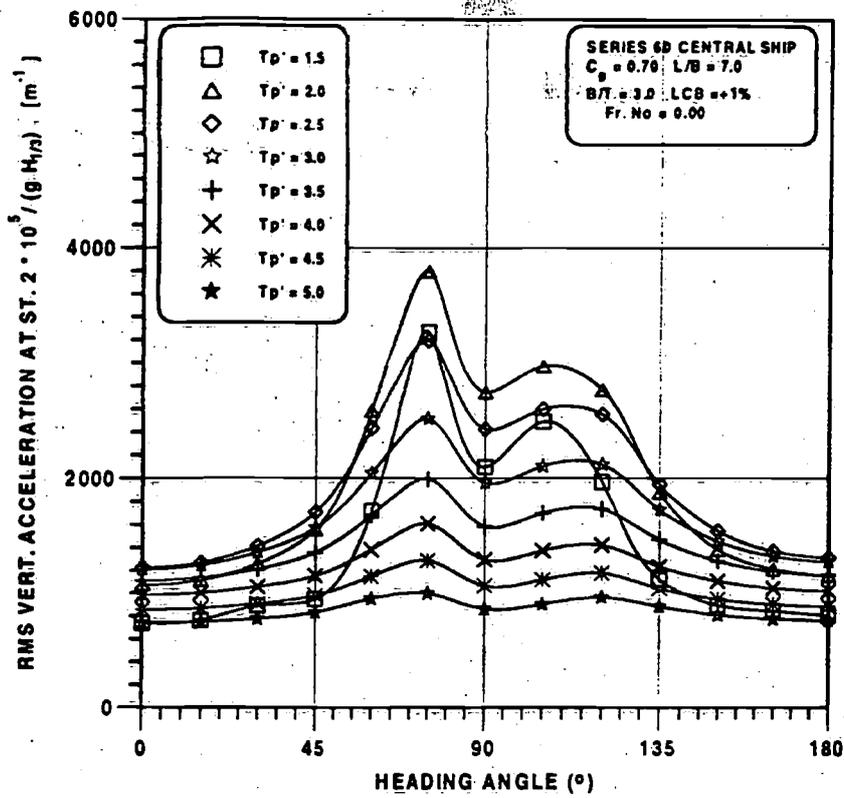


Fig. 11

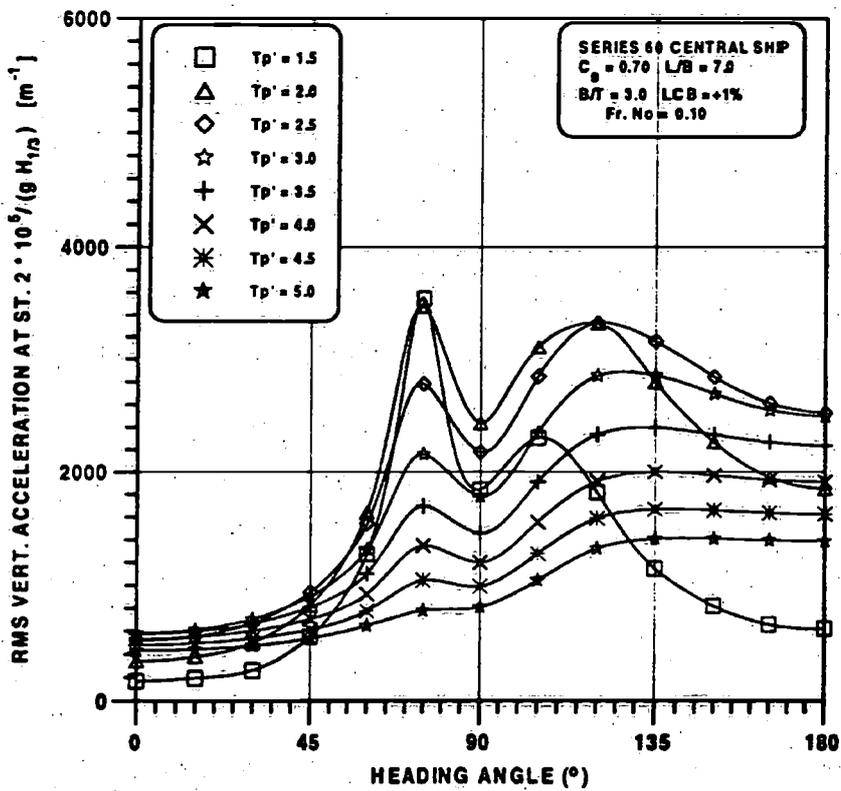


Fig. 12

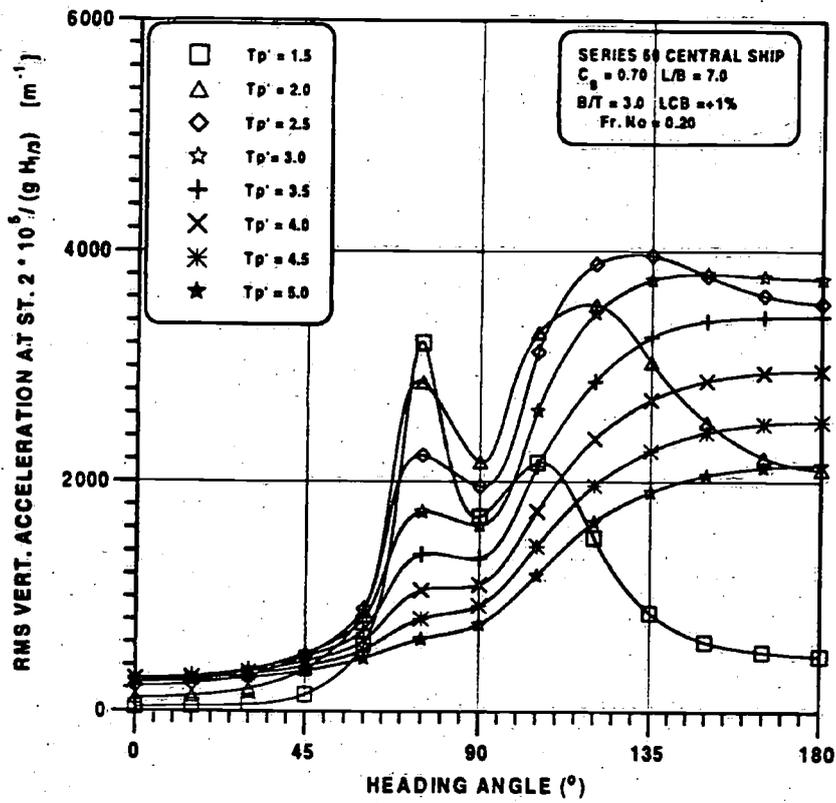


Fig. 13

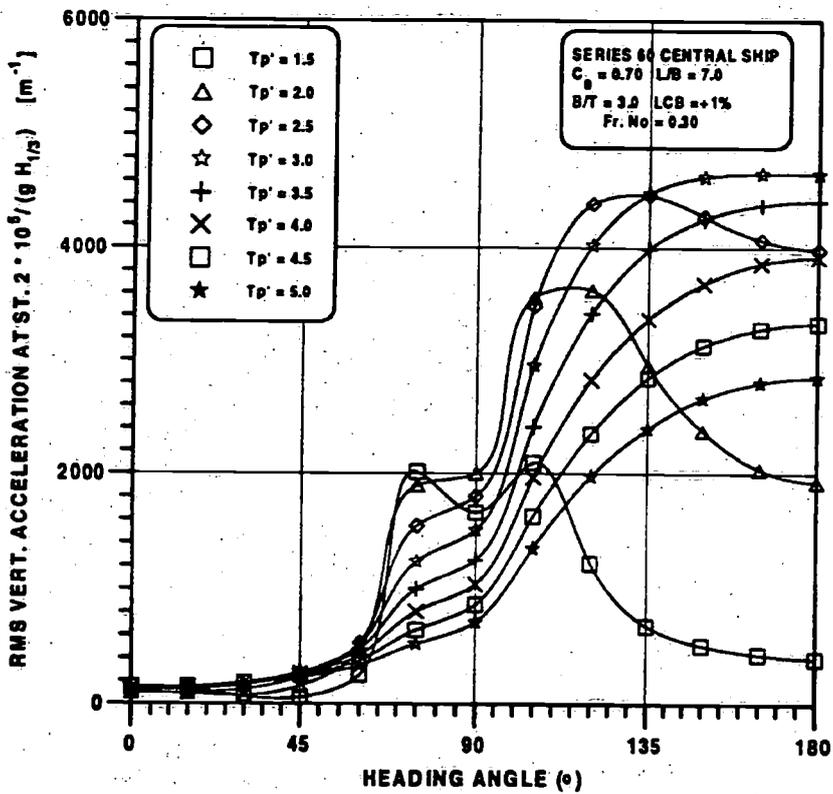


Fig. 14